

# **Release 2022 R1 Highlights**

## **Ansys Mechanical**



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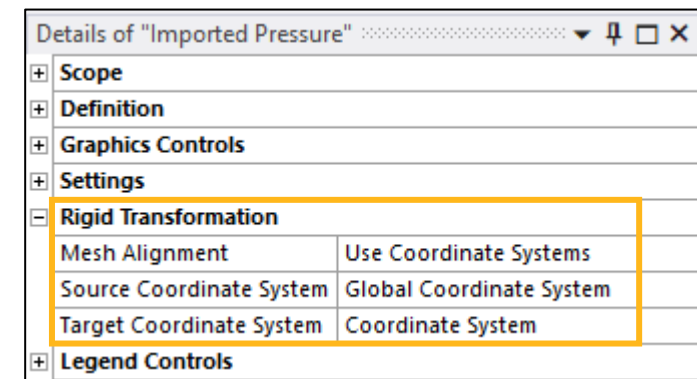
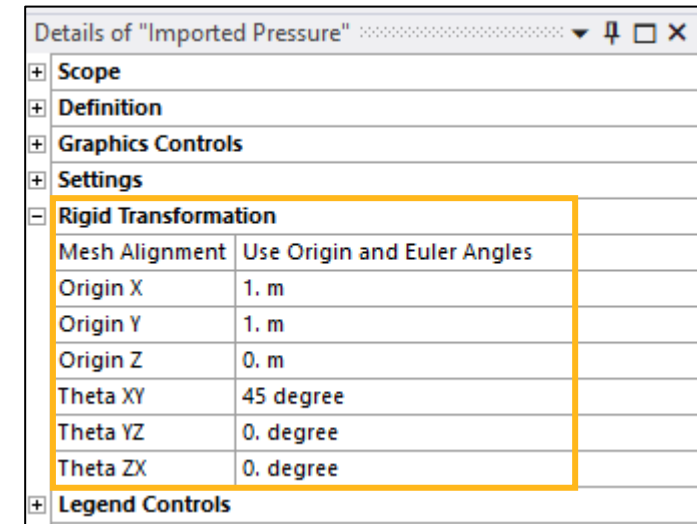


**Mechanical**

**Ansys**

# / Rigid Transformations on External Data

- Rigid Transformation inside Mechanical is supported for data imported from an upstream External Data system
- This transformation is added onto the Rigid Transformation defined in External Data system and can be used to further transform the data
- Two options to define the transformation:
  1. Use Origin and Euler Angles
  2. Use Coordinate Systems

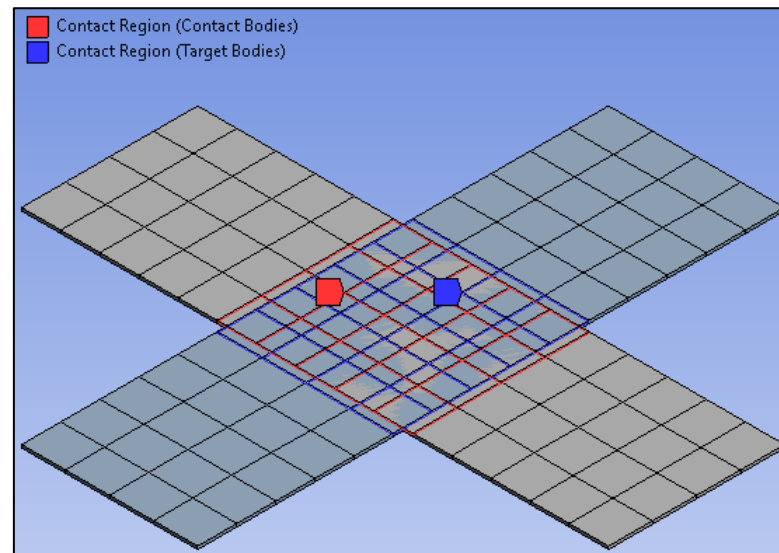




# Contact Shell Element Scoping

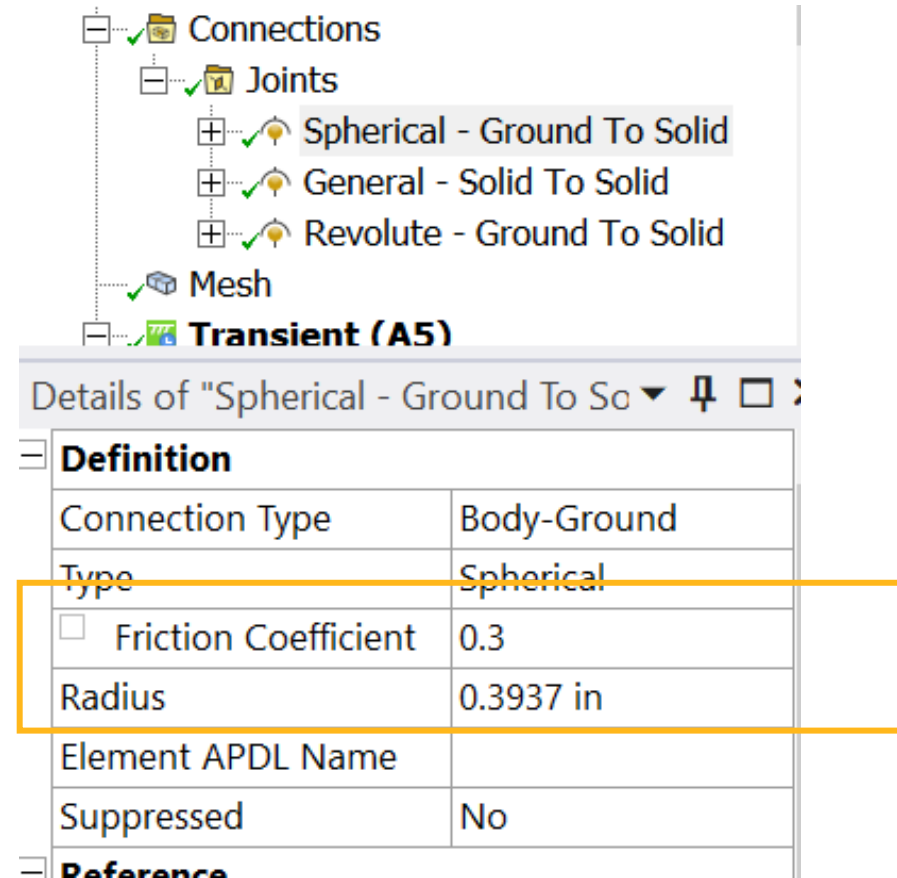
- 3D shell element scoping is supported for both **Contact** and **Target** sides of a “Contact Region” when using the Mechanical APDL solver
- This scoping capability supports both Geometry and Named Selection scoping methods

Scope	
Scoping Method	Geometry Selection
Contact	20 Elements
Target	20 Elements
Contact Bodies	Surface1
Target Bodies	Surface2
Contact Shell Face	Program Controlled
Target Shell Face	Program Controlled
Shell Thickness Effect	No



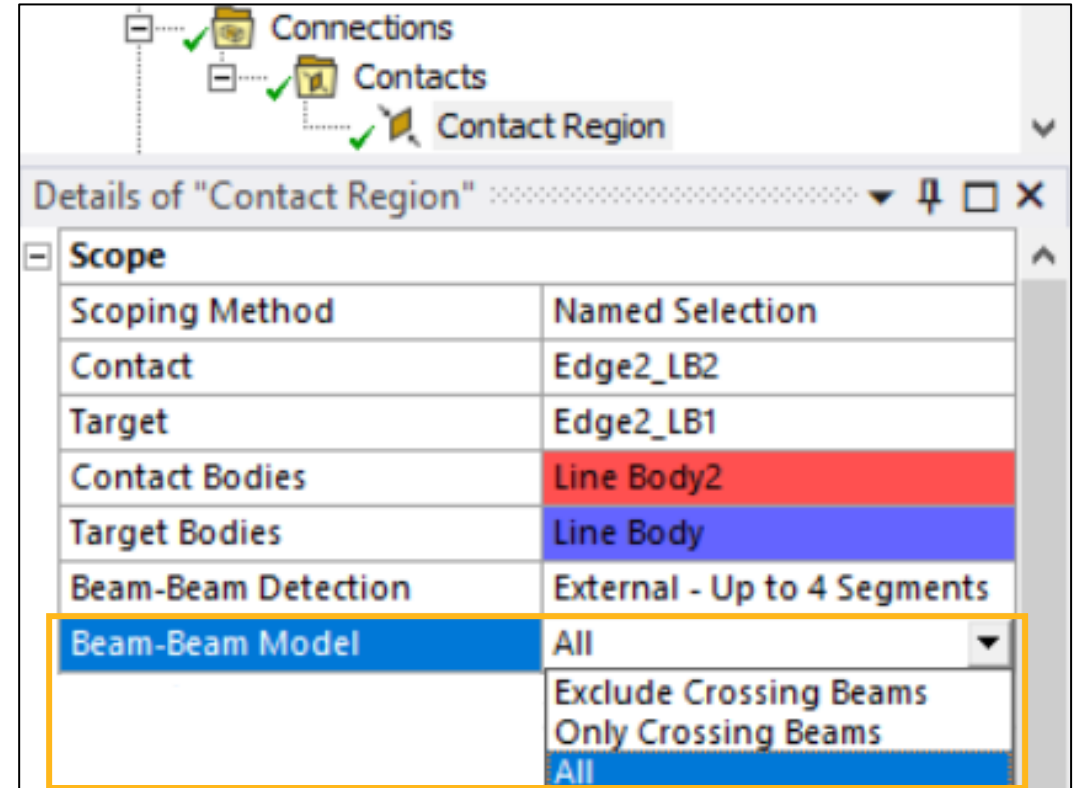
# / Spherical Joint: Friction properties

- Supported the friction for MAPDL solver using Friction Coefficient and Radius properties



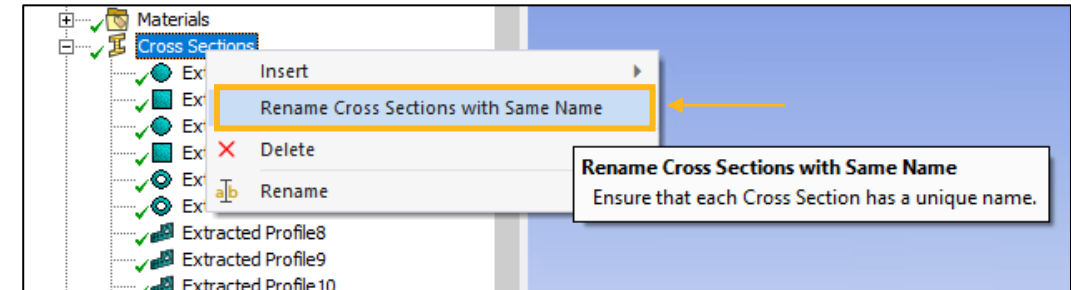
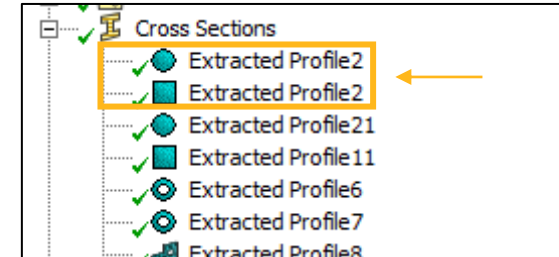
# Beam to Beam Model for Contacts

- Displays this property when it detects beam-to-beam contact.
- Enables you to specify the contact traction-based model the application uses for Beam-to-Beam contact
- Beam-Beam Model:
  - **Exclude Crossing Beams:** Excludes the contact for any beams that cross one another.
  - **Only Crossing Beams:** Includes only the contact for beams that cross one another
  - **All (Default):** includes all beam contact scenarios, which are: beam/edge to surface contact, parallel beam-to-beam contact, and crossing beam-to-beam contact



# Better Handling of Duplicate Cross Section Names

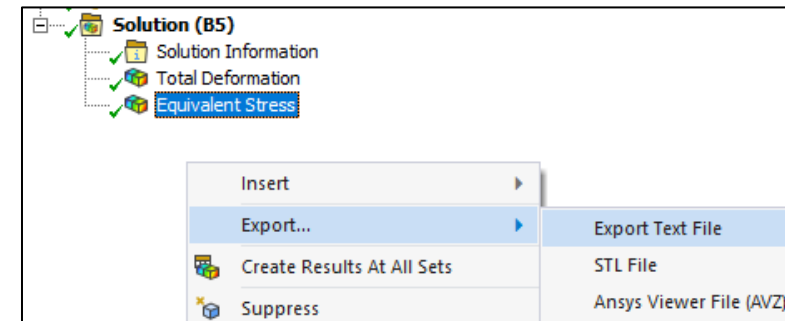
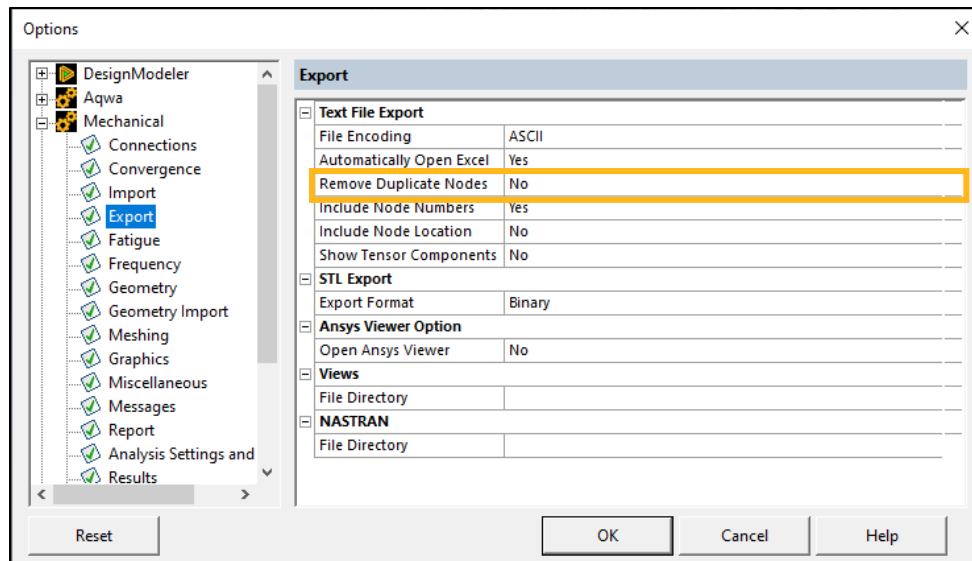
- Immediate error message if a solve is started with duplicate cross sections
- New option on the Cross Sections context menu to automatically rename any duplicate names



Messages		
	Text	Association
Error	One or more Cross Section objects have the same name. Before proceeding with the solution, ensure that all names are unique.	Project>Model>Static Structural>Solution
Info	The status of one or more objects imported from upstream mechanical has been changed from read only to editable. Any changes	Project>Model
Warning	The mesh is in read-only mode and cannot be cleared. Only the data generated in contact tool and downstream environments was	Project>Model

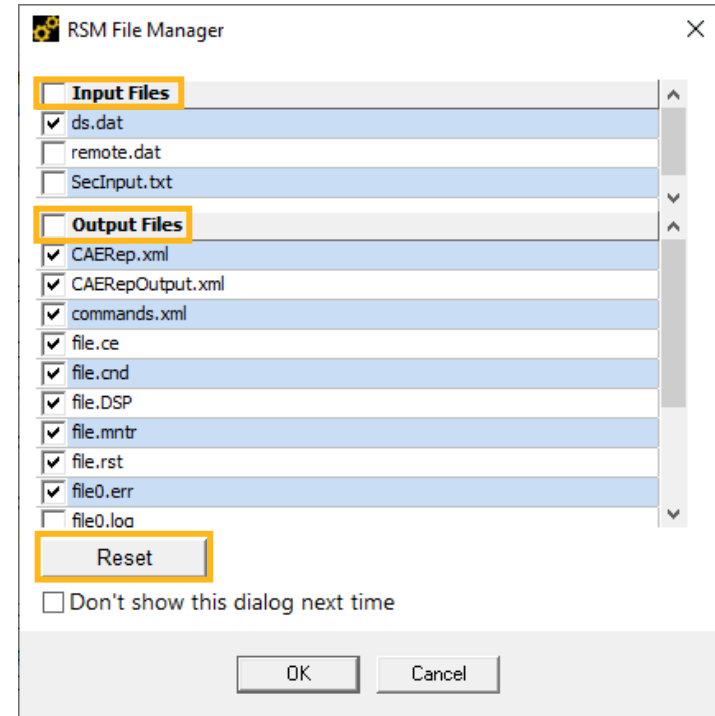
# / Remove Duplicate Nodes Option

- Added option **Remove Duplicate Nodes** under Export
- When set to 'Yes', duplicate nodes will be removed when exporting results to a text file



# / RSM File Manager Enhancements

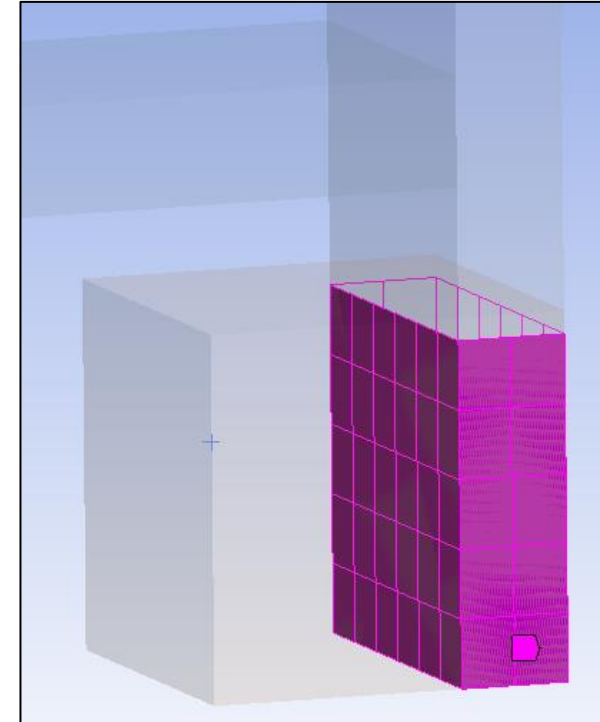
- Added **Select-all option** for both Input and Output files
- Added 'Reset' button to load the default checkbox settings
- Checkbox settings will now persist between solves, including for files that are no longer present
  - For example, input/output files that are present during one run but not present during another will still have their checkbox setting saved



# / Within Body: Named Selections Criterion

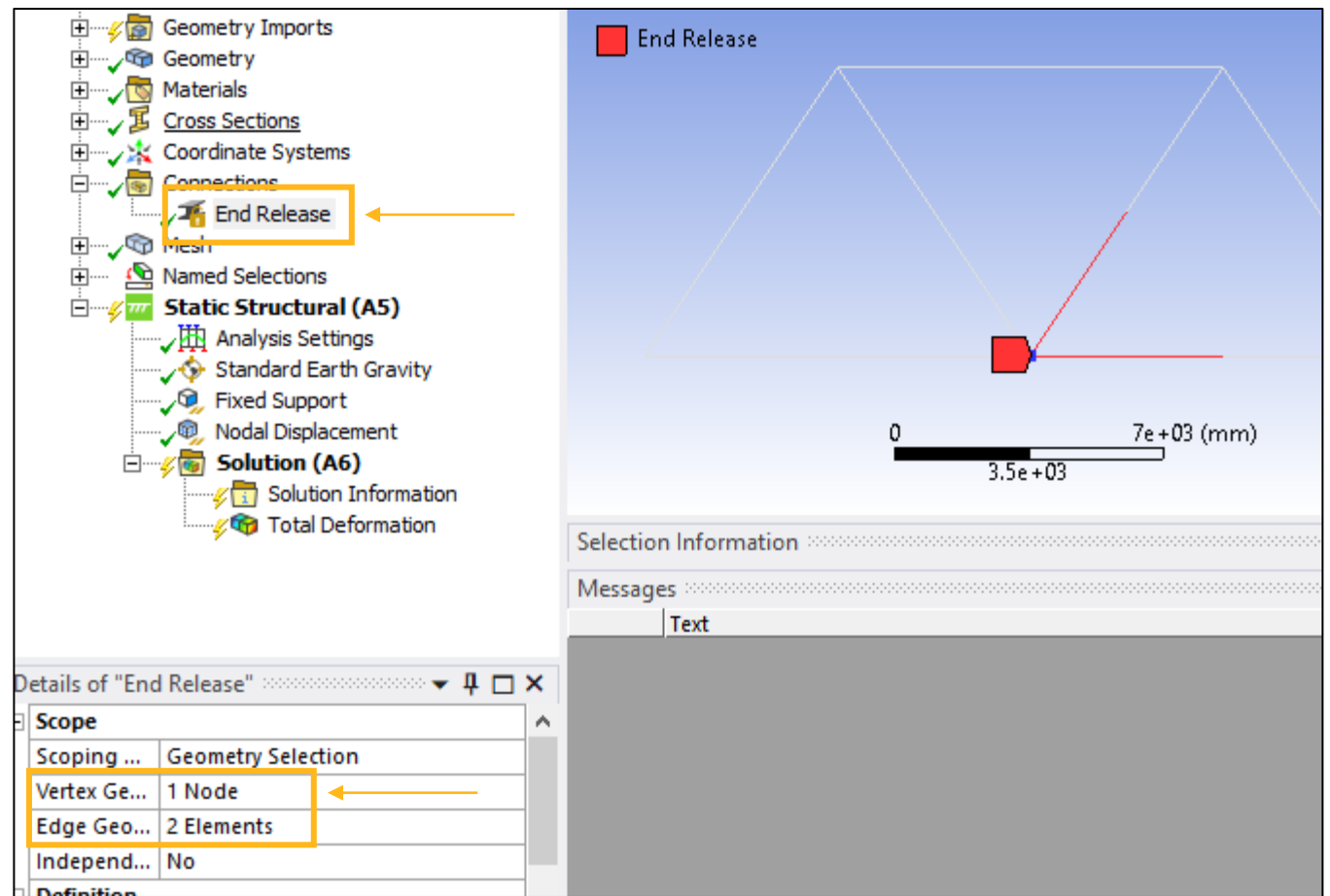
- Added **Within Body** criterion to the Named Selections worksheet. This criterion is only available for nodes, elements, and element faces. Additionally, only geometry bodies with Treatment set to **Construction Body** may be chosen
- This criterion will generate a named selection of the chosen Entity Type that are contained within the **Construction Body**

	Action	Entity Type	Criterion	Operator	Units	Value
<input checked="" type="checkbox"/>	Add	Element Face	Within Body	Equal	N/A	Cube



# End Release FE Scoping

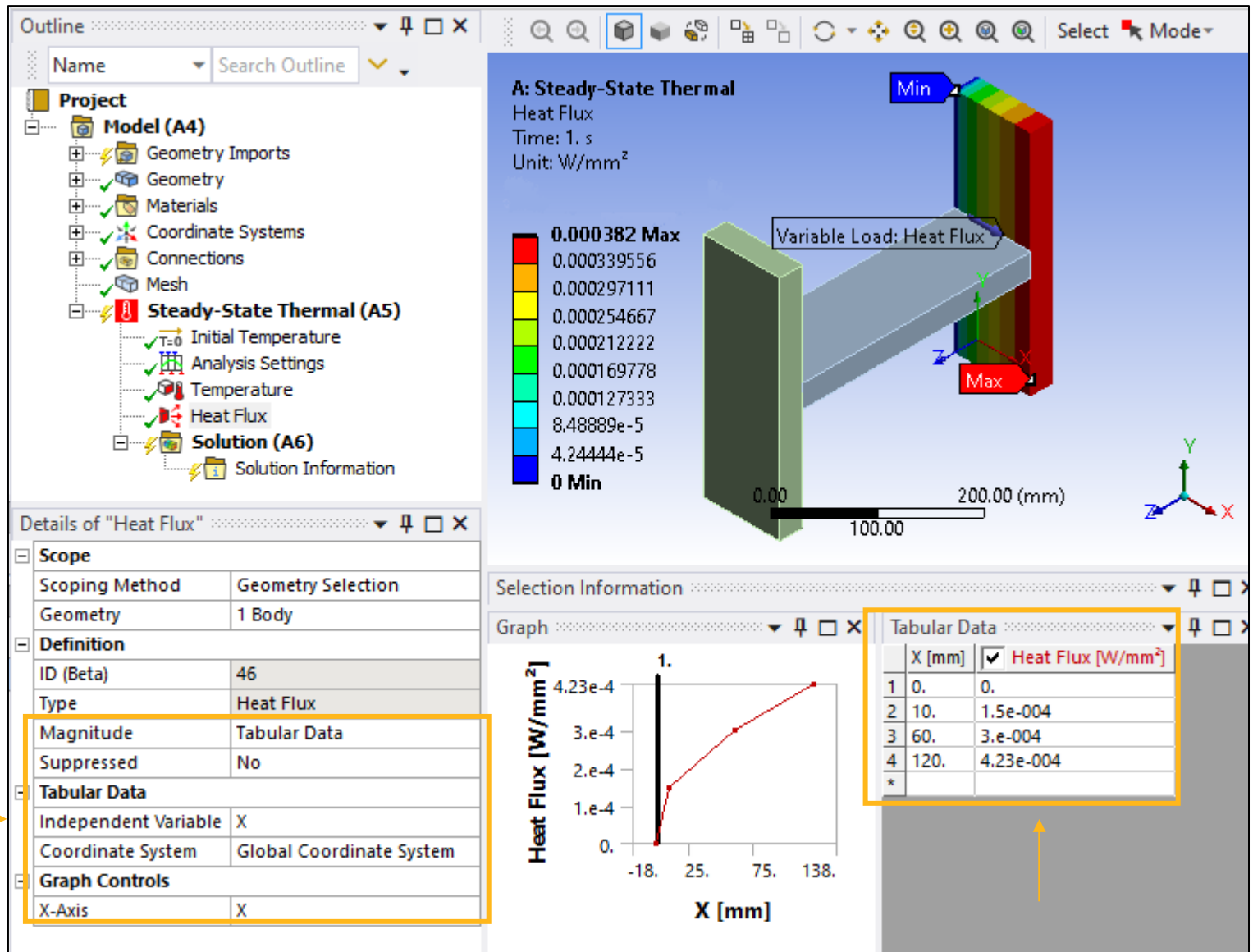
- Exposed the ability to scope nodes and elements to be released via “End Release”





# Heat Flux Tabular Data

- Heat Flux can now be defined via spatially varying tabular data



# Support for Solid-Shell Elements in Trace Mapping

- Geometries meshed with solid-shell elements are now supported in the trace mapping workflow
- The behavior is like that of shell trace mapping, where trace data is controllable on a per-layer basis
- Dielectric and Trace materials can be defined on a per-layer basis
- This approach should provide users the accuracy they are targeting with less computational expense compared to solid element trace mapping

# Seam Weld Worksheet Enhancements

1. Activate all Selections turns on all the curves/edges in the worksheet
2. Deactivate all Selections turns off all the curves/edges in the worksheet
3. Go To Selected Items in Tree activates the weld curves in the geometry folder in the tree
4. Promote to Weld Control cuts and pastes all selected curves and puts them in a new Weld
5. Paste from Clipboard enables copying information from an excel file and pasting the information from a given cell. It is important that the correct cell from where the paste should happen is “right-clicked”. Information flows to the right and then downward from the selected cell
6. Clear Intersection Tag removes the intersection tags for the selected weld curves
7. Add Intersection Tag adds an intersection tag for the selected weld curves. Multiple tags are to be separated by a semi-colon
8. Review Common Intersections selects all the curves that have the same Intersection tag

✓	Step	Weld Curve	Edge Mesh Size (m)	Weld Angle (°)	HAZ Distance (m)	Number Of HAZ	HAZ Growth Rate	Sharp Angle (°)	Connection Tolerance (m)	Material	Smoothing	Intersection Tag
<input checked="" type="checkbox"/>	1	G1Curve 1	8.e-003	45.	4.e-003	0	1.2	30.	0.	Structural Steel	Yes	
<input checked="" type="checkbox"/>	2	G1Curve 2	8.e-003	45.	4.e-003	0	1.2	30.	0.	Structural Steel	Yes	
<input checked="" type="checkbox"/>	3	G1Curve 3	8.e-003	45.	4.e-003	0	1.2	30.	0.	Structural Steel	Yes	
<input checked="" type="checkbox"/>	4	G1Curve 4	8.e-003	45.	4.e-003	0	1.2	30.	0.	Structural Steel	Yes	
<input checked="" type="checkbox"/>	5	G1Curve 5	8.e-003	45.	4.e-003	0	1.2	30.	0.	Structural Steel	Yes	
<input checked="" type="checkbox"/>	6	G1Curve 6	8.e-003	45.	4.e-003	0	1.2	30.	0.	Structural Steel	Yes	
<input checked="" type="checkbox"/>	7	G1Curve 7	8.e-003	45.	4.e-003	0	1.2	30.	0.	Structural Steel	Yes	
<input checked="" type="checkbox"/>	8	G1Curve 8	8.e-003	45.	4.e-003	0	1.2	30.	0.	Structural Steel	Yes	
<input checked="" type="checkbox"/>	9	G1Curve 9	8.e-003	45.	4.e-003	0	1.2	30.	0.	Structural Steel	Yes	
<input checked="" type="checkbox"/>	10	G1Curve 10	8.e-003	45.	4.e-003	0	1.2	30.	0.	Structural Steel	Yes	
<input checked="" type="checkbox"/>	11	G1Curve 11	8.e-003	45.	4.e-003	0	1.2	30.	0.	Structural Steel	Yes	
<input checked="" type="checkbox"/>	12	G1Curve 12	8.e-003	45.	4.e-003	0	1.2	30.	0.	Structural Steel	Yes	

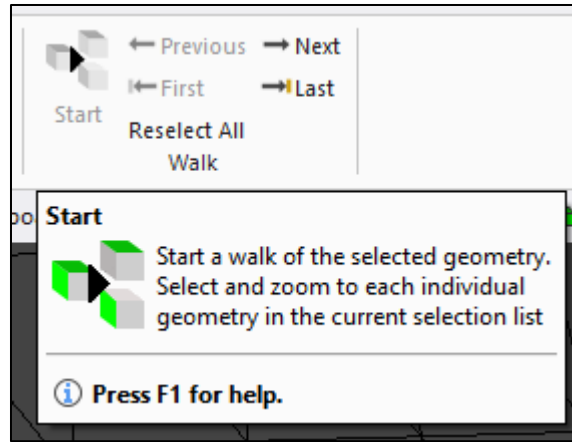
# / Seam Weld Worksheet Enhancements

1. By pressing the **Sort by Selection** button, all selected weld curves (for that corresponding weld object) in the worksheet or graphics window are shown in page 1 in ascending order
2. Enter a string in the **Enter name field** box and click on the **Create Controls for All Curve Bodies** button. All weld curves that contain the string entered in this text field are only displayed in the worksheet. Leave this empty to generate all weld curves. This operation gets recorded/journaled and can be used for playback
3. Selecting one/multiple entries in the worksheet will show the selection in the graphics window and vice-versa

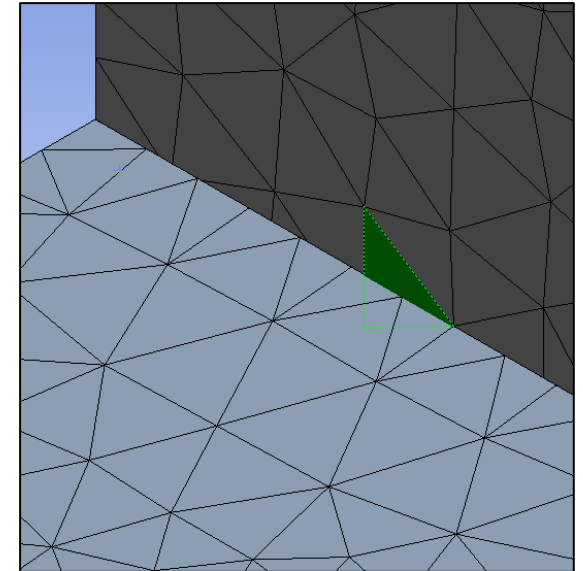
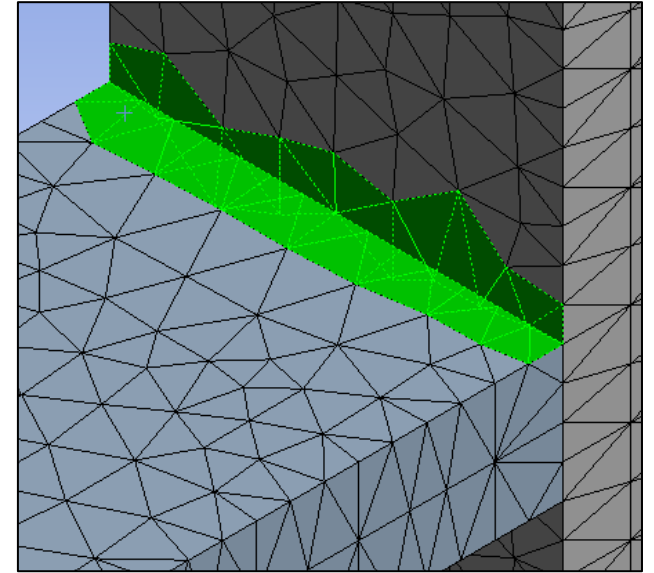


# / Model Walk for FE Data

- Model Walk visits each individual part of a selection
- Expanded to include Nodes, Elements, and Element Faces
- Automatically zooms in to make identification easy

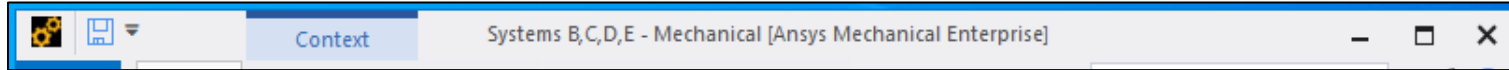


- Especially useful for walking selection sets created by error messages and criteria based named selections

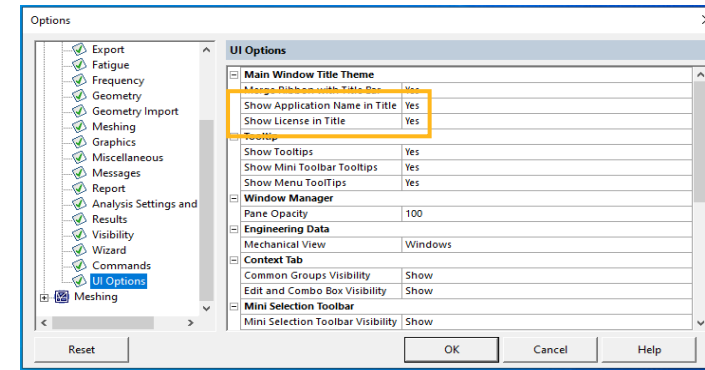
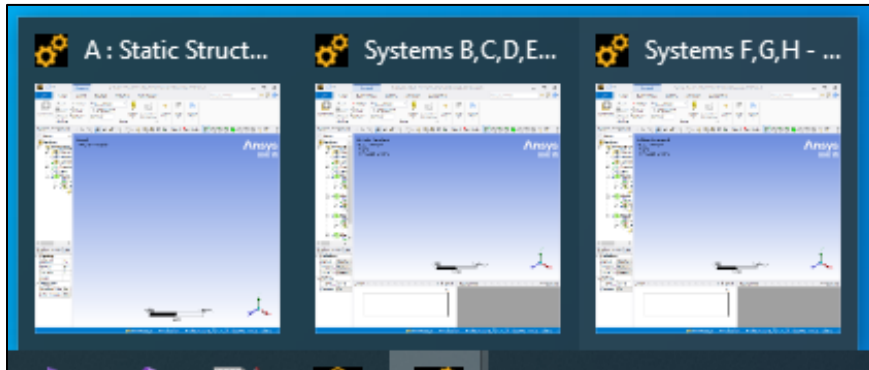


# Mechanical Window Titles

- Instead of only showing “Multiple Systems”, the title bar now shows the system identifiers just as it does for single systems



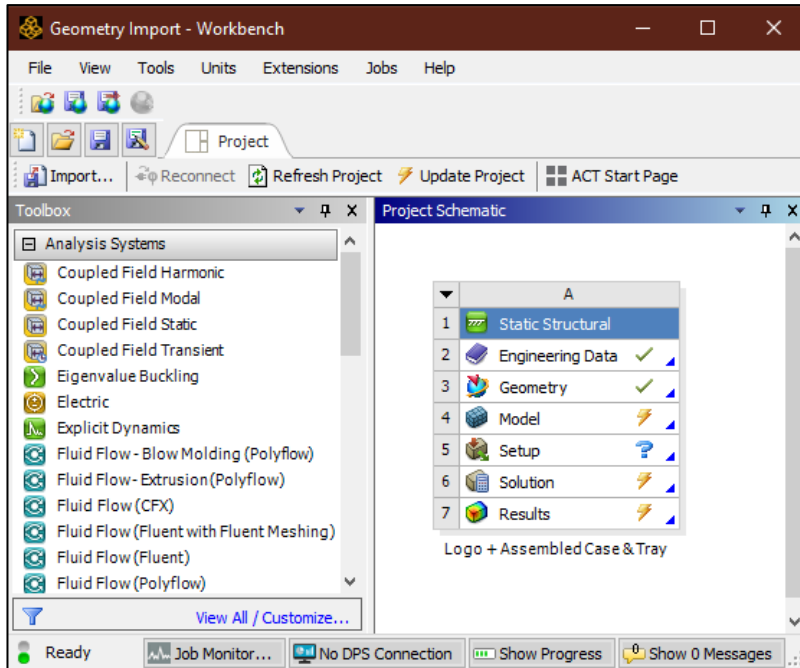
- With several sessions open, it is easier to find a specific window. Hovering the mouse over the Mechanical icon in the task bar will show the entire title



Control setting for window names in “Options” panel

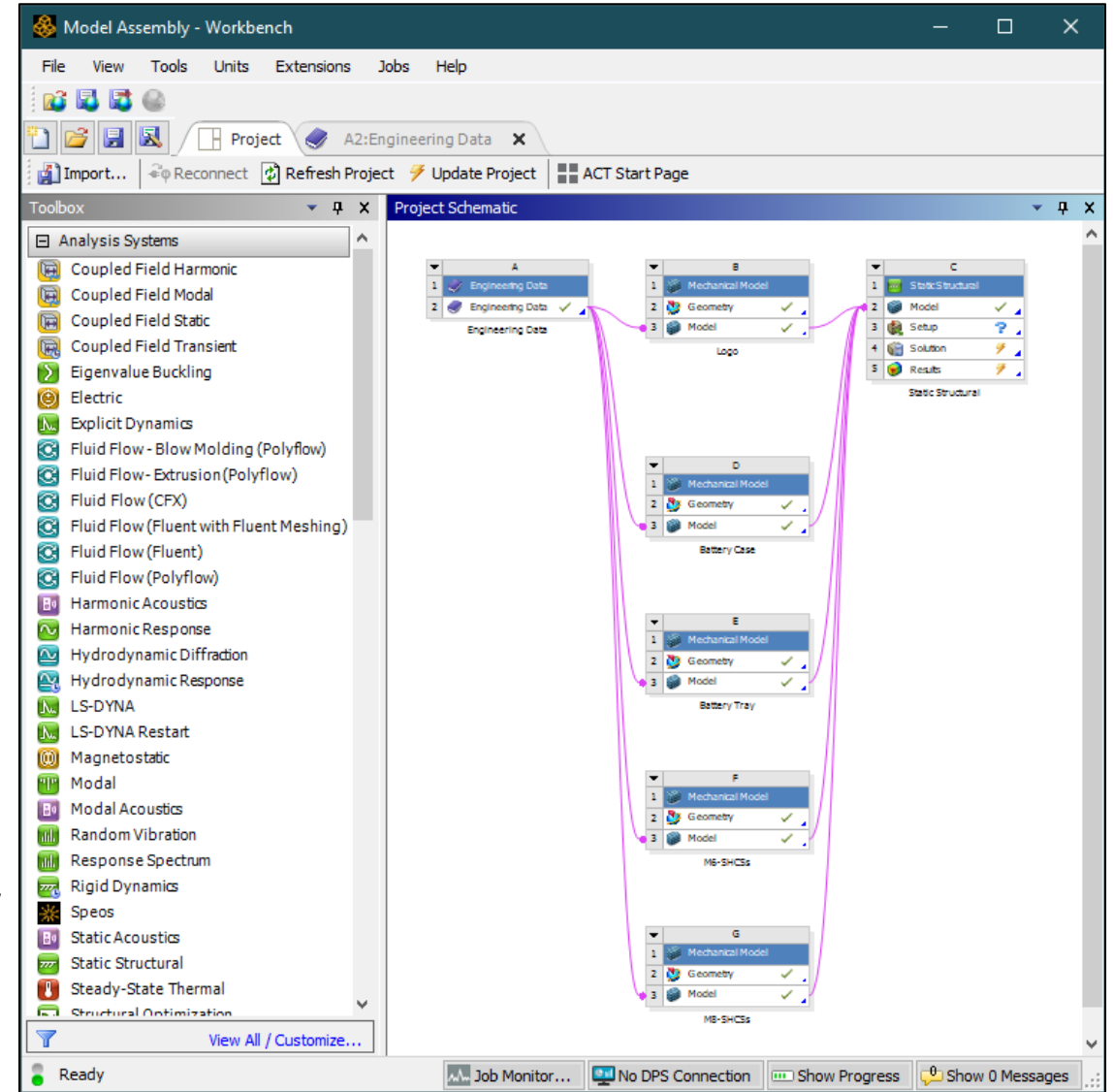
# Geometry Import

- An easier, more unified approach to importing multiple geometry sources into Mechanical



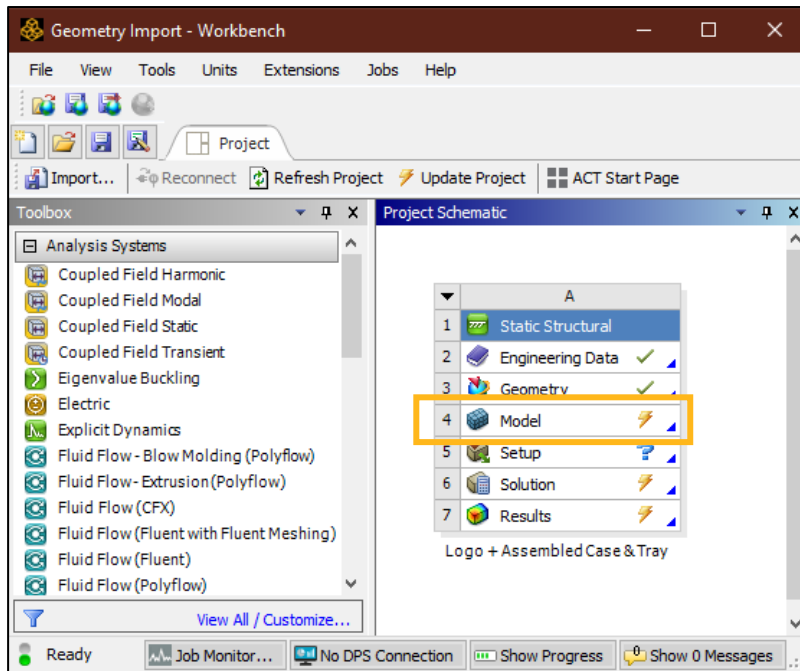
**Geometry Import**

**Model Assembly**



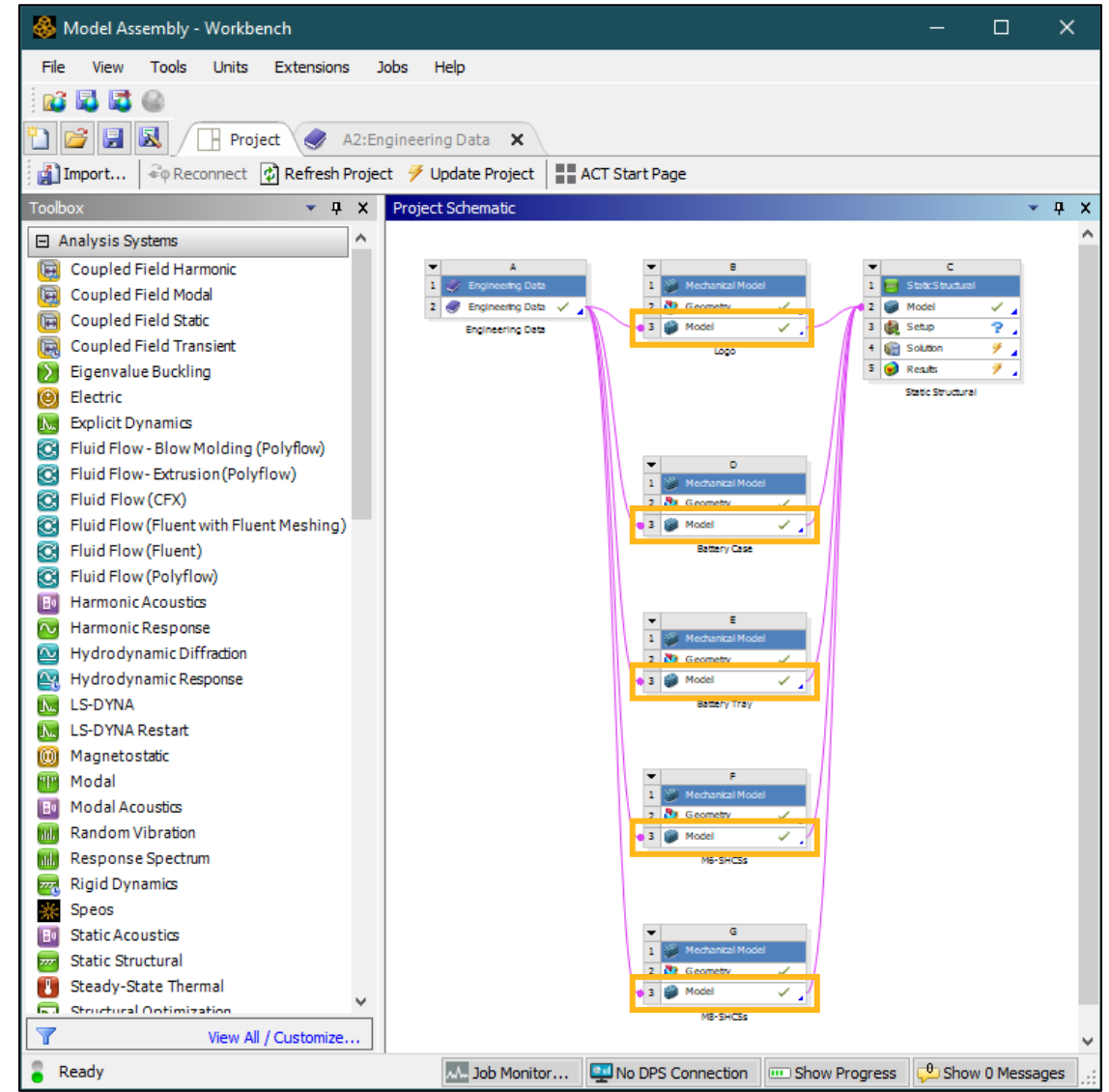
# Geometry Import

- Geometry Import uses an "Assemble First, Mesh Later" workflow, which significantly reduces the "time-to-assembly" footprint



Geometry Import

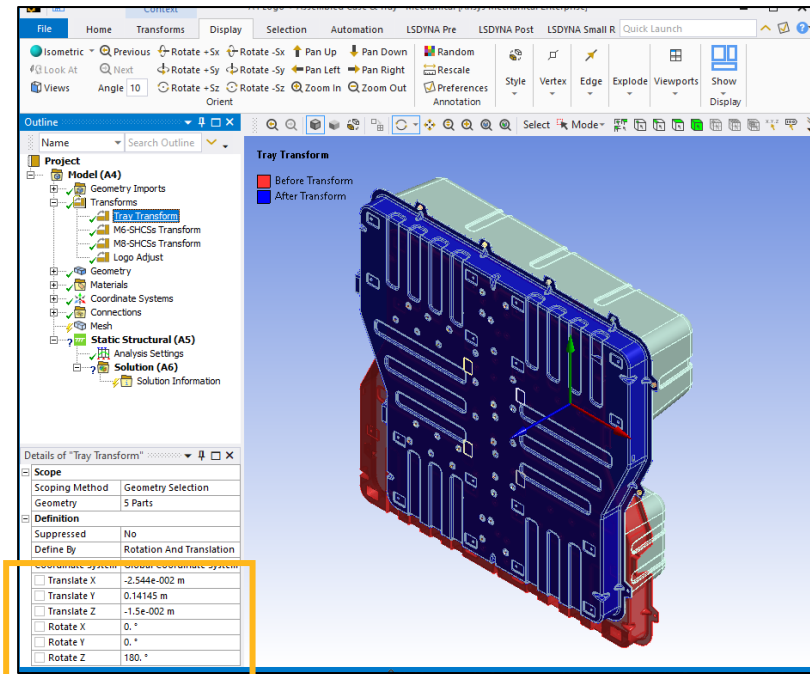
Model Assembly





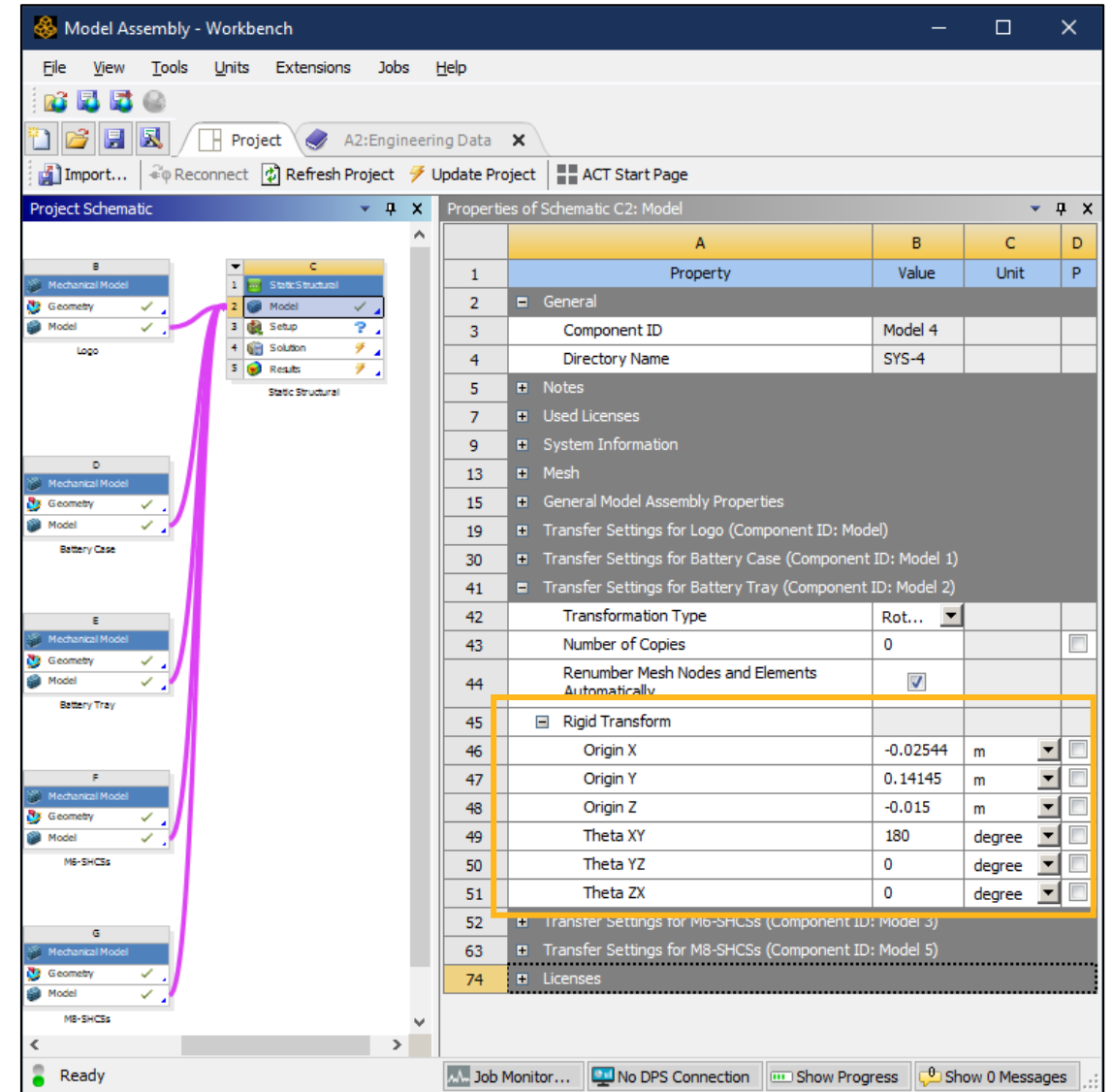
# Geometry Import

- In Mechanical, part transformations provide continuous visual feedback during the assembly process



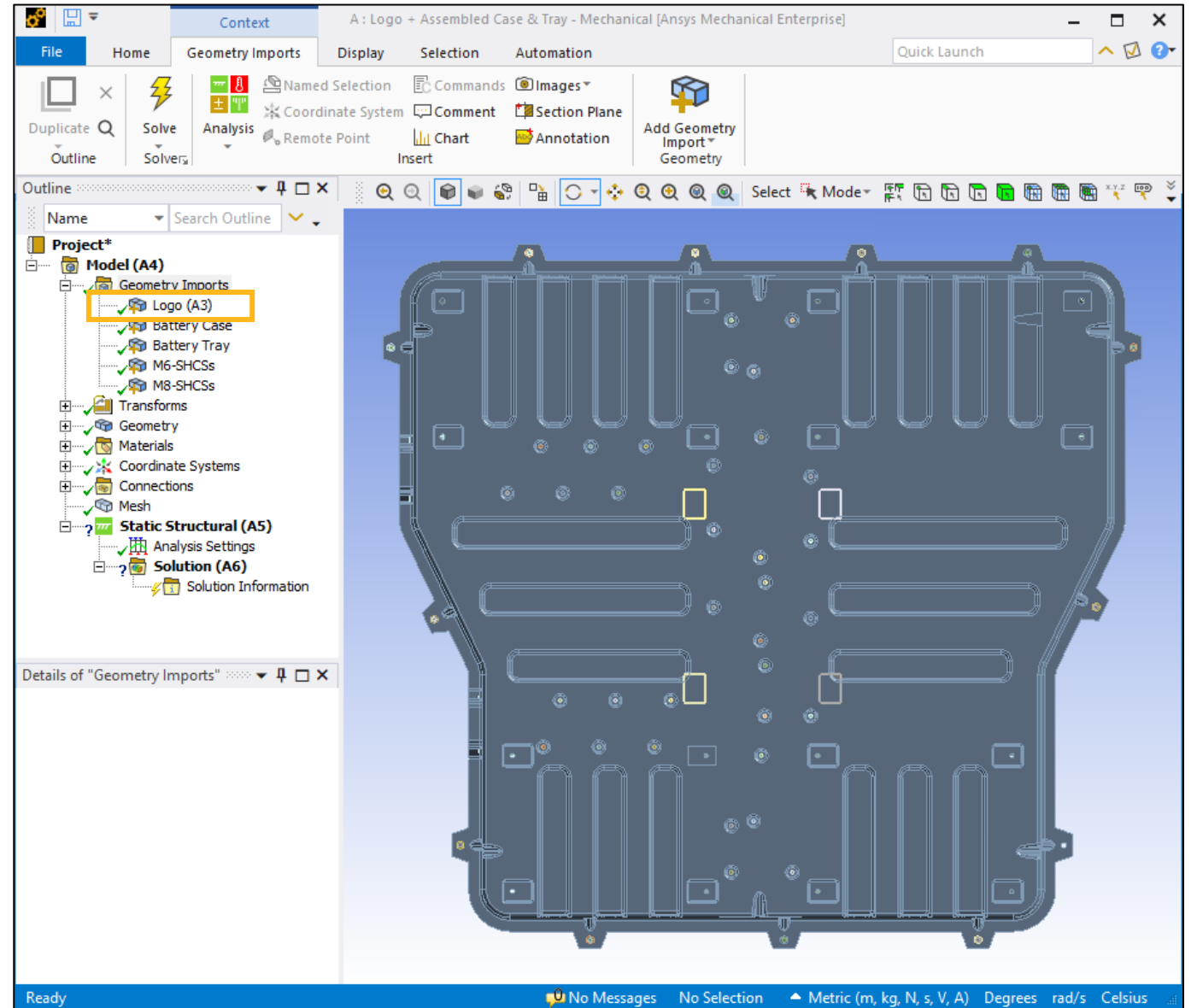
Geometry Import

Model Assembly



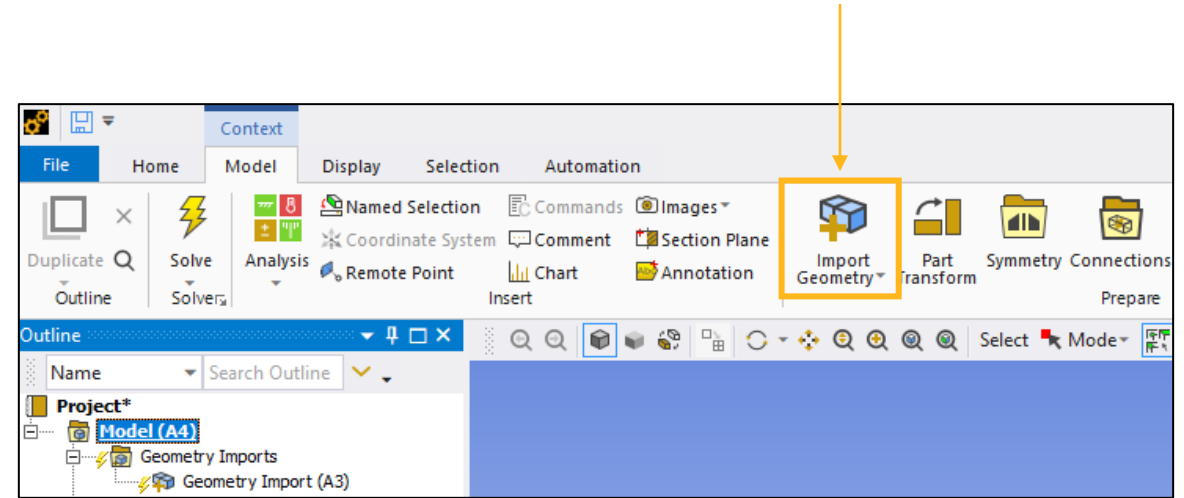
# Geometry Import

- This first Geometry Import object represents the geometry specified in Workbench
- This is sometimes referred to as the "Primary Source" and is always shown in the Tree along with the "Geometry Imports" folder
- The primary source name also includes the cell ID of the corresponding "Geometry" cell in Project Schematic, which is "A3" in this example



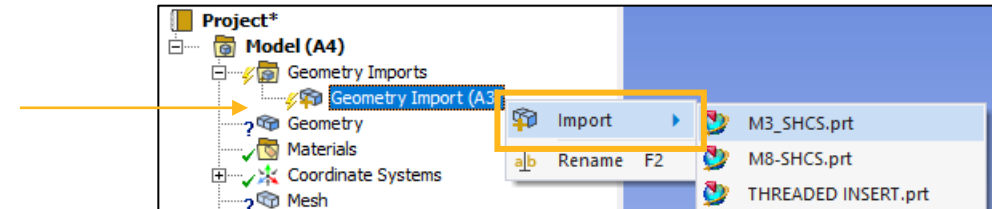
# Geometry Import

- In Mechanical, the "Primary Source" import is started via either the Ribbon or Right-Mouse-Button (RMB) context menus



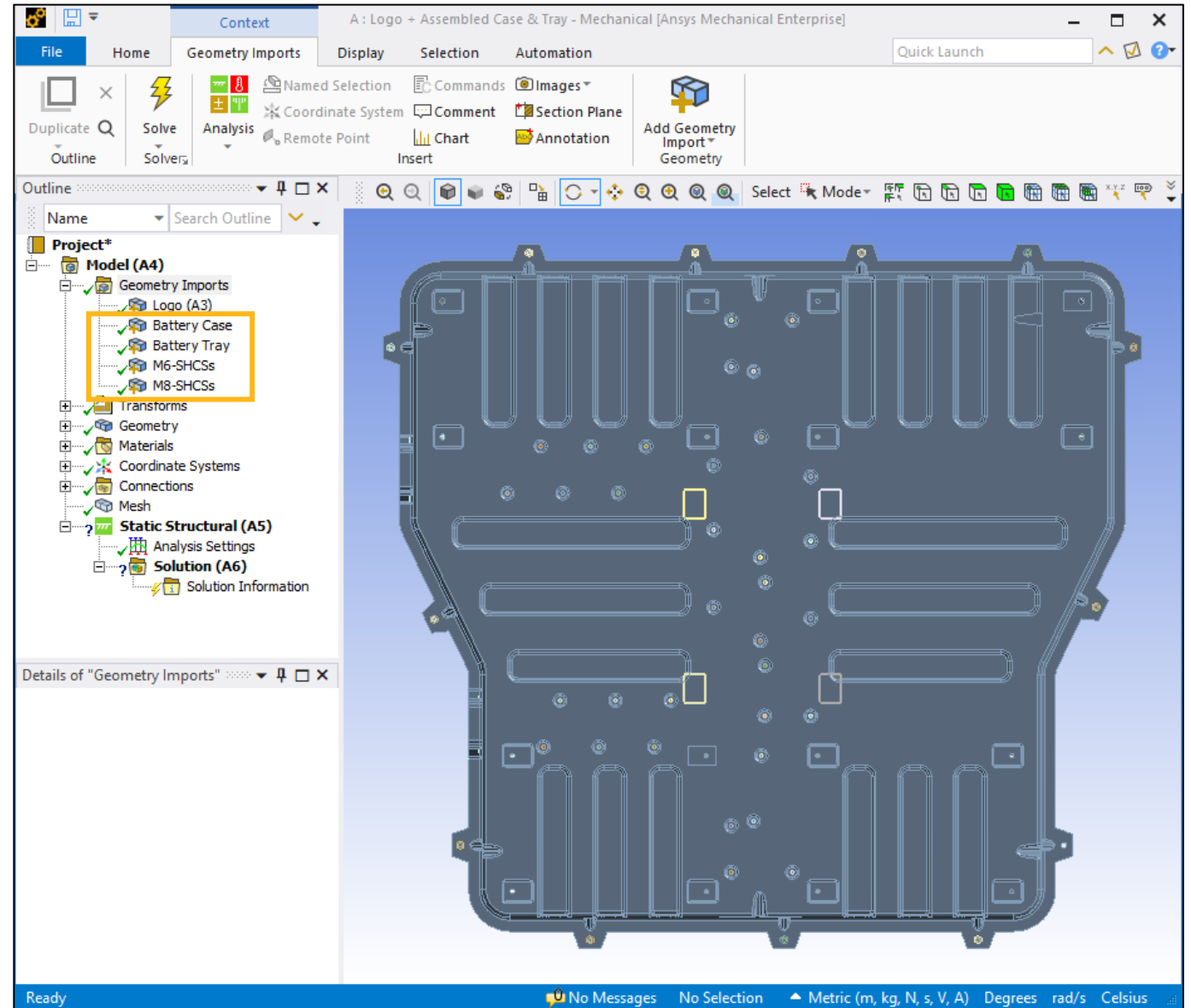
## Pro Tip

Use the expanded menu to select from recently used geometry sources



# Geometry Import

- Additional Geometry Import objects can be added under the "Geometry Imports" folder
- These are sometimes referred to as "Secondary Imports" and will only appear when explicitly added



# Geometry Import

- As before, CAD preferences specify how each import should be processed
- The "Primary Source" import respects the preferences set in the Properties page of the "Geometry" cell in Workbench
- For secondary Geometry Import objects, the preferences are specified *per import* after selecting the geometry source

The image shows two overlapping 'Geometry Import' dialog boxes. The left dialog is in the 'Basic' tab, and the right dialog is in the 'Advanced' tab. Both dialogs have a 'File Name' field set to 'D:\GeometryImport\Battery\_Ansys\M3\_SHCS.prt'.

**Basic Tab (Left Dialog):**

Property	Value
Solid Bodies	Yes
Surface Bodies	Yes
Line Bodies	No
Attributes	No
Named Selections	No
Material Properties	No

**Advanced Tab (Right Dialog):**

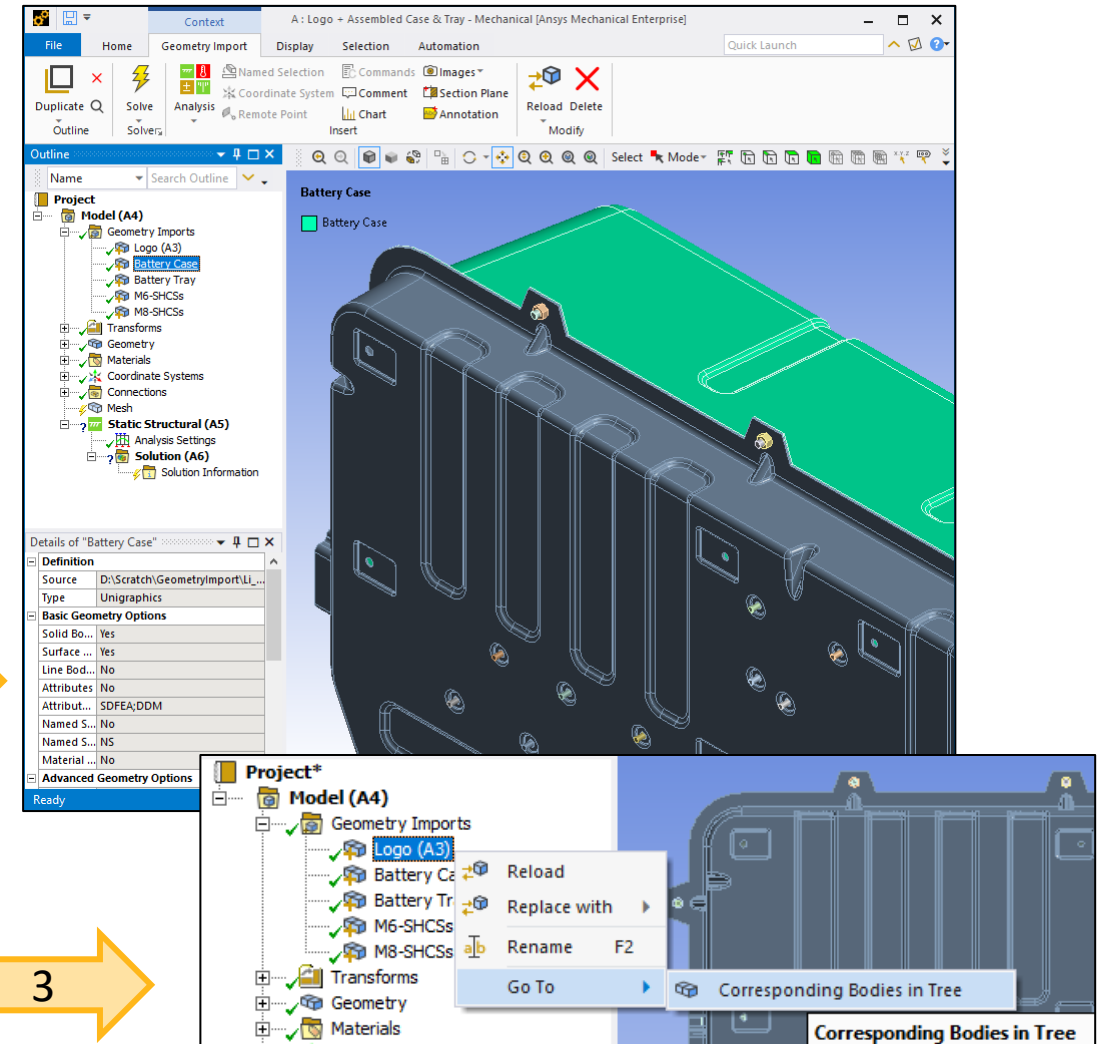
Property	Value
Analysis Type	3D
Use Associativity	Yes
Coordinate Systems	No
Work Points	No
Reader Mode Saves Updated File	No
Use Instances	Yes
Smart CAD Update	Yes
Compare Parts On Update	No
Enclosure and Symmetry Processing	Yes
Decompose Disjoint Geometry	Yes
Mixed Import Resolution	None
Clean Bodies On Import	No
Stitch Surfaces On Import	None
Import Facet Quality	Source

# Geometry Import

1. Selecting a **Geometry Import** objects highlights the corresponding bodies
2. Information about each Geometry Import, such as the source file and CAD preferences, are shown in the details view
3. The corresponding bodies in the Tree can be navigated to via the RMB menu "Go To > Corresponding Bodies in Tree"

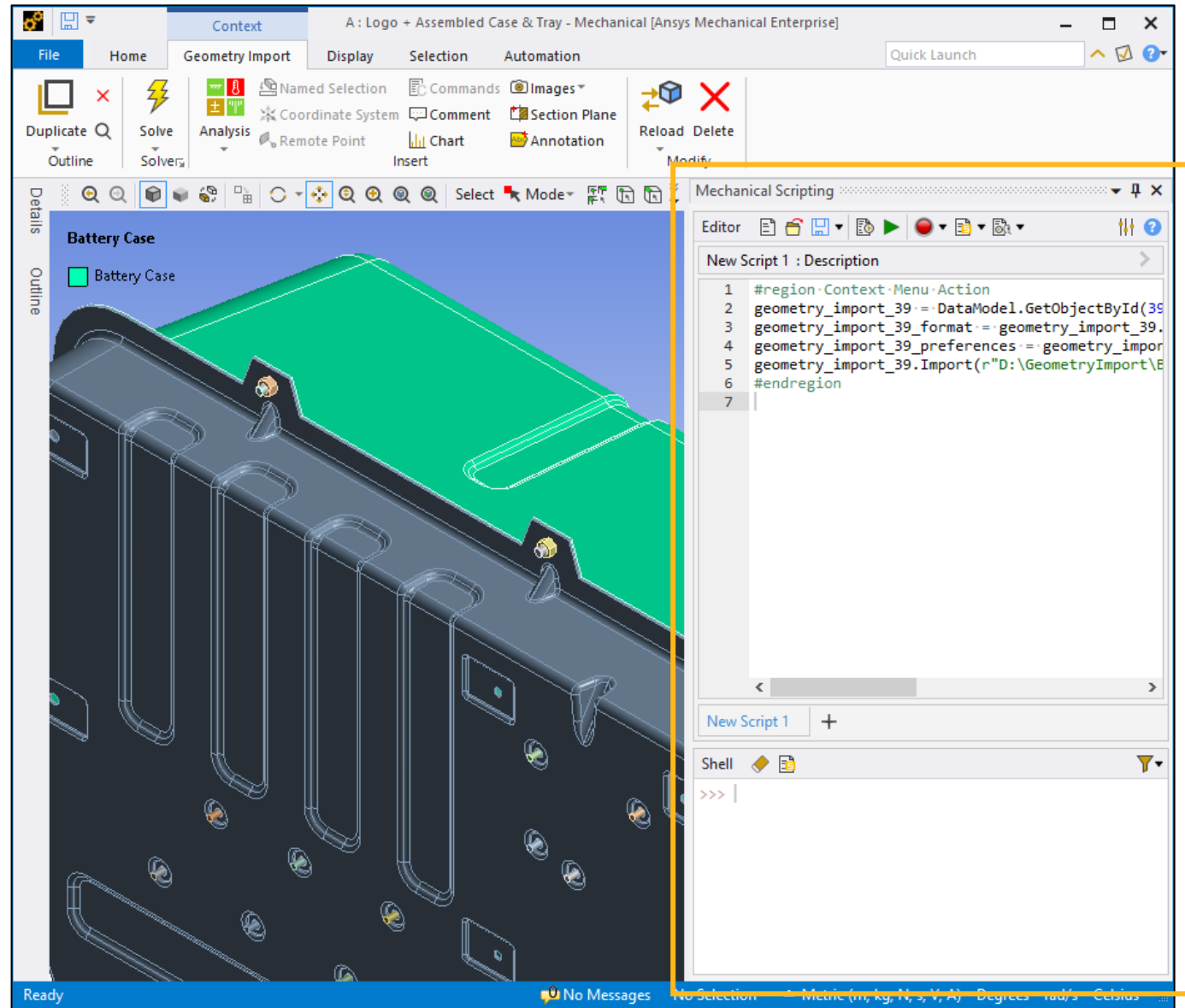
## Pro Tip

Use "Go To > Corresponding Bodies in Tree", then create a group with the selected parts, to easily organize parts under "Geometry" by import



# Geometry Import

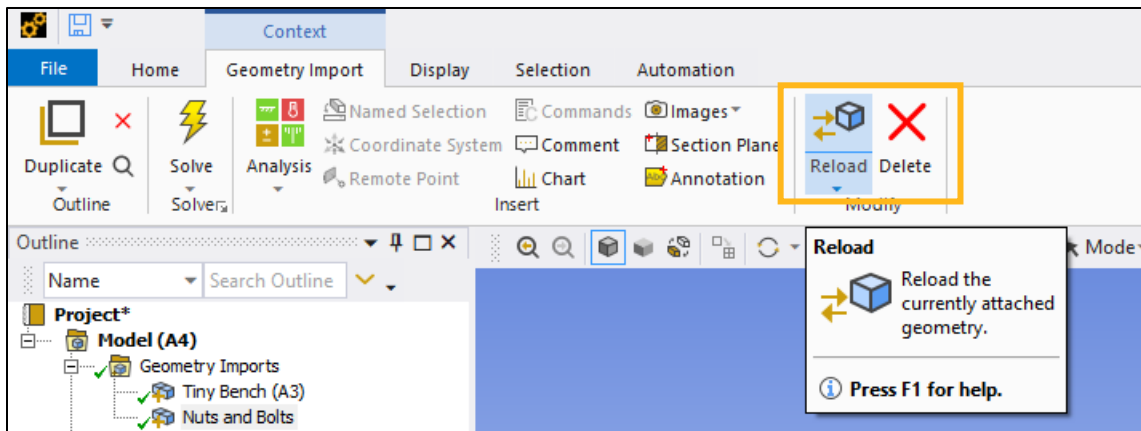
- Geometry Import is a fully-recorded feature
- The included Geometry Import Python API enables powerful, scriptable operations





# Geometry Import

- Geometry sources can easily be reloaded or replaced via the “Reload” button in the Geometry Import Ribbon Context menu
- Secondary sources can also be deleted. Deleting a Geometry Import object deletes the corresponding parts



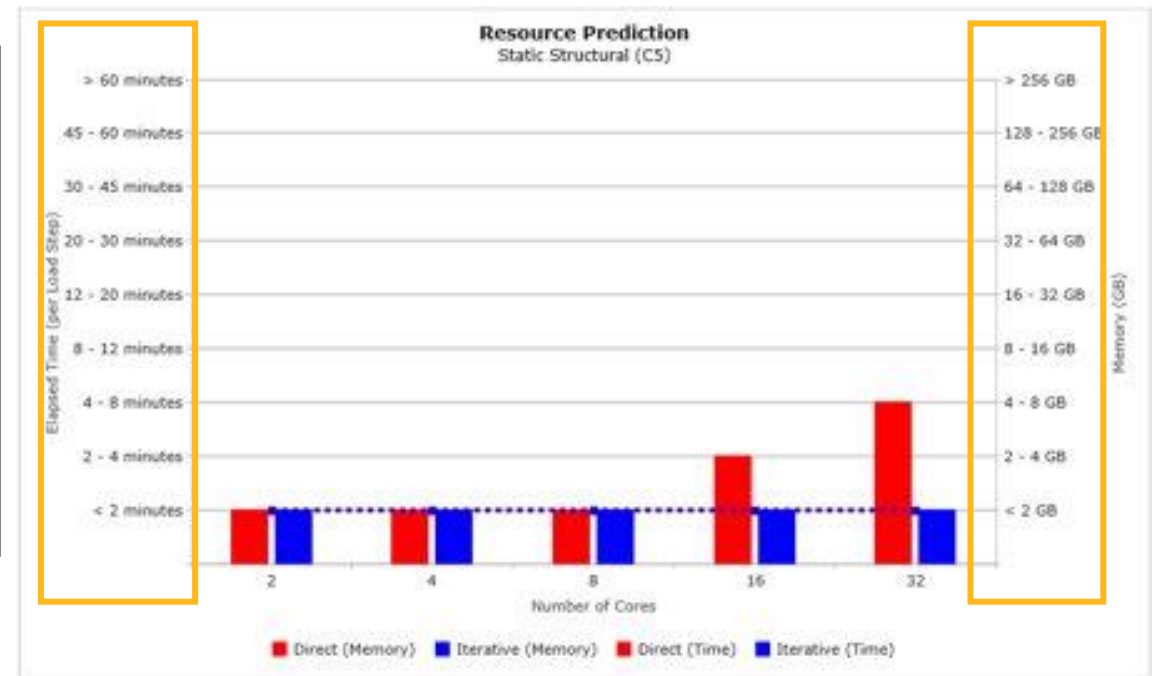
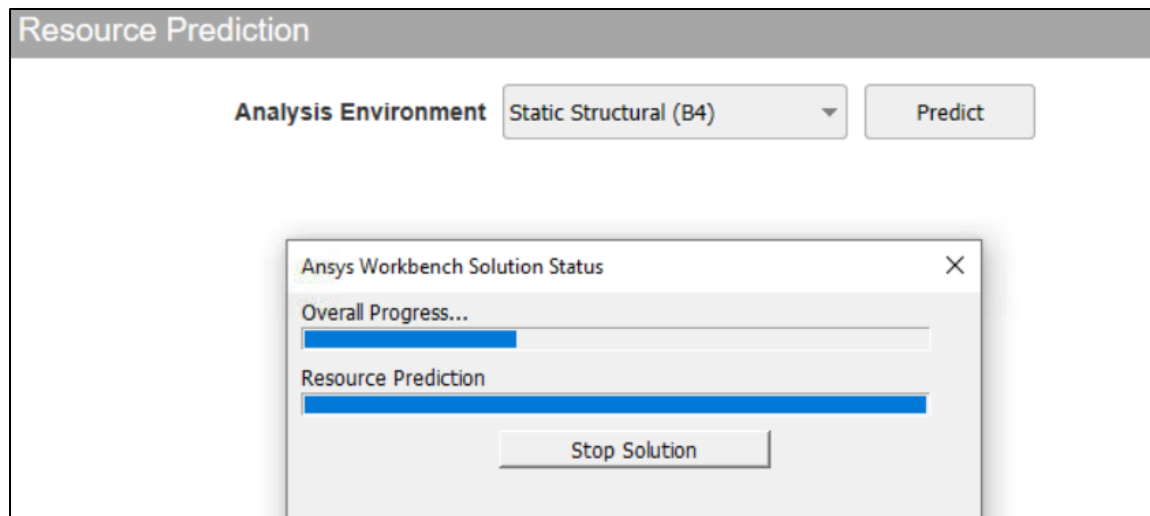
## Pro Tip

Click the top-half of the Ribbon Context "Reload" button to automatically reuse the same file and CAD preferences



# Resource Prediction Enhancements

- Resource Prediction is now run on a separate thread and progress is updated. The default for memory prediction is changed from Random Forest to Neural network algorithm
- With Beta options enabled:  
The number of bins for memory and time is increased to 9 bins. This increase in number of bins helps to better estimate the memory and time requirements for the simulation

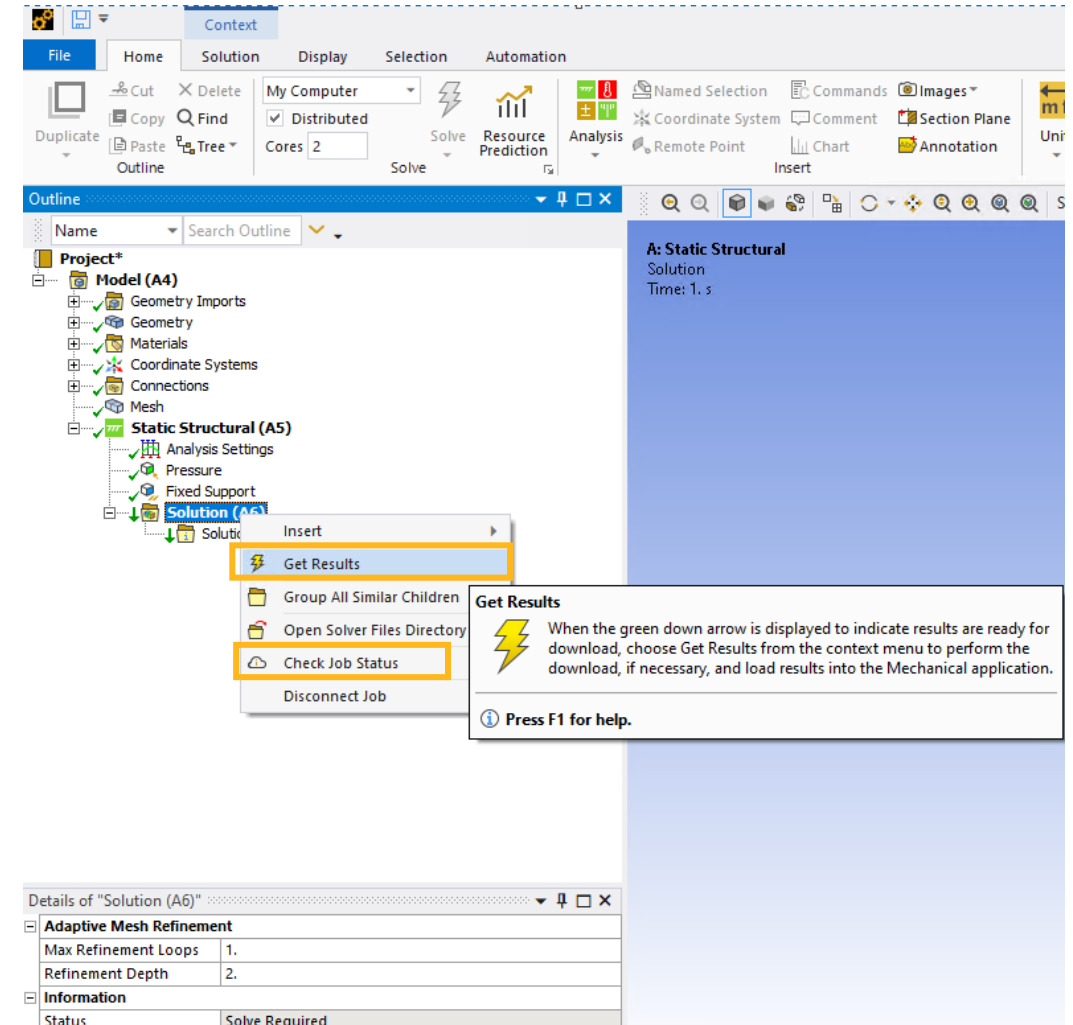
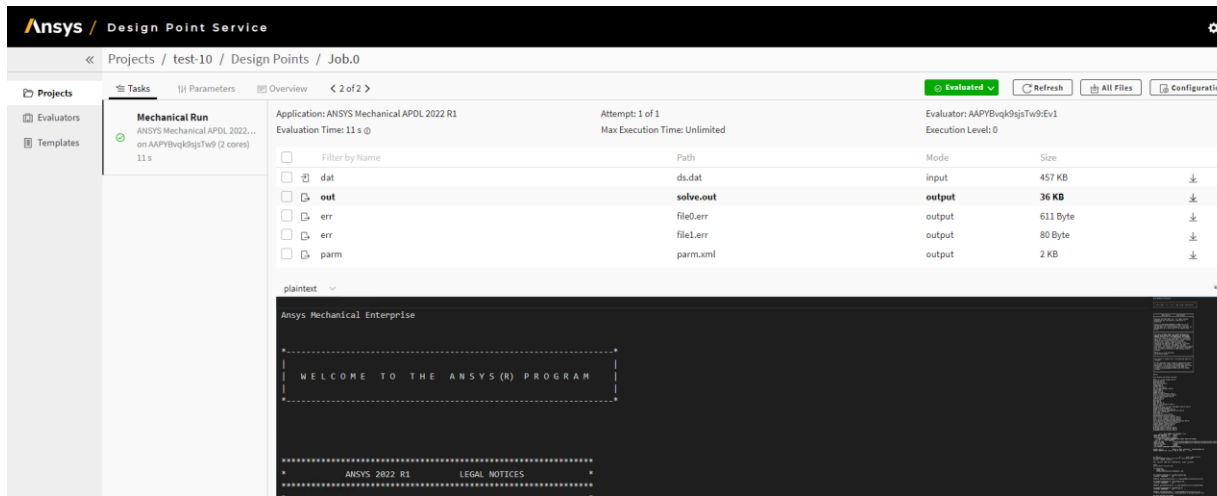


# Core Capabilities of DCS (Distributed Compute Services):

- One or many design points of a Workbench project can be submitted to DPS and solved on an evaluator
- A solution update in Mechanical can be submitted to DPS and solved on an evaluator
- A Mechanical APDL input file can be uploaded directly in DPS and solved on an evaluator
- An LS-DYNA input file can be uploaded directly in DPS and solved on an evaluator
- optiSLang can be used to run optimizations and parameter and sensitivity studies on DPS projects
- Arbitrary clients can interact with DPS
- With the DCS Python API, a custom workflow of an arbitrary executable and a set of inputs can be uploaded to DPS and run on an evaluator

# / DCS (Distributed Compute Services)

- Under “Solver Process Settings”, add a queue and select the target as “DCS”
- To check the status of the job, right click on the solution and select “Check Job Status”
- Once evaluation is done, Mechanical is ready to get the results
- Also review job status on DCS Web UI



# / Workflow to Send APIP Data: Collaboration with RSM

- Previously when jobs were remotely launched through RSM for Mechanical, we were unable to send the APIP data generated in remote nodes to Ansys Analytics Server, because of no internet connection.
- From 2022 R1 the workflow is changed when launching jobs remotely using RSM:
  - APIP executable is launched by RSM after solver has finished running job.
  - APIP executable tries to send already sitting APIP data in HPC cluster. If it fails to send from HPC Cluster (eg: no internet connection), moves the files from app data to HPC staging directory.
  - RSM then moves the files from HPC staging directory to client's project scratch.
  - Files moved from client's solver directory to client's %appdata% by Mechanical.

Note: This is not a Mechanical feature. This enables better APIP data management for RSM workflows

# / Performance and Disk Space Improvements using /fclean

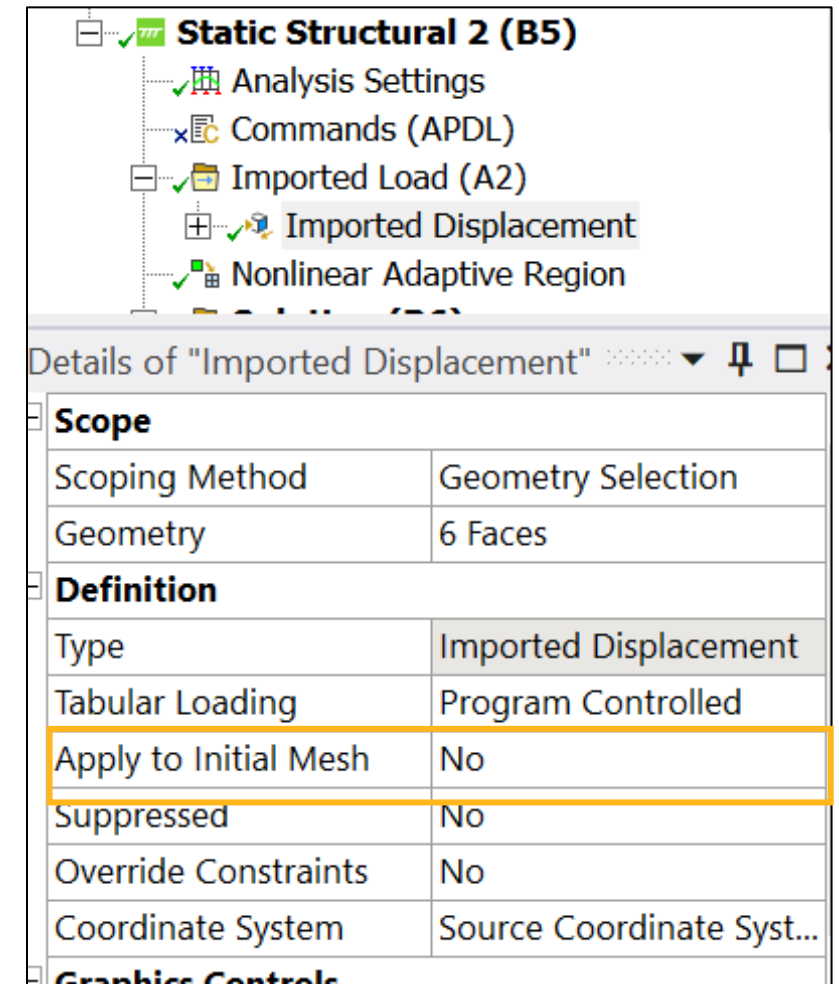
- Performance and disk space improvements by issuing /FCLEAN command which will delete the solution generated redundant distributed files in a distributed parallel processing run. This reduces the disk space to up to half. This gives additional benefit to the users by avoiding the file download when run on a remote machine using RSM
- The files are not cleaned using /FCLEAN command, when files are needed for distributed solution of downstream linked analysis, or when the solution does not run to completion for the existing distributed run

# / NLAD: Apply Loads to Initial Mesh

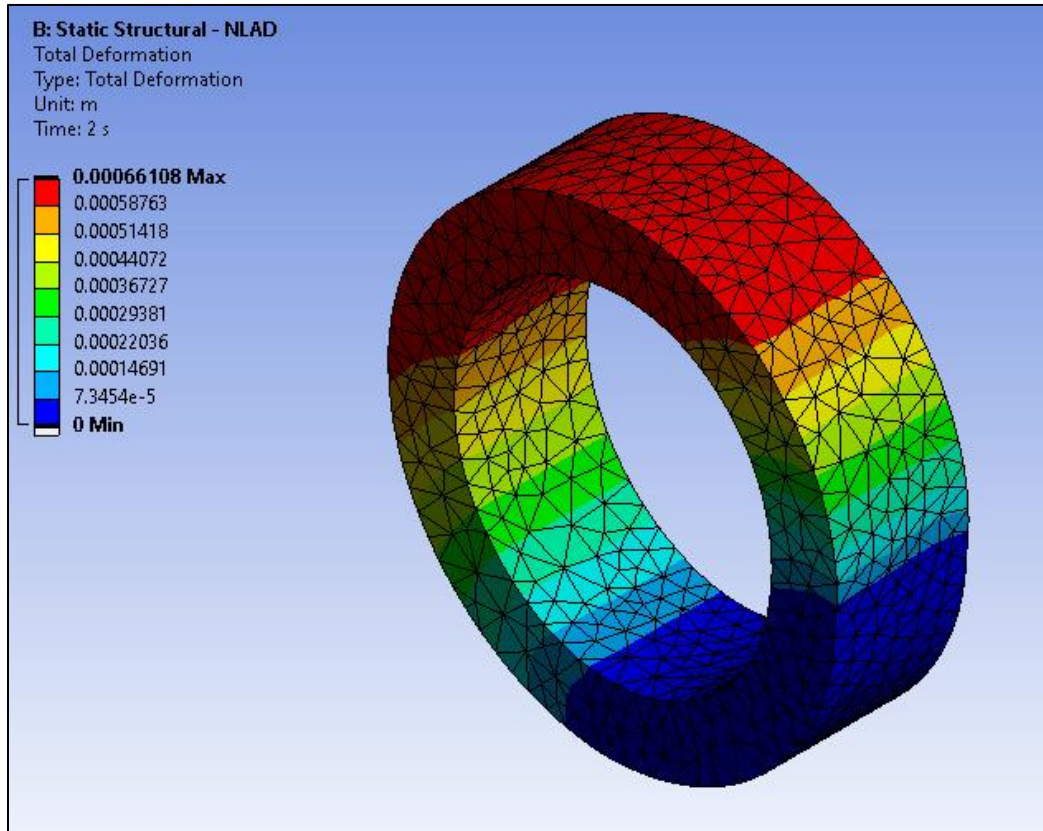
- In the presence of non-linear adaptive region, this property enables you to apply the imported loading and constraint directly on the initial mesh for each load step even when re-mesh occurs. This property enables you to map the loads to initial mesh

## Supported Loads

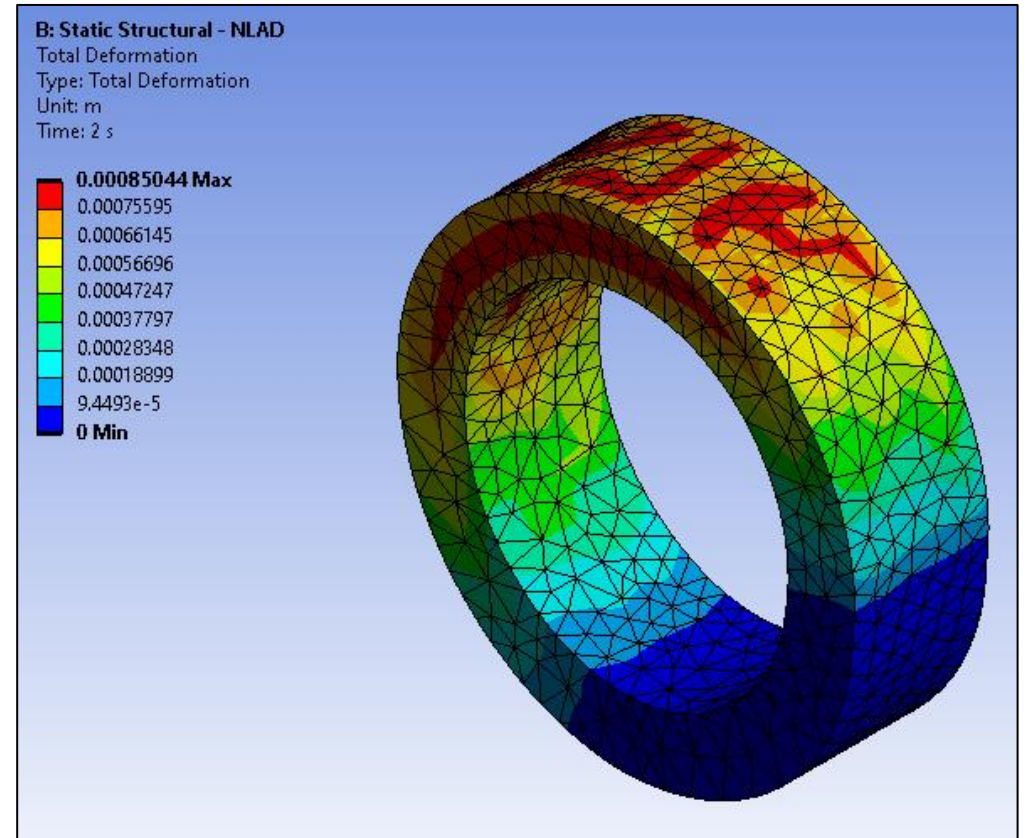
- Imported Displacement
- Imported Body temperature



# / NLAD: Apply Loads to Initial Mesh - Results



Apply To Initial Mesh set to “Yes”



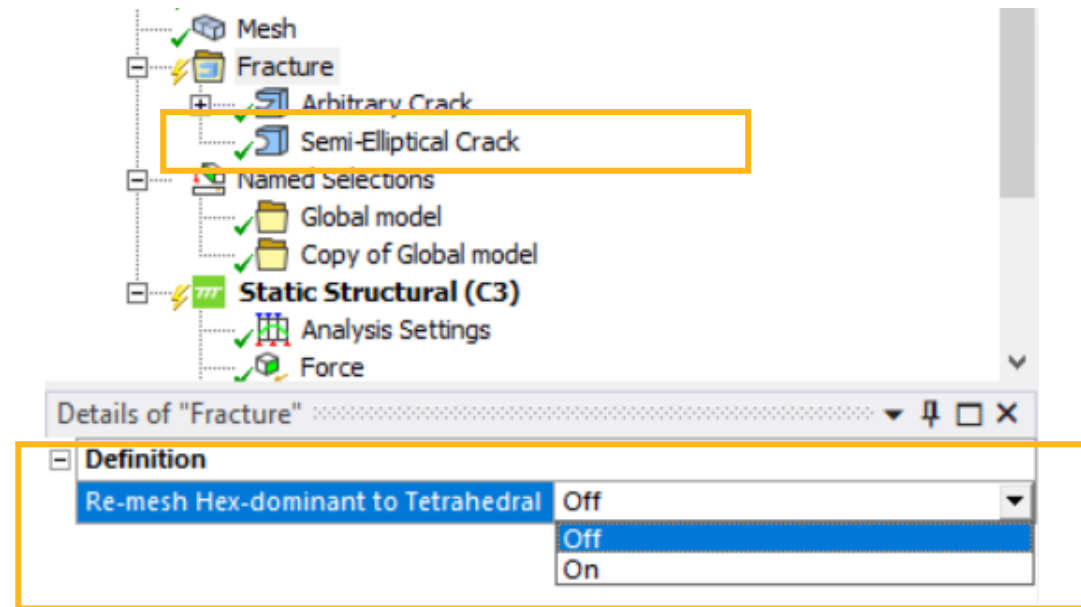
Apply To Initial Mesh set to “No”

Loads are mapped to initial mesh



# Fracture Enhancements

- **Semi-Elliptical Crack** mesh generation on an imported base mesh
- We can automatically re-mesh an imported hex or hex-dominant base mesh as a tetrahedral mesh (as required by the **Semi-Elliptical Crack**) using the **Re-mesh Hex-dominant to Tetrahedral** property of **Fracture** folder





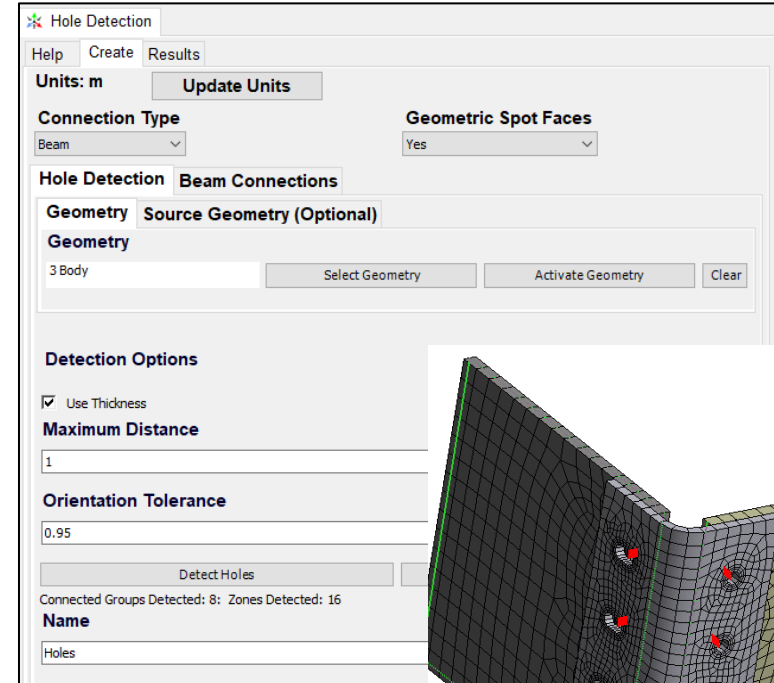
# Bolt Tool

Bolted Joint Simulation



# / Hole Detection/Connection Creation (Surface Bodies)

- Auto hole detection
  - Surface bodies only
- Creation of connections automatically
  - Creates a “spot face” based on mesh inflation
  - Connections via beams or contacts
  - Create mesh controls around hole entities
- Reduces/Eliminates the need for geometric imprints, and tedious creation of multiple connections manually.
- More robust than Object Generator



# / Reaction Probe Wizard

- Auto creation and export of reaction probes for multiple locations
- Allows users to easily export lots of force/moment data for connections to spreadsheet for further processing
- Auto creation of local CS for contextual results
- Multiple scoping/creation methods
  - Contacts
  - Construction surfaces
  - Geometry Faces

The screenshot shows the 'Reaction Probes' wizard interface. It has a top bar with tabs: 'Help', 'Create', 'Select', and 'Reporting'. The 'Create' tab is active. The interface is divided into several sections:

- Units:** Set to 'm'.
- Selection Type:** A dropdown menu showing 'Bodies+Construction Surfaces'.
- Bolt Body Geometry:** Includes buttons for 'Select Geometry', 'Activate Geometry', and 'Clear'.
- Axial Location:** A text input field containing 'MidShank'.
- Analyses:** Includes buttons for 'Select Objects', 'Add To Objects', 'Activate Objects', and 'Clear'.
- Time Points:** A text input field containing 'END'.
- Creation Options:** A section with checkboxes for 'Forces' (checked), 'Moments' (checked), 'Axial Stress' (unchecked), and 'Orient to Local CS' (checked). Below these are tabs for 'Probe Naming', 'Coord. Sys. Naming', and 'Folder Name'.
- Probe Naming:** A text input field showing the template '<Type> - <BodyName> - <Time>'.
- Action Options:** A section with checkboxes for 'Evaluating All Results' (checked), 'Export Summary File' (checked), and 'Group Objects' (checked).

# External Model

# Abaqus: Support Bolt Pretensions on Beam Elements

**Bolt Pretensions(External Model)**

**\*PRE-TENSION SECTION, ELEMENT=8292, NODE=18124**

```
*NSET, NSET=nset_pretnode
18125
*ELSET, ELSET=elset_pretelem
8292
*PRE-TENSION SECTION, ELEMENT=elset_pretelem, NODE=nset_pretnode
```

**Worksheet**

Check/Uncheck	ID	Pretension Node ID	Scoping
<input checked="" type="checkbox"/>	1	18124	Element set 0 (count = 1)
<input checked="" type="checkbox"/>	2	18125	ELSET_PRETELEM(External Model)

Show 2 Showing 1 - 2 of 2 Previous Page 1 of 1 Next

# / Abaqus: Support Bolt Pretensions on Beam Elements

- Connections can be applied to Beam nodes used for Bolt Pretension

**Project\***

- Model (B3, C3)
  - Import Summary
  - Geometry Imports
  - Geometry
  - Materials
  - Cross Sections
  - Coordinate Systems
  - Connections
    - Imported
      - Rigid Remote Connectors(External Model)**
      - Contacts(External Model)
      - Bolt Pretensions(External Model)
  - Mesh
  - Named Selections
  - Boundary Conditions
  - Constraints(External Model)
  - Nodal Loads(External Model)
  - Static Structural (C4)
    - Analysis Settings
    - Solution (C5)**
      - Solution Information

**Details of "Rigid Remote Connectors(External Model)"**

**Definition**

Suppressed: No

**Graphics Properties**

Kinematic\_Coupling

Show Rows: From Current Page

**Transfer Properties**

Source: A2::External Model

Read Only: Yes

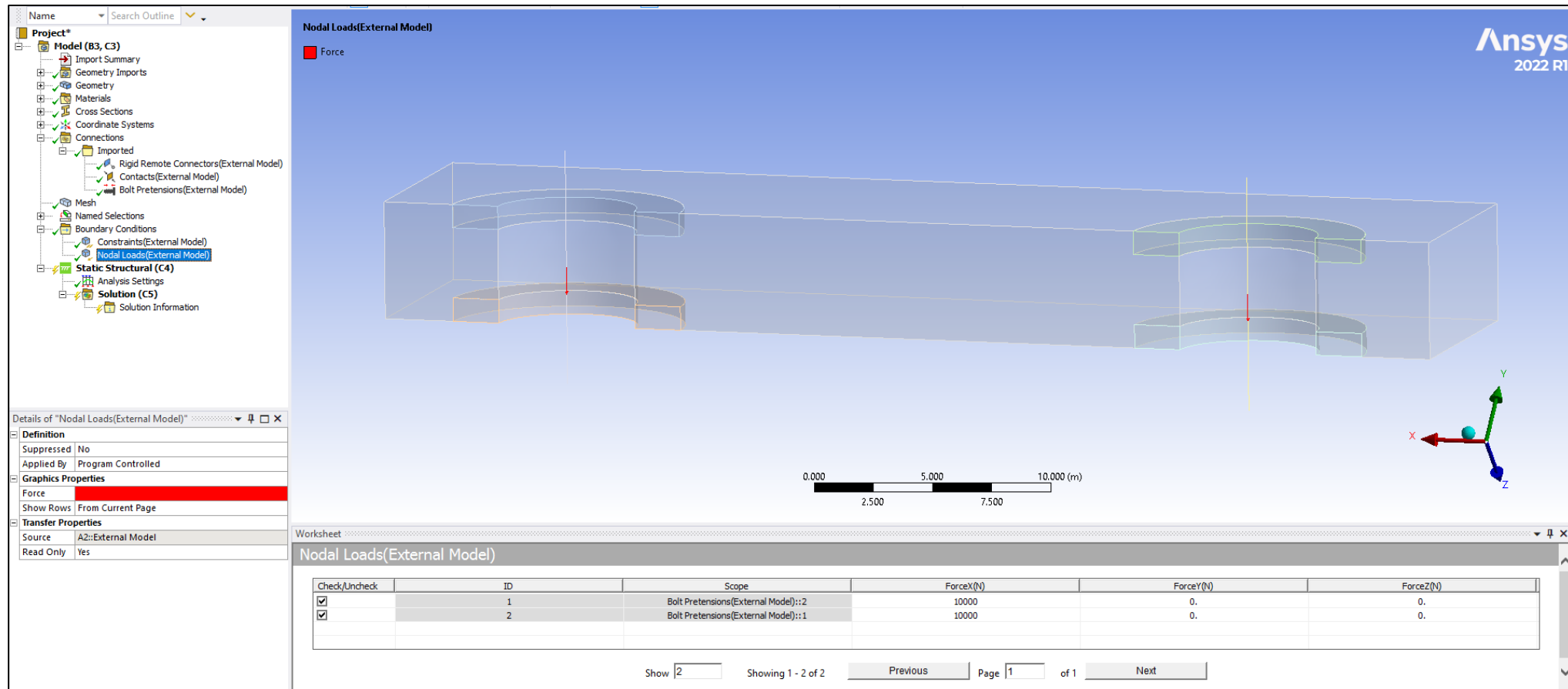
**Rigid Remote Connectors(External Model)**

Check/Uncheck	Type	ID	Reference Node Id	DOFs	Participating nodes
<input checked="" type="checkbox"/>	Kinematic_Coupling	1	4090	ux; uy; uz; rot x; rot y; rot z	Count = 693
<input checked="" type="checkbox"/>	Kinematic_Coupling	2	4091	ux; uy; uz; rot x; rot y; rot z	Count = 693
<input checked="" type="checkbox"/>	Kinematic_Coupling	3	18121	ux; uy; uz; rot x; rot y; rot z	Count = 693
<input checked="" type="checkbox"/>	Kinematic_Coupling	4	18122	ux; uy; uz; rot x; rot y; rot z	Count = 693

Show 4 Showing 1 - 4 of 4 Previous Page 1 of 1 Next

# / Abaqus: Support Bolt Pretensions on Beam Elements

- Loads can be applied to the Beam Pretension nodes



The screenshot displays the Abaqus/CAE software interface. The left-hand pane shows the 'Project' tree with the following structure:

- Project\*
- Model (B3, C3)
  - Import Summary
  - Geometry Imports
  - Geometry
  - Materials
  - Cross Sections
  - Coordinate Systems
  - Connections
    - Imported
    - Rigid Remote Connectors(External Model)
    - Contacts(External Model)
    - Bolt Pretensions(External Model)
  - Mesh
  - Named Selections
  - Boundary Conditions
  - Constraints(External Model)
  - Nodal Loads(External Model)
- Static Structural (C4)
  - Analysis Settings
  - Solution (C5)
    - Solution Information

The main viewport shows a 3D model of a mechanical assembly with a coordinate system (X, Y, Z) and a scale bar (0.000 to 10.000 m). The 'Nodal Loads(External Model)' section is active, showing a 'Force' load applied to the model.

The 'Details of "Nodal Loads(External Model)"' pane shows the following properties:

- Definition
  - Suppressed: No
  - Applied By: Program Controlled
- Graphics Properties
  - Force: Force
  - Show Rows: From Current Page
- Transfer Properties
  - Source: A2:External Model
  - Read Only: Yes

The 'Worksheet' pane displays the following table:

Check/Uncheck	ID	Scope	ForceX(N)	ForceY(N)	ForceZ(N)
<input checked="" type="checkbox"/>	1	Bolt Pretensions(External Model)::2	10000	0.	0.
<input checked="" type="checkbox"/>	2	Bolt Pretensions(External Model)::1	10000	0.	0.

At the bottom of the worksheet, it shows 'Show 2', 'Showing 1 - 2 of 2', 'Previous', 'Page 1 of 1', and 'Next'.

# / Abaqus: Support \*HYPERELASTIC Keyword for Materials

The following mutually exclusive parameters (material models) are supported:

- MOONEY-RIVLIN
- NEO HOOKE: This parameter is equivalent to \*HYPERELASTIC, REDUCED POLYNOMIAL and as a result, is imported as Yeoh 1st Order.
- OGDEN: Supports parameter N = 1 (default), 2, or 3.
- POLYNOMIAL (default): Supports parameter N = 1 (default), 2, or 3.
- REDUCED POLYNOMIAL: Supports parameter N = 1 (default), 2, or 3.
- YEOH

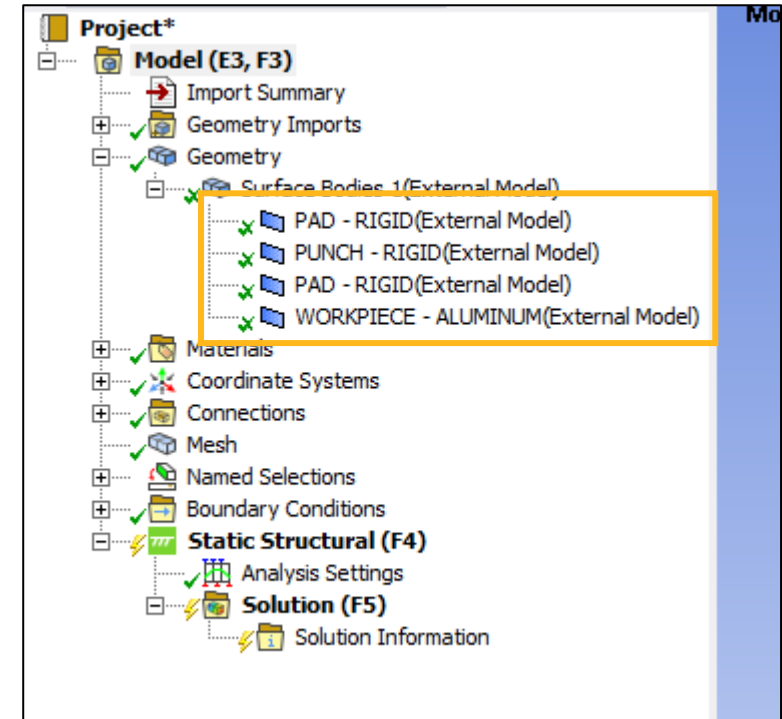
**Limitation:** Temperature dependence is not supported



# LS-DYNA: Set Body Names Using \*PART Names

- Imported Bodies are named using \*PART names from the .k input files

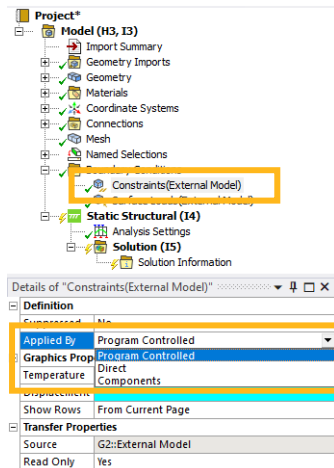
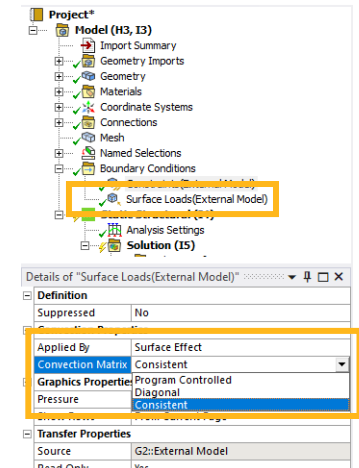
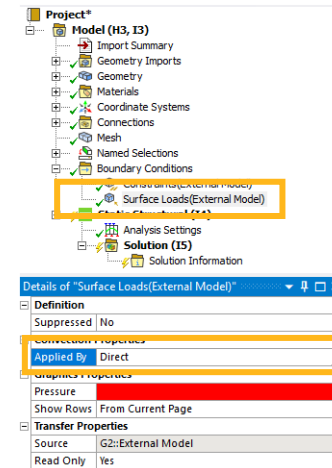
```
*PART
$ c
$ head
PUNCH - RIGID
$
$ pid sid mid eosid hgid adpopt
$ 1 2 2 0 0 0
*PART
$ c
$ head
PAD - RIGID
$
$ pid sid mid eosid hgid adpopt
$ 2 2 2 0 0 0
*PART
$ c
$ head
WORKPIECE - ALUMINUM
$
$ pid sid mid eosid hgid adpopt
$ 3 1 1 0 0 0
*PART
$ c
$ head
PAD - RIGID
$
$ pid sid mid eosid hgid adpopt
$ 4 2 2 0 0 0
```



# New Imported Boundary Conditions Options

## Thermal convection

- Surface Loads object now has 2 new properties for "Applied By": Surface Effect or Direct
  - This specifies how to send the thermal convections to the solver file
  - "Surface Effect" has the options:
    - Convection Matrix: Program Controlled, Diagonal or Consistent
    - This specifies which film coefficient to use for the convection loads



## Imported Nodal Loads and Constraints

- New property available:
- Definition → Applied By: Program Controlled, Direct or Components
  - This specifies how to send the nodal loads to the solver file

# Meshing



# New Features

- **Shell Meshing**

- Tri Reduction
- Support for new metric visualization:
  - Warping angle
  - Min quad/tri angle
  - Max quad/tri angle
  - Min/Max Element Edge Length

- **Welds Meshing**

- Enhancements to Weld Worksheet
- Usability and error handling improvements
- Visualisation improvements
- HAZ layer Named Selections

- **Pull**

- Line Coating
- Quadratic mesh support

- **Explicit Physics Preference**

- New defaults/behaviours for Tet meshing
- New defaults/behaviours for Hex meshing
- Quality targeting for Aspect Ratio (Explicit)
- New default metric visibility per physics preference
- Support for new metric visualization:
  - Characteristic Length (LS Dyna)
  - Aspect Ratio (Explicit)
  - Tet Collapse

- **General Tet Meshing**

- Improved robustness of defeaturing
- Improved Error/Warning messages
- Feature Detection for solid holes and fillets
- Proximity Gap Factor for AR control in coarse mesh
- Diagnostics-based Named Selection Worksheet tools:
  - Intersecting surface mesh failures
  - Free edge mesh
  - Sharp angle
  - Body Interference
  - Defeatured Topology
- Model Walk extension to mesh elements/element clusters

- **Hex Meshing**

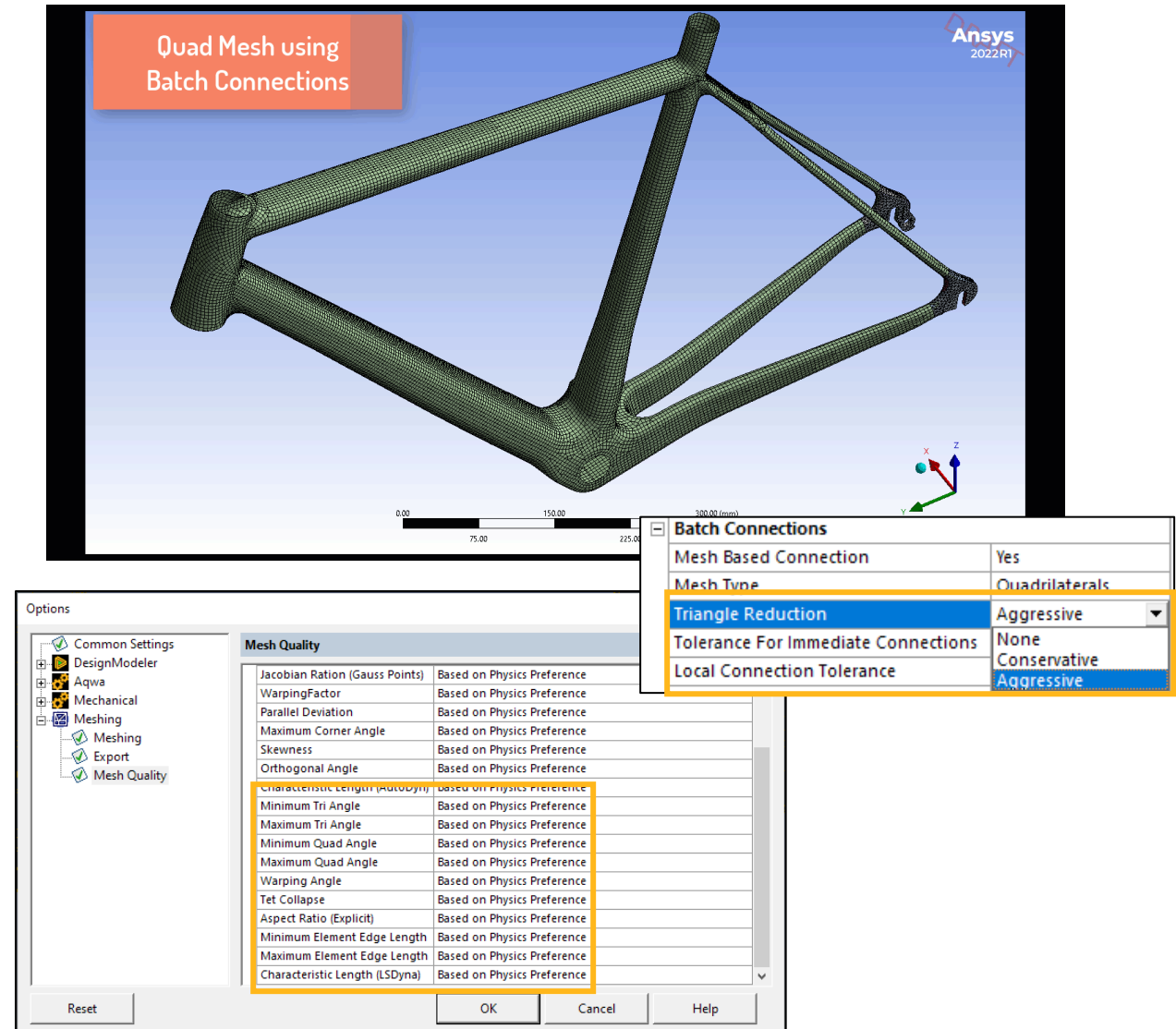
- Improved pave mesh in Multizone
- Improved default mesh with MultiZone for simple shapes e.g. cylinder, sphere, pipe, etc
- Split Angle for less decomposition
- Body-Fitted Cartesian
  - Support for Edge Sizing
- Beta Multizone Options:
  - CartSweep Decomposition (2.5D Geometry)
  - ThinSweep Decomposition (Thin Geometry)

- **SpaceClaim Meshing**

- Thin Body Meshing
- Robustness, Performance, Usability
- Meshing for Explicit Improvements

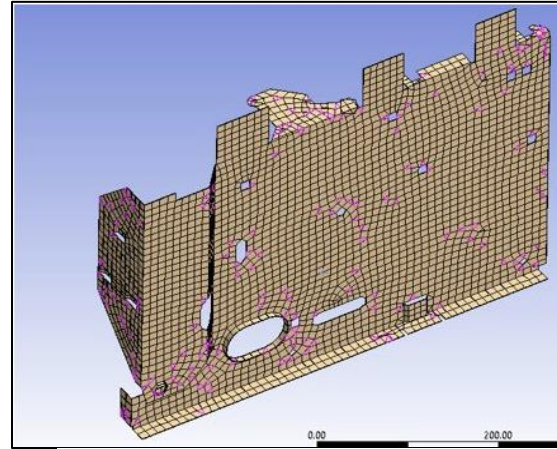
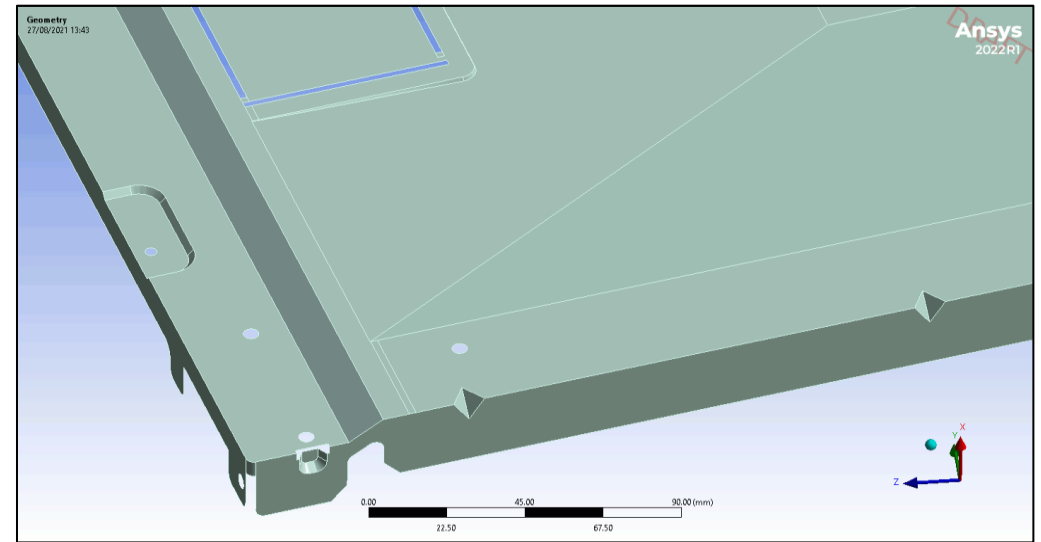
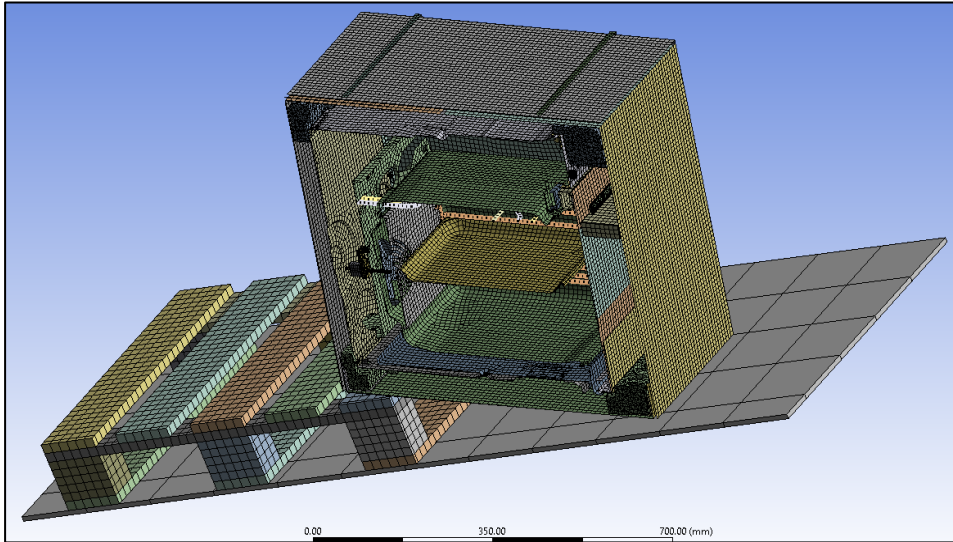
# Shell Meshing: Triangle Reduction & Quality

- Available with Batch Connections
- Option to control level of triangles in quad mesh
  - None
  - Conservative (default)
    - Remove triangles near shell edges
  - Aggressive
    - Remove as many triangles as possible sometimes at cost of quality
    - Seen to reduce tri count by up to 80% in some cases and below 1-3%
- New Quality Metrics
  - Hidden/Shown based on Physics Preference chosen
  - User can customise

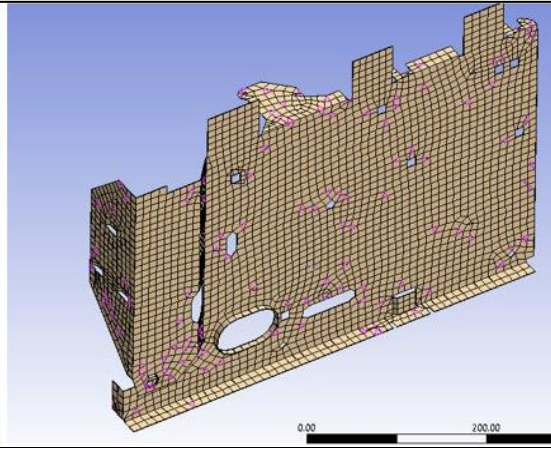




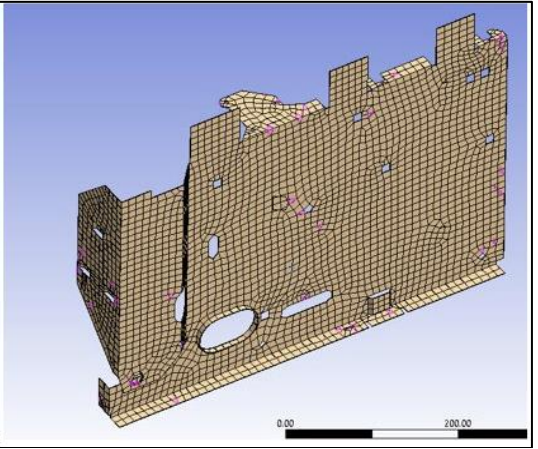
# Steamer Shell Mesh: Tri Reduction



Tri Reduction = None  
Number of tri = 184



Tri Reduction = Conservative  
Number of tri = 126



Tri Reduction = Aggressive  
Number of tri = 48

# / Terminology Changes to Align with Industry

Old Terminology	New Terminology
Tent	Angled
Extension	Normal
Tent and Extension	Normal and Angled
Seam	Continuous Seam
Skip	Intermittent Seam
Create Offset Layer	Create HAZ Layer
Number of Layer	Number of HAZ [1,2,3, No]
Offset Layer Growth Rate	HAZ Growth Rate
Offset Layer Height	HAZ Distance
Weld Width	Weld Width (Leg01)
Weld Height	Weld Height (Leg02)
Default Changes	
Weld Creation Criteria	Default change to Width Based instead of Angle Based for Creation Criteria

# Weld Meshing: Worksheet Enhancements

- Work with multiple rows from weld worksheet
  - Selection, Activation, Deactivation etc...
  - Go to selected items in tree
- Promote to weld control
  - creates new weld control with worksheet by removing selected rows from original.

Worksheet

Weld

Import Export

✓	Step	Weld Curve	Edge Mesh Size (mm)	Weld Angle (°)	Offset Layer Height (mm)	Number Of Layers
<input checked="" type="checkbox"/>	1	LineBody1	75.	45.	75.	1
<input checked="" type="checkbox"/>	2	LineBody2	75.	45.	75.	1
<input checked="" type="checkbox"/>	3	LineBody3	75.	45.	75.	1
<input checked="" type="checkbox"/>	4	LineBody4	75.	45.	75.	1
<input checked="" type="checkbox"/>	5	LineBody5	75.	45.	75.	1
<input checked="" type="checkbox"/>	6	LineBody6	75.	45.	75.	1
<input checked="" type="checkbox"/>	7	LineBody7	75.	45.	75.	1
<input checked="" type="checkbox"/>	8	LineBody8	75.	45.	75.	1
<input checked="" type="checkbox"/>	9	LineBody9	75.	45.	75.	1
<input checked="" type="checkbox"/>	10	LineBody10	75.	45.	75.	1
<input checked="" type="checkbox"/>	11	LineBody11	75.	45.	75.	1
<input checked="" type="checkbox"/>	12	LineBody12	75.	45.	75.	1

Add  
Delete  
Activate all Selections  
Deactivate all Selections  
Go To Selected Items in Tree  
Promote to Weld Control

Promote to Weld Control

Outline

Name Search Outline

Project\*

- Model (A4)
  - Geometry
  - Materials
  - Coordinate Systems
  - Connections
  - Mesh
    - Connect
    - Weld
    - Weld 2

Worksheet

Weld 2

Import Export

✓	Step	Weld Curve	Edge Mesh Size (mm)	Weld Angle (°)	Offset Layer Height (mm)	Number Of Layers
<input checked="" type="checkbox"/>	1	LineBody10	75.	45.	75.	1
<input checked="" type="checkbox"/>	2	LineBody11	75.	45.	75.	1
<input checked="" type="checkbox"/>	3	LineBody12	75.	45.	75.	1

Worksheet

Weld

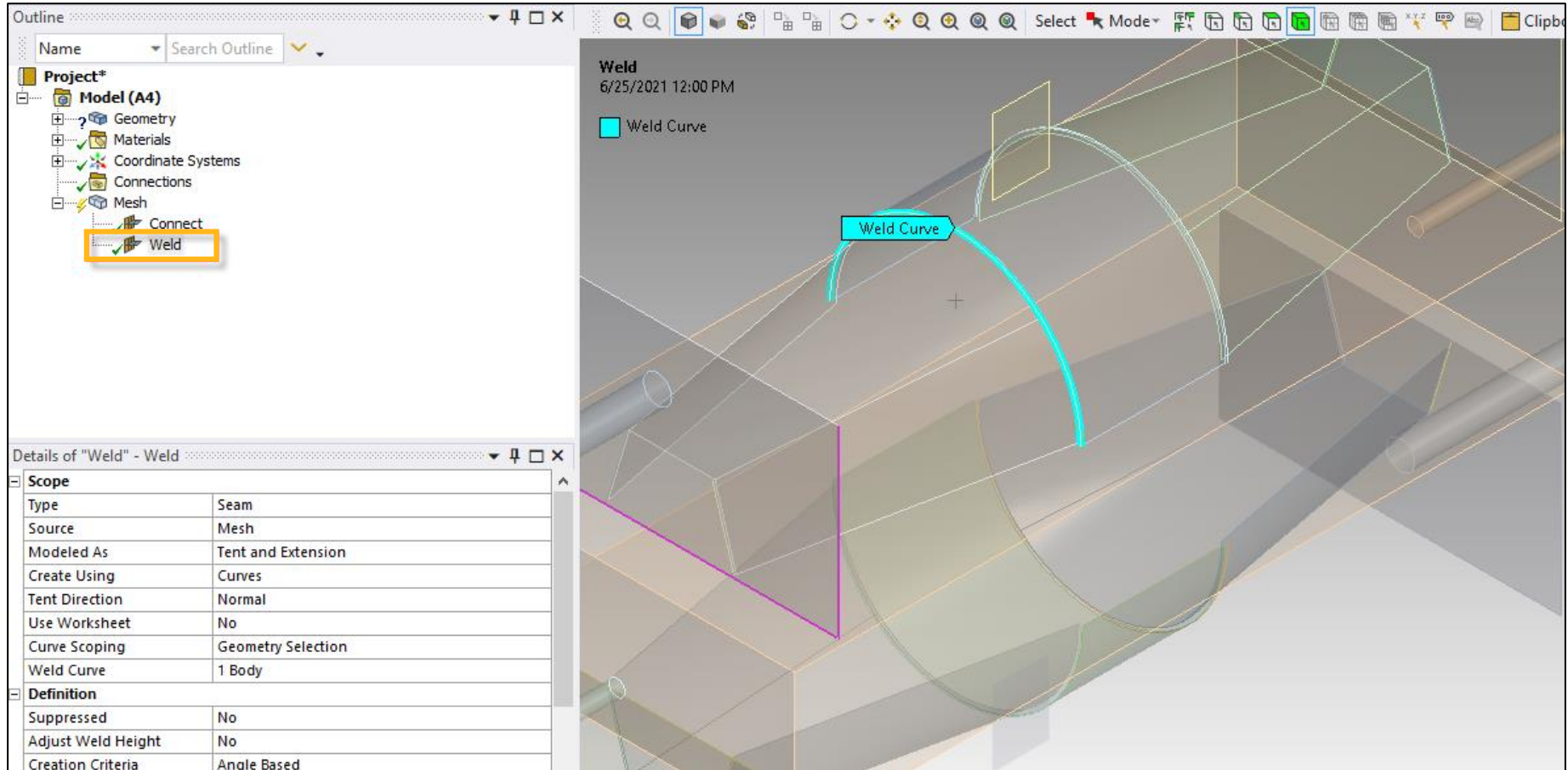
Import Export

✓	Step	Weld Curve	Edge Mesh Size (mm)	Weld Angle (°)	Offset Layer Height (mm)	Number Of Layers	Offset
<input checked="" type="checkbox"/>	1	LineBody1	75.	45.	75.	1	
<input checked="" type="checkbox"/>	2	LineBody2	75.	45.	75.	1	
<input checked="" type="checkbox"/>	3	LineBody3	75.	45.	75.	1	
<input checked="" type="checkbox"/>	4	LineBody4	75.	45.	75.	1	
<input checked="" type="checkbox"/>	5	LineBody5	75.	45.	75.	1	
<input checked="" type="checkbox"/>	6	LineBody6	75.	45.	75.	1	
<input checked="" type="checkbox"/>	7	LineBody7	75.	45.	75.	1	
<input checked="" type="checkbox"/>	8	LineBody8	75.	45.	75.	1	
<input checked="" type="checkbox"/>	9	LineBody9	75.	45.	75.	1	
<input checked="" type="checkbox"/>	10	LineBody10	75.	45.	75.	1	
<input checked="" type="checkbox"/>	11	LineBody11	75.	45.	75.	1	
<input checked="" type="checkbox"/>	12	LineBody12	75.	45.	75.	1	

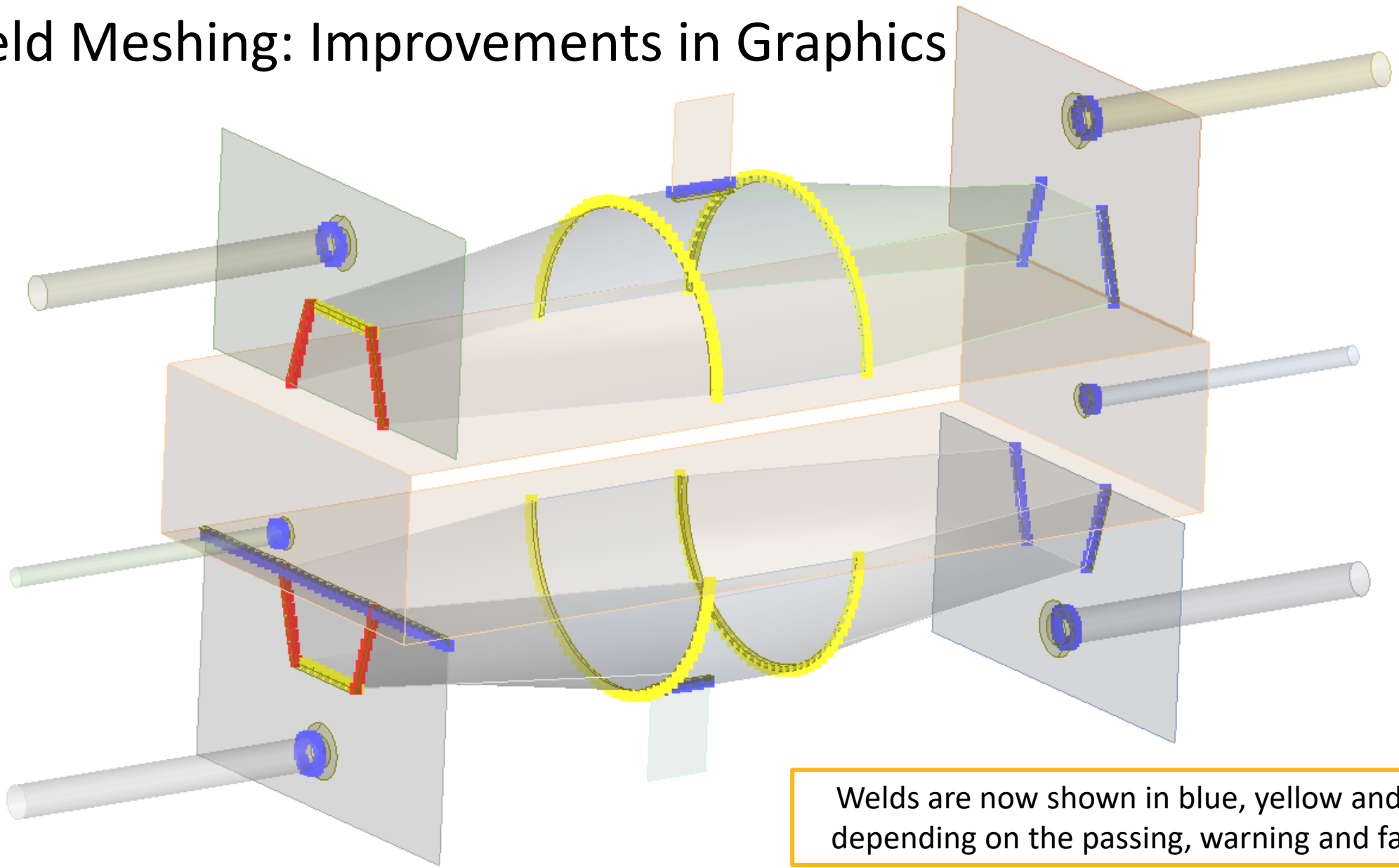


# Highlighting Weld Curve with Annotations

- Select weld control object corresponding weld curve body is highlighted with annotation



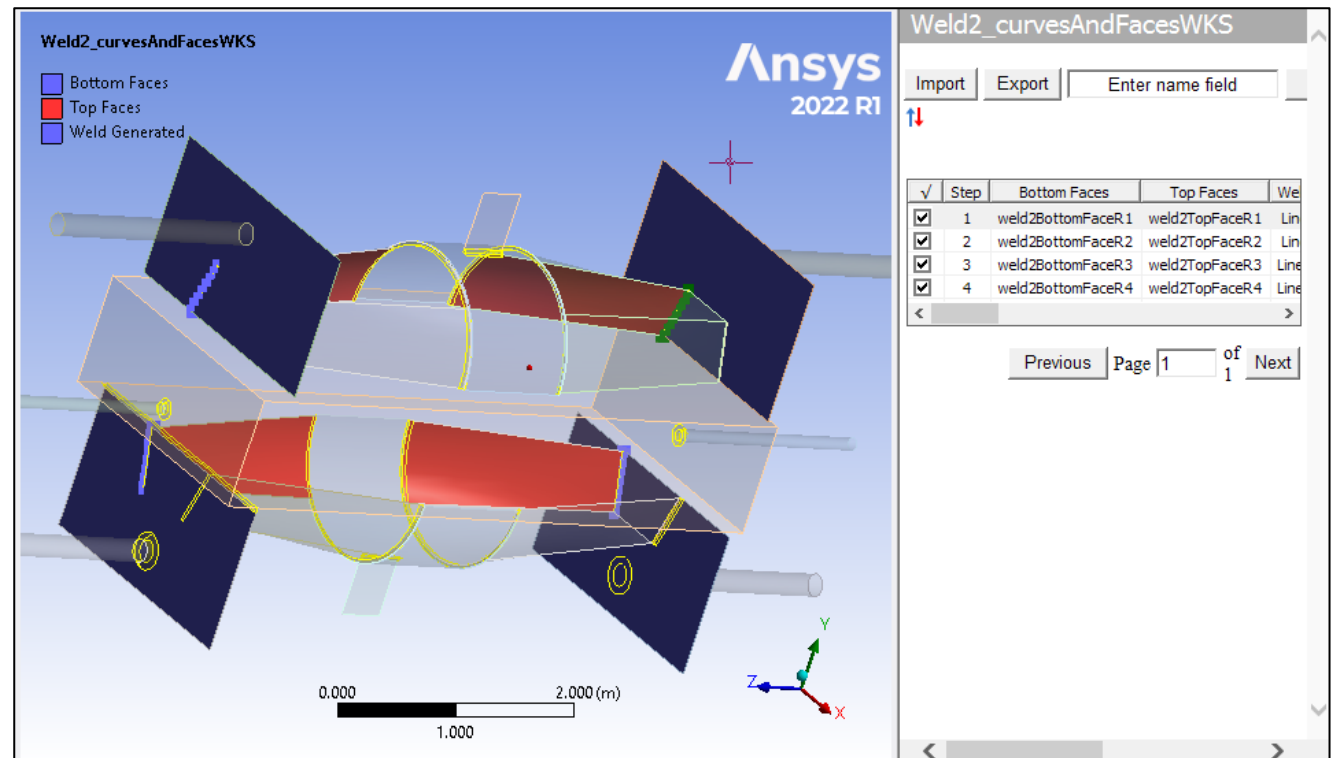
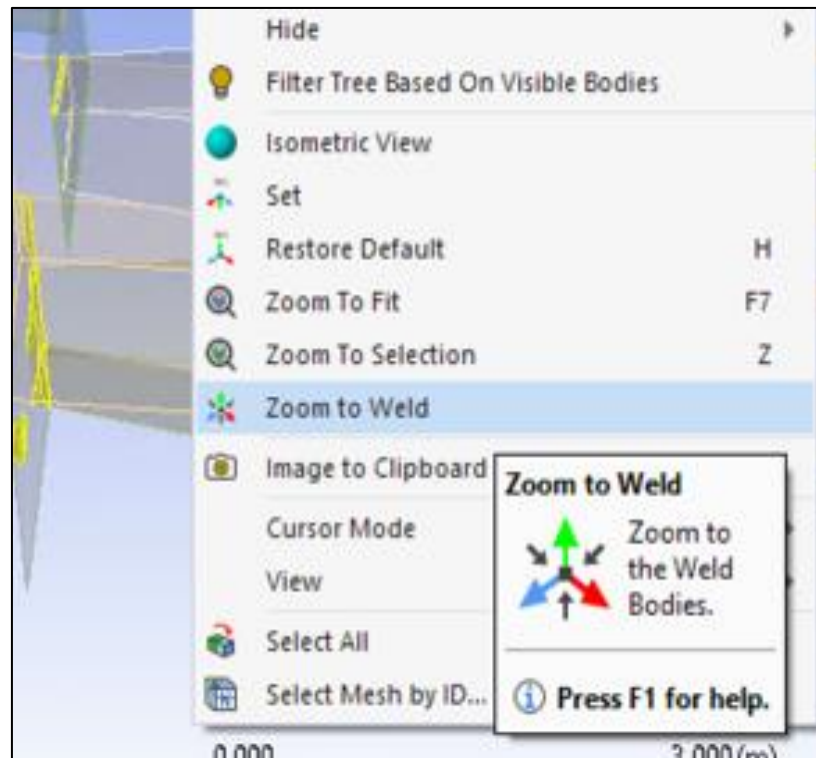
# / Weld Meshing: Improvements in Graphics



Welds are now shown in blue, yellow and red colors depending on the passing, warning and failing status

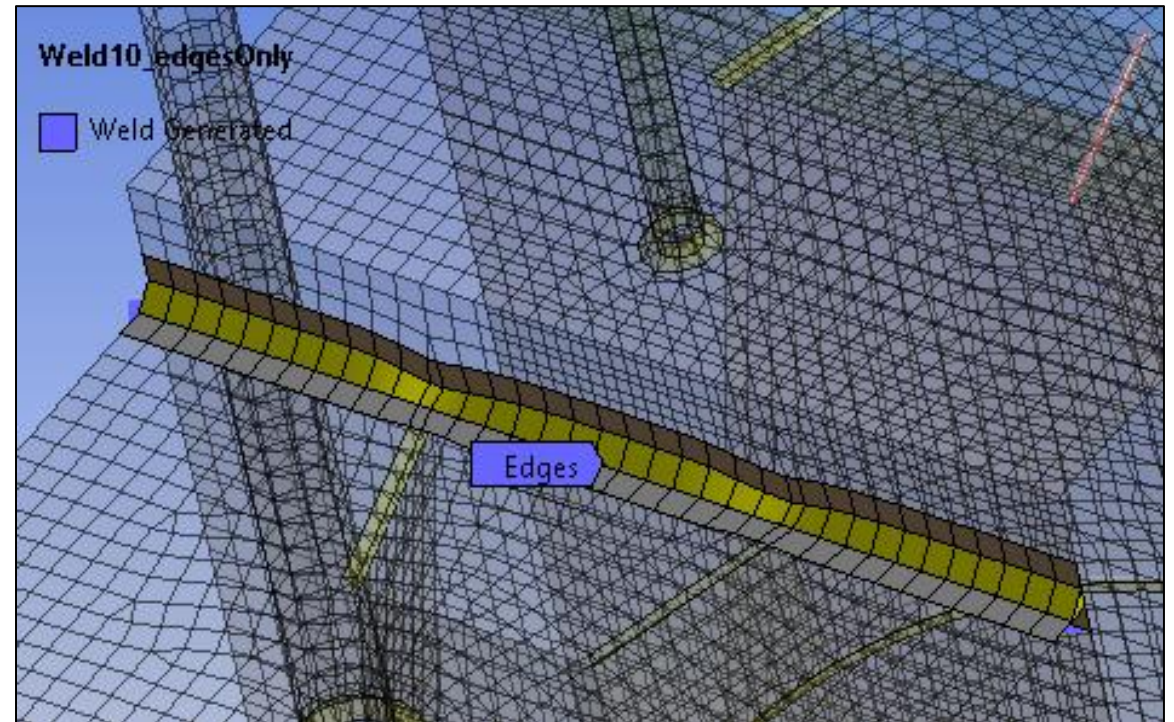
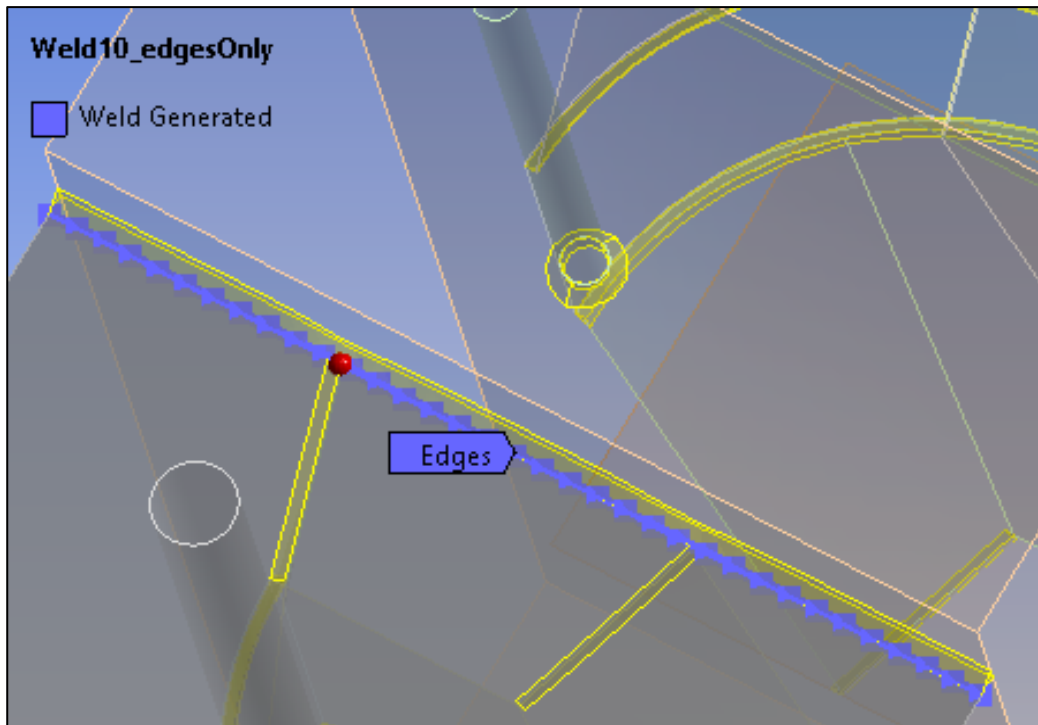
# / Weld Meshing: Improvements in Graphics

- “Zoom to Weld” option is added in the graphics context menu which zooms to fit the scoped references of the weld component. For the object using worksheet “Zoom to Weld” will be applied to the selected items in the worksheet
- For interaction with the worksheet, Picking and Highlight of the weld components can be done using the Imported Data Highlight button in the Graphics Toolbar



# / Weld Meshing: Improvements in Graphics

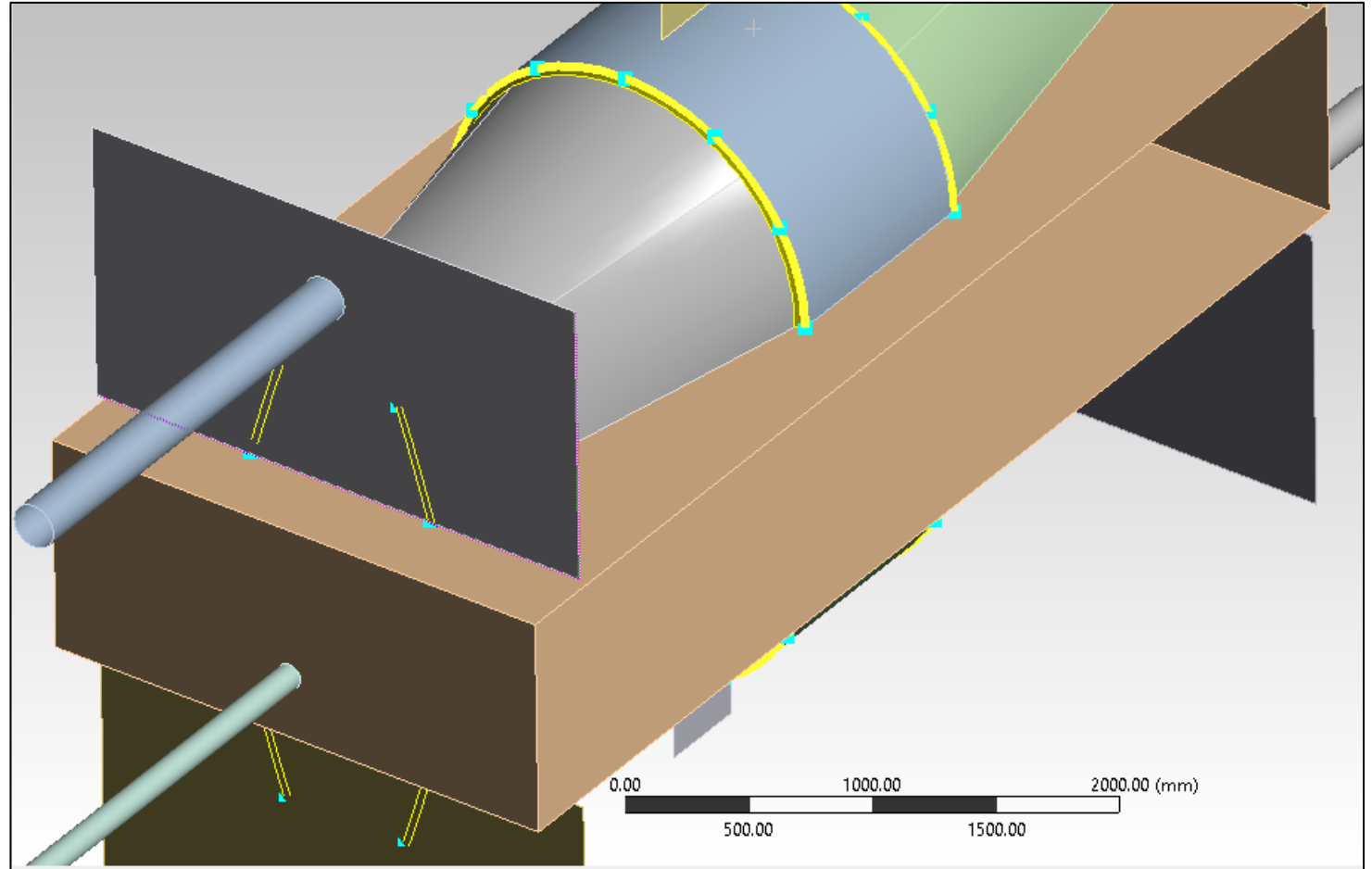
- The geometry references for different weld types scoping are highlighted based on the Mesh States. Dots along the weld edges/curves is displayed which represent the edge mesh size
- Turn on “Show Mesh” displays the Weld and HAZ layer elements as solid and other elements as translucent





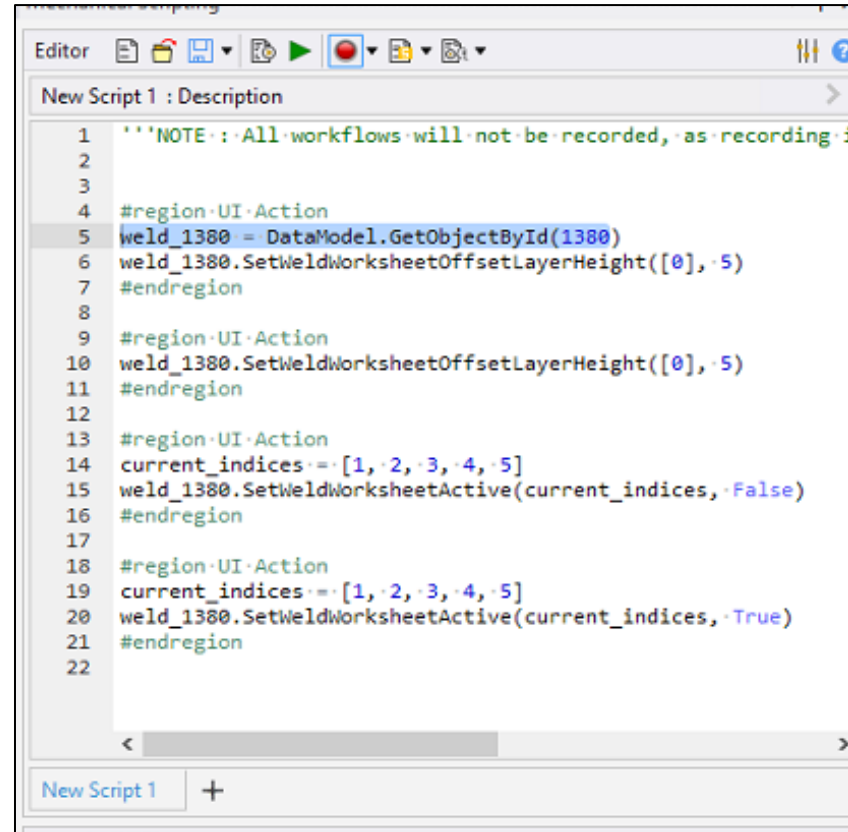
# Preview Weld

- Weld faces are created without mesh
- Weld face failure or warning messages on preview is being implemented



# / Weld Meshing: Scripting and Recording with Worksheet Editing

- All functions related to worksheet editing can now be recorded for automation



```
1  '''NOTE: All workflows will not be recorded, as recording i
2
3
4  #region UI Action
5  weld_1380 = DataModel.GetObjectById(1380)
6  weld_1380.SetWeldWorksheetOffsetLayerHeight([0], 5)
7  #endregion
8
9  #region UI Action
10 weld_1380.SetWeldWorksheetOffsetLayerHeight([0], 5)
11 #endregion
12
13 #region UI Action
14 current_indices = [1, 2, 3, 4, 5]
15 weld_1380.SetWeldWorksheetActive(current_indices, False)
16 #endregion
17
18 #region UI Action
19 current_indices = [1, 2, 3, 4, 5]
20 weld_1380.SetWeldWorksheetActive(current_indices, True)
21 #endregion
22
```

# / Weld Meshing: Editing of Multiple Cells for a Parameter Through Scripting

- Current worksheet has limitations in selecting/editing multiple cells in a column
- The new scripting capability can help us to overcome the limitation

Here is a code snippet to show how we can change offset layer height for row #2 to 6



```
current_indices_test = [1, 2, 3, 4, 5]  
weld_1380 = DataModel.GetObjectById(1380)  
weld_1380.SetWeldWorksheetOffsetLayerHeight(current_indices_test, 4)
```

# Weld Meshing: Criteria Based HAZ Named Selection

Name Search Outline

**Project\***

- Model (A4)
  - Geometry
  - Materials
  - Coordinate Systems
  - Connections
  - Mesh
    - Connect
    - Weld
    - Named Selections
    - Selection

Details of "Selection"

**Scope**

Scoping Method	Worksheet
Geometry	500 Elements

**Definition**

Send to Solver	Yes
Visible	Yes
Program Controlled Inflation	Exclude
Preserve During Solve	No

**Statistics**

Type	Manual
<input type="checkbox"/> Total Selection	500 Elements
Suppressed	0
Used by Mesh Worksheet	No

**Tolerance**

Tolerance Type	Program Controlled
Zero Tolerance	1.e-008
Relative Tolerance	1.e-003
Angular Tolerance	1. °

**Selection**  
7/28/2021 11:45 AM

Selection

0 1e+03 2e+03 3e+03 4e+03 (mm)

Worksheet

**Selection**

Generate

Note: Internal comparisons of values that have units are done in the CAD Unit System. See help for more information.  
Current CAD Unit System: Metric (m, kg, N, s, V, A)

	Action	Entity Type	Criterion	Operator	Units	Value	Lower Bound
<input checked="" type="checkbox"/>	Add	Mesh Element	Seam Weld HAZ1	Equal	N/A	All Welds	N/A



# Weld Meshing: Criteria Based HAZ Named Selection

Mesh

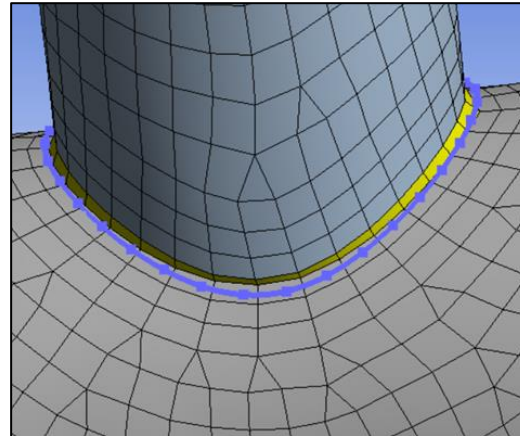
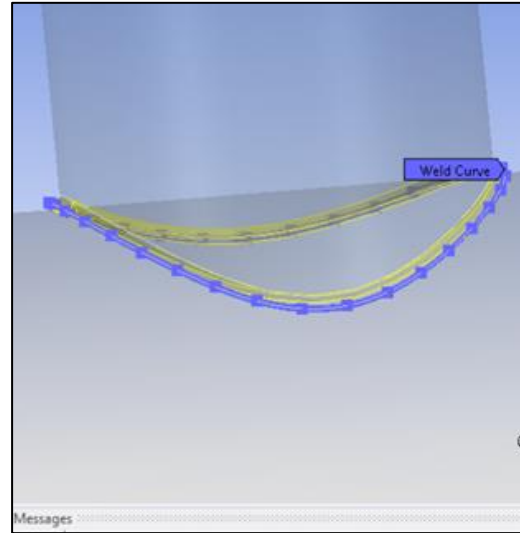
Connect

Weld

Named Selections

Details of "Weld" - Weld

Scope	
Type	Seam
Source	Mesh
Modeled As	Tent and Extension
Create Using	Curves
Tent Direction	Normal
Use Worksheet	No
Curve Scoping	Geometry Selection
Weld Curve	1 Body
Definition	
Suppressed	No
Adjust Weld Height	No
Creation Criteria	Angle Based
Weld Angle	Default (45.0°)
Edge Mesh Size	Default (20.0 mm)
Create Offset Layer	Yes
Offset Layer Height	10.0 mm
Number Of Layers	3



Weld:HAZ1:3

0.00 50.00 100.00

Worksheet

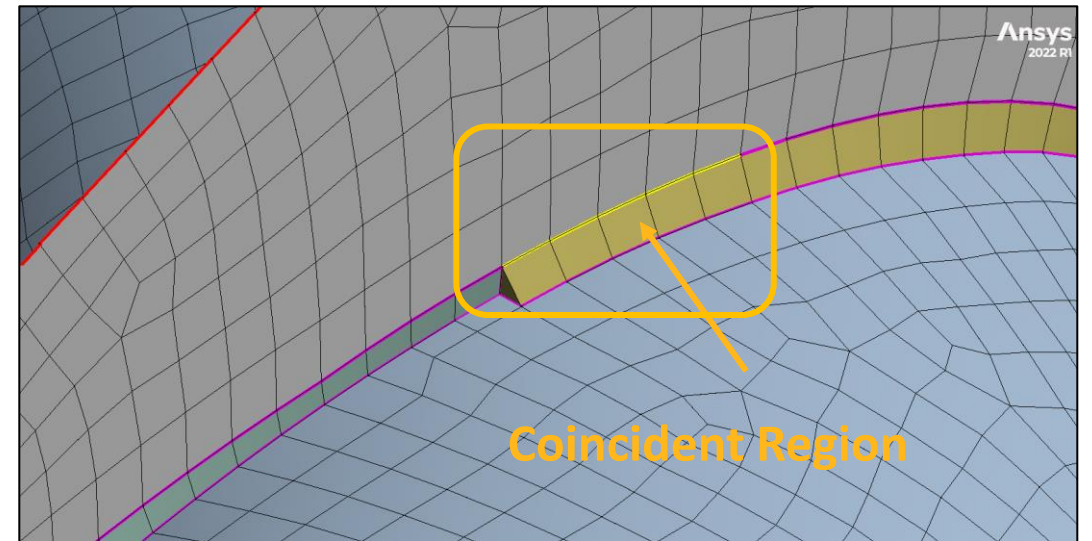
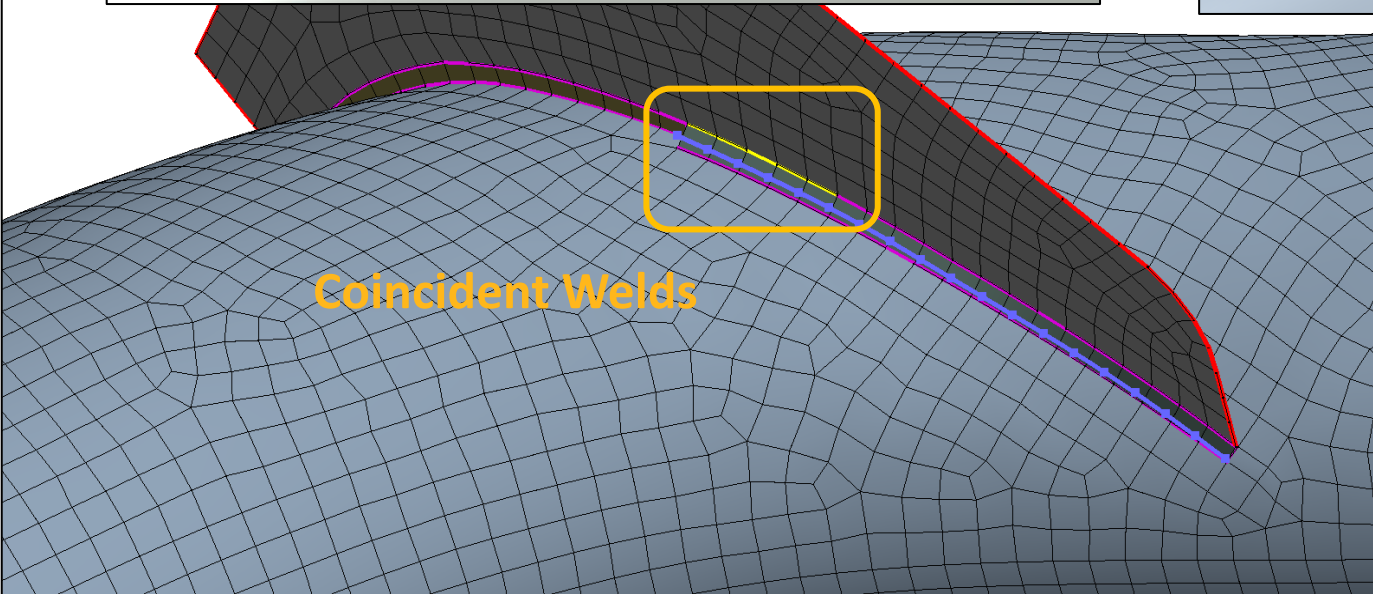
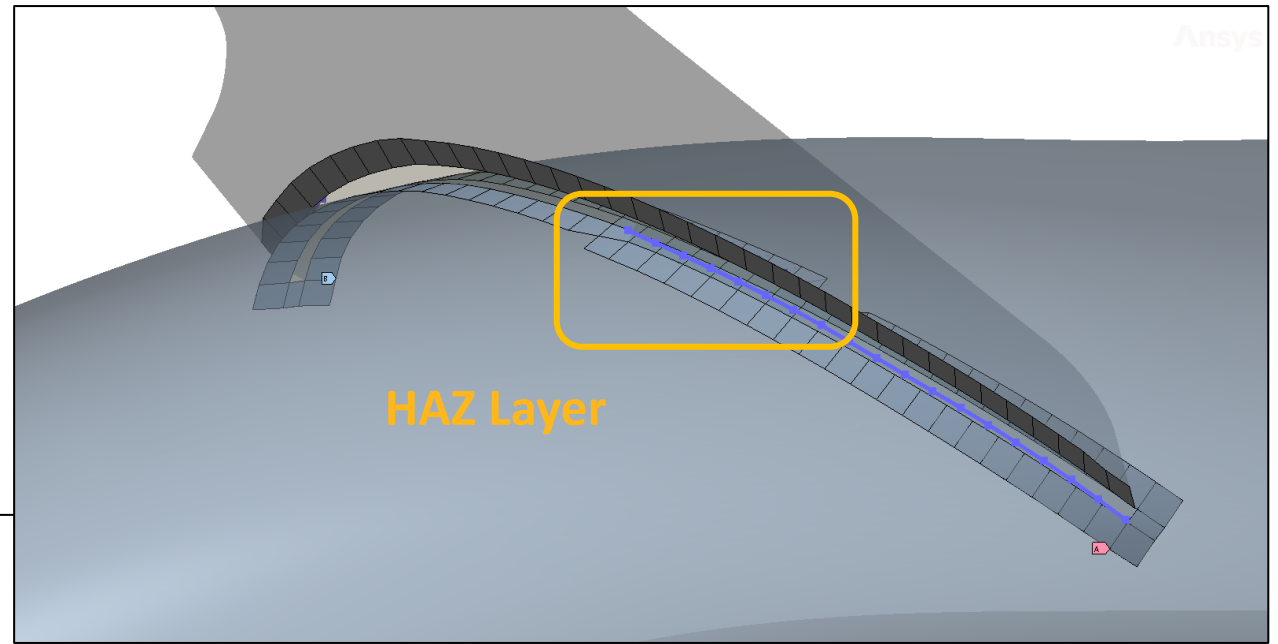
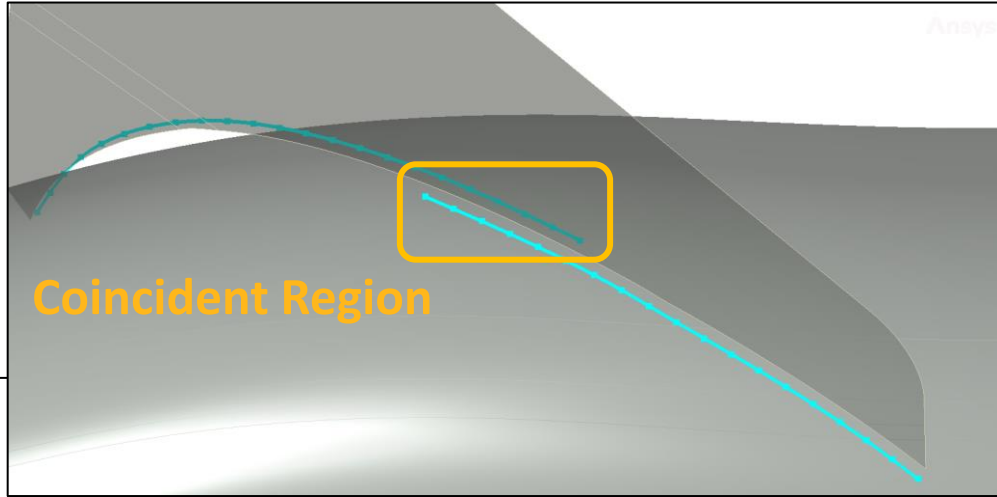
Weld:HAZ1:3

Generate

Note: Internal comparisons of values that have units are done in the Current CAD Unit System: Metric (m, kg, N, s, V, A)

	Action	Entity Type	Criterion	Operator	Units	Value
<input checked="" type="checkbox"/>	Add	Mesh Element	Seam Weld All HAZ	Equal	N/A	All Welds
<input checked="" type="checkbox"/>	Remove	Mesh Element	Seam Weld HAZ2	Equal	N/A	Weld

# Coincident Welds



# / Pull (Line Coating)

- Pull (Line Coating) allows user to create a "Line Coating" Line bodies using the Boundary Edges of 2D Axisymmetric bodies. The Line Coating bodies share nodes with the underlying elements

Line Coating bodies sharing nodes with underlying elements

Details of "Pull (Surface Coating)"

Scope	
Scoping Method	Geometry Selection
Geometry	4 Edges

Definition

Method	Line Coating
Suppressed	No

Part Properties

Material	Structural Steel
Stiffness Option	Stress Evaluation Only



# Pull: Support for Quadratic Elements (Curvilinear Mid-Nodes)

## Pull (Extrude): Merge Profile Nodes

Pull (Extrude) allows user to Merge Profile Nodes, for Extrude by element faces selection on solid bodies with quadratic elements

Support for Quadratic elements

Pull Extrude bodies sharing nodes with underlying body element faces

**Project\***

- Model (A4)
  - Geometry Imports
    - Geometry
      - test\_mb-1
        - Pull (Revolve)
          - Pull (Revolve):Body\_1
          - Pull (Revolve):Body\_2
          - Pull (Extrude)
            - Pull (Extrude):Body\_1
            - Pull (Extrude):Body\_2
            - Pull (Extrude):Body\_3
            - Pull (Extrude):Body\_4
- Materials
- Coordinate Systems
- Symmetry
- Cyclic Region
- Connections
- Mesh
  - Inflation
  - Mesh Edit
    - Pull (Revolve)
    - Pull (Extrude)
- Named Selections

**Details of "Mesh"**

Display

Defaults

Physics Preference Mechanical

Element Order Quadratic

☐ Element Size 2.0 mm

**Details of "Pull (Extrude)"**

Scope

Scoping Method Named Selection

Named Selection 3D\_Coating\_Elements

Definition

Method Extrude

Height 0.5 mm

Number of Layers 1

Merge Profile Nodes Yes

Extrude By Face Normal

Extrude UpTo No

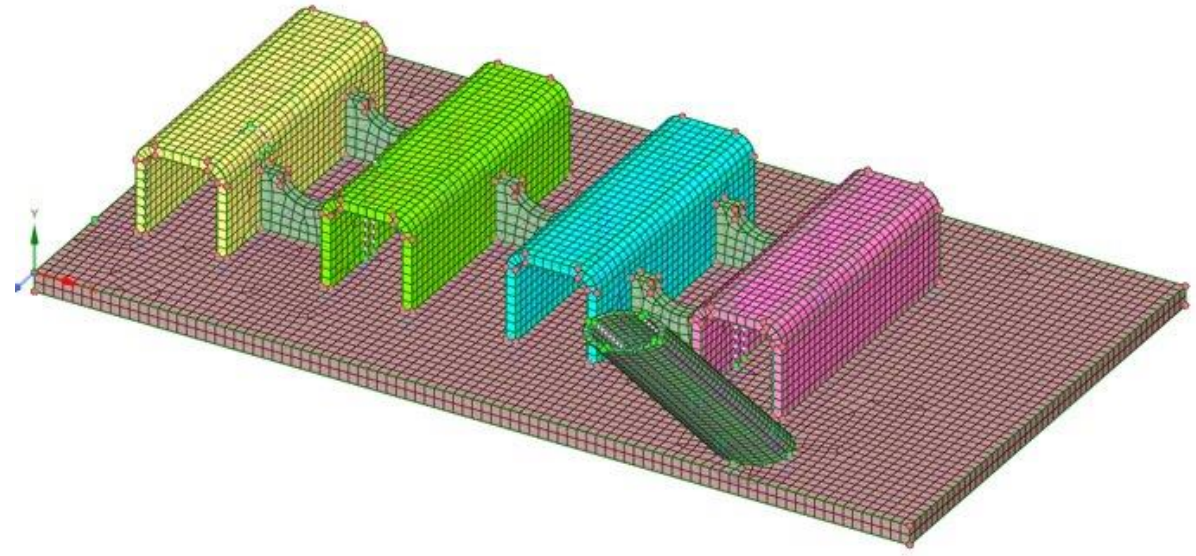
Suppressed No

Part Properties

Material Structural Steel

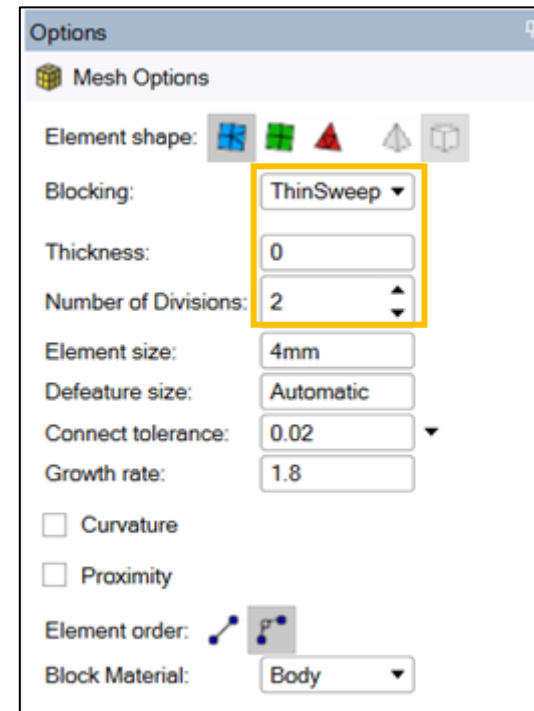
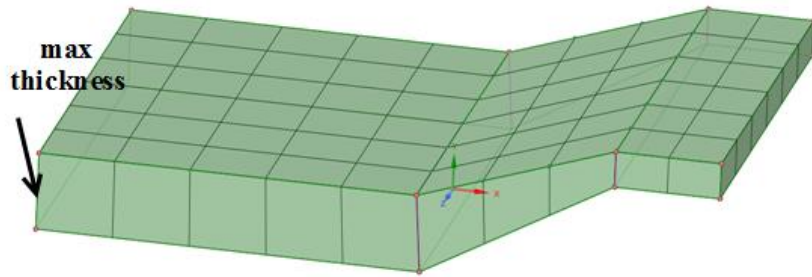
# Spaceclaim Meshing: Overview

- Thin Body Meshing
  - Full Release of ThinSweep Block Decomposition
- Robustness, Performance, Usability
  - Pull improvements
  - New Mapping Options at block controls level
  - Improved quality for All Quadrilateral method
  - Improved performance
    - Faster “activate” for meshing in SpaceClaim
    - Faster surface meshing of circuit board type models
    - Faster model transfer to Ansys Mechanical
  - Improved robustness for external documents
  - Better support for \*.cdb/\*.inp files
- Meshing for Explicit Improvements
  - New Explicit Physics option (Beta)
  - Better uniformity of mesh
  - Characteristic Length calculation consistent with LS-DYNA
  - Better support for LS-DYNA \*.k files



# Spaceclaim Meshing: ThinSweep Blocking

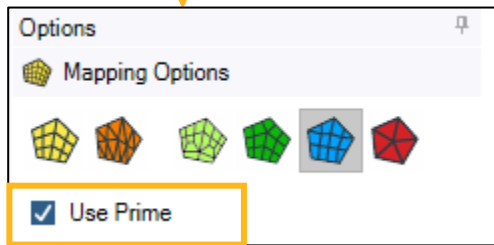
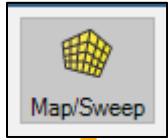
- Full release of ThinSweep option which was previously Beta
- Set number of divisions across thickness and (optionally) thickness of material
- For models with varying thickness, specify a Thickness value slightly more than the maximum thickness of the model.



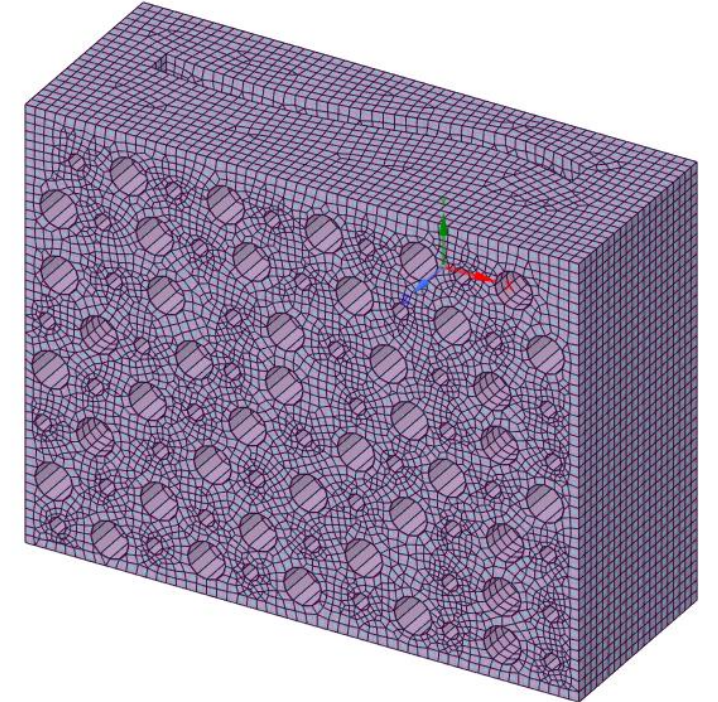
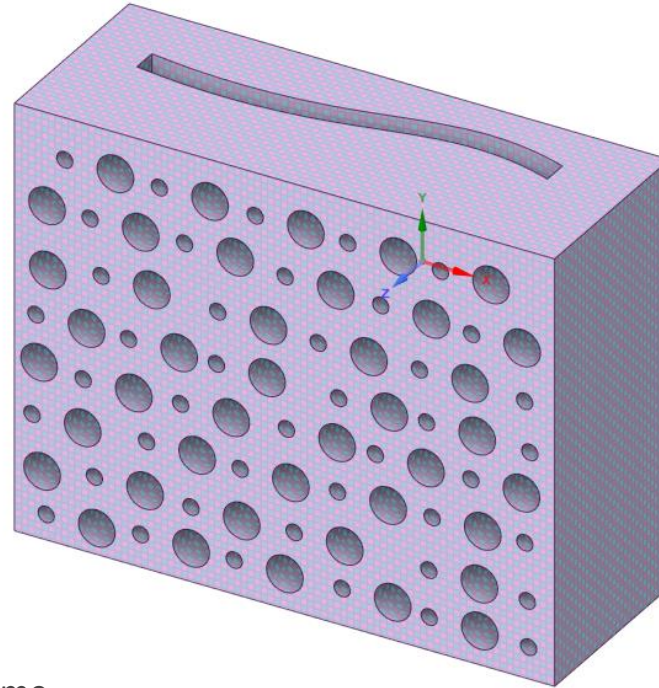


# Spaceclaim Meshing: Surface Meshing Performance

**New option:** Prime surface meshing at local control level



Apply faster Prime  
Surface meshing by  
Toggling ON the "Use  
Prime" option



## Geometry Surfaces

`options.Method = MapMeshMethod.QuadsPrime`

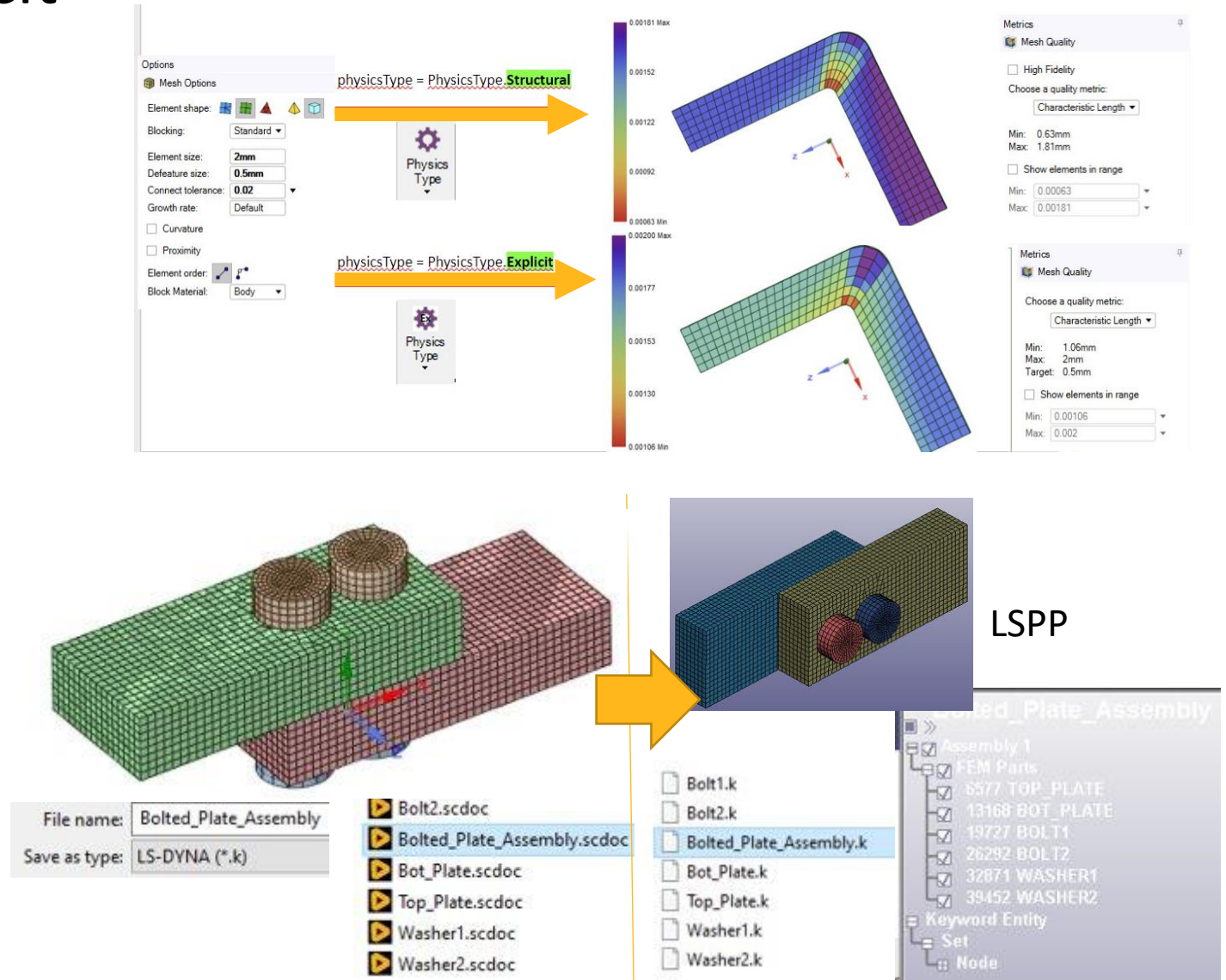
## Block Faces:

`options.ConvertType = ConvertBlockType.ToFreeQuadDom`

`options.ElementShape = ElementShapeType.PrimeQuadDominant`

# Spaceclaim Meshing: Explicit

- New Explicit Physics preference in dropdown
  - Heavier default defeaturing tolerance
  - Target characteristic length and optimisation for better uniformity
- Better support for /input \*include files
  - Export .k assembly and include file



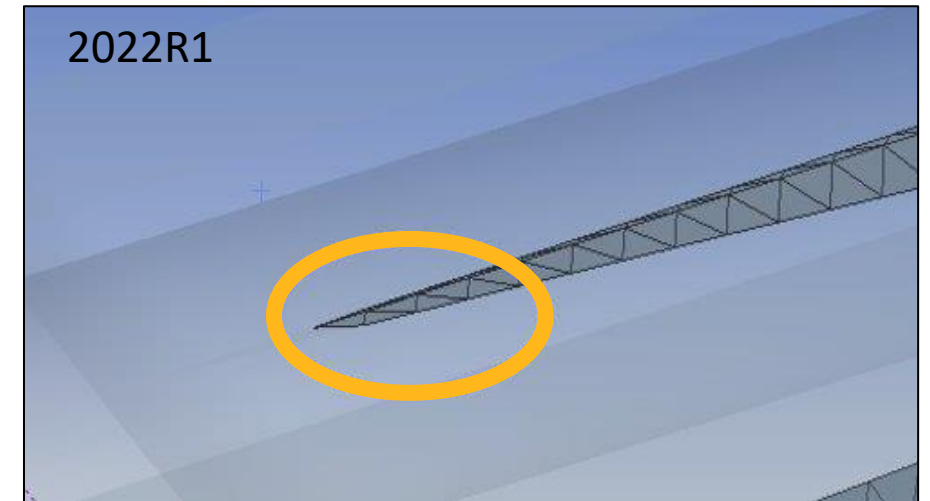
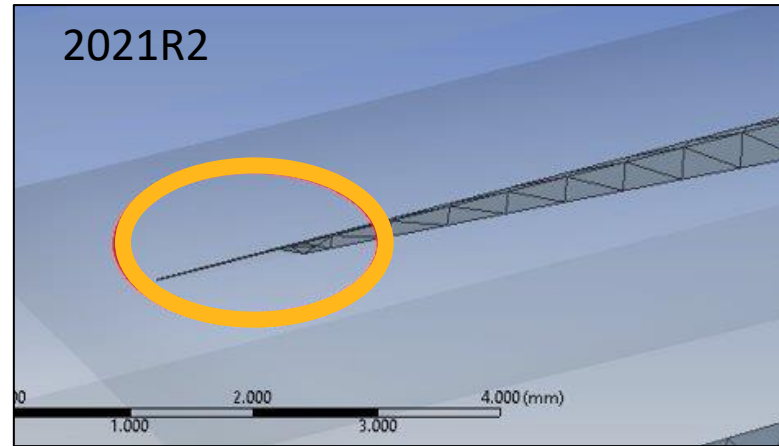


**Explicit Meshing**  
**Tetrahedral Meshing**  
**Mesh Diagnostics**  
**Feature Based Meshing**



# Improved Robustness for Patch Conforming Tetrahedra

- Increasing the defeaturing tolerance avoids small elements but also causes more instability in the meshing algorithms
- In 2022 R1, many improvements to robustness have been made for heavy defeaturing required by Explicit users



# Improved Error Handling

- Intersecting Surface Mesh RMB Options

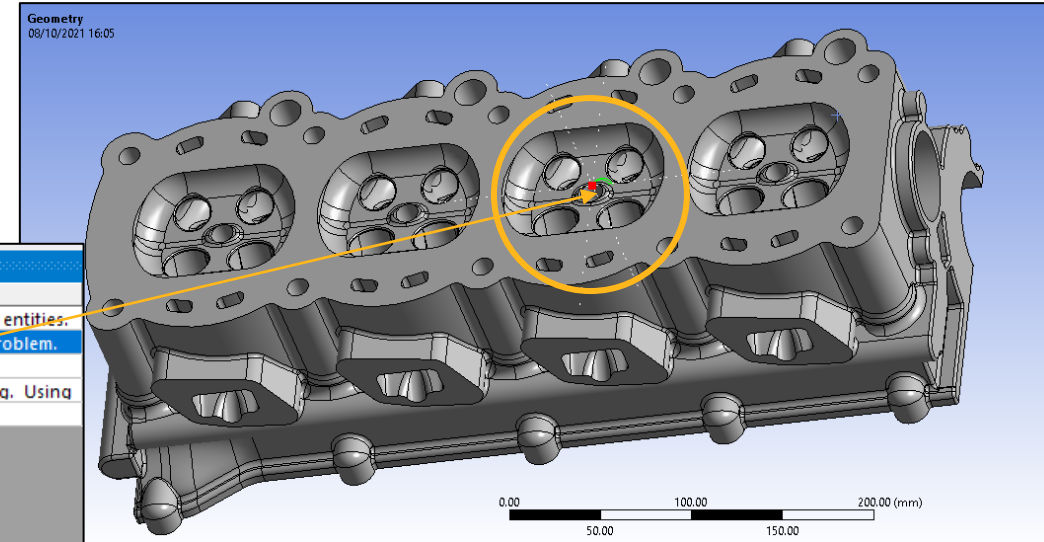
Messages

	Text
Error	One or more entities failed to mesh. The mesh of the bodies containing these entities may not be up-to-date. However, meshing might be successful on the other entities.
Error	The surface mesh is intersecting or close to intersecting, making it difficult to create a volume mesh. Please adjust the mesh size or adjust the geometry to fix the problem.
Warning	Some boundaries of protected topologies have been defeatured. Right click on this message and select "Go To Object" to see the affected edges.
Warning	Quad map meshing failed on one or more surfaces. A surface could be narrow or have been improperly paired up for meshing. Using
Error	A mesh could not be generated using the current meshing options and settings.

Right-click context menu options:

- Go To Object
- Show Problematic Geometry
- Show Intersecting Surface Mesh
- Show Message
- Copy
- Delete
- Refresh

**Show Problematic Geometry**



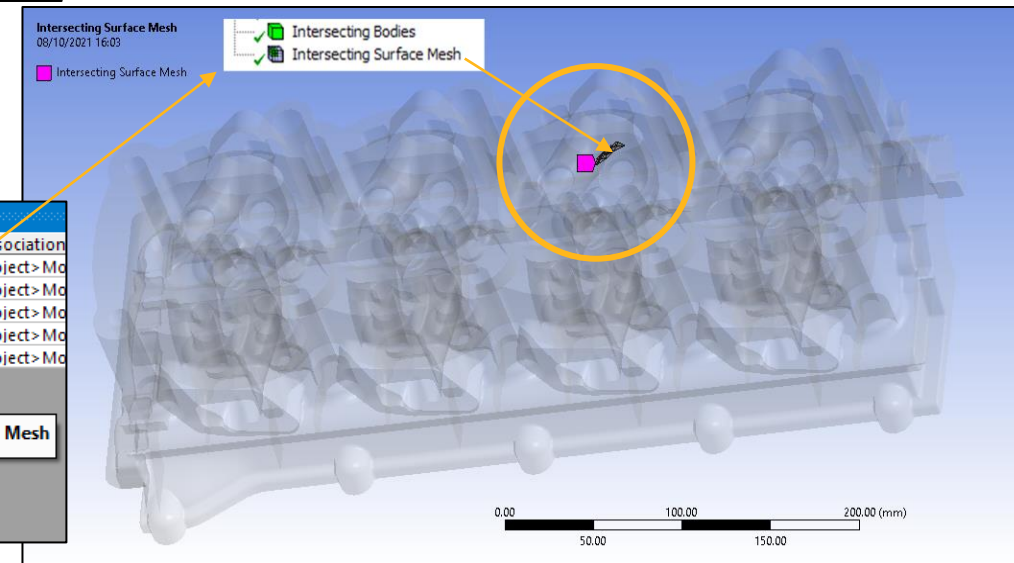
Messages

	Text
Error	One or more entities failed to mesh. The mesh of the bodies containing these entities may not be up-to-date. However, meshing might be successful on the other entities.
Error	The surface mesh is intersecting or close to intersecting, making it difficult to create a volume mesh. Please adjust the mesh size or adjust the geometry to fix the problem.
Warning	Some boundaries of protected topologies have been defeatured. Right click on this message and select "Go To Object" to see the affected edges.
Warning	Quad map meshing failed on one or more surfaces. A surface could be narrow with many boundary edges
Error	A mesh could not be generated using the current meshing options and settings.

Right-click context menu options:

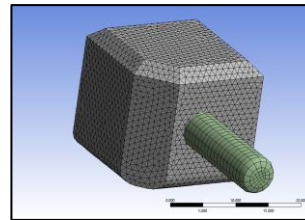
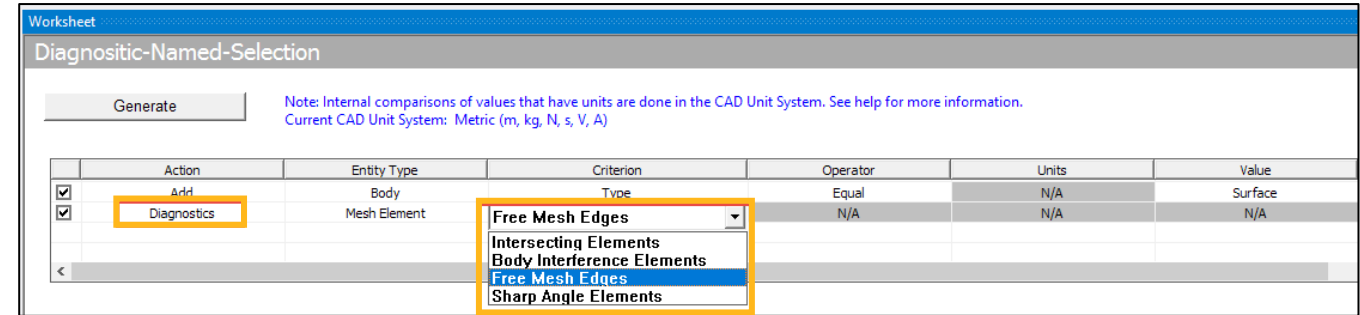
- Go To Object
- Show Problematic Geometry
- Show Intersecting Surface Mesh
- Show Message
- Copy
- Delete
- Refresh

**Show Intersecting Surface Mesh**

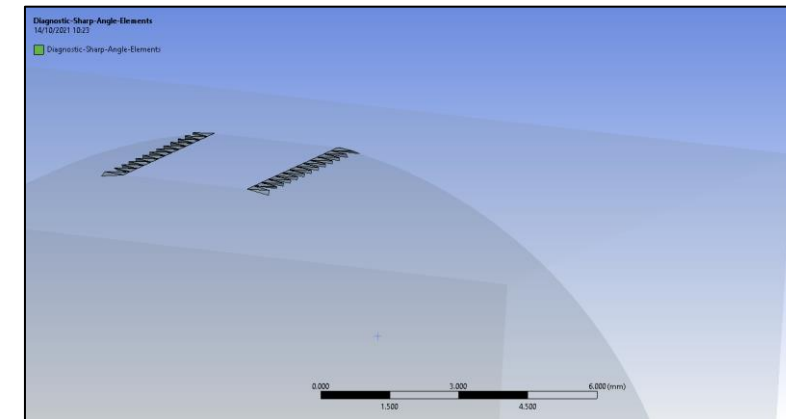
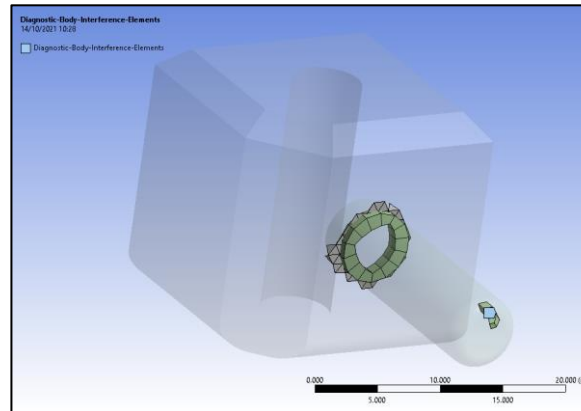


# / Diagnostics Tools

- Use worksheet Named Selection to select bodies and then run Diagnostics for visualisation of the problem
- New tools to find issues and fix via additional settings or return to geometry tool for modifications
- Options available at 2022 R1:
  - Mesh Element:
    - Intersecting surface mesh failures
    - Free edge mesh
    - Sharp angle
    - Body Interference
  - Topology
    - Defeatured Faces

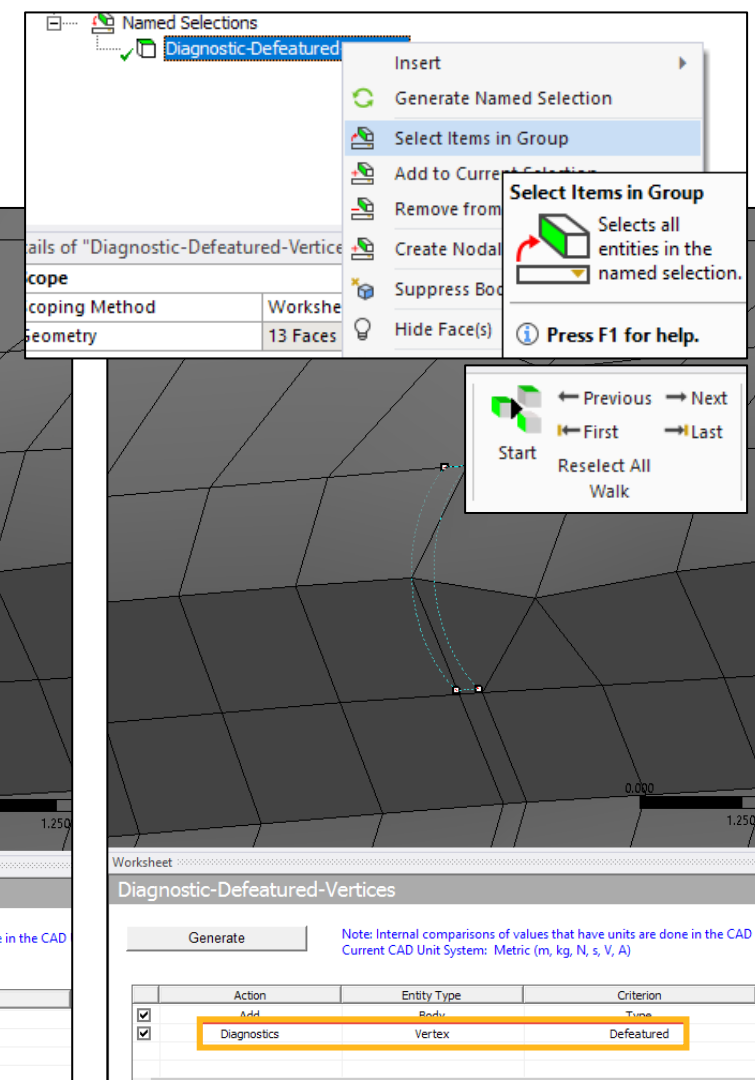
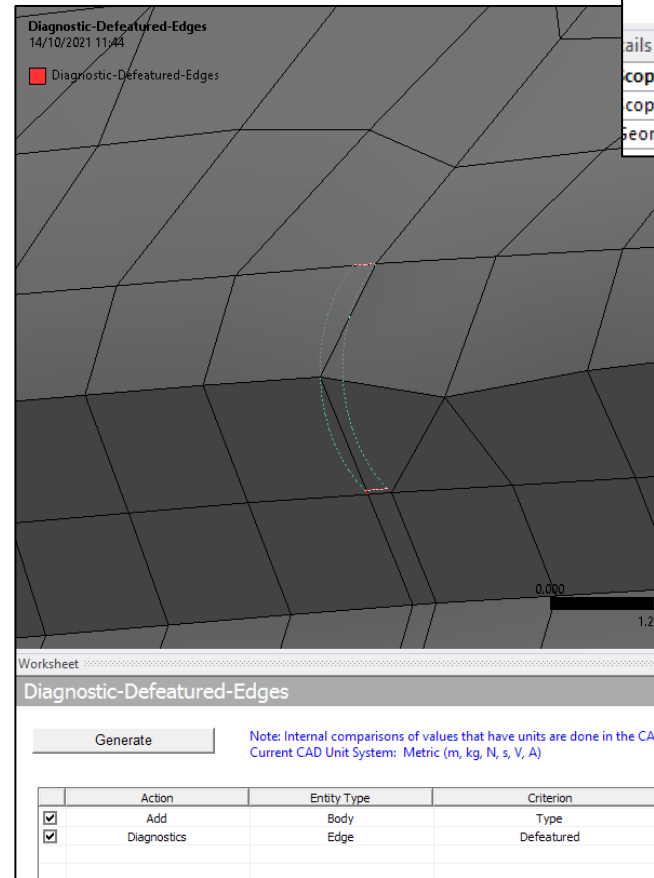
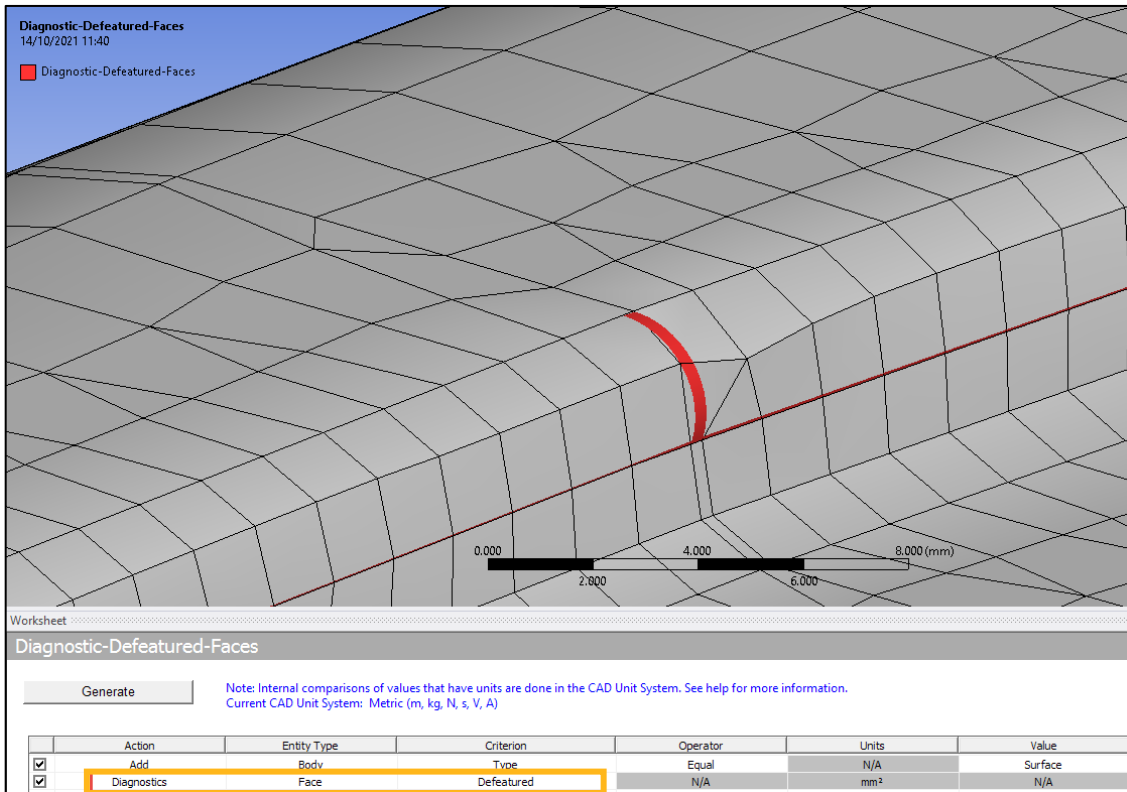


*Body Interference*



*Sharp Angle Elements*

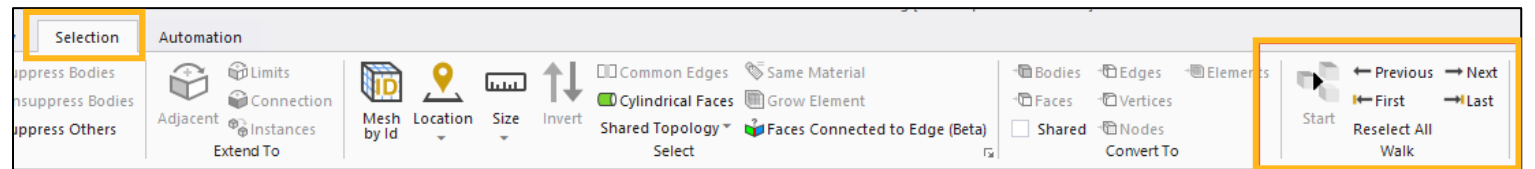
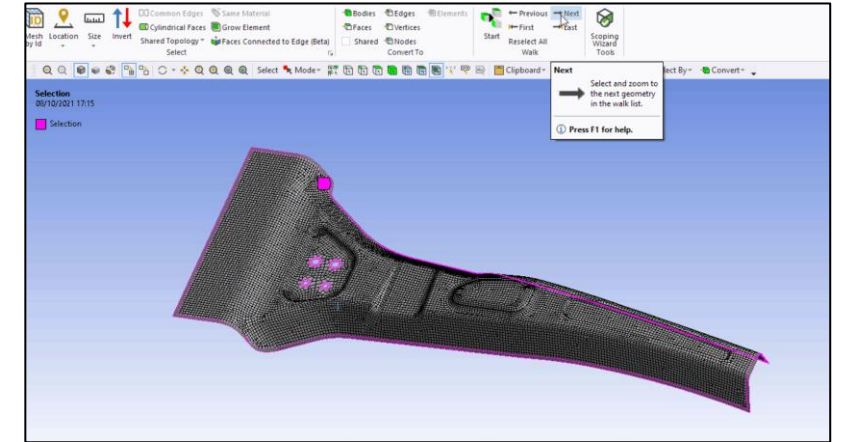
# Mesh Diagnostics Tools: Defeaturing Checks



- Detect and examine where defeaturing has occurred after meshing
- Use Selection Walk to traverse defeatured faces

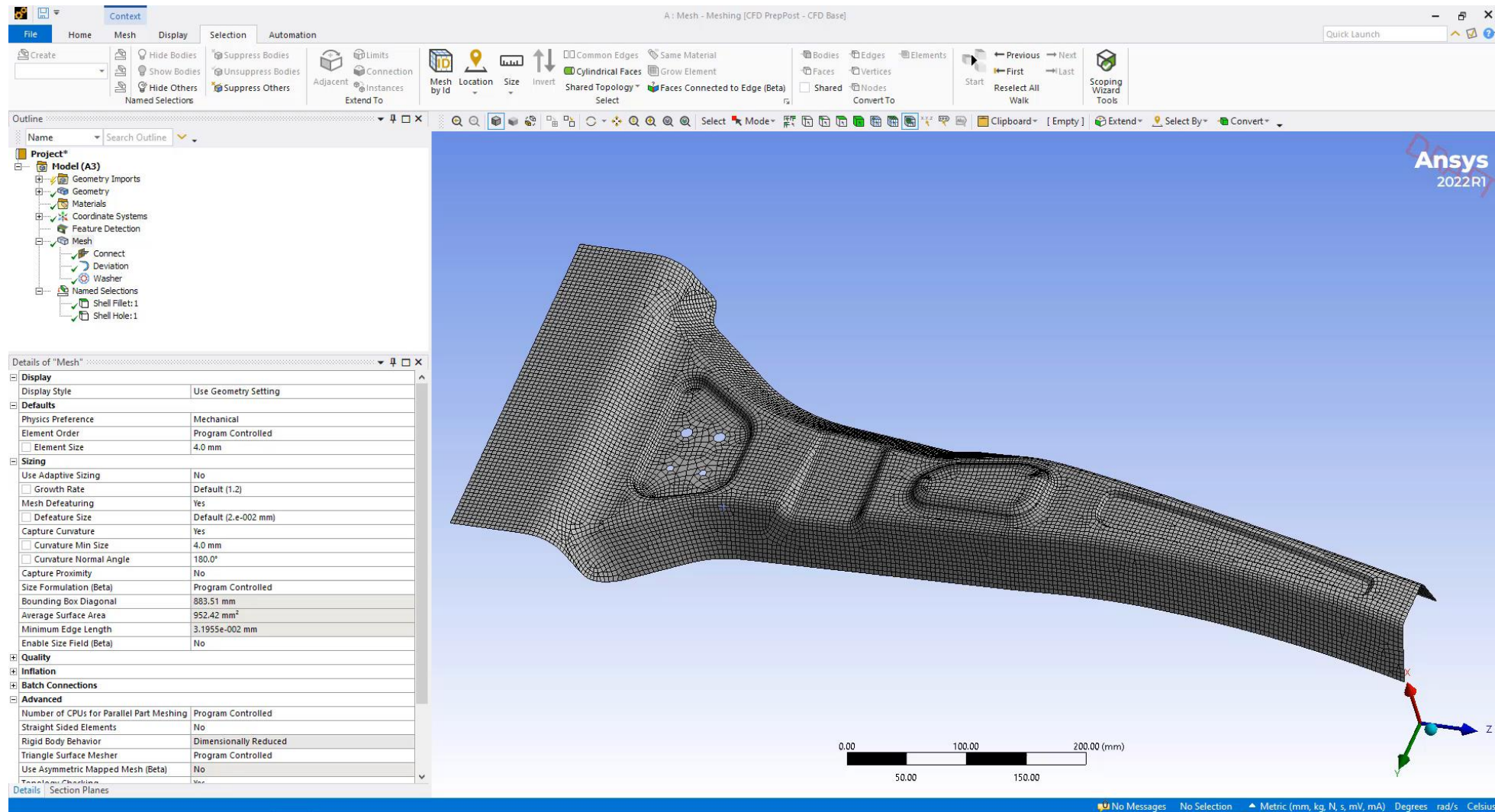
# / Mesh Cluster Walk

- Model Walk has been enhanced to walk through
  - Mesh Elements
  - Mesh Element Clusters
- Useful for new diagnostics tools to traverse multiple issues
- New RMB option on Element based NS from Diagnostics
  - Select Mesh Clusters in Group





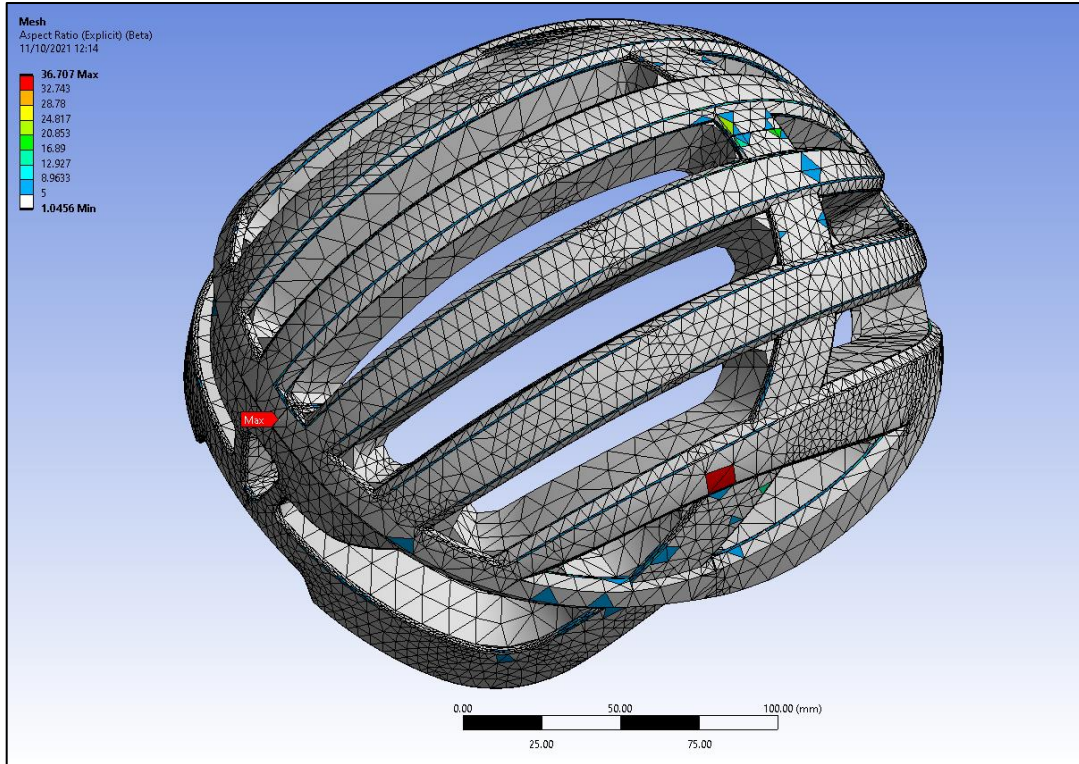
# Mesh Cluster Walk: Short Demo with Free Face Diagnostics



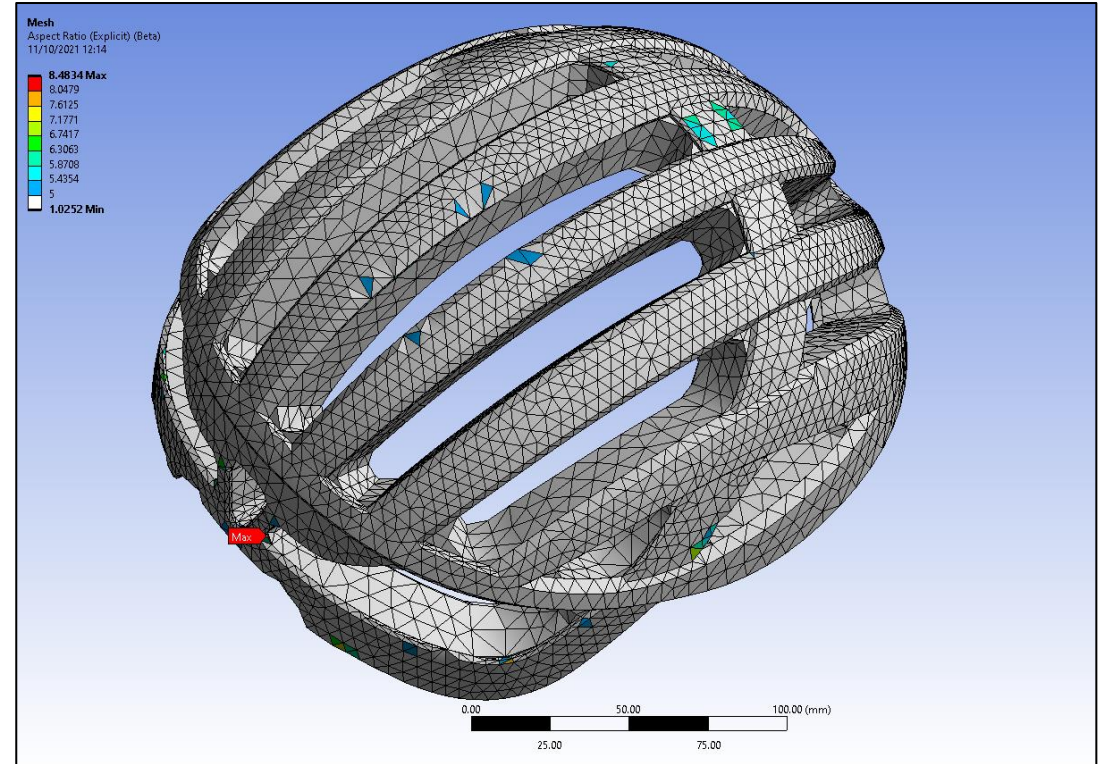
# Default Surface Mesher Explicit Physics Pref. = Adv. Front

- More uniform and smooth triangular surface mesh

Advanced	
Number of CPUs for Parallel Part Meshing	Program Controlled
Straight Sided Elements	
Rigid Body Behavior	Full Mesh
Triangle Surface Mesher	Advancing Front



*Previous Default Surface Mesh*

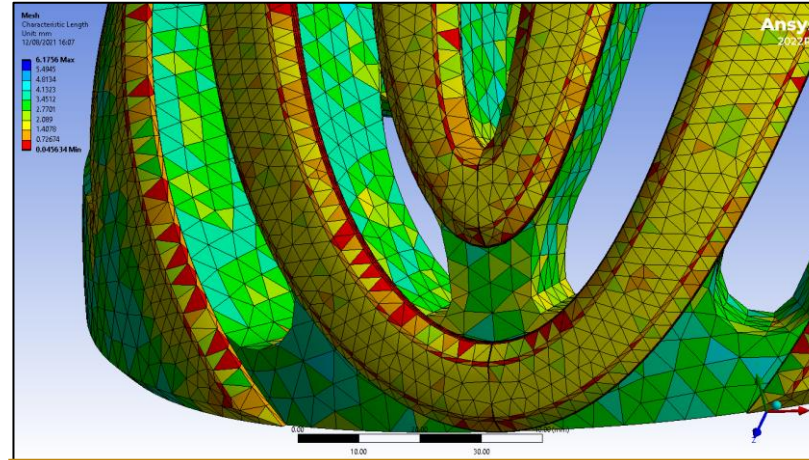


*New Default Surface Mesh*

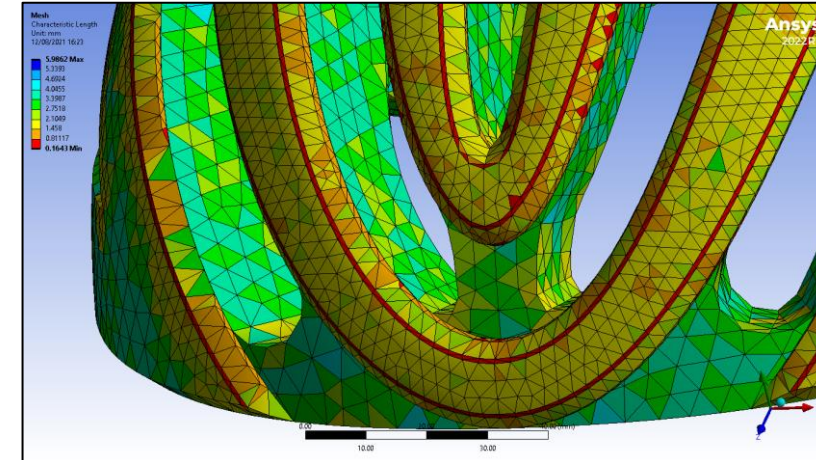


# Tet Meshing Aspect Ratio Targeting (Explicit Physics Pref.)

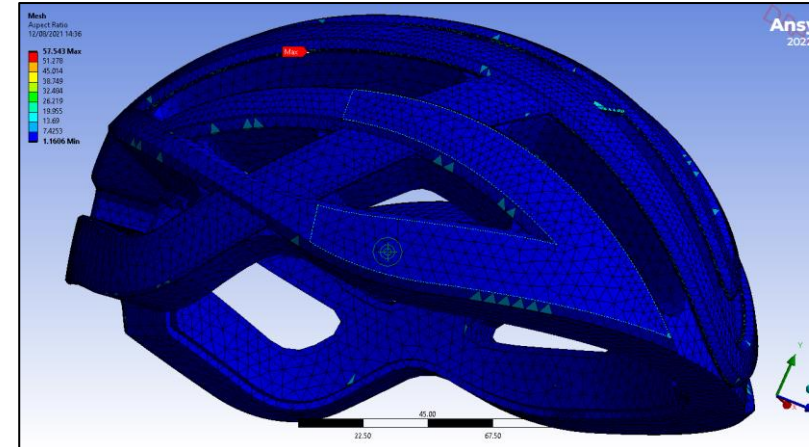
- Aspect Ratio based meshing criteria drastically reduces the max. Aspect Ratio (AR)
- This help improve the Characteristic Length (CL)significantly which has a big impact of the Explicit CFL Time-Step ( $\Delta t = \frac{\text{Characteristic Length}}{\text{Speed of Sound}}$ )
- Help run the analysis without much mass-scaling



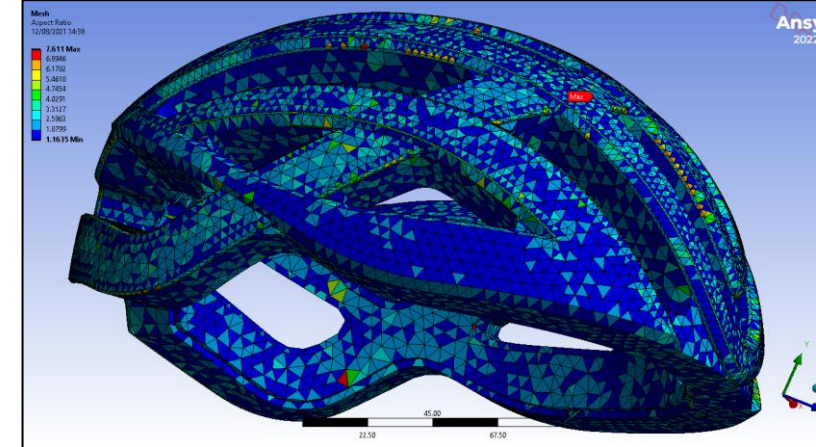
2021 R2 – Max AR=53, Min CL=0.05mm



2022 R1 – Max AR=7.6, Min CL=0.16mm



BEFORE



AFTER

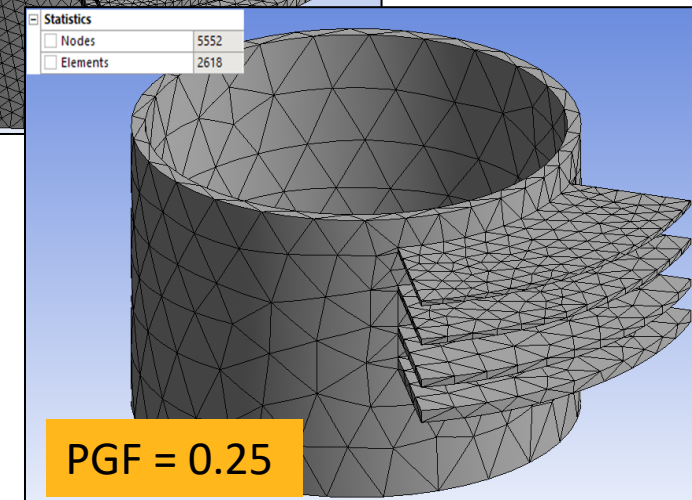
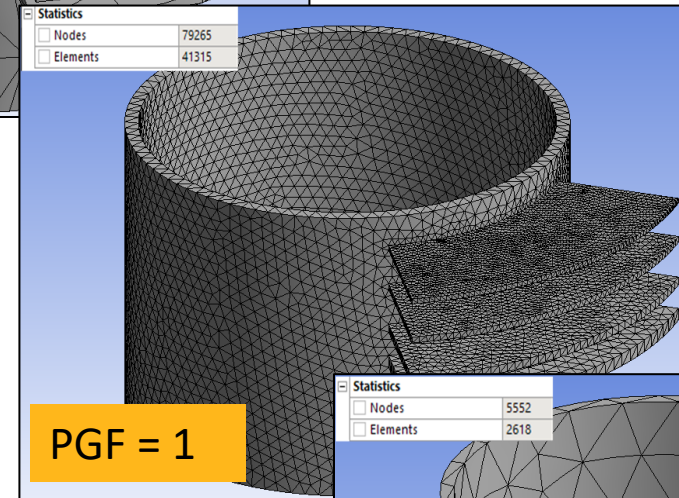
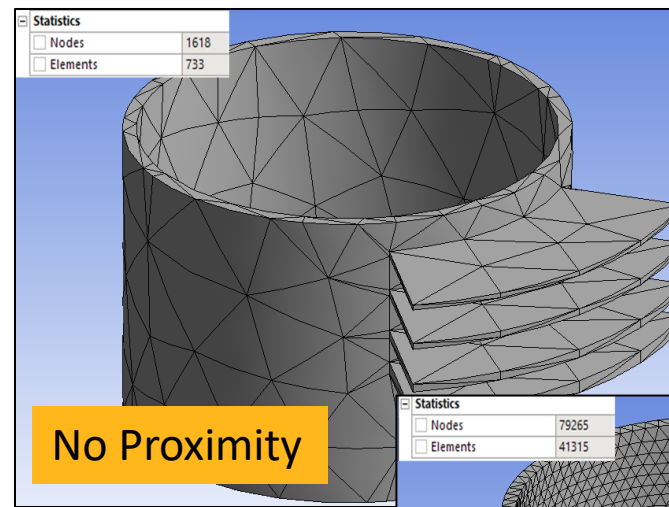
Quality	
Check Mesh Quality	Yes, Errors and Warnings
Target Element Quality	Default (0.200000)
Target Characteristic Length (LSDyna)	Default (0.5 mm)
Target Aspect Ratio (Explicit)	5.

## Challenges:

- Bike Helmet Example: A complex geometry with several intricate features

# Thin Solids

- Thin solids offer specific challenges:
  - How to avoid very high AR elements without increasing cell count considerably?
- Proximity Gap Factor now allows user to control mesh size across thin regions without requirement for full isotropic element (Non integer is allowed)
- e.g. Use Proximity Gap Factor = 0.25 to aim for aspect ratio of  $\sim 4$  in thin regions

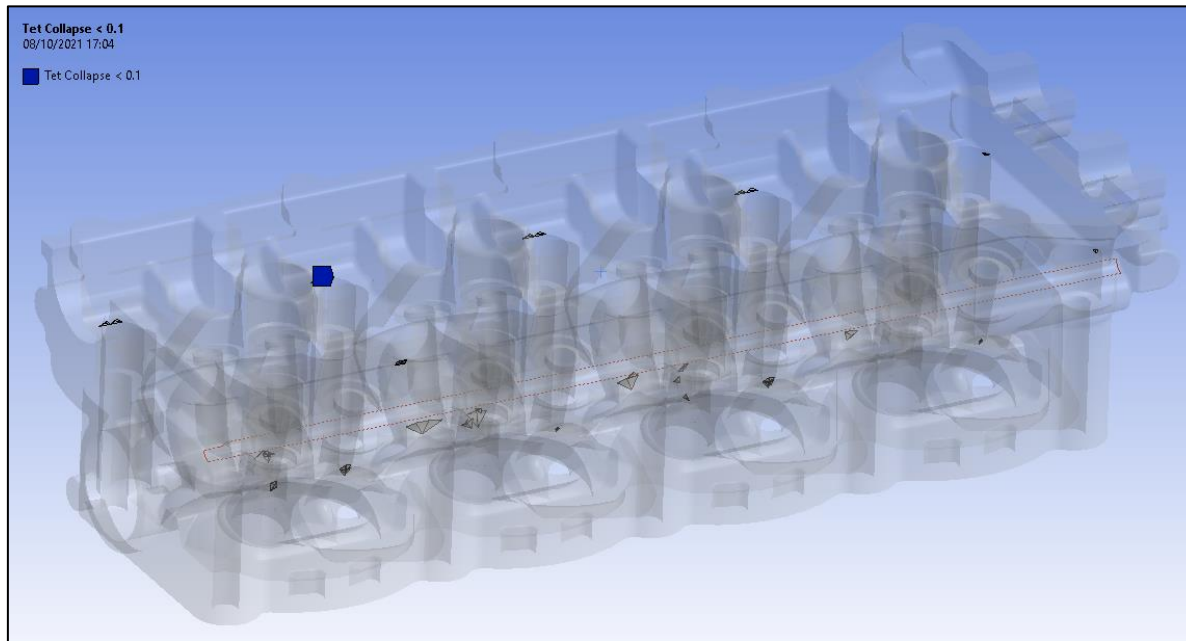


Capture Proximity	Yes
<input type="checkbox"/> Proximity Min Size	Default (1.0 mm)
<input checked="" type="checkbox"/> Proximity Gap Factor	0.5

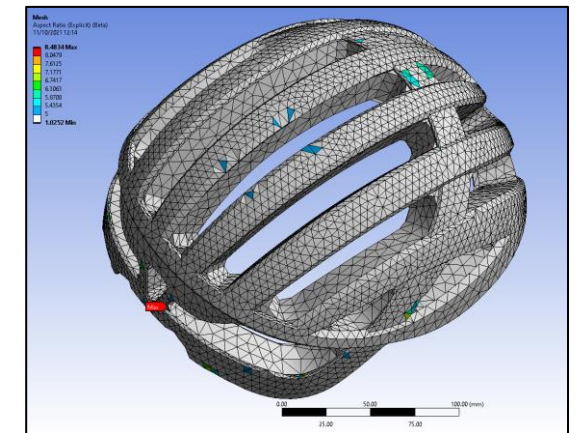
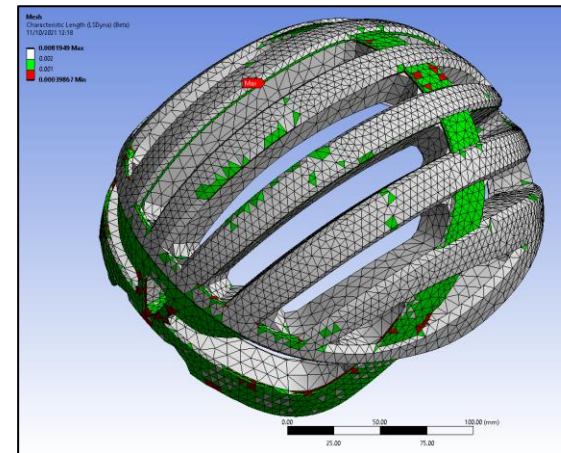
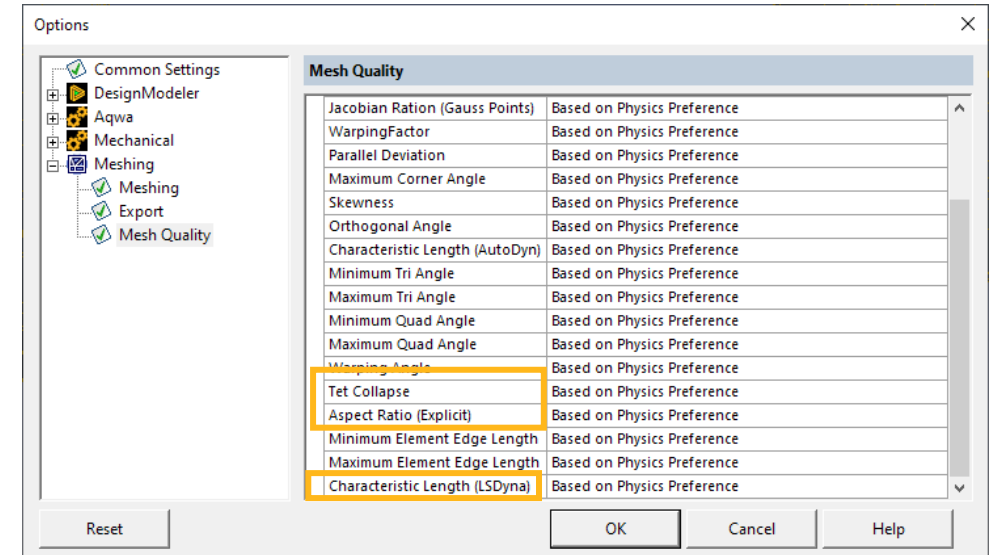


# Quality

- Exposure of many more metrics as mentioned
  - Visibility of metrics is based on physics preference
  - New include LS Dyna Characteristic Length and Explicit Aspect Ratio



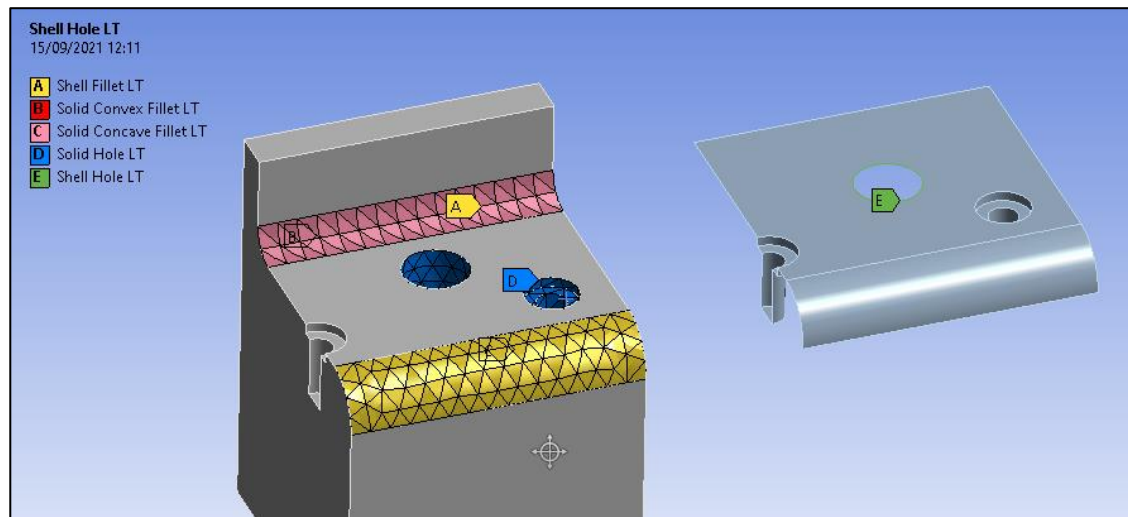
Quality	
Check Mesh Quality	Yes, Errors and Warnings
<input type="checkbox"/> Target Element Quality	Default (0.200000)
<input type="checkbox"/> Target Characteristic Length (LSDyna)	Default (0.5 mm)
<input type="checkbox"/> Target Aspect Ratio (Explicit)	5.



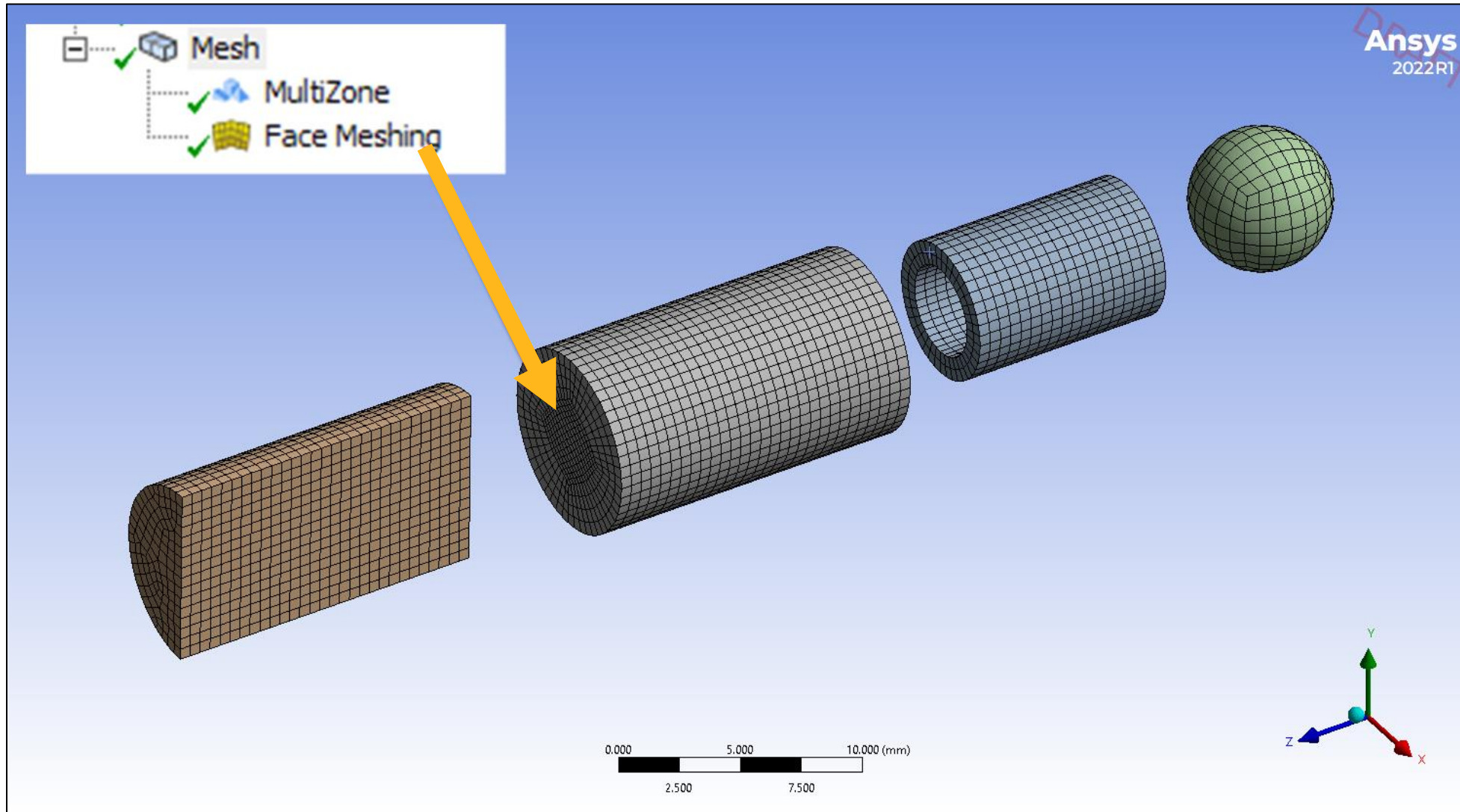
# Feature Detection and Treatment

- Now available for **Solid** bodies
  - **3D hole** detection with option for mapped mesh treatment
  - **3D fillet** detection with option for mapped mesh treatment

Worksheet									
Feature Detection									
*Right click on the grid to add/delete a row.									
	Name	Type	Criteria	Operator	Value	Angle	Min Bound	Max Bound	Mesh Treatment
1	Shell Fillet LT 5mm	Shell Fillet	Radius	Less Than	5	0	0	0	Deviation Control
2	Solid Convex Fillet LT 5mm	Solid Convex Fillet	Radius	Less Than	5	0	0	0	Mapped Meshing
3	Solid Concave Fillet LT 5mm	Solid Concave Fillet	Radius	Less Than	5	0	0	0	Mapped Meshing
4	Solid Hole LT 10mm	Solid Holes	Radius	Less Than	10	0	0	0	Mapped Meshing
5	Shell Hole LT 10mm	Shell Holes	Radius	Less Than	10	0	0	0	Washer Control
+ Add Feature Detection									



# Hex Meshing: Less Decomposition in Mechanical MultiZone

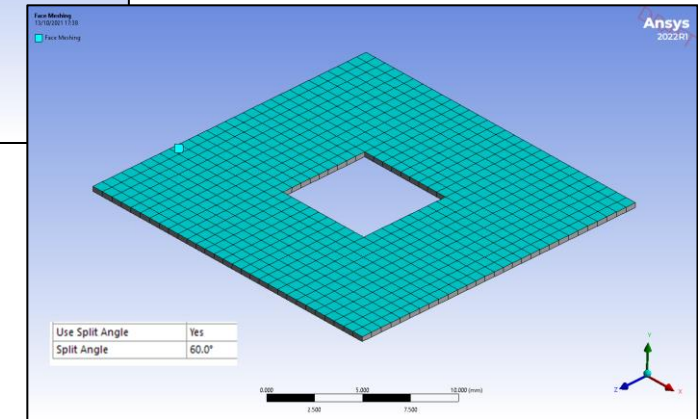
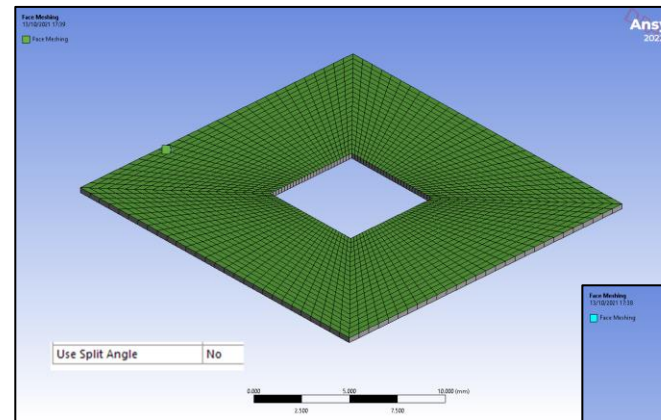
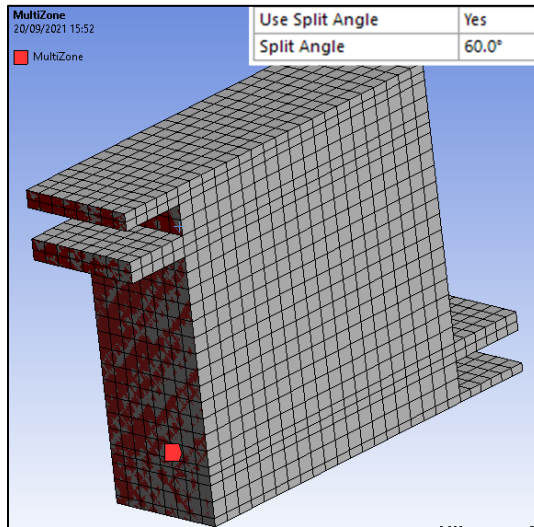
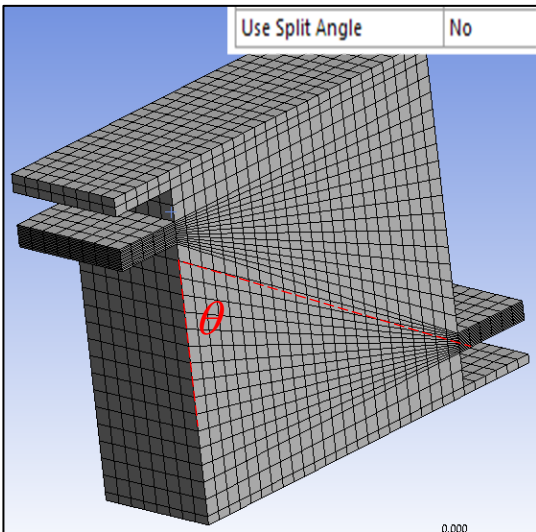




# Hex Meshing: Split Angle

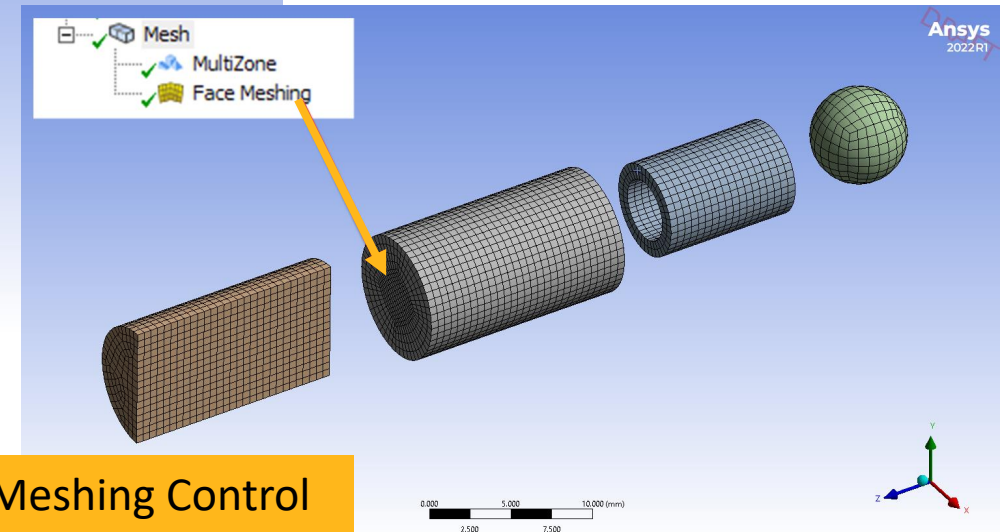
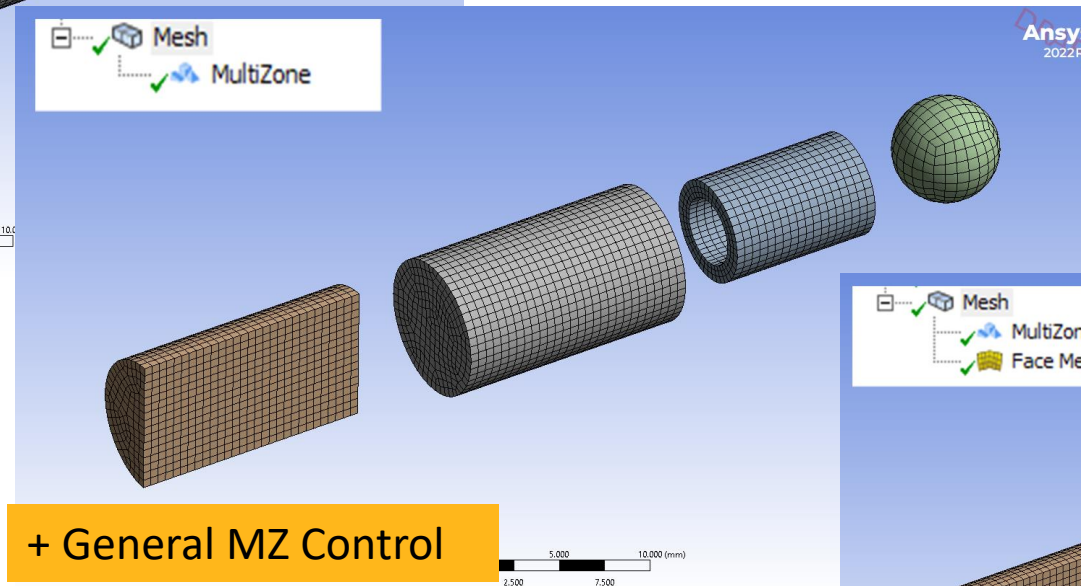
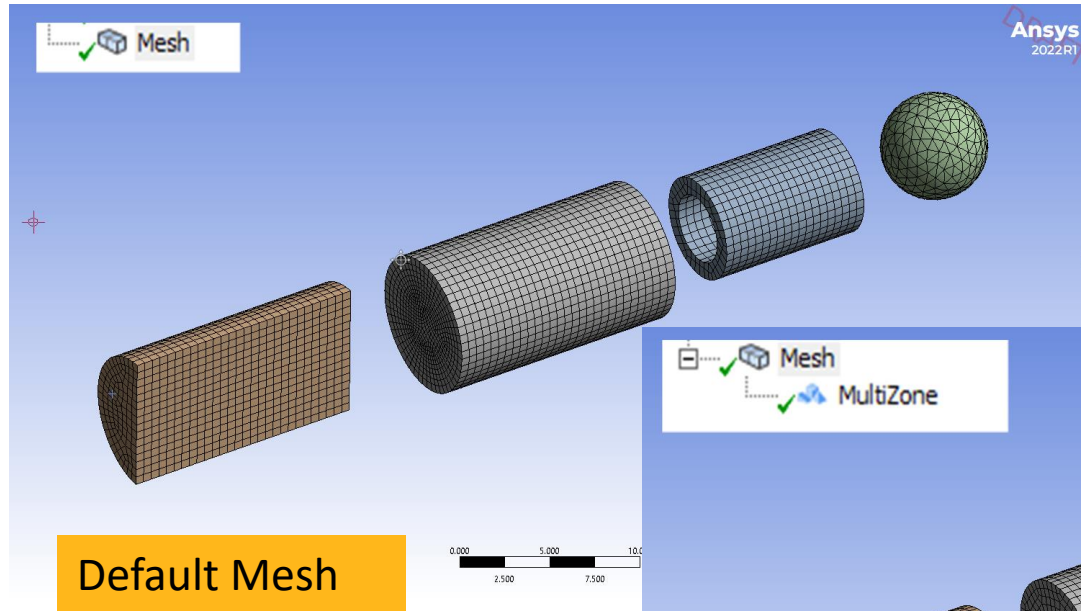
- Detect “skewed” blocks with bad angles and automatically cut them to yield better orthogonal meshes in a more automated way
  - Reduce need for decomposition in geometry tool
  - **Yes** by default for MZ controls created after Explicit Physics Preference is enabled

Details of "MultiZone" - Method	
Scope	
Scoping Method	Geometry Selection
Geometry	1 Body
Definition	
Suppressed	No
Method	MultiZone
Decomposition Type (Beta)	Standard
Mapped Mesh Type	Hexa
Surface Mesh Method	Program Controlled
Free Mesh Type	Not Allowed
Element Order	Use Global Setting
Src/Trg Selection	Automatic
Source Scoping Method	Program Controlled
Source	Program Controlled
Sweep Size Behavior	Sweep Element Size
<input type="checkbox"/> Sweep Element Size	Default
Element Option	Solid
Advanced	
Preserve Boundaries	Protected
Mesh Based Defeaturing	Off
Minimum Edge Length	1.0 mm
Write ICEM CFD Files	No
Reuse Blocking (Beta)	Off
Use Split Angle	Yes
Split Angle	60



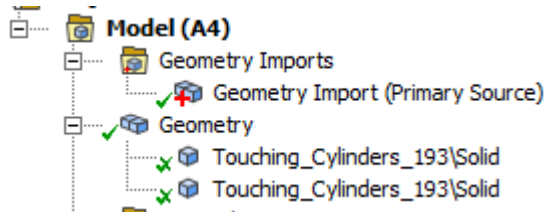


# Hex Meshing: Better Orthogonality and Default Meshing Explicit Preferences



# Hex Meshing: Better Default Mesh for Cylinders, Circles, and Spheres

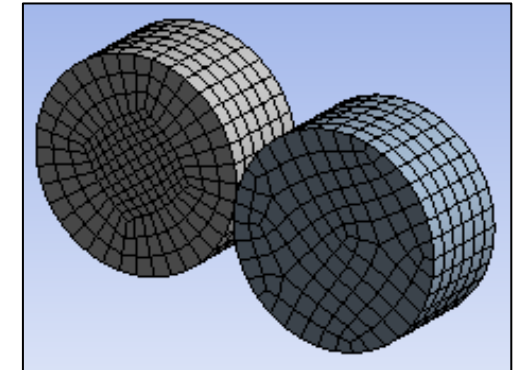
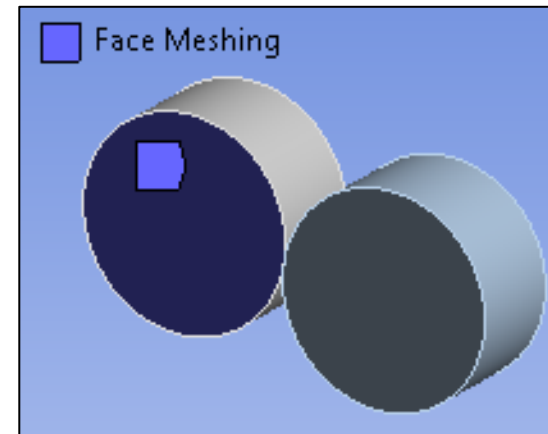
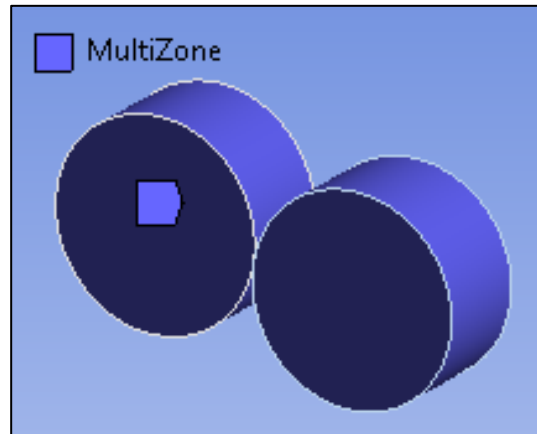
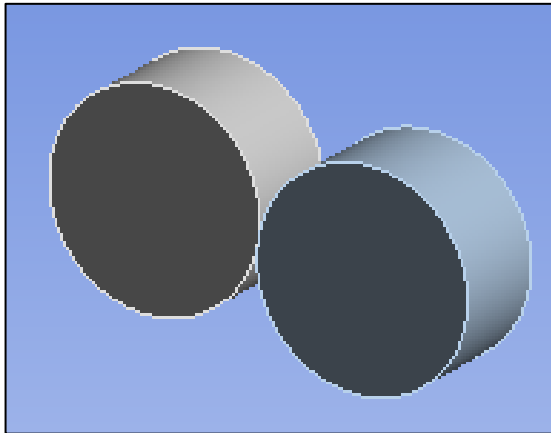
## Touching Cylinders



MultiZone Applied with no special inputs/selections – Automatic Hex



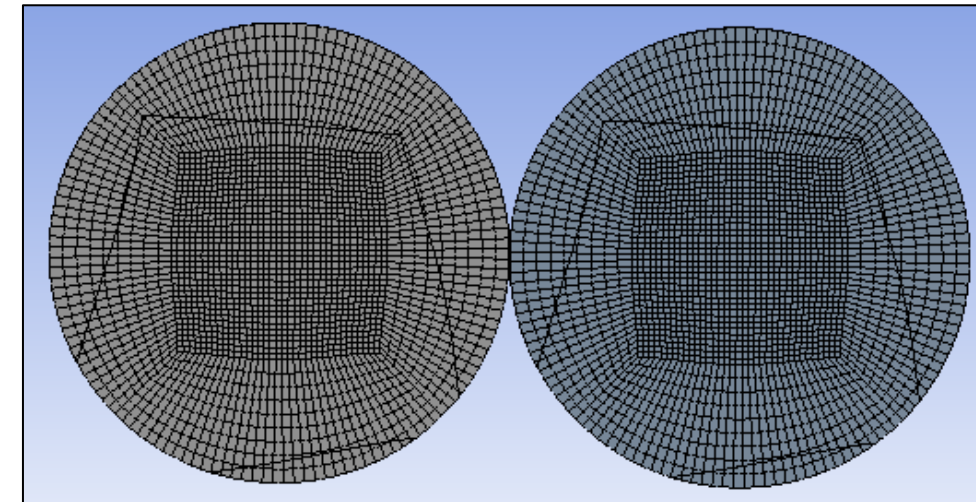
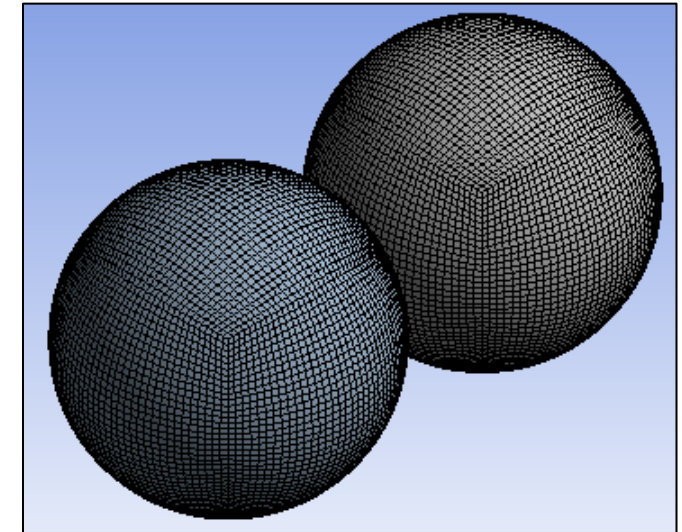
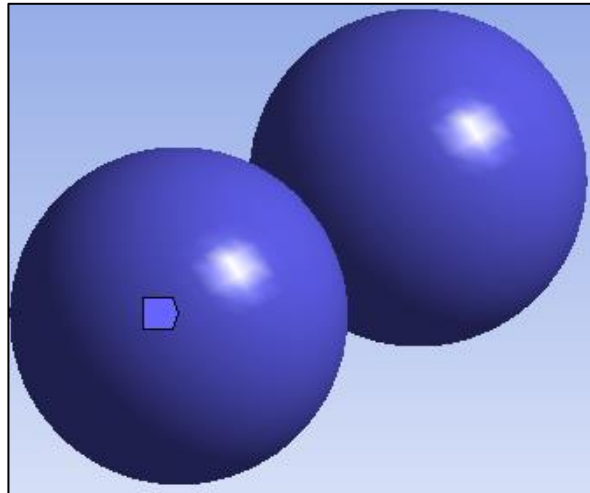
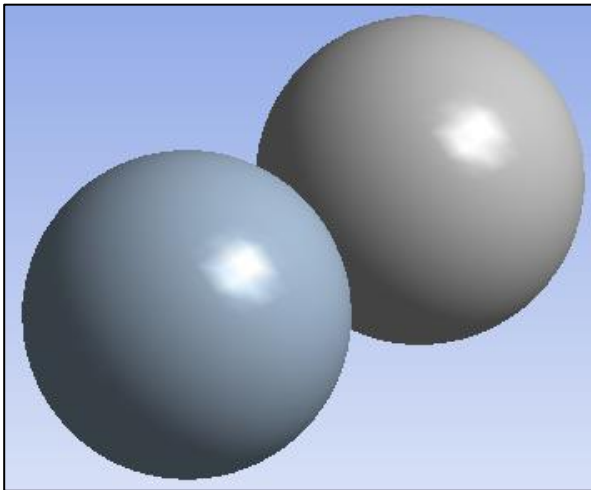
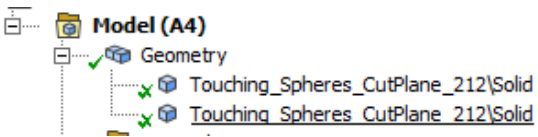
If Face Meshing is Applied O-Grid is automatic with no decomposition needed



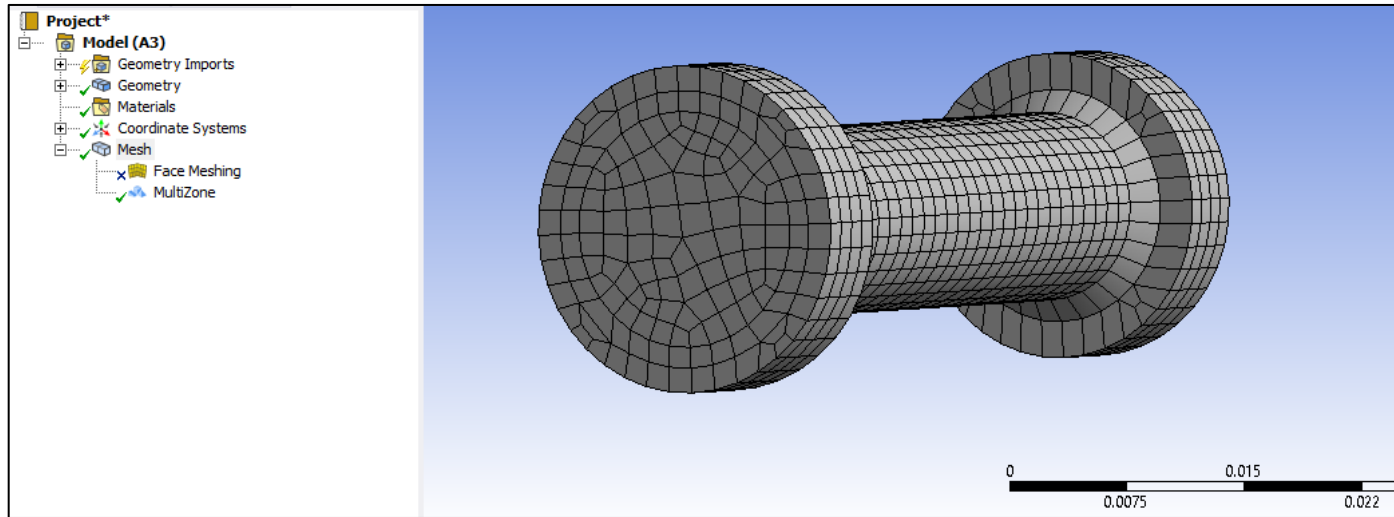
# Hex Meshing: Better Default Mesh for Cylinders, Circles, and Spheres

## Touching Spheres

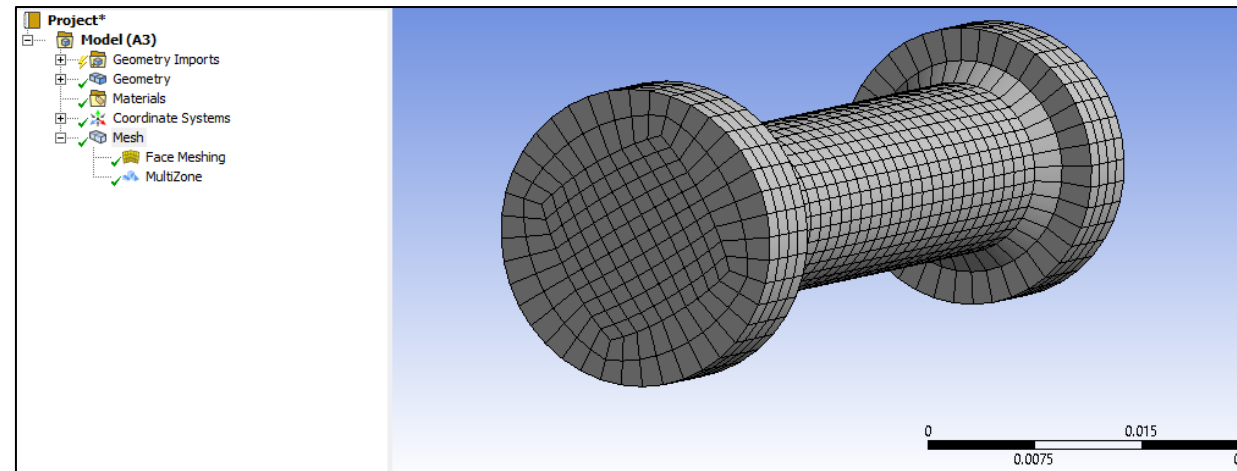
MultiZone Applied with no special inputs/selections – Automatic Hex



# Hex Meshing: Improved Hex Mesh for Simple Shapes

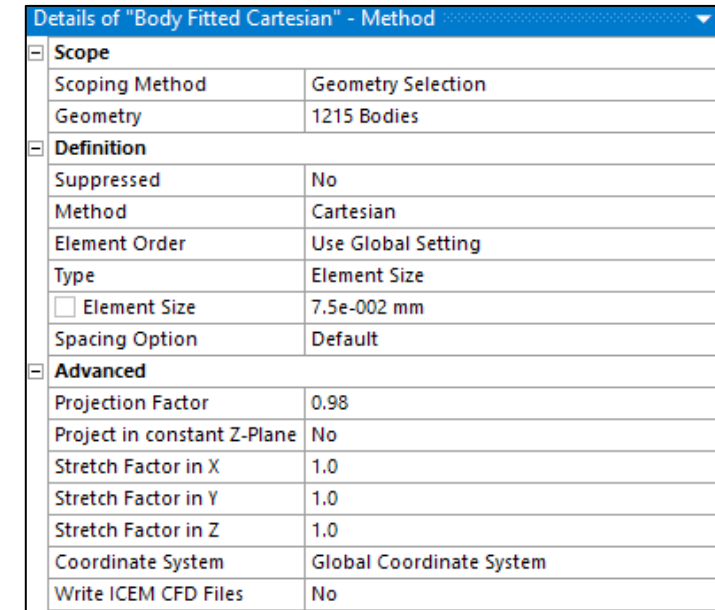
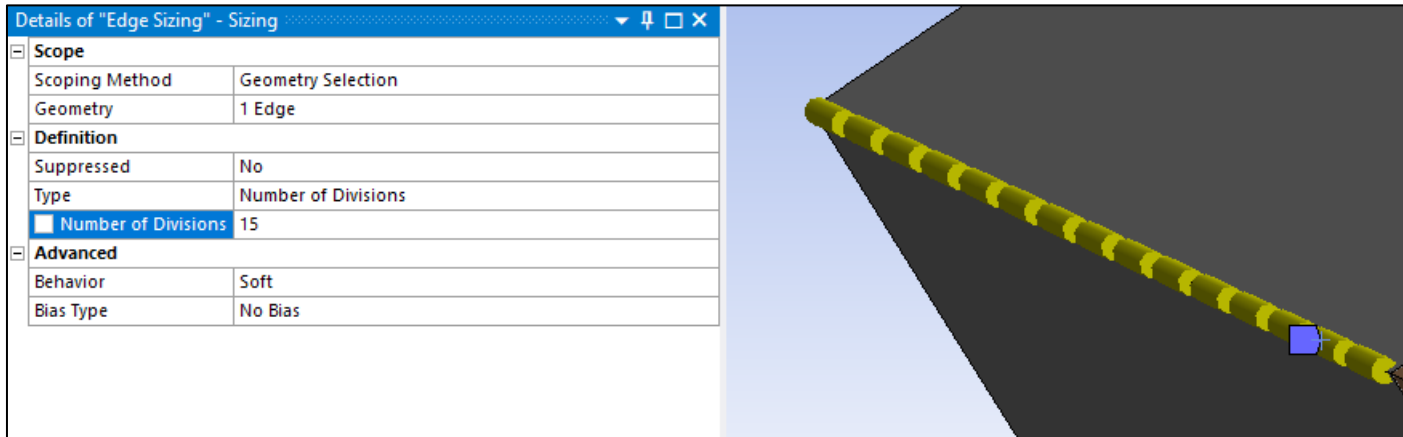


*O-Grid results are more smoothed for Explicit Phys. Pref.*



# Hex Meshing: Body Fitted Cartesian - Edge Sizing Support

- BF Cart now supports “Edge Sizing” control





# **Post Processing/Graphics**



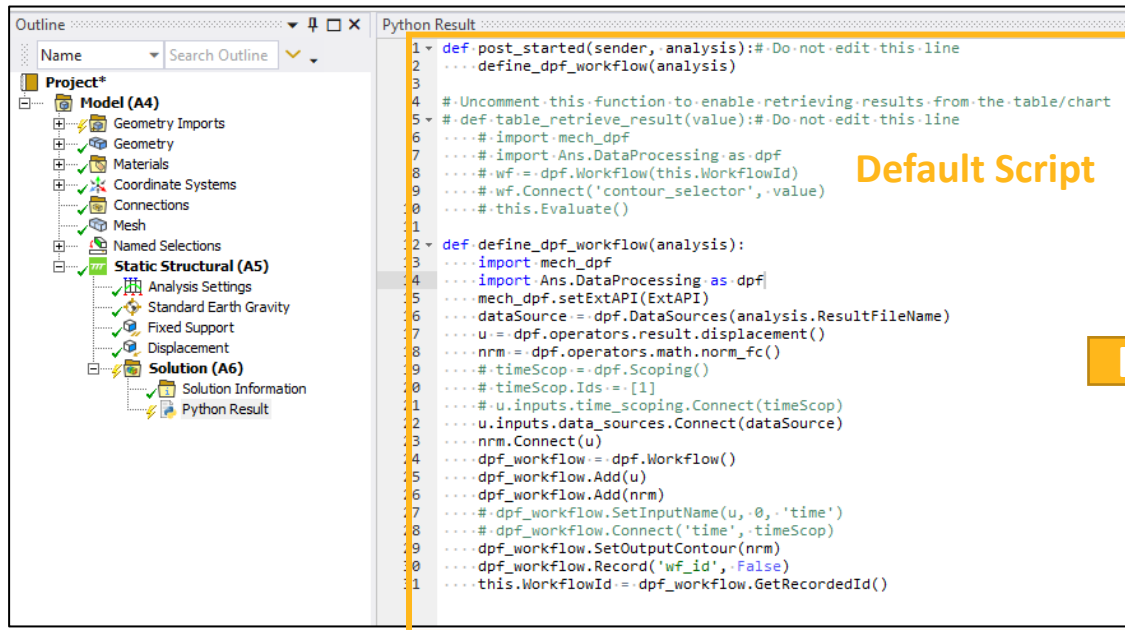


# / Performance Improvements

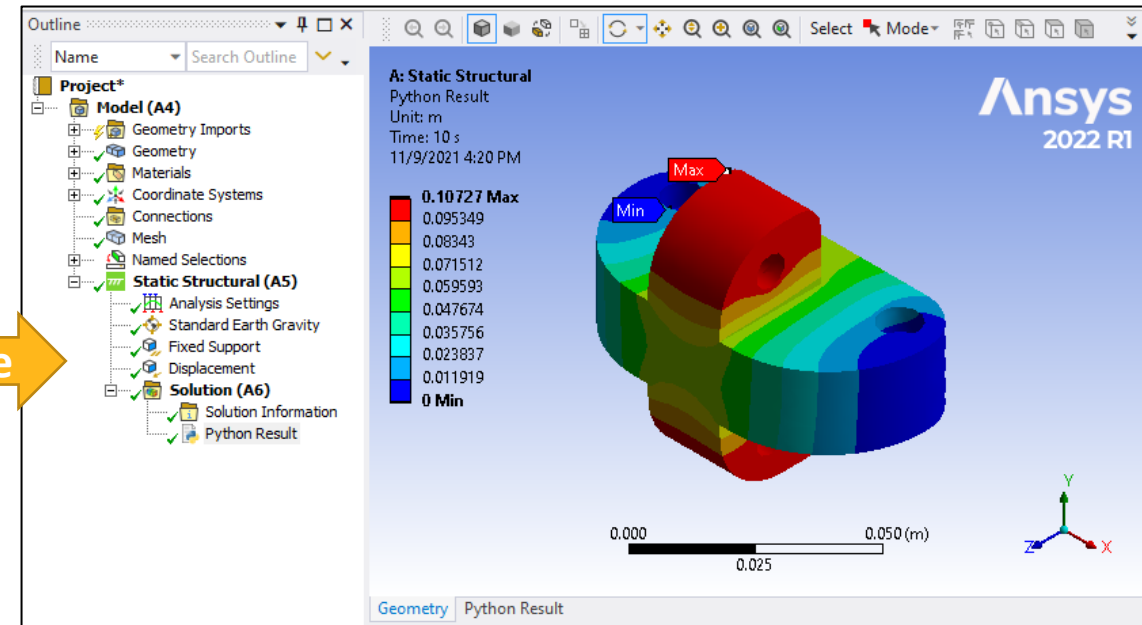
- Speedup time to click on Named Selections and loads for some models by 90%
- Speedup time to hide and show body groups for some models by 75%
- Improve time to click on Imported Contact on some models from infinity (impossible to click earlier – Mechanical would hang) down to a few seconds
- Improve UI behavior of “Thick Shells and Beams” button for some models – from minutes to instantaneous if there are no shells, beams or SPH particles present (no-op behavior)
- Speedup time to click on mesh for some models with lots of bodies (meshed + unmeshed parts) by 70% when there was a section plane enabled and by 45% when the section plane was disabled
- Speedup time to click on mesh for some models by 37%
- Utilize bulk-data APIs to further speedup the time (on top of the above-mentioned improvement) to click on Imported Contact for some models by 64%

# Python Result

- The **Python Result** object enables users to evaluate output quantities by executing an Iron-python script based on the Data Processing Framework (DPF) post-processing toolbox
- A **Python Result** object with a default script can be inserted under the **Solution** object. Users can create and evaluate a variety of scripts and display the results in Mechanical

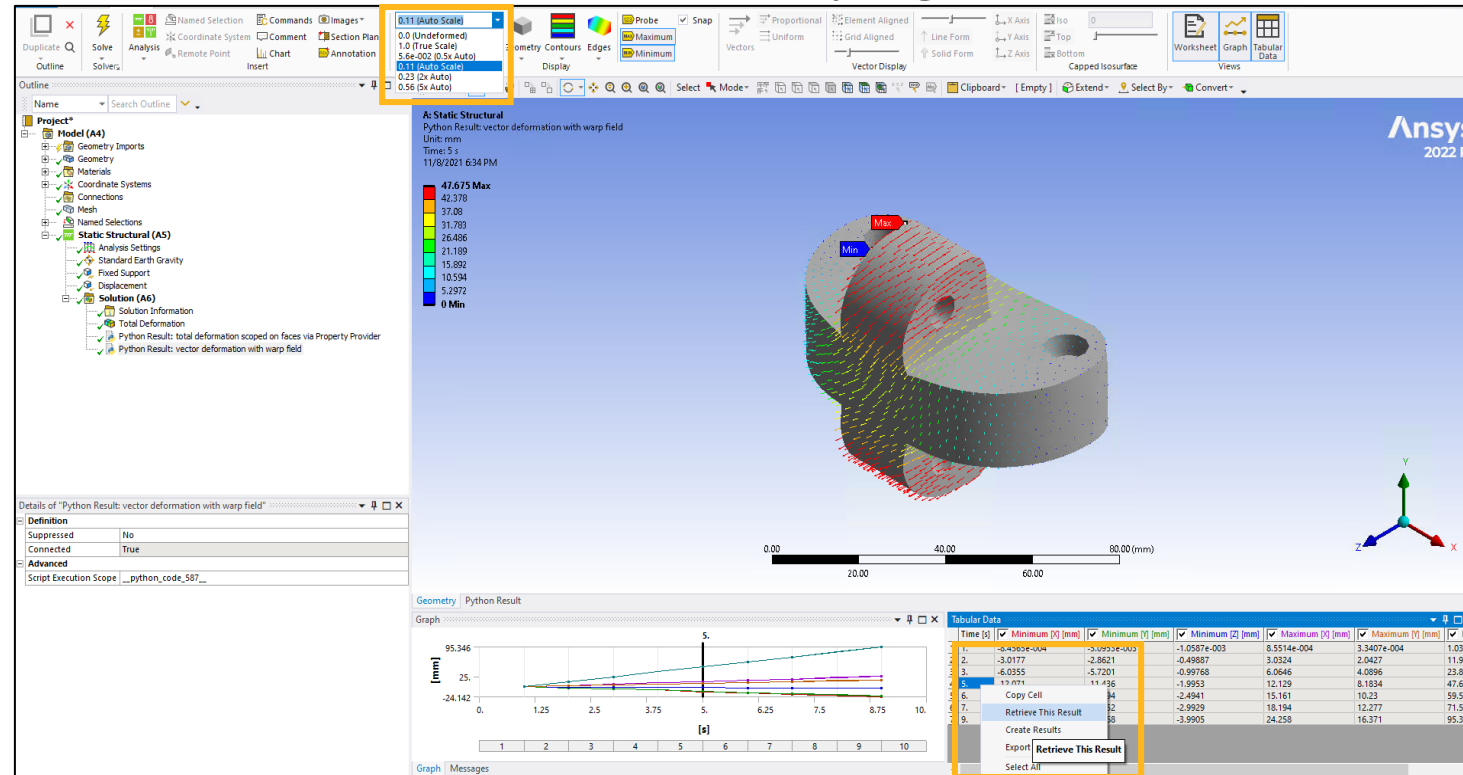


Evaluate



# Python Result

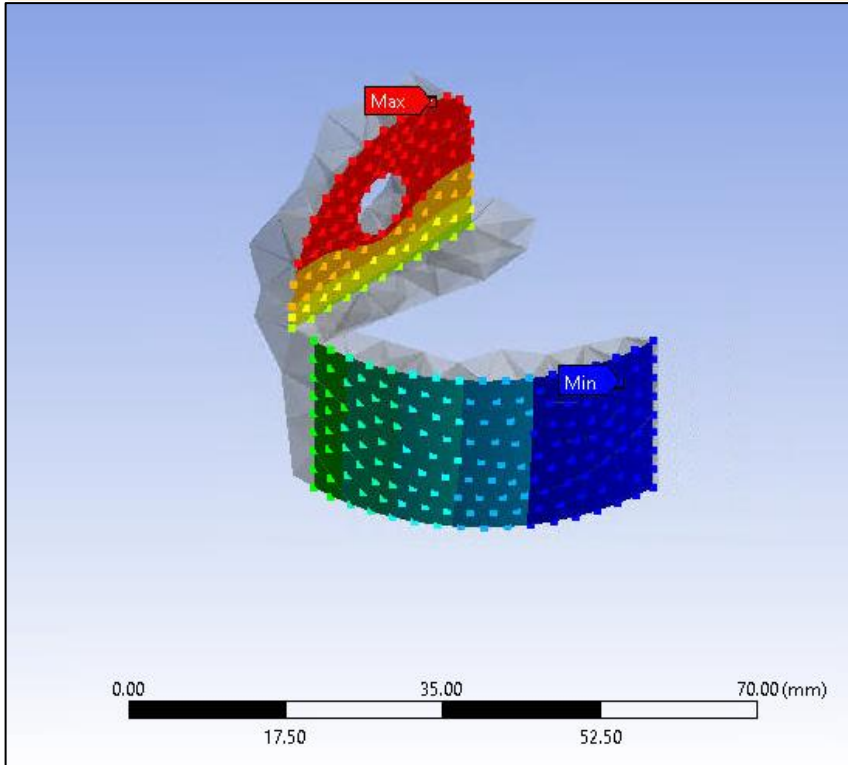
- Vector result is supported
- Retrieve result from the **Graph** window or **Tabular Data** window to view the results at a desired time point
- Deformation Scale Factor is available if the warping field is defined



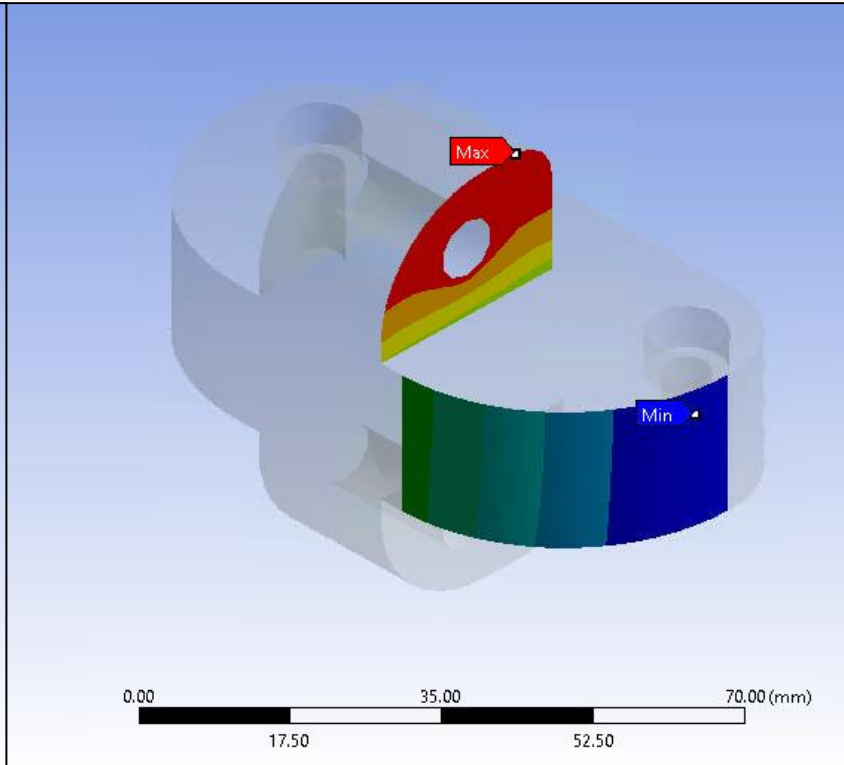
# / Python Result

- Different types are provided for a better contour result display

`dpf_workflow.SetOutputContour(nrm)`



`dpf_workflow.SetOutputContour(nrm,  
dpf.enums.GFXContourType.GeomFaceScoping)`



`dpf.enums.GFXContourType.  
FENodalScoping`

`dpf.enums.GFXContourType.  
FEElementalScoping`

`dpf.enums.GFXContourType.  
FEElementalFaceScoping`

`dpf.enums.GFXContourType.  
GeomBodyScoping`

`dpf.enums.GFXContourType.  
GeomFaceScoping`

`dpf.enums.GFXContourType.  
GeomEdgeScoping`

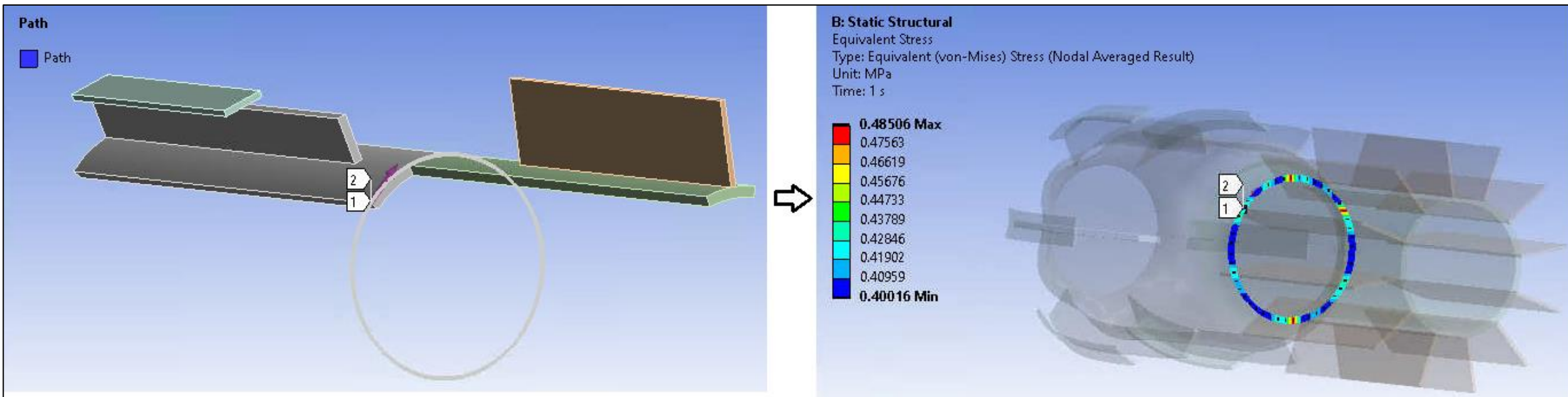
`dpf.enums.GFXContourType.  
GeomVertexScoping`

# Path Results Extensions: Results scoped to Paths Available for Models with Symmetry Conditions

- Create Paths and scope results to them, even if there are symmetry conditions present
- The model will be expanded, and results will be mapped wherever the Path intersects with the expanded model

Symmetry conditions include:

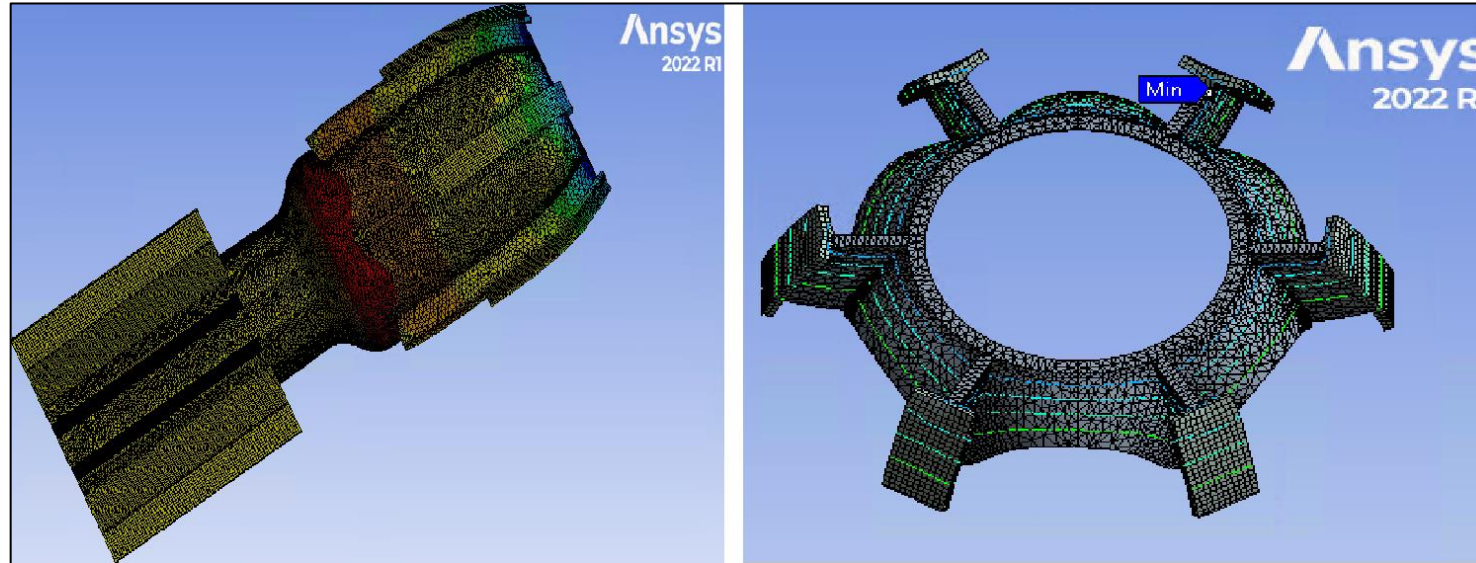
- Cyclic
- Pre-meshed cyclic
- General Axisymmetric
- Multi-stage cyclic





# / Result Preferences API

- Using the ResultPreference API, we can control some display properties of result objects



```
# Displays the result minimum annotation label
Graphics.ViewOptions.ResultPreference.ShowMinimum = True

# Sets the geometry view to CappedIsoSurface
Graphics.ViewOptions.ResultPreference.GeometryView = MechanicalEnums.Graphics.GeometryView.CappedIsosurface

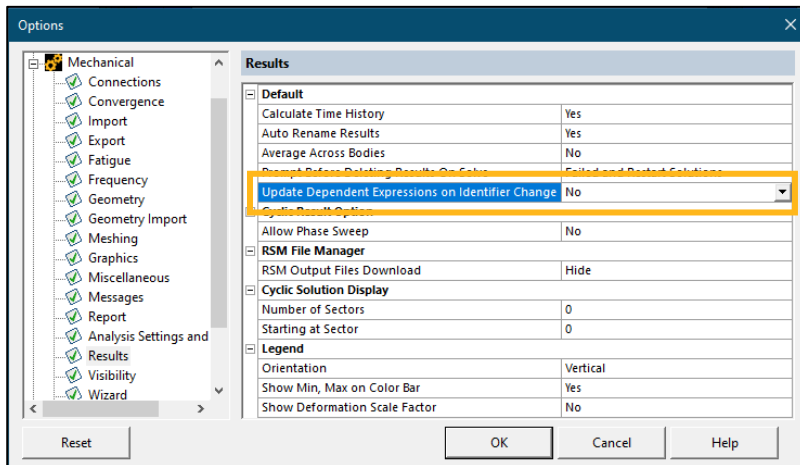
# Sets the capping value for CappedIsoSurface view
Graphics.ViewOptions.ResultPreference.IsoSurfaceValue = Quantity('0.55767')

# Sets the contour view to Isolines
Graphics.ViewOptions.ResultPreference.ContourView = MechanicalEnums.Graphics.ContourView.Isolines
```

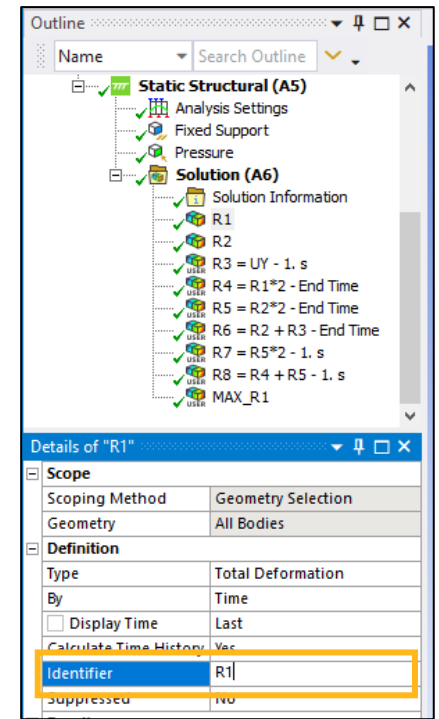


# Result - Identifiers Editing

- Users can now edit Identifiers after a result has been evaluated.
- Changes will automatically update and reevaluate all the expressions where that Identifier was used.
- Changing the Identifier in a result will never invalidate that result. The effect on the dependent results is controlled by the setting:

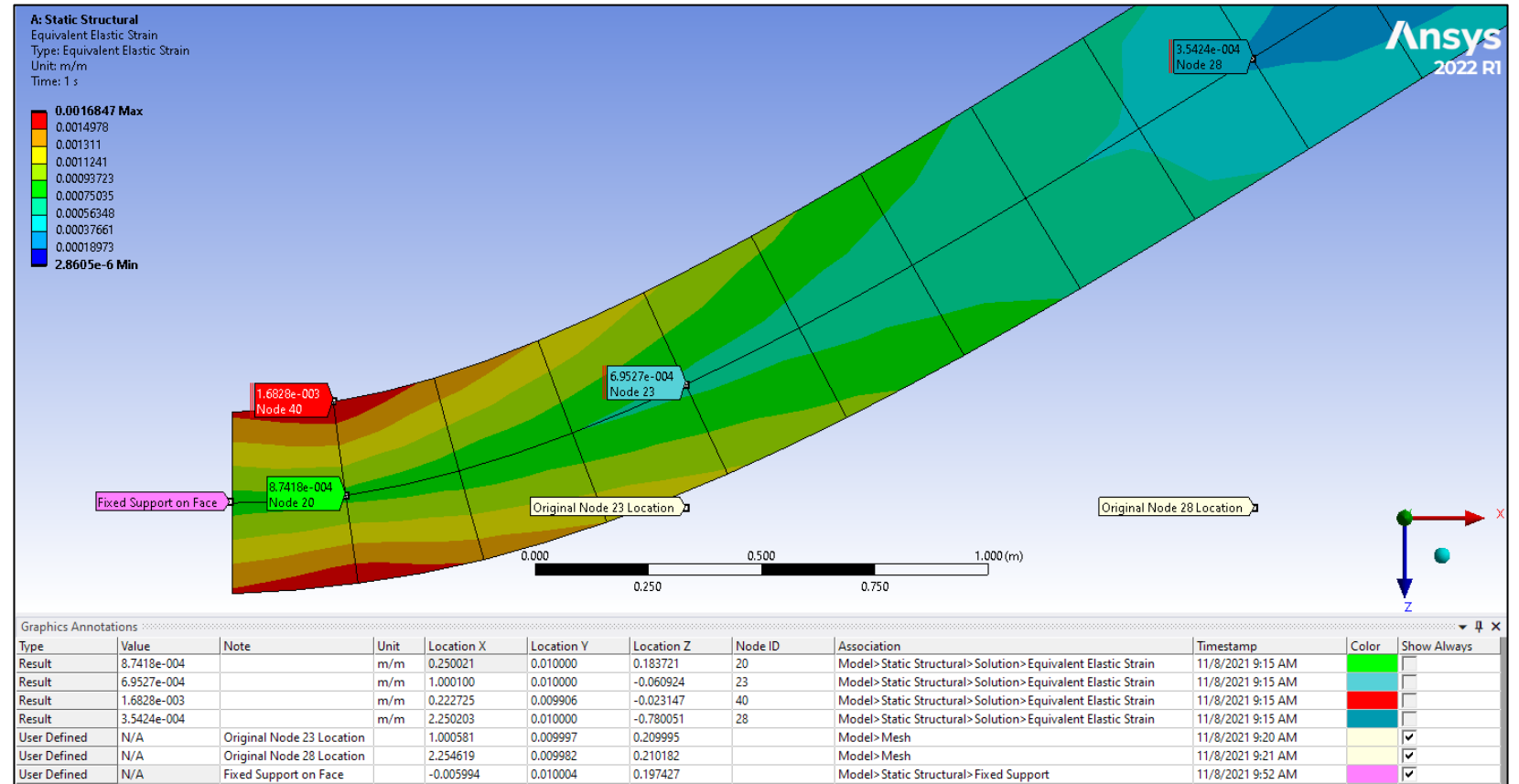


- Mechanical → Results → “Update Dependent Expressions on Identifier Change”



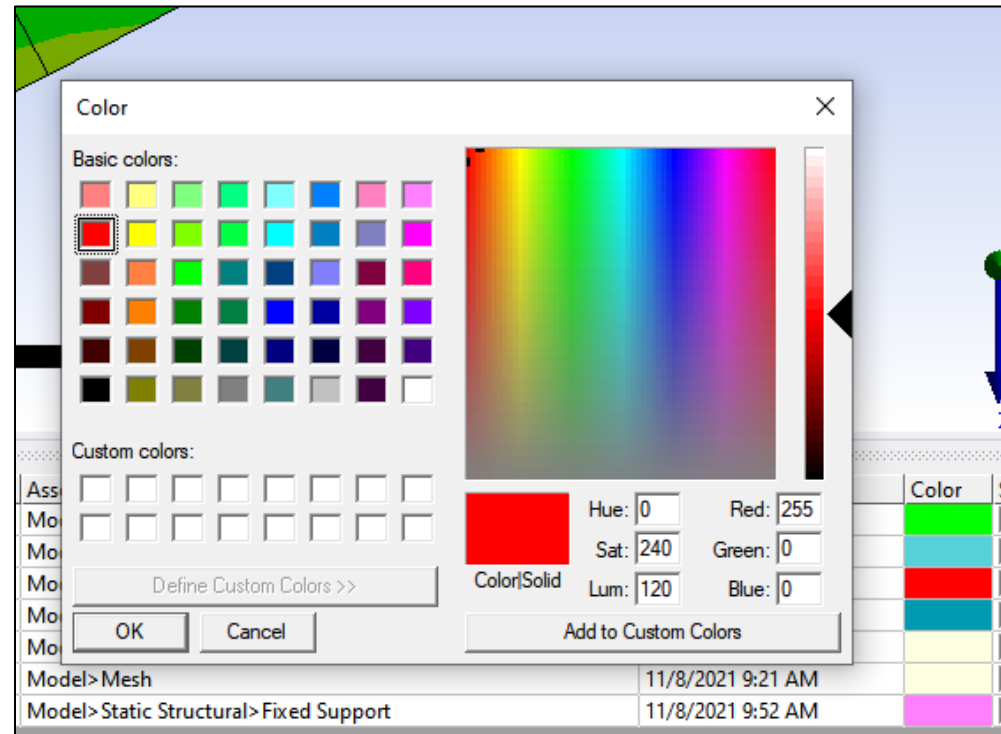
# Enhancements to User Annotations and Result Probe Labels

- The new color column allows user annotations and result probe labels to have customized colors
- The new “Show Always” column allows a user annotation to be displayed while on any object in the tree
- Snapping result probe labels now show the snapped-to node’s ID (if any mesh node was hit)



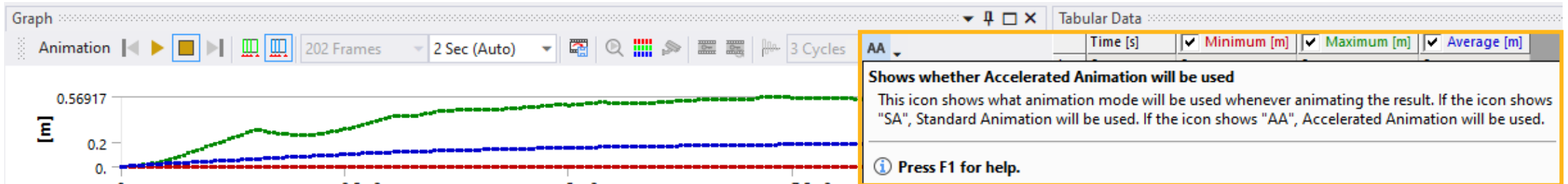
# Enhancements to User Annotations and Result Probe Labels

- To change the color of a label, double-click its color cell and select a new color in the pop-up window



# Accelerated Animation

- While not all results, models, or display modes are supported, those that do (most standard structural contour results scoped to bodies displayed with no wireframe, for example) can animate much faster with less memory overhead. Furthermore, it supports full user interaction during animation creation (such as model rotation) rather than locking up the UI as in Standard Animation.

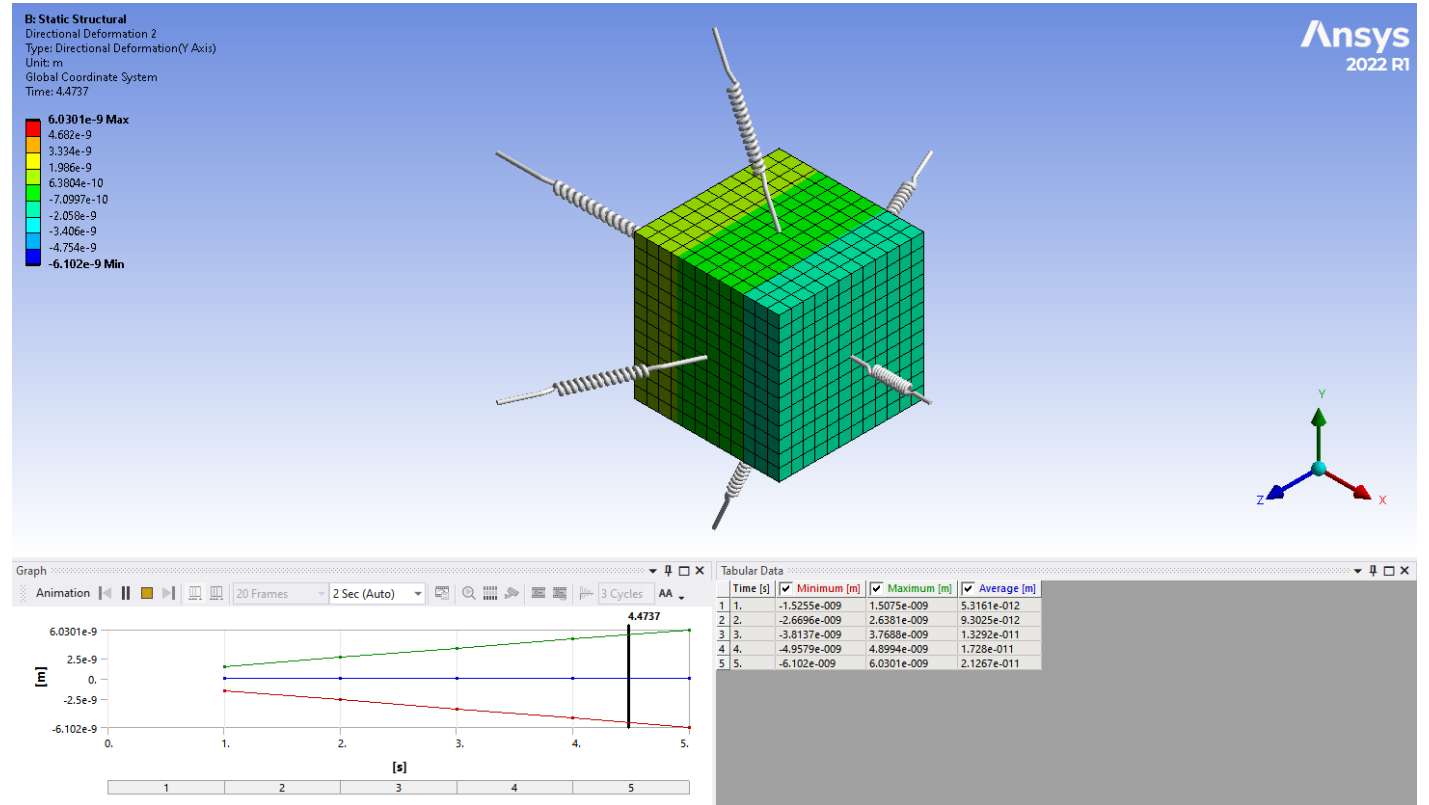


- Can be controlled via the Animation Mode Mechanical Graphics preference

Graphics	
Use Deformed Edge For Slice ISO Option	Yes
Animation Mode	Standard
Mesh Translucency	Program Controlled
Probe Font (Windows Only)	Standard
Font	Accelerated (Beta)
	Arial

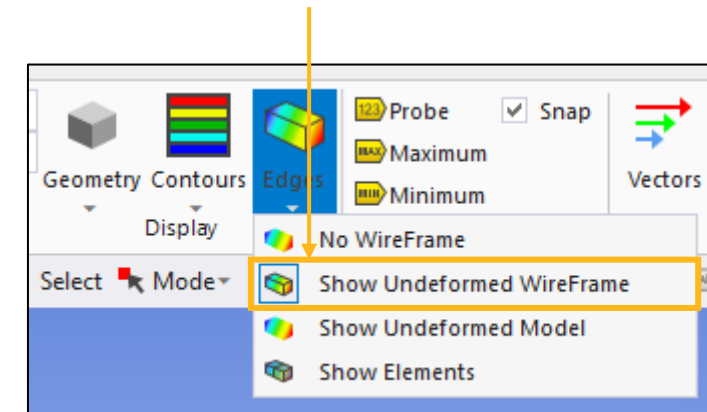
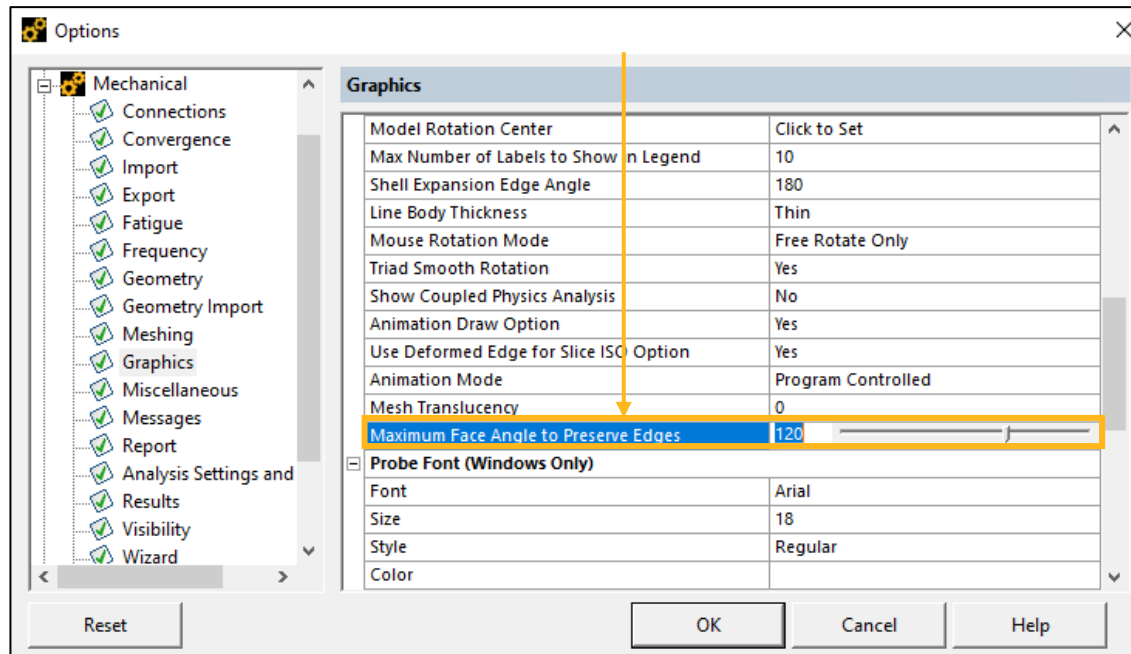
# Accelerated Animation

- Accelerated Animation is now the default animator for supported scenarios
- Accelerated Animation now supports rendering mesh element outlines and springs



# Face Angle Display Preference

- Provide display preference to change the maximum face angle in order to better preserve the edges between faces when showing undeformed wireframe on results
- Default is 120 degrees and range is from 0 to 180 degrees

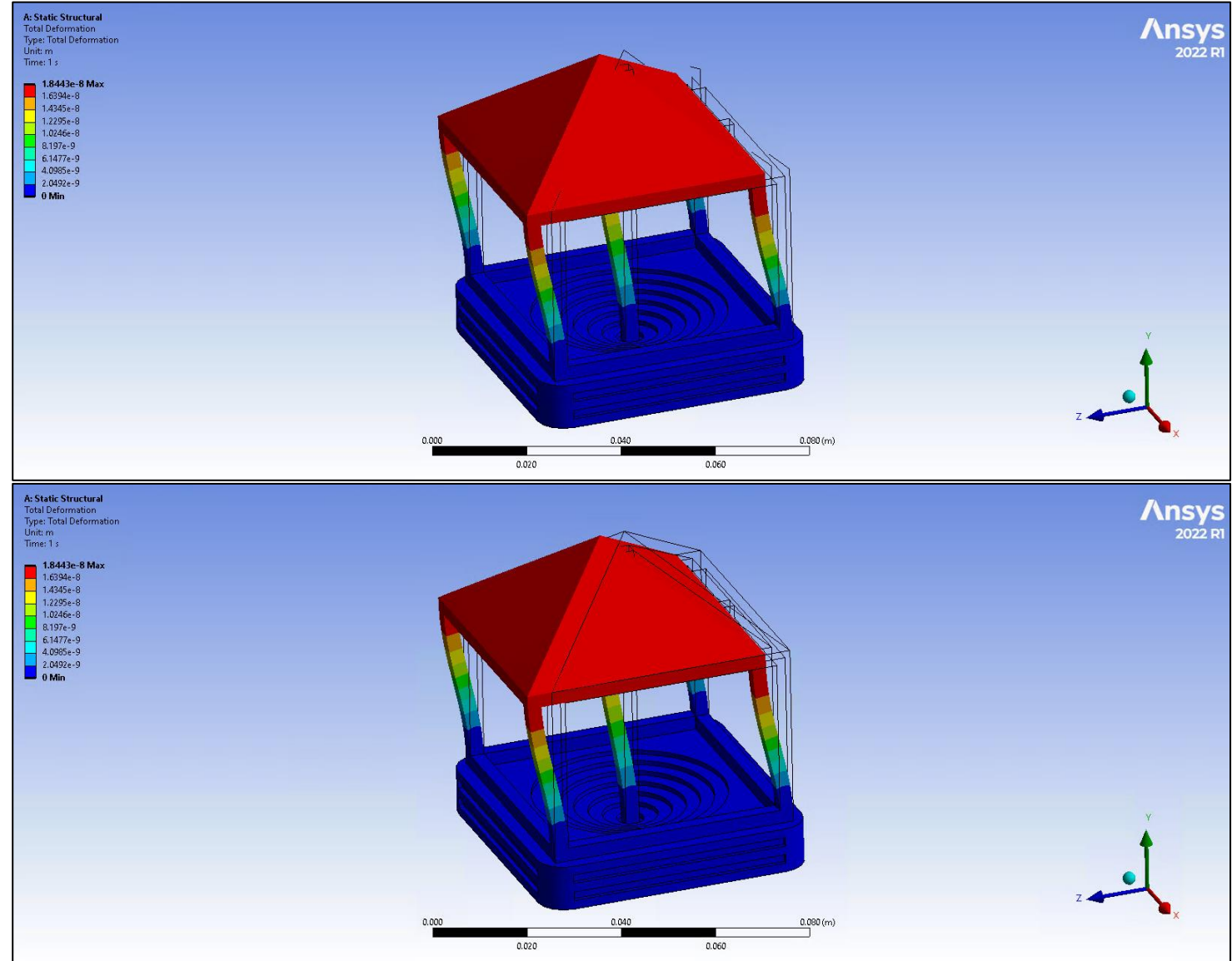




# Face Angle Display Preference

Default undeformed wireframe on result shows missing edges due to insufficient maximum face angle of 120 degrees

Undeformed wireframe display is fixed with a maximum face angle of 130 degrees



# Composites

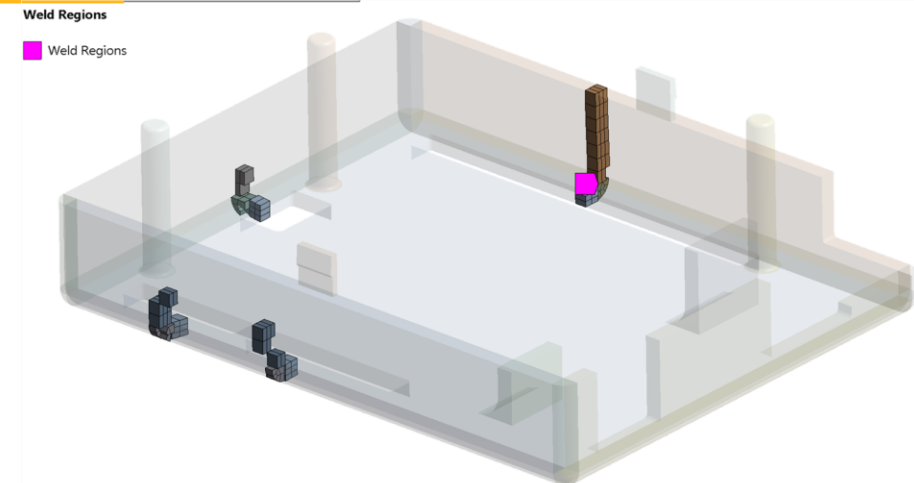
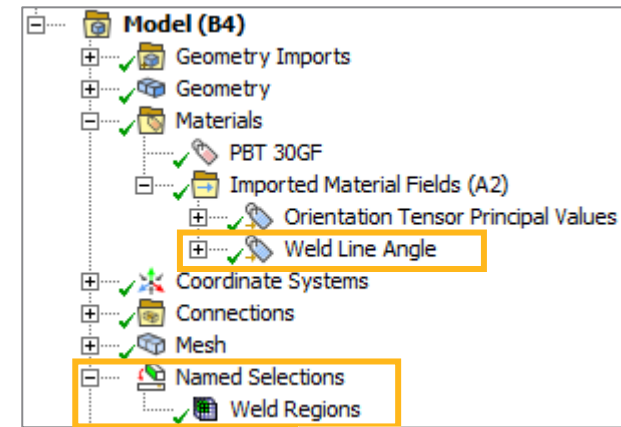
Short Fiber Workflow

**Ansys**

# Import of Weld Lines

- With the Injection Molding Data system, you can now import Weld Lines results from Moldex3D® and Moldflow®
- You can map Weld Lines as element-based Named Selections in Mechanical. You can use them for visualization and post-processing purposes, as well as to assign degraded material properties
- The Weld Line Angle is imported and mapped as a Material Field

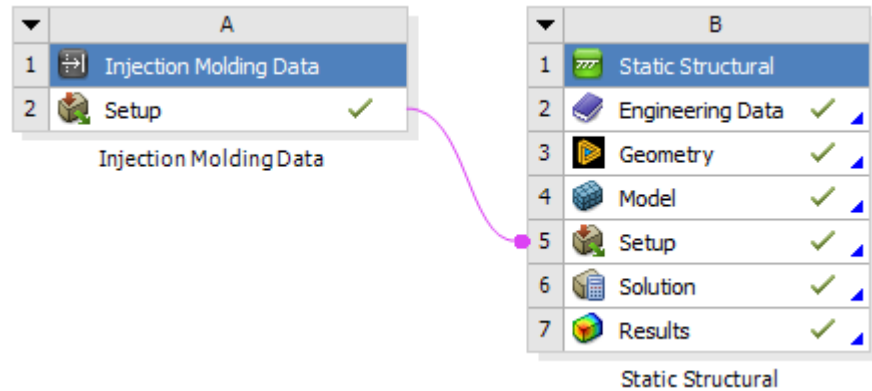
17	Injection Molding Simulation Data	
18	Mesh File	D:\casing.cdb
19	Fiber Orientation Tensor File	D:\casing.o2d
20	Initial Stress File	
21	Fiber Volume Fraction File	
22	Weld Lines File	D:\casing.nwd



# / Injection Molding Data: Support for Unreinforced Materials

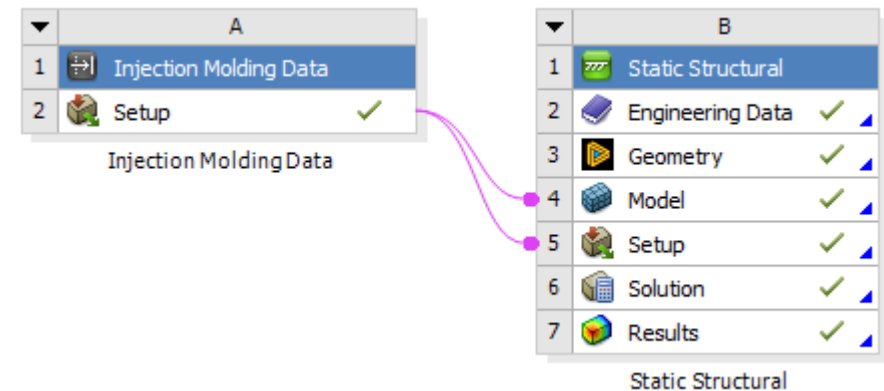
- Import Initial Stress and Weld Line results independently of the fiber orientation tensor. This is especially useful if you are dealing with unreinforced materials

## Import Initial Stress



Injection Molding Simulation Data	
Mesh File	D:\casing.cdb
Fiber Orientation Tensor File	
Initial Stress File	D:\casing.ist
Fiber Volume Fraction File	
Weld Lines File	

## Import Initial Stress and Weld Lines



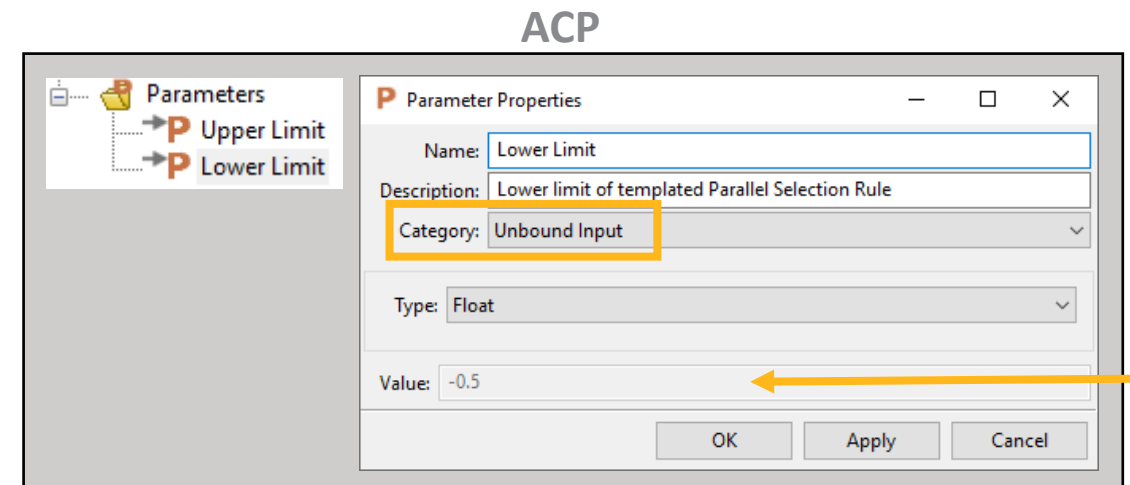
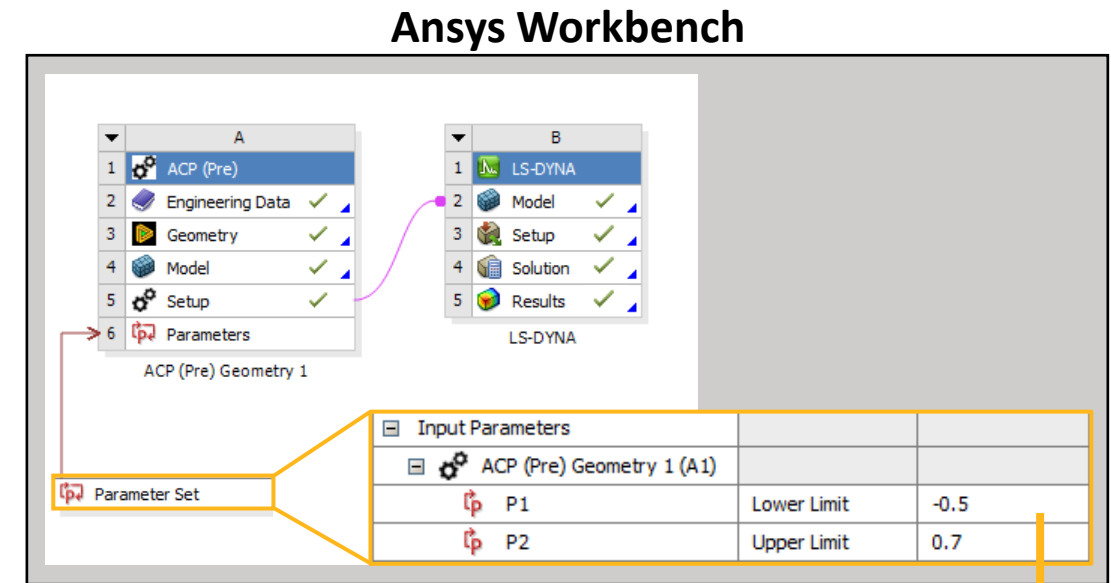
Injection Molding Simulation Data	
Mesh File	D:\casing.cdb
Fiber Orientation Tensor File	
Initial Stress File	D:\casing.ist
Fiber Volume Fraction File	
Weld Lines File	D:\casing.nwd

# **Ansys Composite PrepPost**



# / Unbound Input Parameter

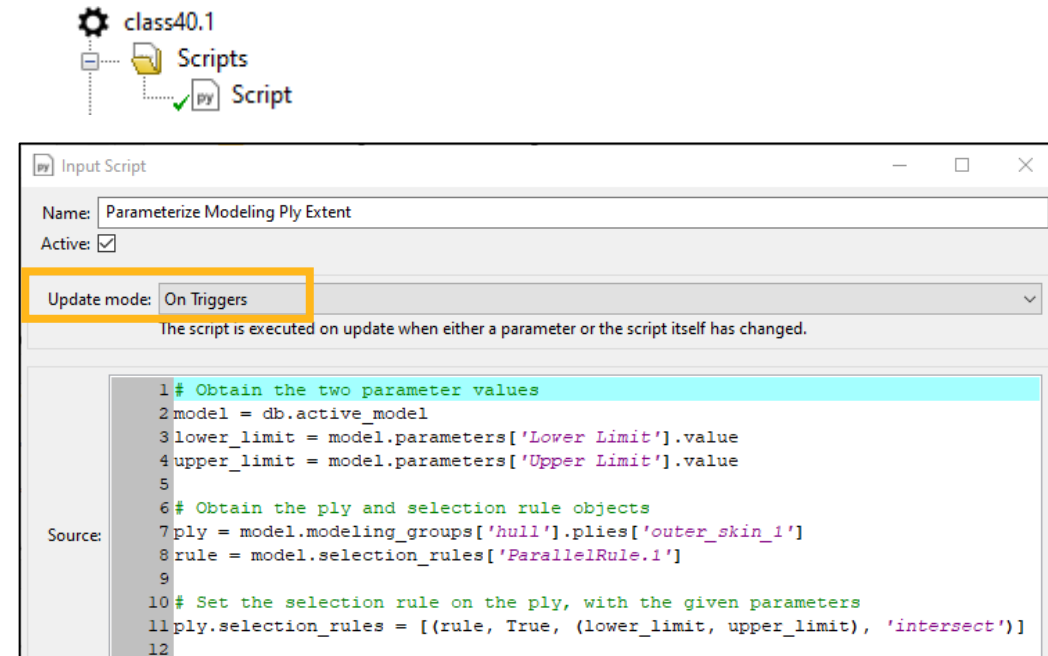
- New **Parameter** type **Unbound Input** which is not linked to an ACP object
- It can be used for any parameterization or customization
- The main use of this parameter type is in combination with the new Python Script object (see next slide)
- Supported by Ansys Workbench





# / New Python Script Object

- The **Script Object** serves as a command snippet. It allows the execution of arbitrary Python code.
- The **Update Mode** defines when the code is run:
  - a) Either as part of the update logic of ACP (**on triggers**)
  - b) or before each update (**always**)
  - c) or if manually triggered (**manual**)
- Use cases:
  - Complex parametrization
  - Customization and automatization of the lay-up modelling in ACP Pre
  - Implementation of custom post-processing features in ACP Post



# How to Parametrize Ply Coverage (independent of the CAD geometry)

**1.** Create **Unbound Parameters** in ACP

**2.** Set values in WB

**3.** Retrieve them in a **Python Script**

**4.** Change the template Rule of the Modeling Ply

→ Parametrization of template rules now possible!

**Parameter Set**

Input Parameters		
ACP (Pre) Geometry 1 (A1)		
P1	Lower Limit	-0.5
P2	Upper Limit	0.7

**ACP Model**

Scripts

Ply Extent

Update mode: On Triggers  
The script is executed on update when either a parameter or the script itself has changed.

```
1 # Obtain the two parameter values
2 model = db.active_model
3 lower_limit = model.parameters['Lower Limit'].value
4 upper_limit = model.parameters['Upper Limit'].value
5
6 # Obtain the ply and selection rule objects
7 ply = model.modeling_groups['hull'].plies['outer_skin_1']
8 rule = model.selection_rules['ParallelRule.1']
9
10 # Set the selection rule on the ply, with the given parameters
11 ply.selection_rules = [(rule, True, (lower_limit, upper_limit), 'intersect')]
12
```

**Modeling Ply Properties**

Name: outer\_skin\_1  
ID: outer\_skin\_1

General | Draping | Rules | Thickness

Selection rule	Type	Template	Parameter 1	Parameter 2
ParallelRule.1	Intersect	True	-0.5	0.7

**Outcome**

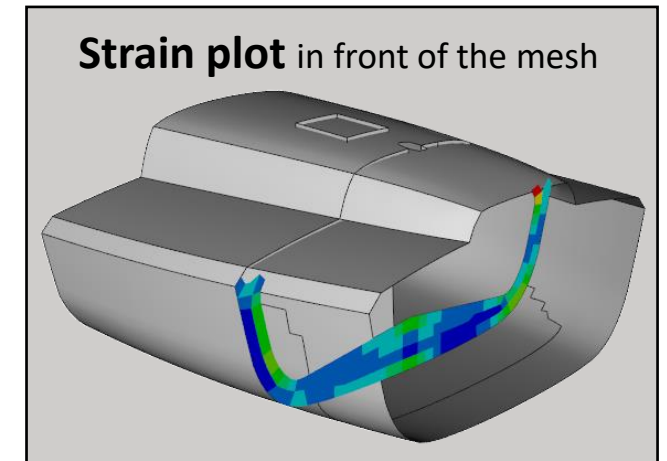
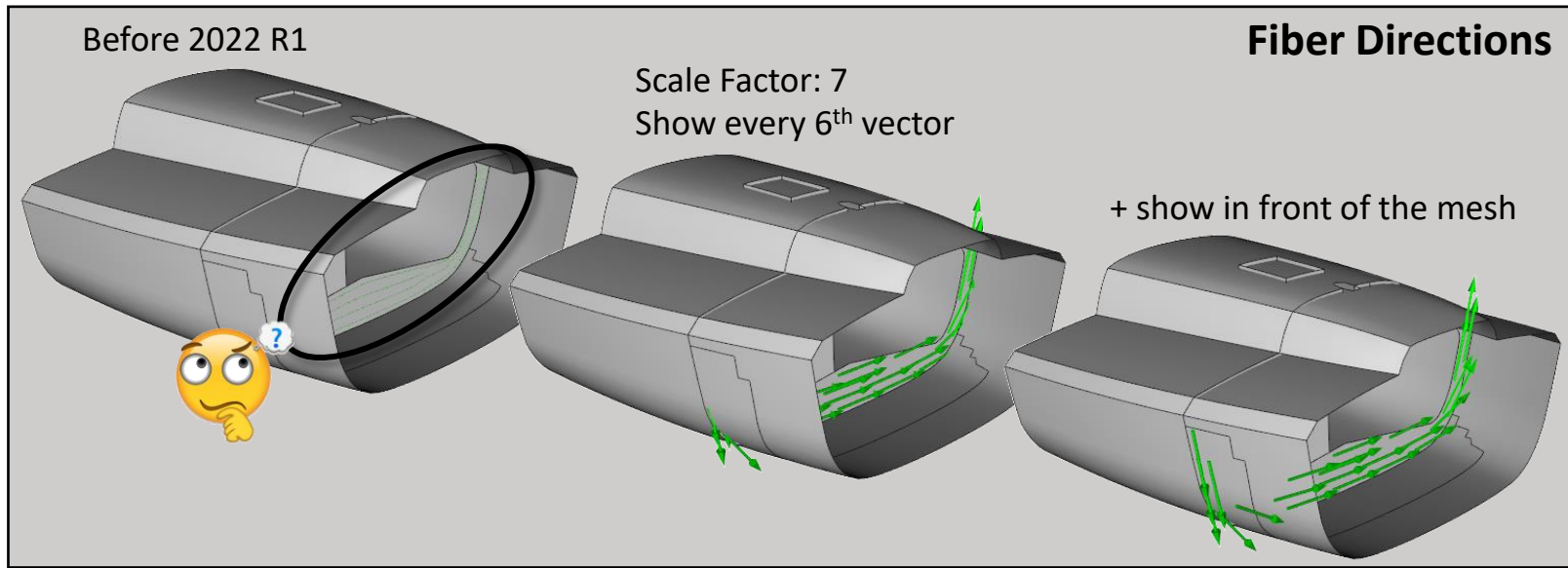
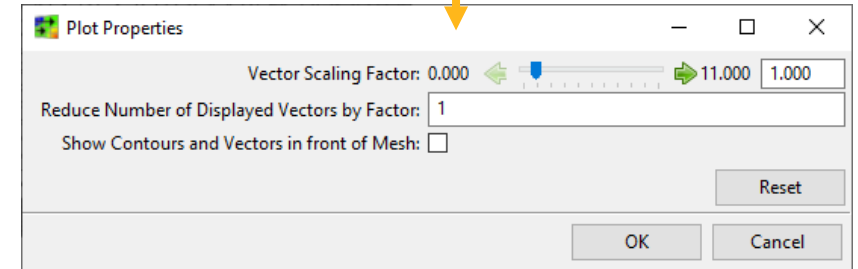
Modeling Groups

- hull
- outer\_skin\_1

3D Model: A car body model with a purple ply coverage on the side panel.

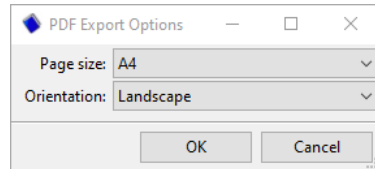
# Plot Visualization

- New options to customize all direction plots
  - Scale the vectors
  - Set number of displayed vectors
- And show a plot in front or behind of the mesh!



# / Ply Book

- The Ply Book allows you to create a report for production
- It is improved in various areas
  - Customization
  - New automatic visualization of Rosettes (reference systems)
  - New live preview
  - New PDF export options

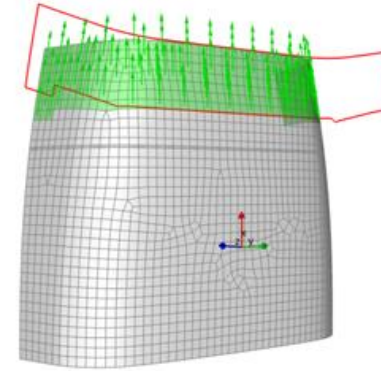


- The new plot visualization also improves the readability of the ply book

Ansys Composite PrepPost

## Chapter - hull

P1\_\_outer\_skin\_1

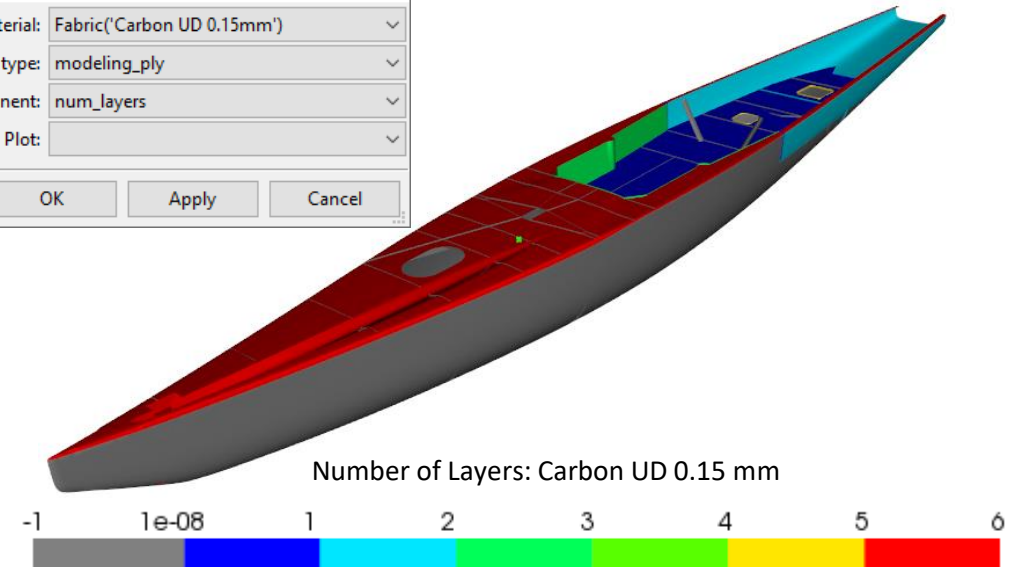
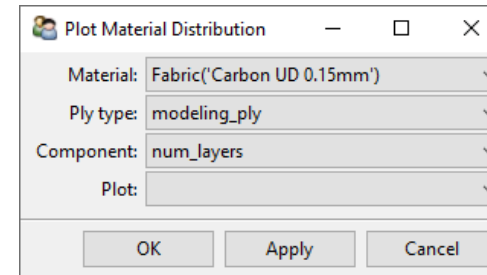
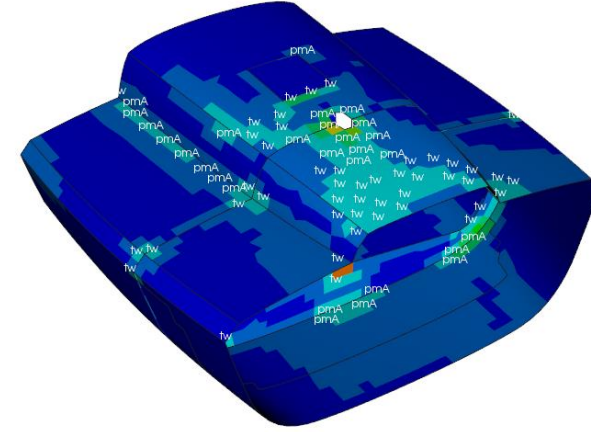
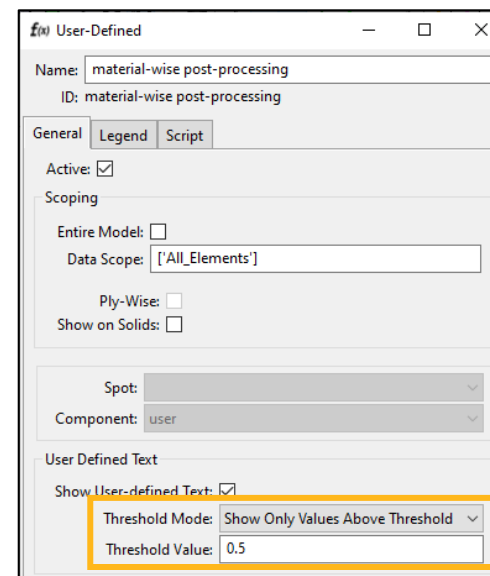


- Ply Group Name: hull (hull)
- Modeling Ply Name: outer\_skin\_1 (outer\_skin\_1)
- Ply Name: P1\_\_outer\_skin\_1 (ProductionPly)
- Orientation: 0.0
- Material: E-Glas
- Draping Seed Point: (-4.7643, -0.0, -0.2401)

Parameter	Value
Thickness	0.00027000000000000006
Area	3.703239453349462
Cost	0.0
Weight	1.9998242293937767

# Other Improvements

- User-Defined Plots
  - Threshold for elemental text labels to hide labels above or below the value
- Python Scripts and Wizards for customers
  - New examples to get started with scripting
  - Wizard to plot material distributions (see example)
  - Wizard to compute material-wise failure





# Structural Optimization

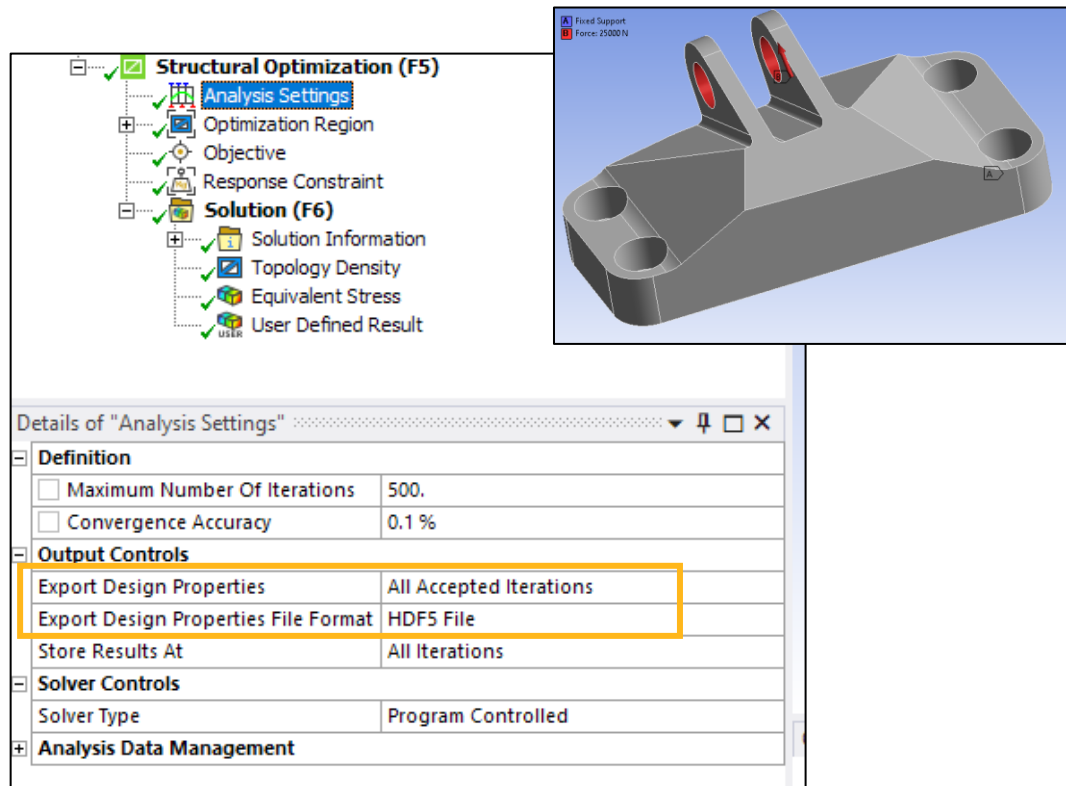
**Ansys**

# Export Design Properties

This new feature permits users to visualize the results (deformation, stress, eigen-mode, etc) on the final design, giving them the opportunity to quickly examine and validate the mechanical behavior.

## (1) Optimization set-up:

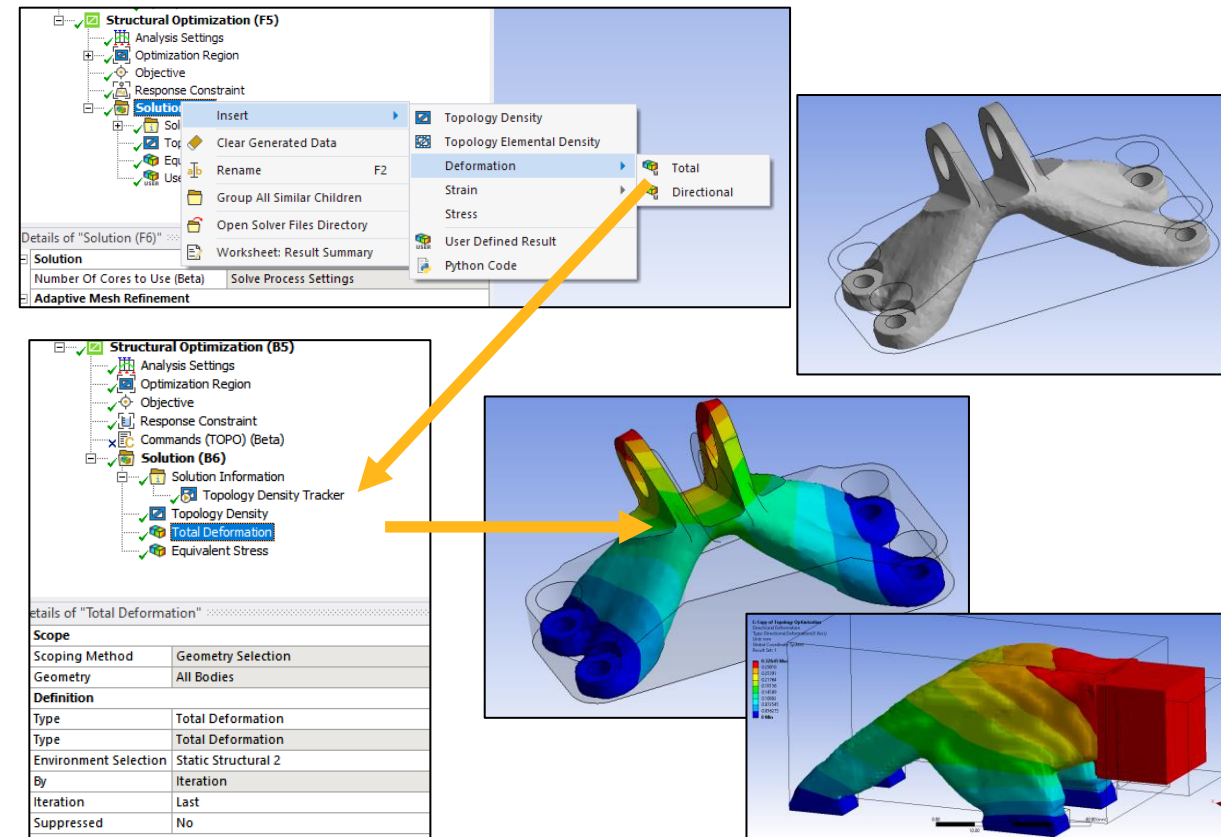
Activate the option



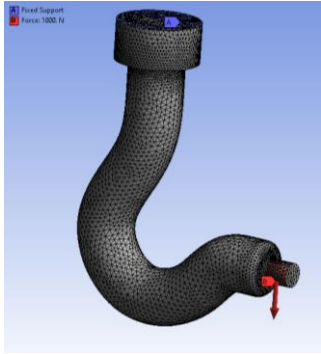
## (2) Run the optimization

## (3) Post:

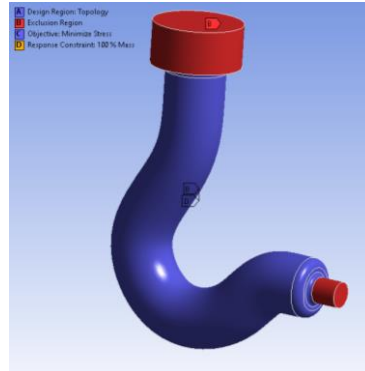
Define your display



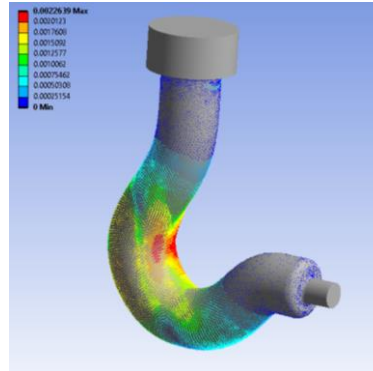
# Maximum Principal Stress



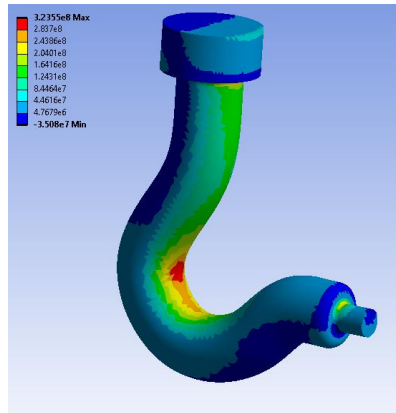
Mesh &  
Structural  
Set-up



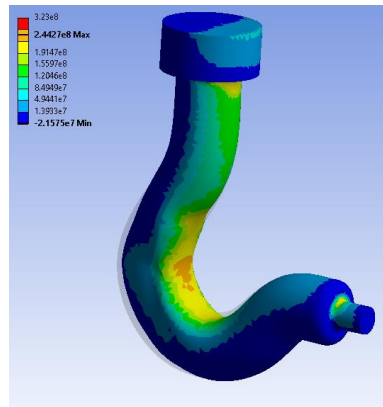
Shape  
Optimization  
Set-up



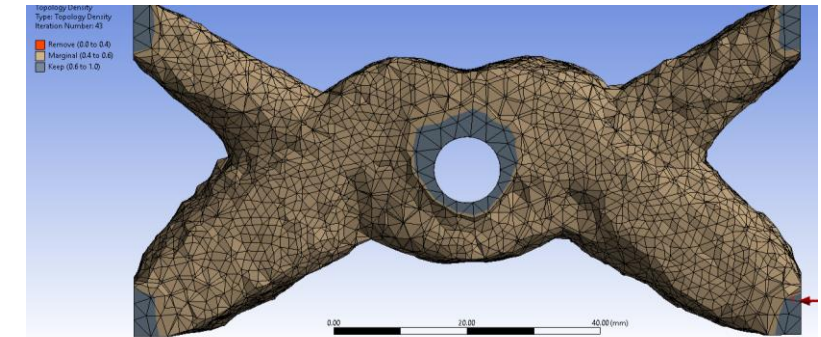
Results &  
direct  
validation



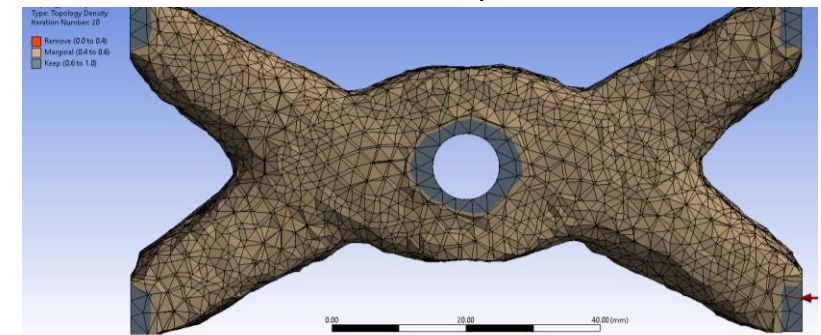
MPS  
– 25 %



- ✓ MPS is available alongside Von Mises eq. norm
- ✓ Available for Topology (SIMP or Level-Set), Shape and Lattice Optimization
- ✓ Can be consumed as an objective or as a constraint



MPS-driven optimization



VonMises-driven optimization

# / User Defined Criterion

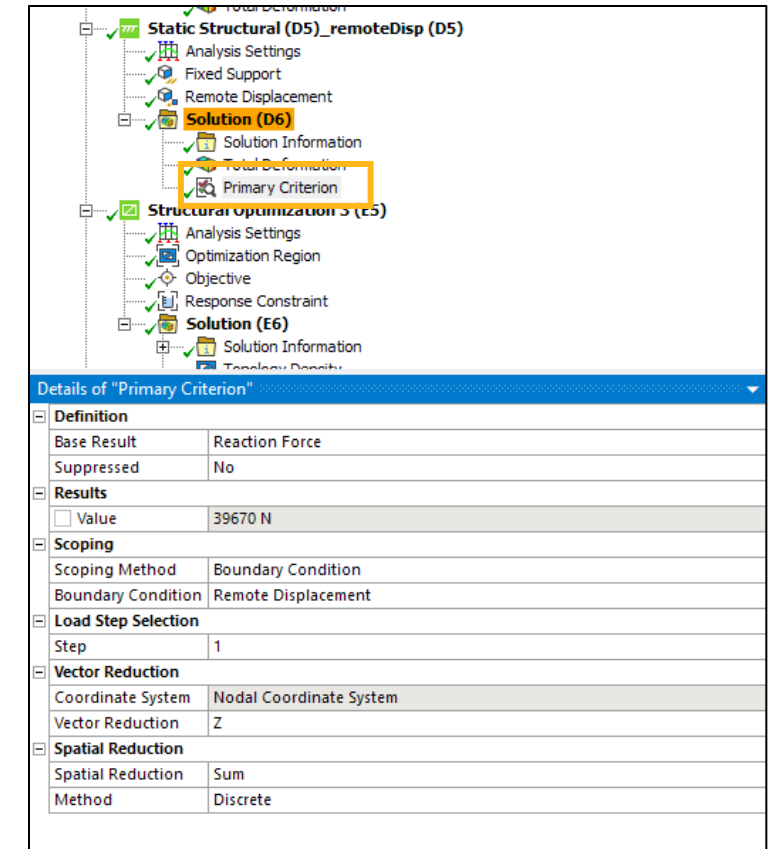
Since 2021 R1, a new capability has been introduced in a bid to create user-defined criterion in the upstream Static Linear Analysis system. This criterion (primary/composite) can then be consumed as objective or constraint in the downstream Structural Optimization

This feature has been extended:

- New base result: displacement, **rotation**, reaction-force or **reaction-moment**
- New scoping: **remote point** or support of boundary condition (eg **remote-force, moment, remote-displacement**)

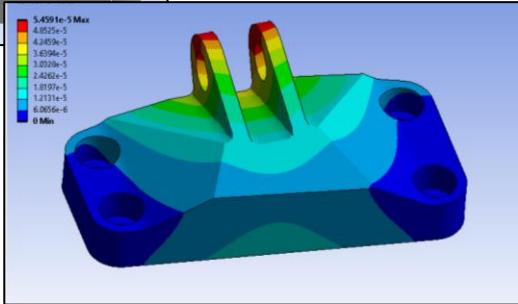
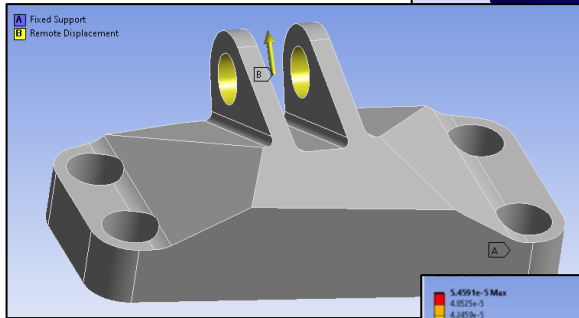
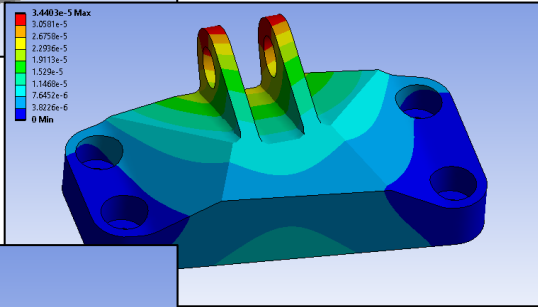
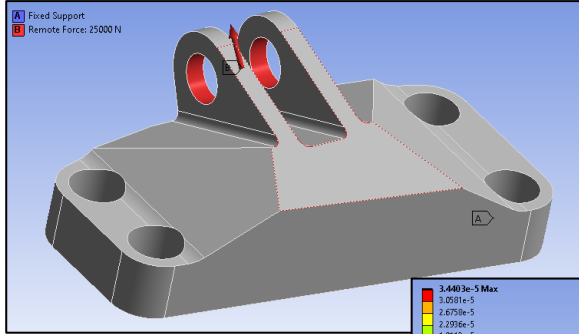
UDC has also been introduced in **Modal-Analysis**:

- **Single Frequency** criterion that aims to control the i-th eigenfrequency of interest
- **Robust Frequency** criterion that aims to control the i-th eigenfrequency of interest while handling efficiently mode-crossing





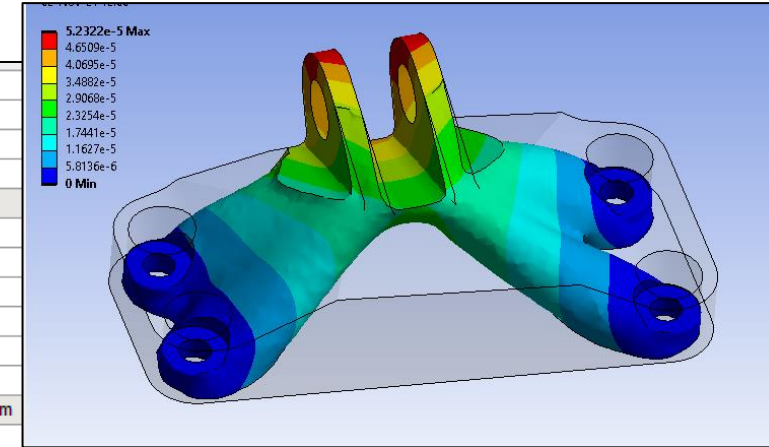
# Example #1-2



$\left\{ \begin{array}{l} \min_{\Omega} (primary\ criterion) \\ vol < 40\% \end{array} \right.$

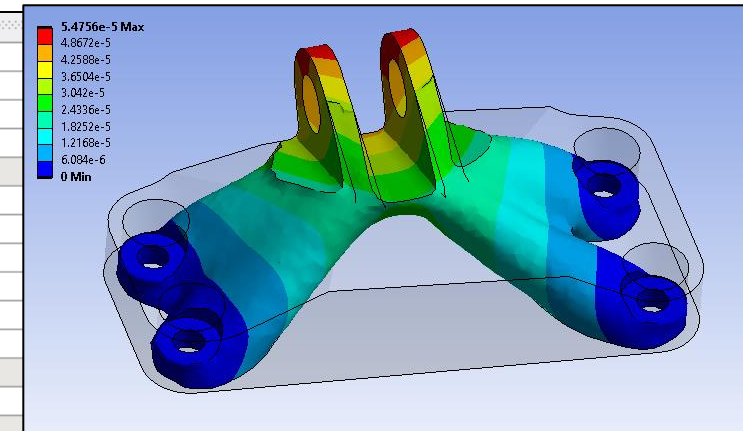
$\left\{ \begin{array}{l} \max_{\Omega} (primary\ criterion) \\ vol < 40\% \end{array} \right.$

Definition	
Base Result	Displacement
Suppressed	No
Results	
<input type="checkbox"/> Value	2.1427e-005 m
Scoping	
Scoping Method	Boundary Condition
Boundary Condition	Remote Force
Load Step Selection	
Step	1
Spatial Reduction	
Coordinate System	Nodal Coordinate System
Reference Type	Constant
Reference Value	0.
Vector Reduction	Z
Spatial Reduction	
Spatial Reduction	Average
Method	Discrete



z-disp of the remote-force support

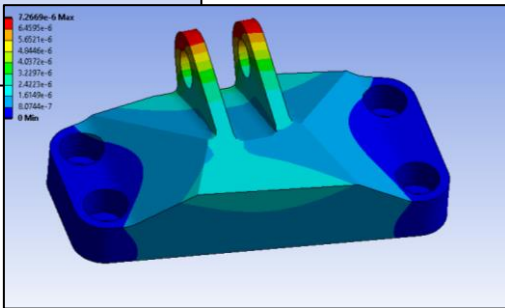
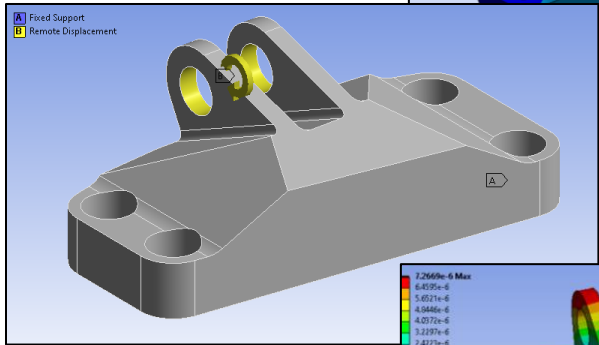
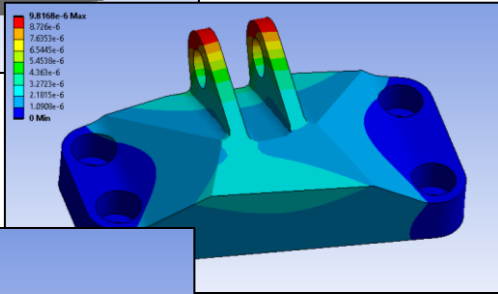
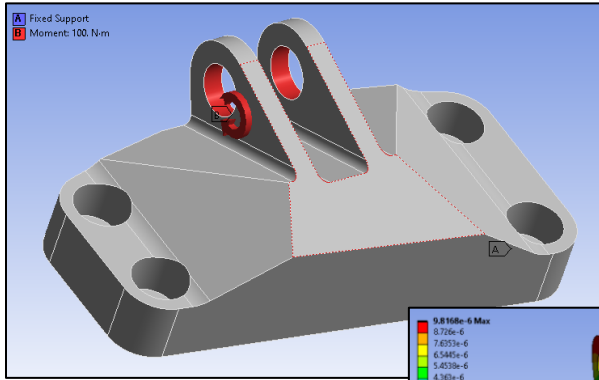
Details of "Primary Criterion"	
Definition	
Base Result	Reaction Force
Suppressed	No
Results	
<input type="checkbox"/> Value	39670 N
Scoping	
Scoping Method	Boundary Condition
Boundary Condition	Remote Displacement
Load Step Selection	
Step	1
Spatial Reduction	
Coordinate System	Nodal Coordinate System
Reference Type	Constant
Reference Value	0.
Vector Reduction	Z
Spatial Reduction	
Spatial Reduction	Sum
Method	Discrete



z-reaction-force  
at the remote-displacement support

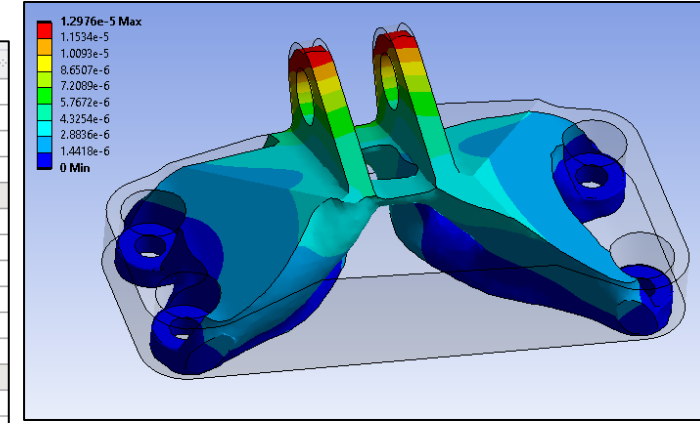


# Example #3-4



$\left\{ \begin{array}{l} \max_{\Omega} (primary\ criterion) \\ vol < 40\% \end{array} \right.$

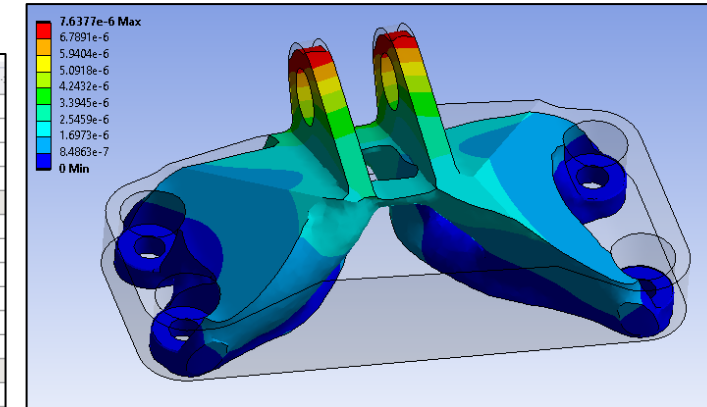
Details of "Primary Criterion"	
Definition	
Base Result	Rotation
Suppressed	No
Results	
Value	-1.3509e-002 °
Scoping	
Scoping Method	Boundary Condition
Boundary Condition	Moment
Load Step Selection	1
Vector Reduction	
Coordinate System	Nodal Coordinate System
Vector Reduction	X
Spatial Reduction	
Spatial Reduction	Average
Method	Discrete



x-rotation of the remote-moment support

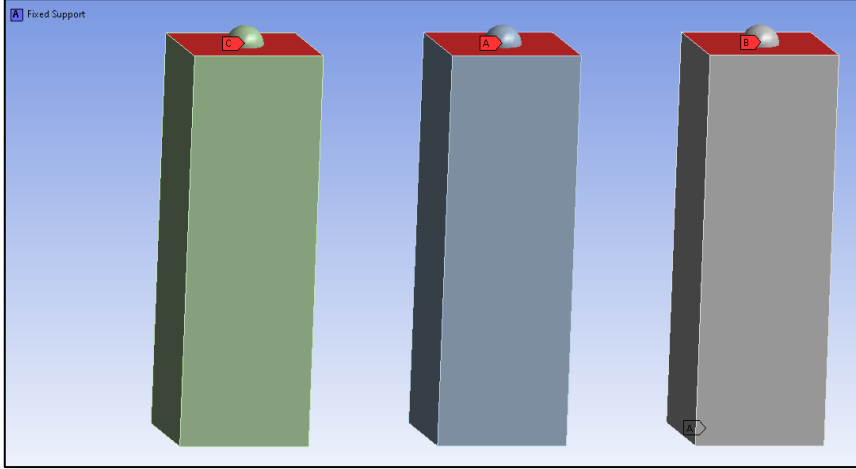
$\left\{ \begin{array}{l} \min_{\Omega} (primary\ criterion) \\ vol < 40\% \end{array} \right.$

Details of "Primary Criterion"	
Definition	
Base Result	Reaction Moment
Suppressed	No
Results	
Value	-74.025 N-m
Scoping	
Scoping Method	Boundary Condition
Boundary Condition	Remote Displacement
Load Step Selection	1
Vector Reduction	
Coordinate System	Nodal Coordinate System
Vector Reduction	X
Spatial Reduction	
Spatial Reduction	Sum
Method	Discrete

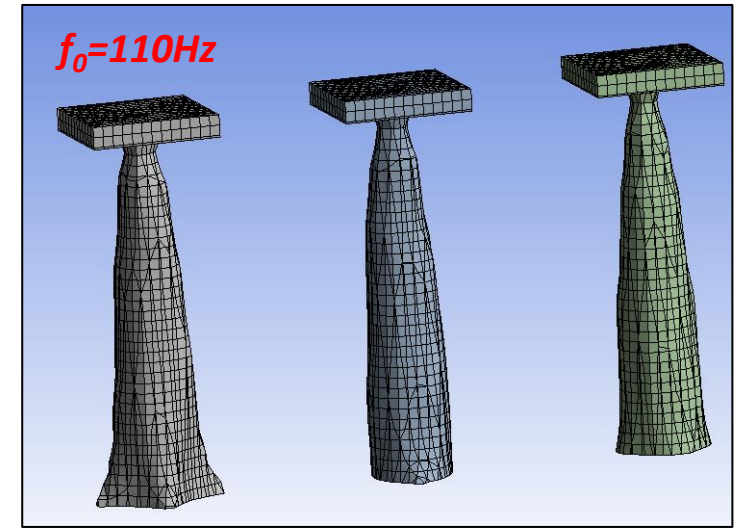


x-reaction-moment  
at the remote-displacement support

# UDC Modal: Robust Frequency Example



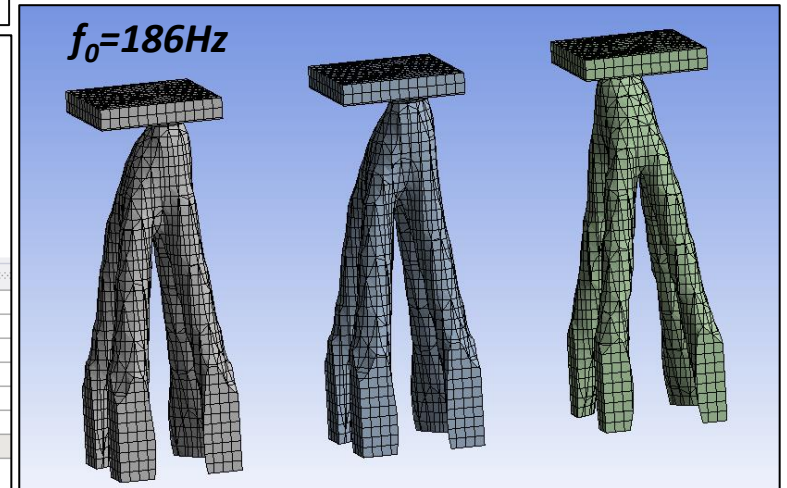
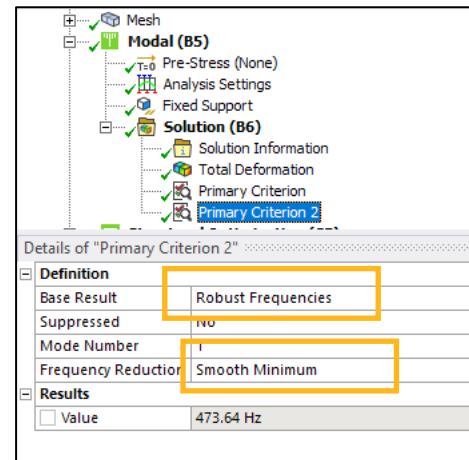
$$\begin{cases} \max_{\Omega} f_0 & [\text{single freq}] \\ \text{mass} \leq 25\% \end{cases}$$



+70%  
better

*Better performance thanks to  
the robust-frequency criterion*

$$\begin{cases} \max_{\Omega} \tilde{f}_0 & [\text{robust freq}] \\ \text{mass} \leq 25\% \end{cases}$$



## Context of this example

- 3 bodies, fixed at the bottom, remote mass at the top
- **6 first modes** share the same eigenfrequency, ie 465Hz
- The purpose is to maximize the first eigenfrequency

## Notes

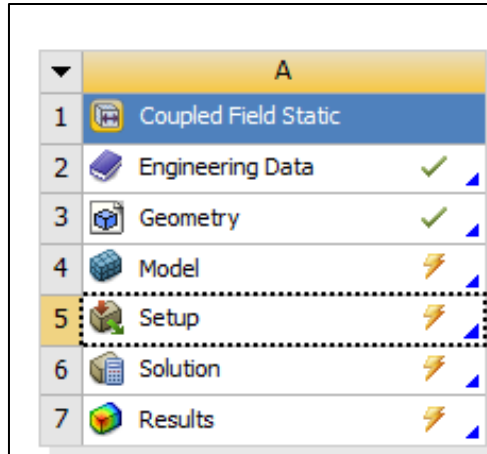
- The optimizer may converge prematurely due to the “mode-crossing” phenomenon
- Roughly speaking, the modes order changes during the optimization and confuses the optimizer
- The “smooth minimum” option manages this context

# Linear Dynamics



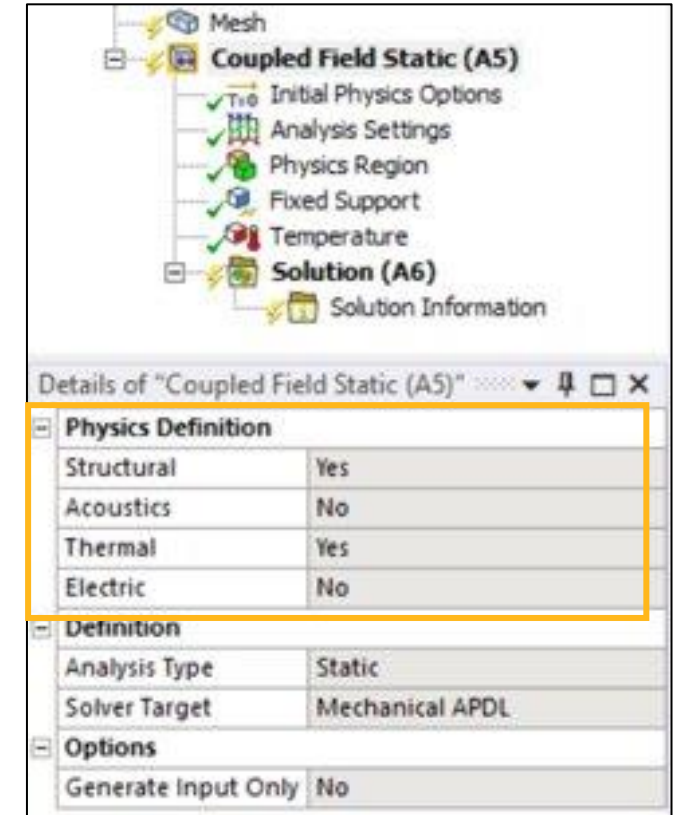
# / Coupled Field Static Analysis

- Piezoelectric coupling and Acoustics physics is supported for Coupled Field Static analysis



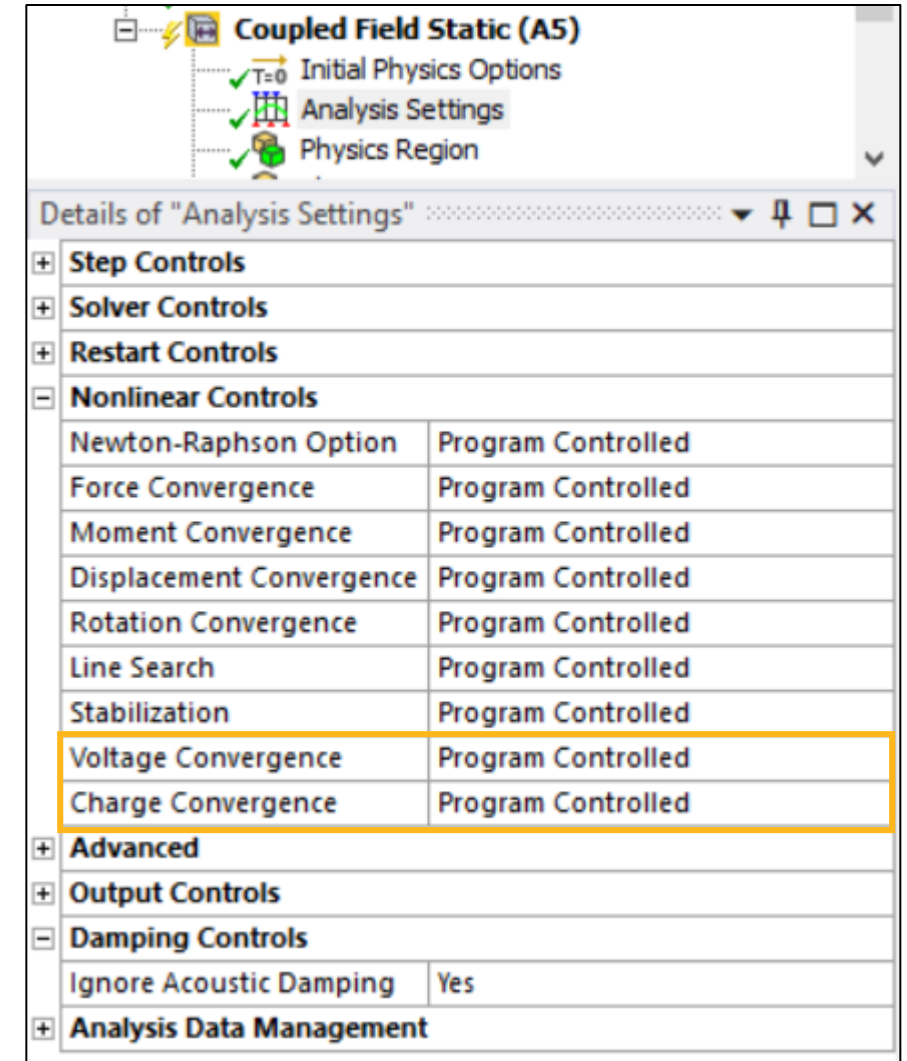
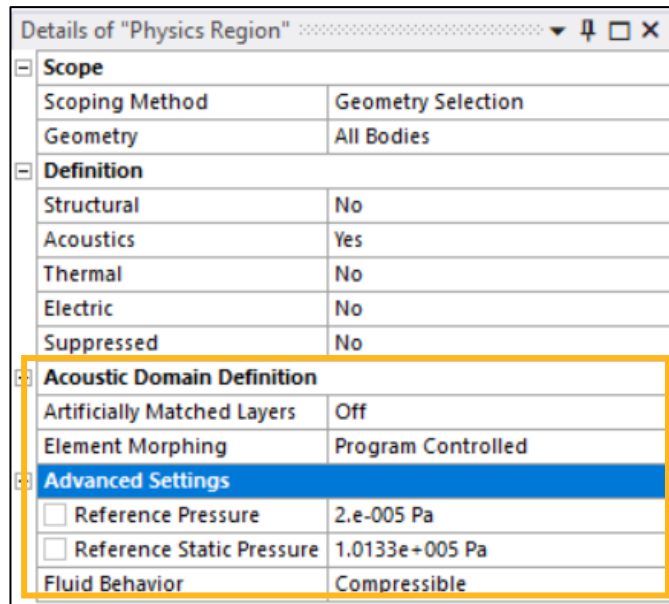
Coupled Field Static

1	Property	Value
2	General	
3	Component ID	Setup
4	Directory Name	SYS
5	Update Condition Parameter (Beta)	None
6	Notes	
7	Notes	
8	Used Licenses	
9	Last Update Used Licenses	
10	System Information	
11	Physics	Multiphysics
12	Analysis	Static
13	Solver	Mechanical APDL
14	Physics	
15	Structural	<input checked="" type="checkbox"/>
16	Acoustics	<input type="checkbox"/>
17	Thermal	<input checked="" type="checkbox"/>
18	Electric	<input type="checkbox"/>



# Physics Region and Analysis Settings

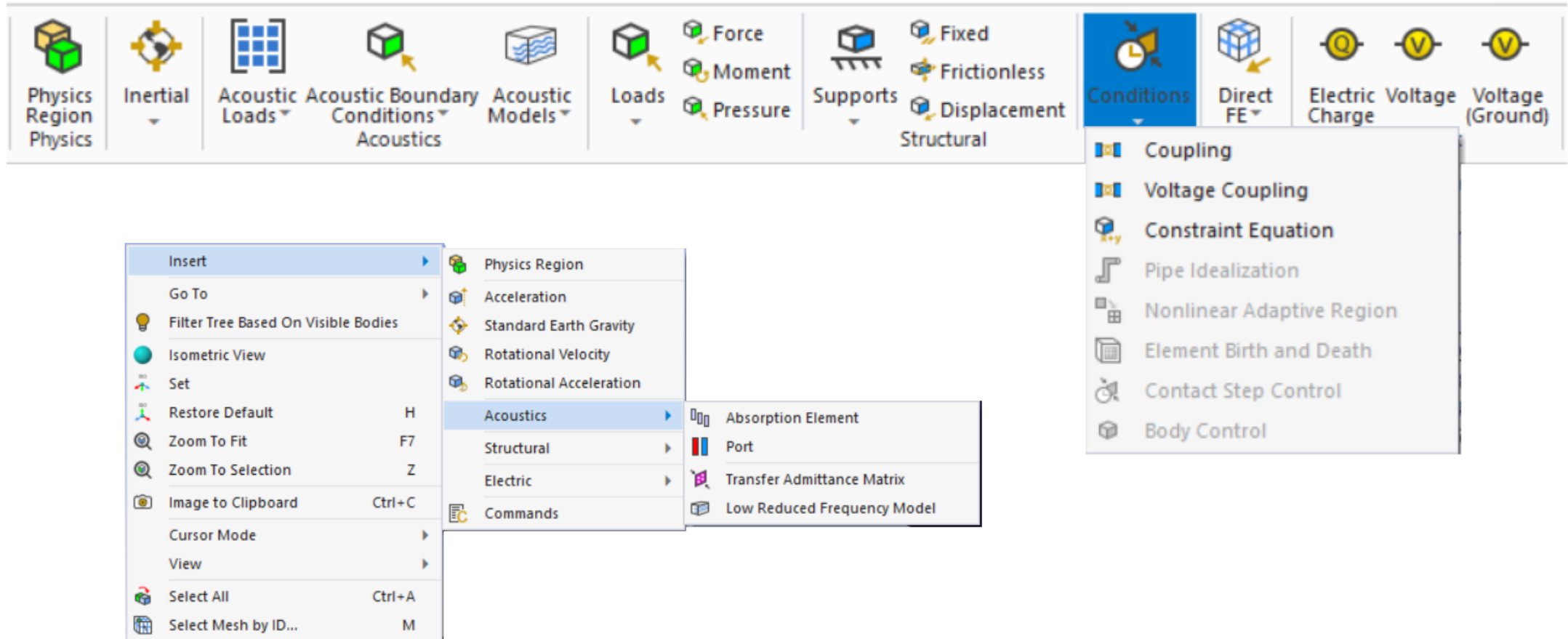
- PML settings are supported for respective Acoustics physics in the Physics region object
- Voltage and Charge convergence supported on Analysis settings
  - Program Controlled
  - On
  - Remove





# / Boundary Conditions

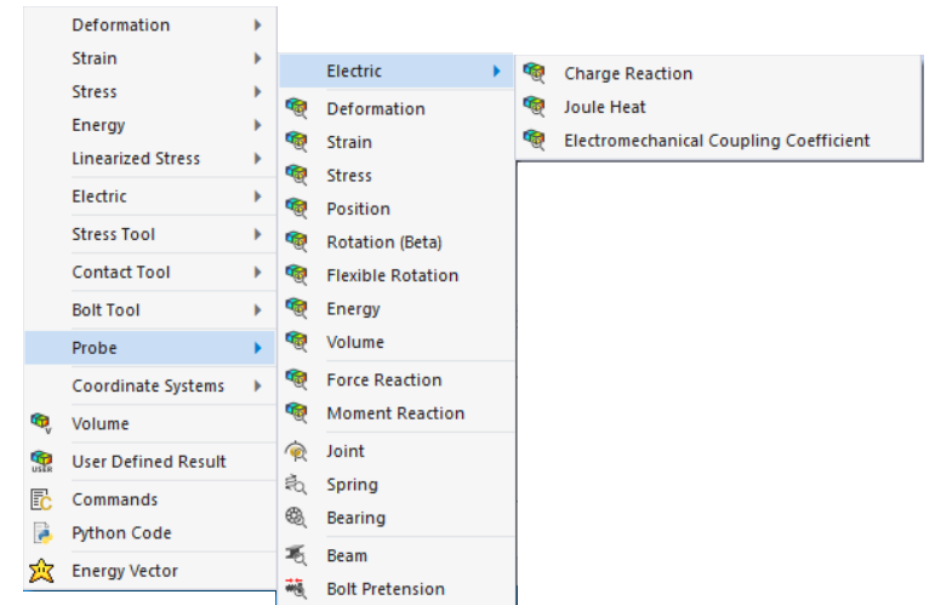
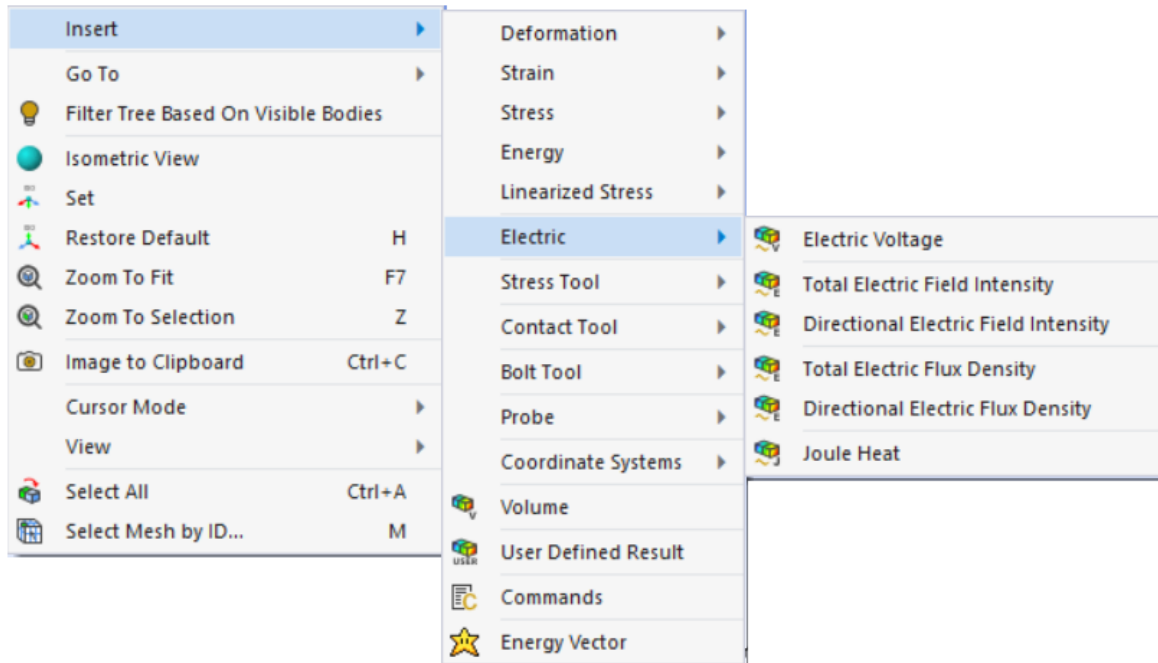
- Electric and Acoustics boundary conditions are supported





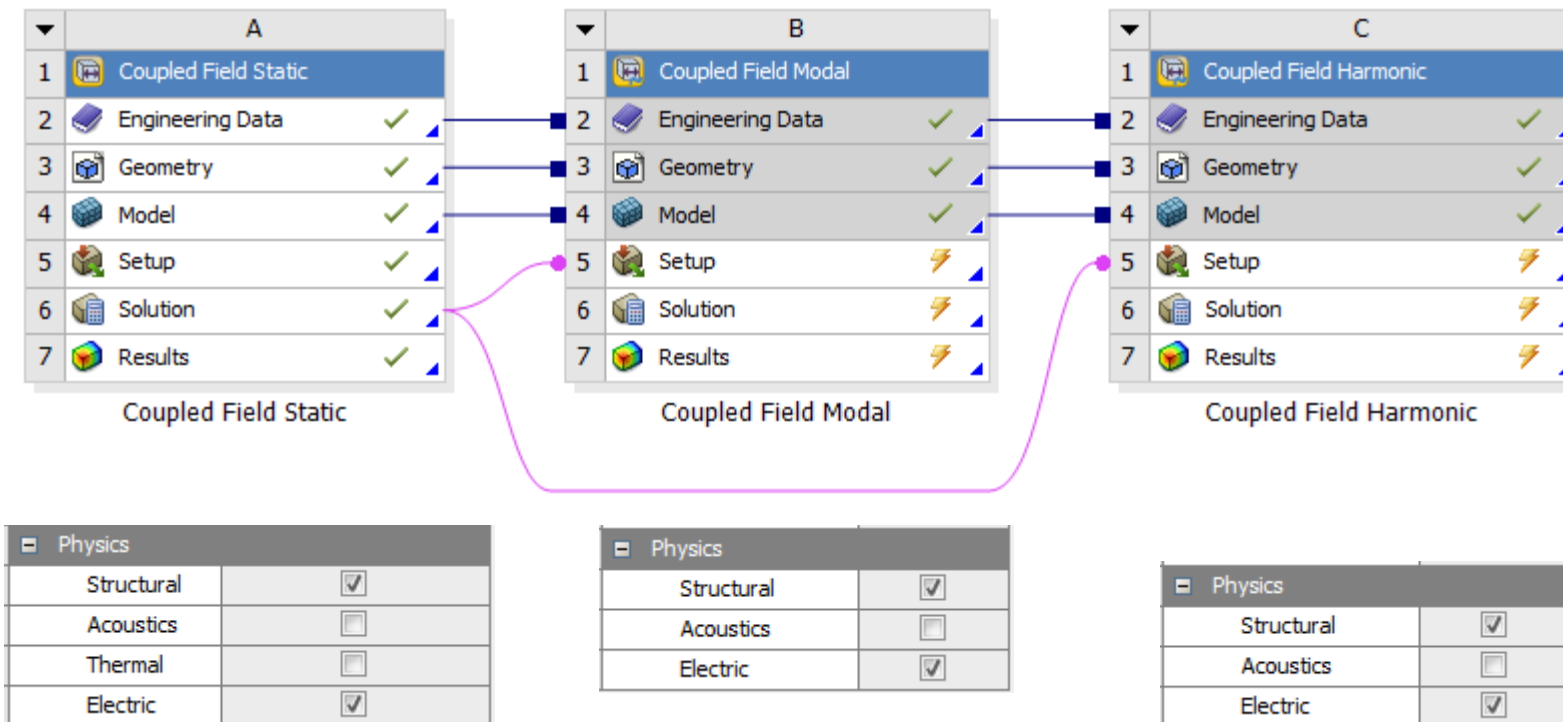
# Results

- Electric results and probes are supported



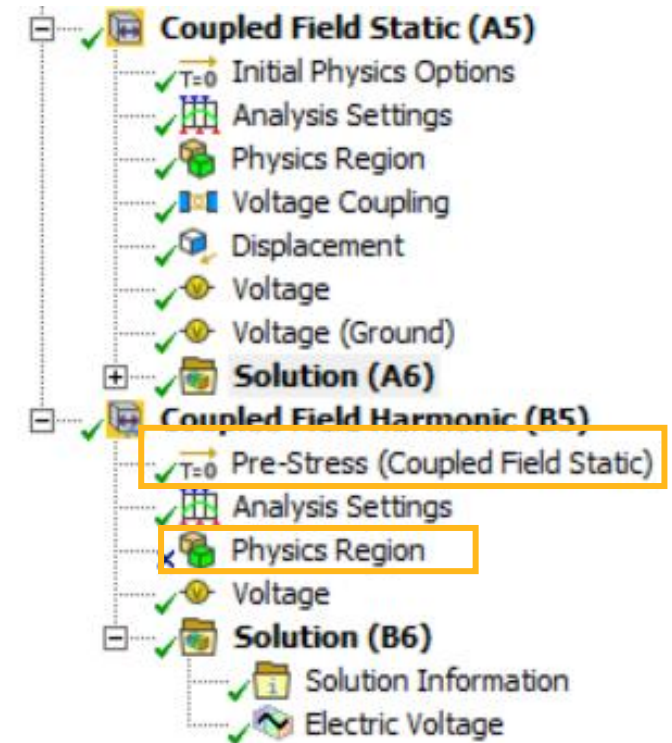
# / Pre-stress Coupled Field Analysis Enhancements

- Pre-stressed Coupled Field Modal and Pre-stress Coupled Field Harmonic (full harmonic) is supported by linking to upstream a Coupled Field Static analysis



# / Pre-stress Coupled Field Enhancements

- The physics combination which can be performed in Pre-stress Coupled Field analysis are:
  - Structural – Acoustics
  - Structural – Electric with Piezoelectric coupling
  - Acoustics with Piezoelectric coupling
- Physics region specified in the upstream coupled field static analysis will be automatically selected on downstream linked environment.
- Thermal physics selection in the coupled field static analysis will not support pre-stress workflow
- Linking on Coupled Field Modal and Harmonic can be done by selecting the pre-stress environment on the Initial conditions

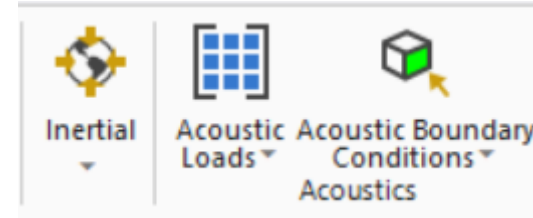


# Boundary Conditions and Results for Pre-Stressed Analysis

Pre-stressed Coupled Field analysis supports these boundary conditions

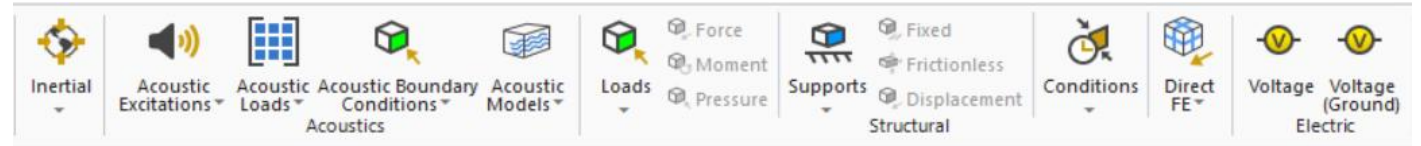
## Pre-stress Coupled Field Modal

- Acoustic Boundary conditions



## Pre-stress Coupled Field Harmonic

- Inertial Load, FE Loads
- Acoustic Boundary conditions
- Electric: Voltage and Voltage Coupling

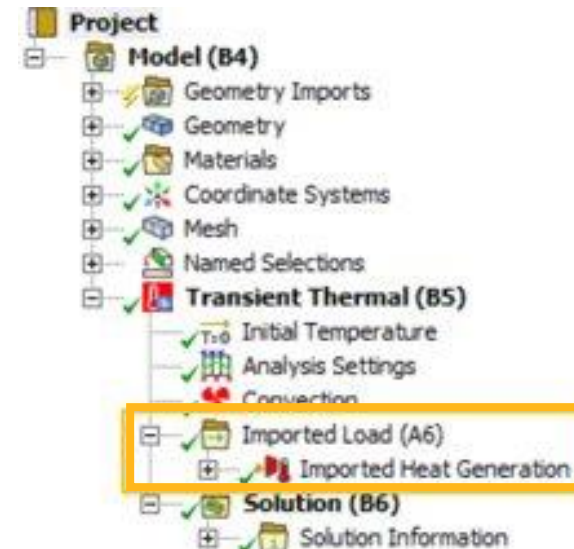
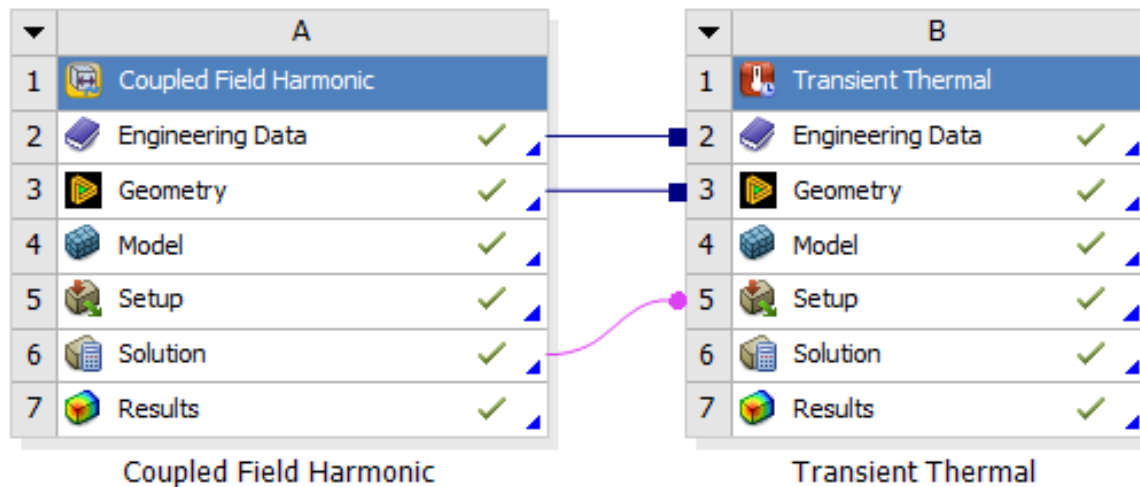


**Results:** All results supported for standalone analysis is applicable for pre-stress case



# Imported Heat Generation Load from Coupled Field Harmonic

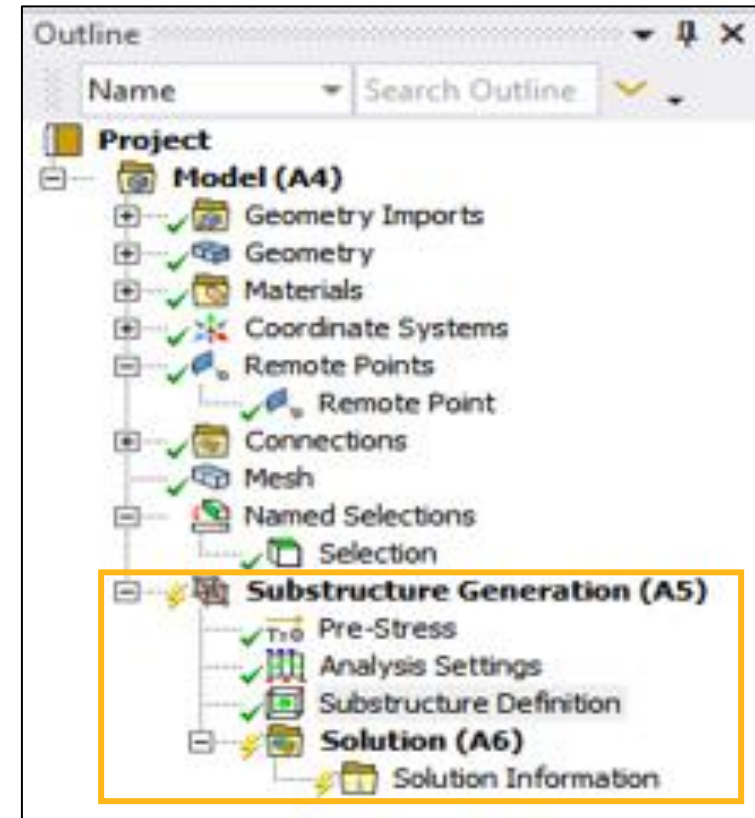
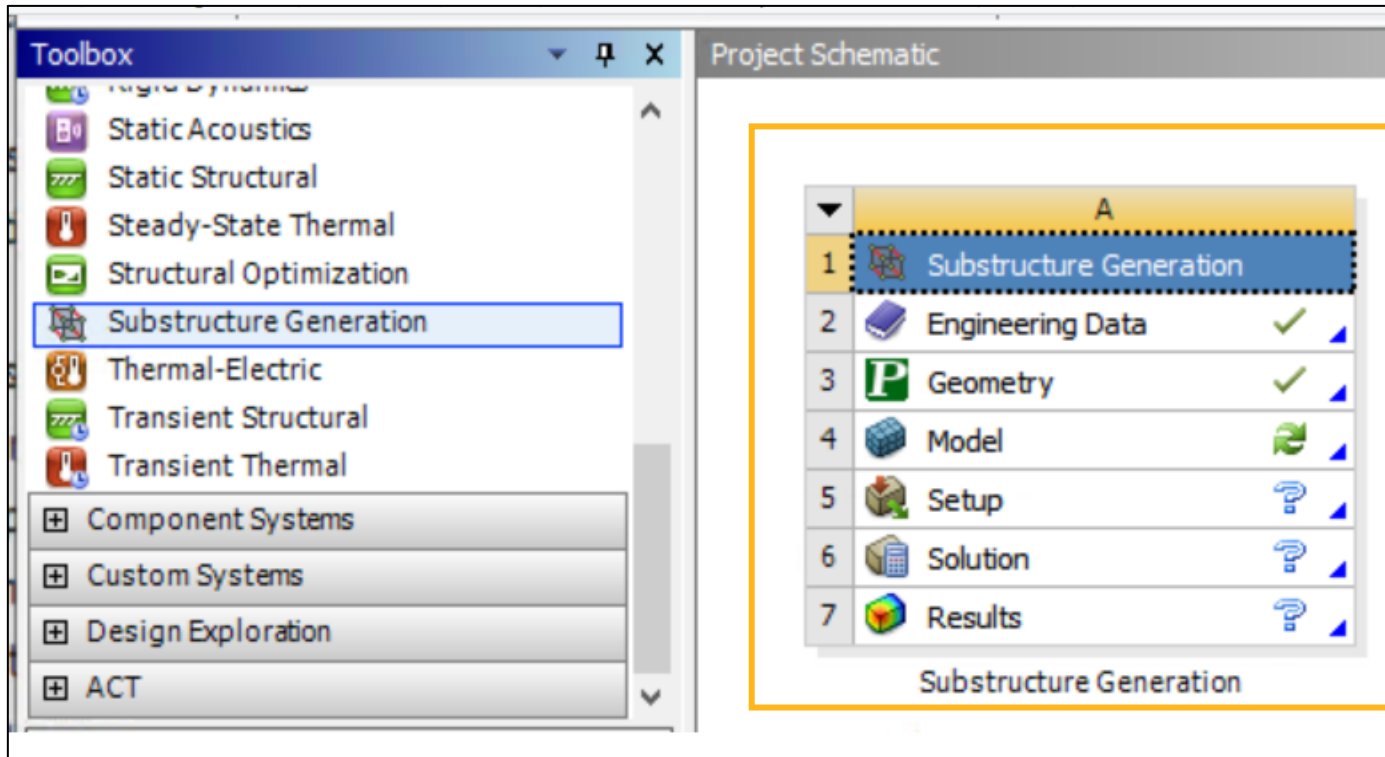
- Loss due to damping in the upstream Coupled Field Harmonic analysis can be imported as Heat Generation load in Transient Thermal analysis by linking the solution cell of Coupled Field Harmonic to setup of Transient Thermal analysis
- The losses are only considered from the coupled structural-electric bodies with Piezoelectric coupling and is applicable for dissimilar meshes. The source frequencies is split over equal time intervals in the transient thermal analysis when All option is selected from Worksheet of Imported Heat generation object





# Substructure Generation Analysis

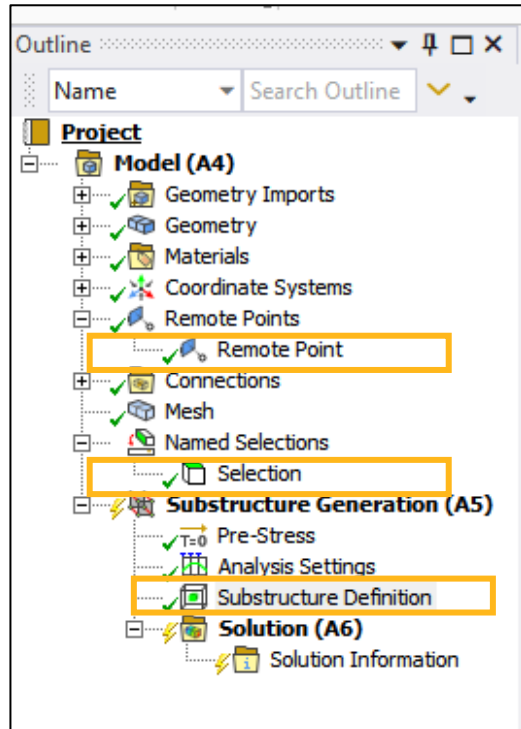
- Substructure Generation analysis is supported to condense the finite elements of your entire model using Component Mode Synthesis methodology into one “superelement”. These superelements can then be composed using Bottom-up Substructuring approach for use pass in another mechanical session





# Substructure Generation

- Master degrees of freedom is specified using Remote point and Named selection added as Interface on Substructure Definition Worksheet. The user can drag and drop these objects on worksheet to add it as Interface



**A: Substructure Generation**  
Substructure Definition  
Frequency: N/A  
11/11/2021 5:07 PM

Selection  
Remote Point

Worksheet

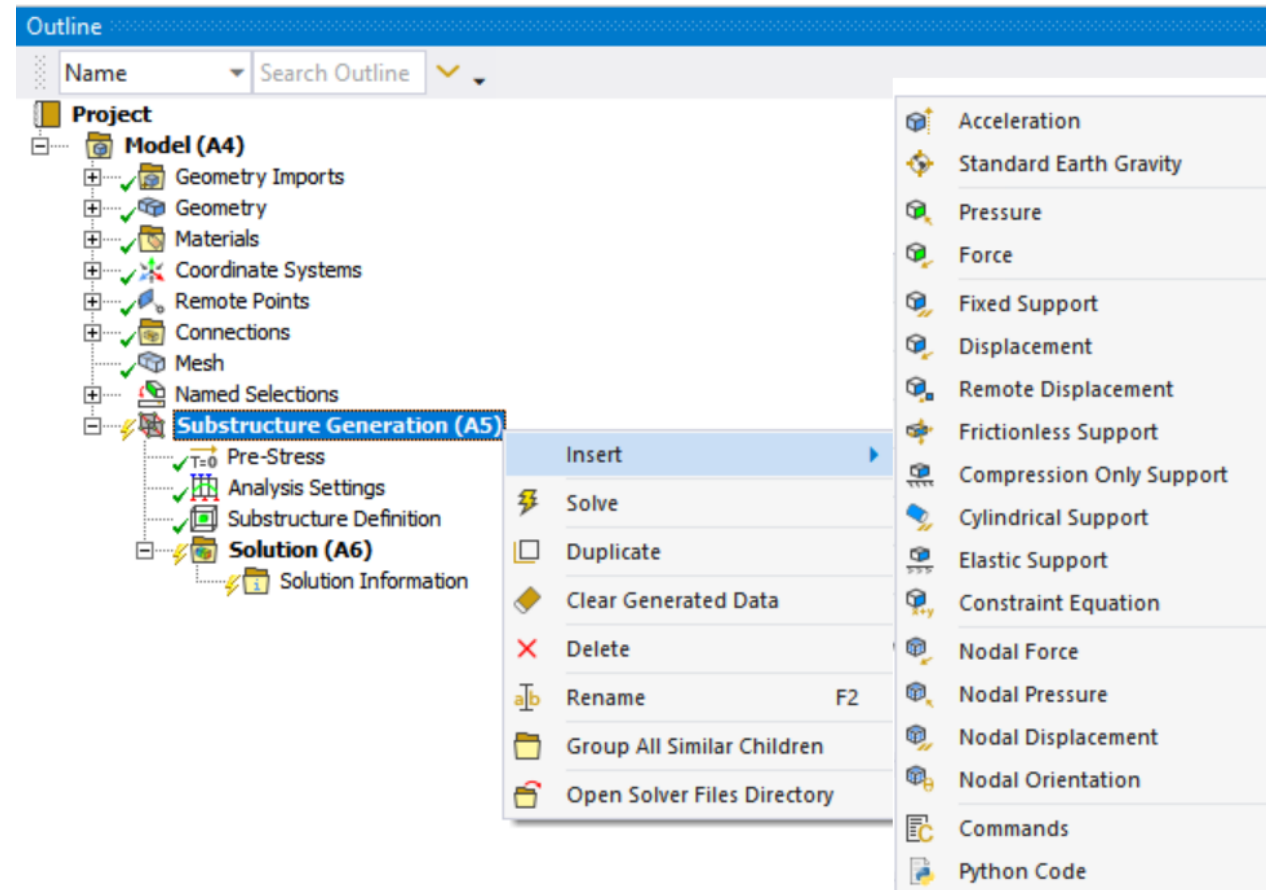
Substructure Definition

Clear

Interfaces						
Name	Scope Method	Environment Name	Source	Type	Condition	Side
Selection	Geometry Selection	Substructure Generation	Manual	General	Named Selection	N/A
Remote Point	Geometry Selection	Substructure Generation	Manual	Remote	Remote Point	N/A

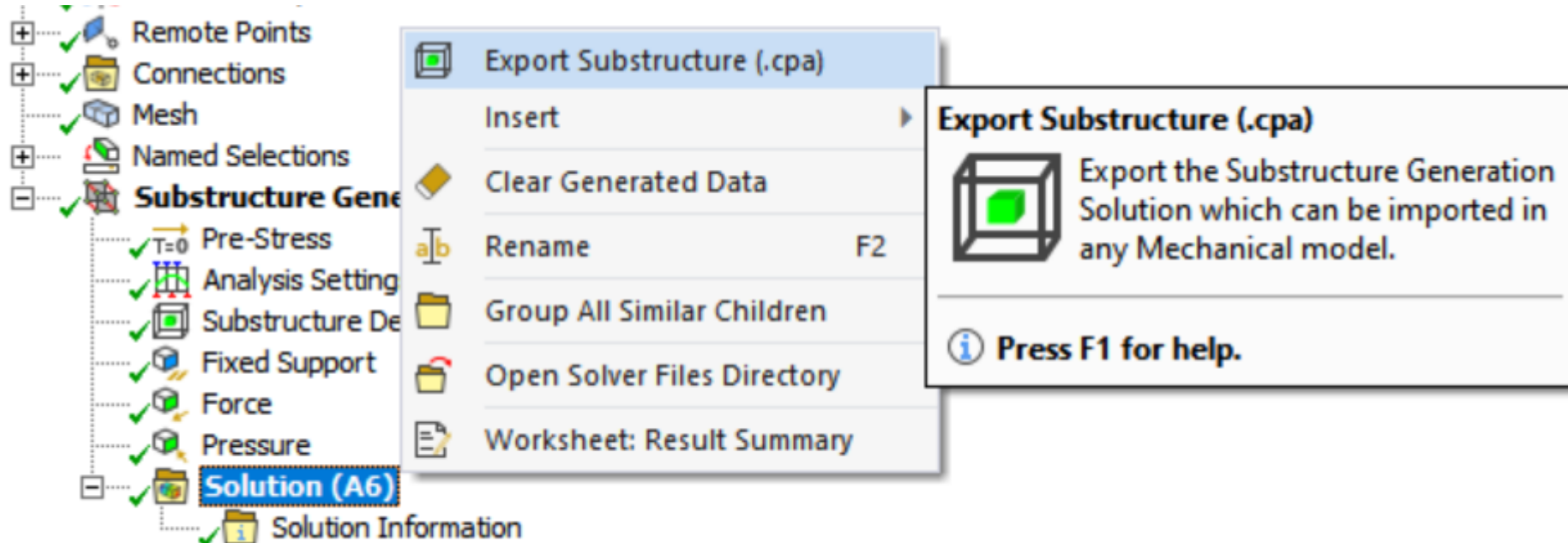
# Substructure Generation

- Supports constant loads and boundary conditions.
- Solving the analysis will generate individual load vector for each load applied in the Substructure Generation analysis. These load vectors can then be applied in the use pass of MSUP harmonic analysis



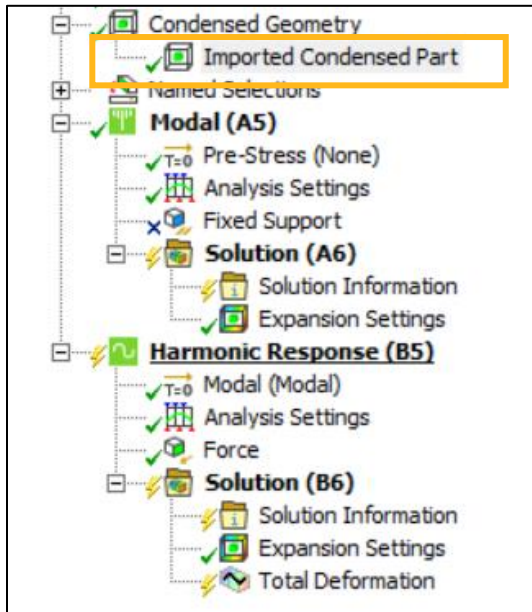
# Substructure Generation

- These superelements can then be exported using Export Substructure (.CPA ) option



# Substructure Generation

- These superelements can then be imported using Imported Condensed Part and will display the interfaces and load vectors in the Worksheet. These superelements will then be used for use pass in Modal, MSUP harmonic , Response Spectrum and Random Vibration analysis. The load vectors will be applied as unit loads only for MSUP harmonic analysis



Worksheet

### Imported Condensed Part

Clear

☒ **Interfaces** ☐ **Load Vectors**

ID	Name	Node Connectivity	Number of Nodes
1	Selection	Quadrilaterals - Quadratic	96
2	Remote Point	Point	1

Worksheet

### Imported Condensed Part

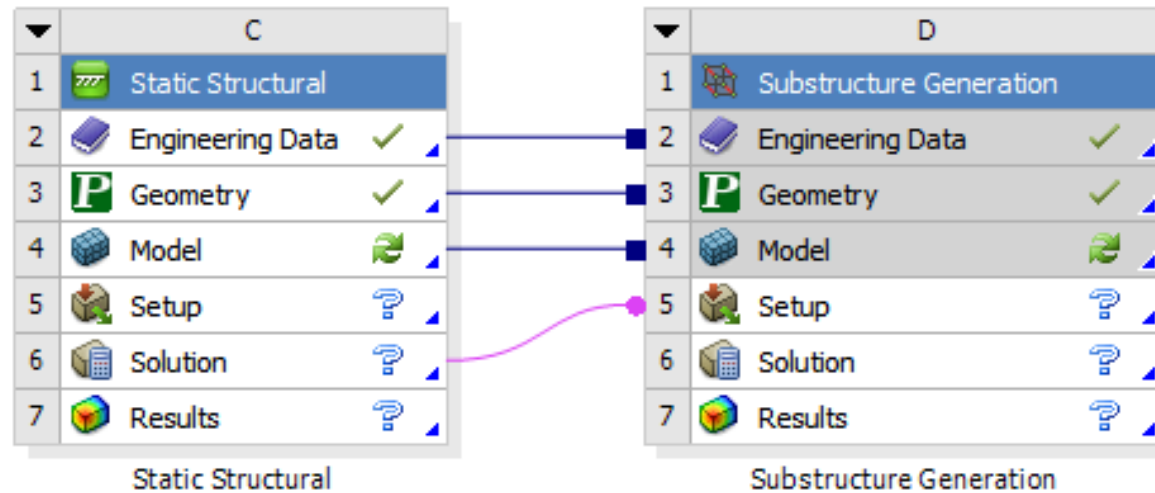
Clear

☐ **Interfaces** ☒ **Load Vectors**

	Load Vector ID	Name	Magnitude	X Component	Y Component	Z Component
	1	Pressure	500. Pa	N/A	N/A	N/A
	2	Force	1000. N	N/A	N/A	N/A

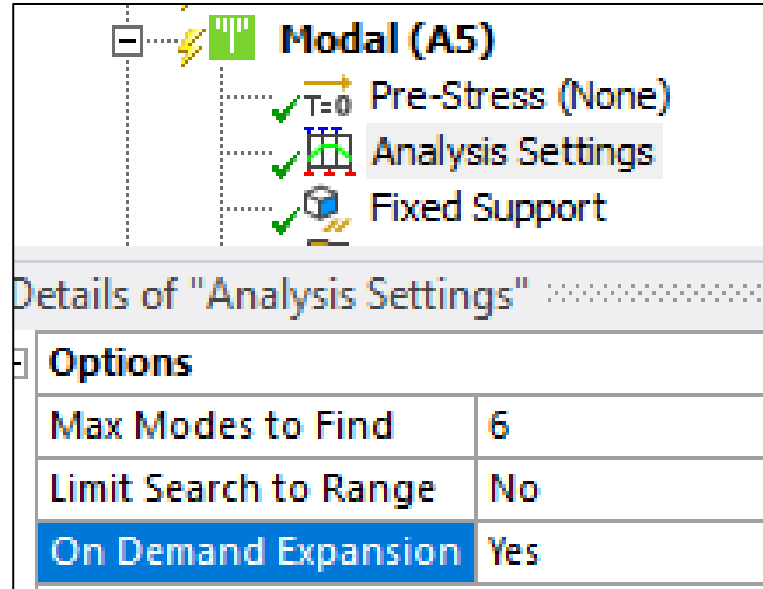
# Substructure Generation

- Prestress effects can be included in the Substructure Generation analysis by linking it to upstream Static Structural analysis
- When linked to Static Structural analysis, the first load vector in the Substructure Generation analysis will be included from the pre-stress effects
- Substructure analysis is only supported for MAPDL solver and is supported in Beta mode when linked with External model



# / On Demand Expansion for Top-down Substructuring

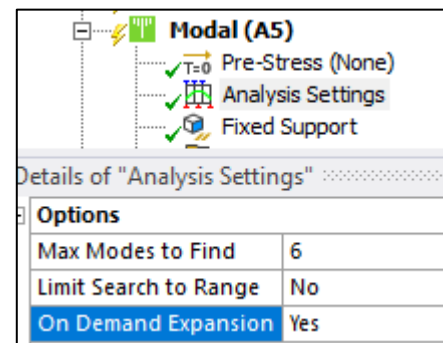
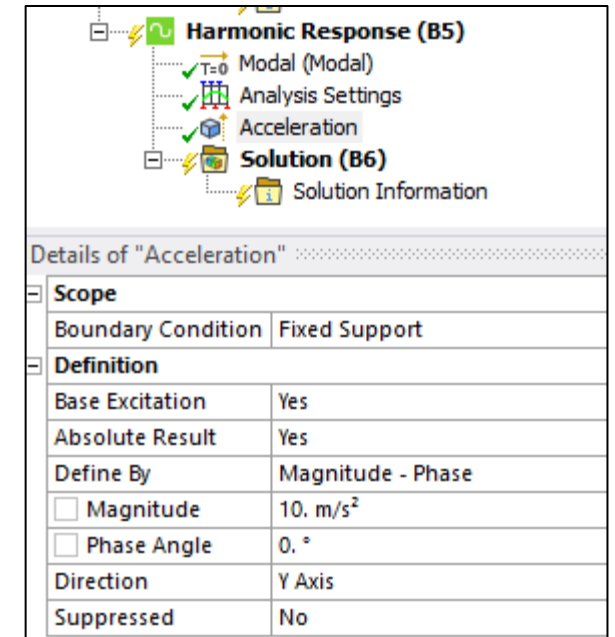
- On Demand Expansion option is available in Modal analysis which when selected will evaluate Condensed Part results (Top-down Substructuring) “on the fly” and expansion pass through MAPDL solver is not needed for deformation results on the Condensed Parts





# Mode Superposition Analysis Enhancements

- Enforced Motion with On Demand Expansion: Base Excitation (Relative & Absolute) can be used with On Demand Expansion
- On Demand Expansion option is available in Modal analysis allowing to post-process results using distributed files (dividing overall solution directory size by 2) and to evaluate Condensed Part results “on the fly” (Expansion Pass files not generated)



# Mode Superposition Analysis Enhancements

- Analyses using On Demand Expansion now only retains the solution files required for post-processing which allows to global reduce the solution directory size
- A preference has been added to avoid storing displacement in the rst file using On Demand Expansion which can significantly reduce rst files size

Options (Modal, Harmonic and Transient Mode Superposition)	
On Demand Expansion	No
On Demand Mode Shapes	Yes

file0.rst	7,552 KB	→	file0.rst	3,840 KB
file1.rst	7,872 KB		file1.rst	4,160 KB

2021 R2

ds.dat  
ds\_file.xml  
dummy.dat  
file.db  
file.mlv  
file.mode  
file.rfrq  
file0.err  
file0.esav  
file0.full  
file0.log  
file0.mlv  
file0.mode  
file0.rfrq  
file0.rst  
file1.err  
file1.esav  
file1.full  
file1.mlv  
file1.mode  
file1.out  
file1.rfrq  
file1.rst  
MatML.xml  
parm.xml  
solve.out

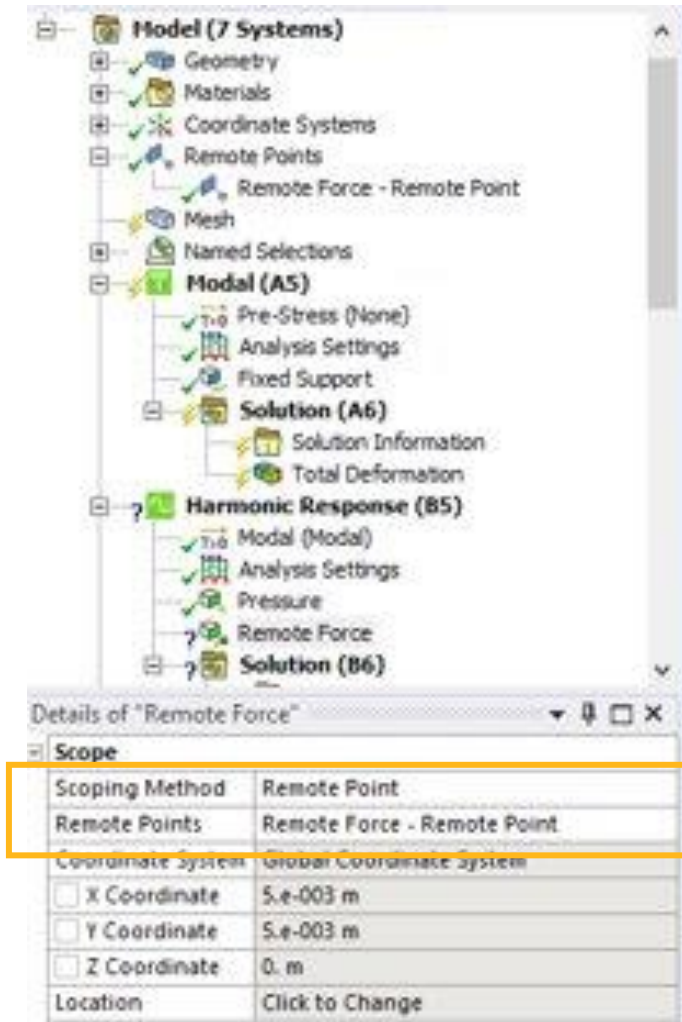
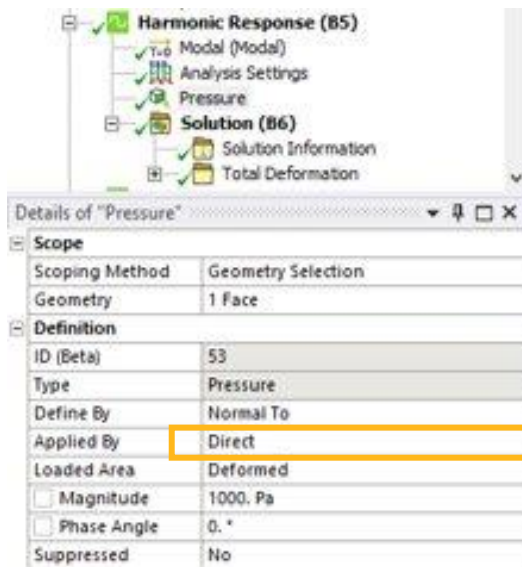
2022 R1

ds.dat  
file.rfrq  
file0.err  
file0.mode  
file0.rst  
file1.mode  
file1.rst  
MatML.xml  
solve.out



# Mode Superposition Analysis Enhancements

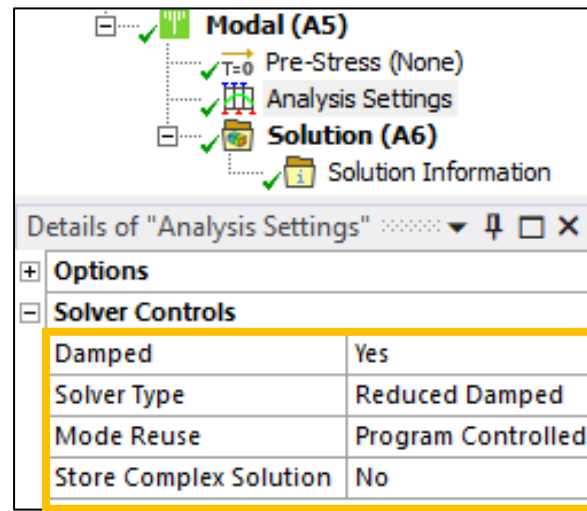
- MSUP analysis will use the Pressure and Force load default Applied by option of Direct and this will not create surface effect elements
- Remote force and moment when created will promote to remote point by default when no analysis is solved in the session and location is selected before creation



# Material Damping losses in MSUP Harmonic Analysis

- In a linked Harmonic Response (MSUP) analysis when upstream Modal analysis uses Reduced Damped solver type and has Store Complex Solution property set to No
- Then, the losses associated with Material Dependent Damping under Engineering Data will be contributed by **Constant Structural Damping Coefficient (MP,DMPS)**
- For a Damped Modal analysis, the application now sends the Material-Dependent Structural Damping Coefficient (**MP,DMPS**) instead of Material-Dependent Damping Ratio (**MP,DMPR**)

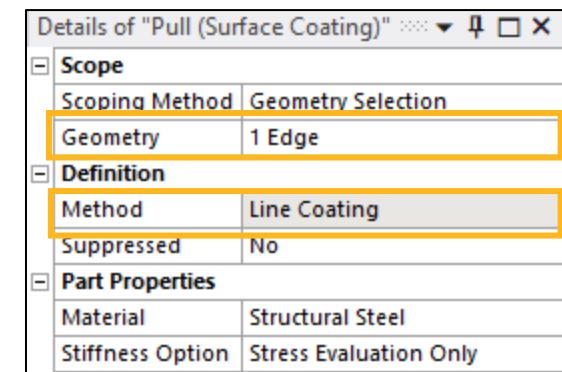
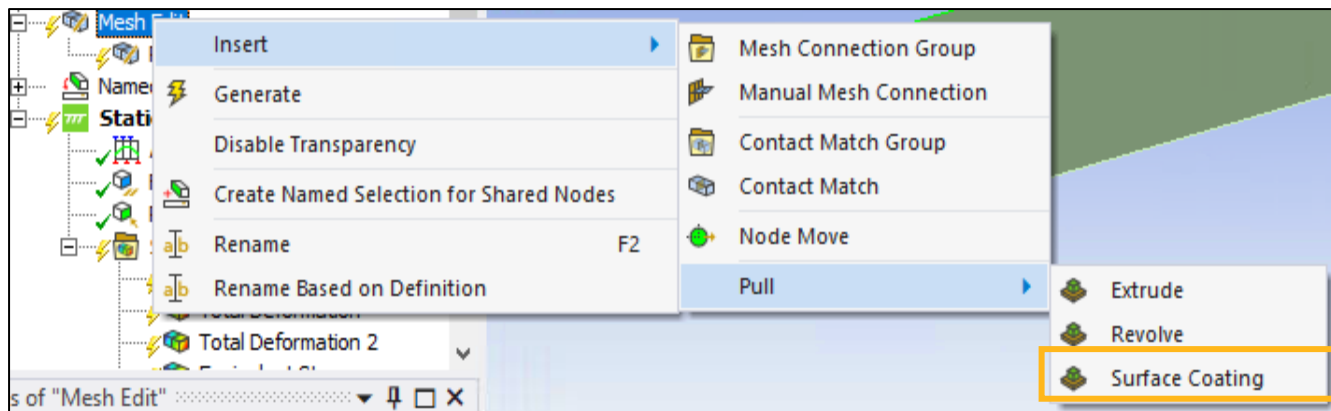
Properties of Outline Row 3: Structural Steel		
	A	B
1	Property	Value
2	Material Field Variables	Table
3	Density	7850
4	Isotropic Secant Coefficient of Thermal Expansion	
6	Material Dependent Damping	
7	Damping Ratio	0.25
8	Constant Structural Damping Coefficient	= 0.5



```
/com,***** Send Materials *****
Temperature = 'TEMP' ! Temperature
MP,DENS,1,7850, ! kg m^-3
MP,ALPX,1,1.2e-05, ! C^-1
MP,C,1,434, ! J kg^-1 C^-1
MP,KXX,1,60.5, ! W m^-1 C^-1
MP,RSVX,1,1.7e-07, ! kg m^3 A^-2 s^-3
MP,EX,1,200000000000, ! Pa
MP,NUXY,1,0.3,
MP,MURX,1,10000,
MP,DMPS,1,0.5,
```

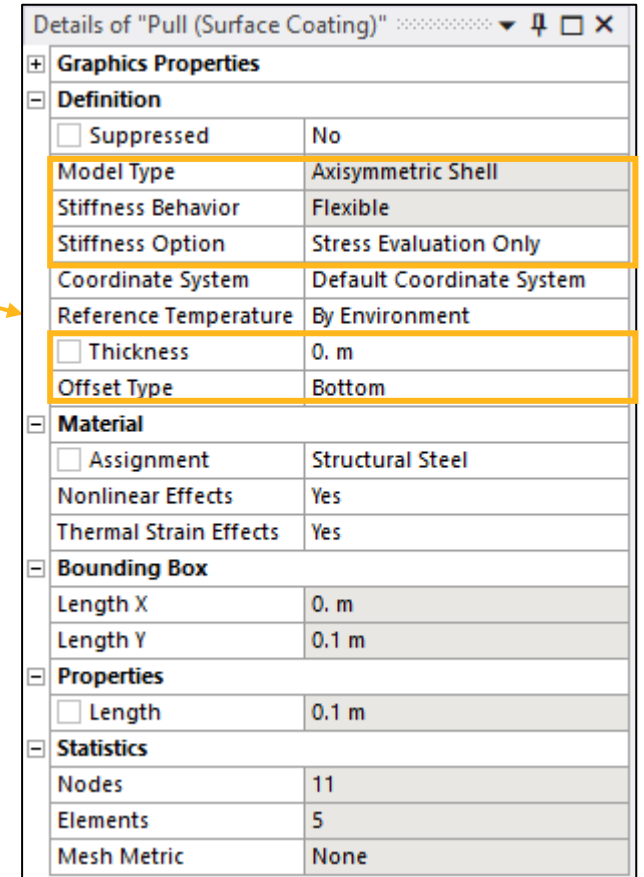
# Line Coatings Using Pull

- Surface Coating can be added under Mesh edit folder
- Pull method set to **Line Coating** when scoped to edge of surface body
  - In 2D analysis: Scope to edge of surface body.
  - In 3D analysis: Scope to edge of surface body with **Dimension** set to **2D**
- Scoping multiple line coatings on same edge is supported
- Supported only for analyses that use Mechanical APDL solver. i.e., Solver Target property on environment set to Mechanical APDL.



# Line Coatings Using Pull

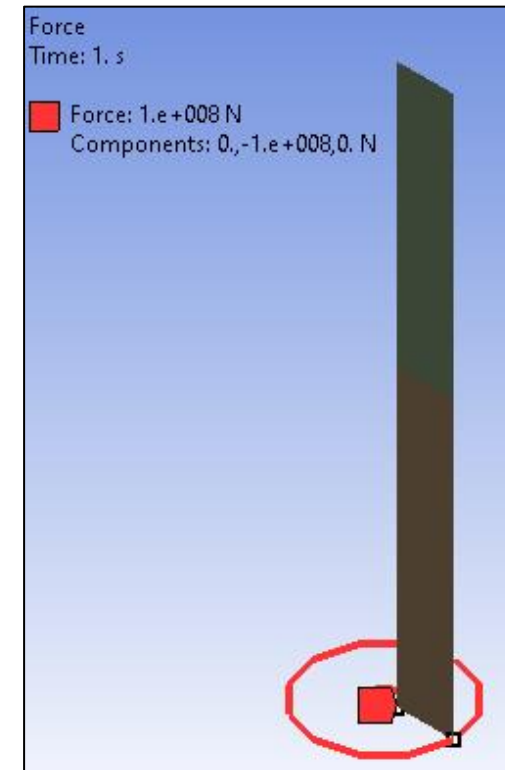
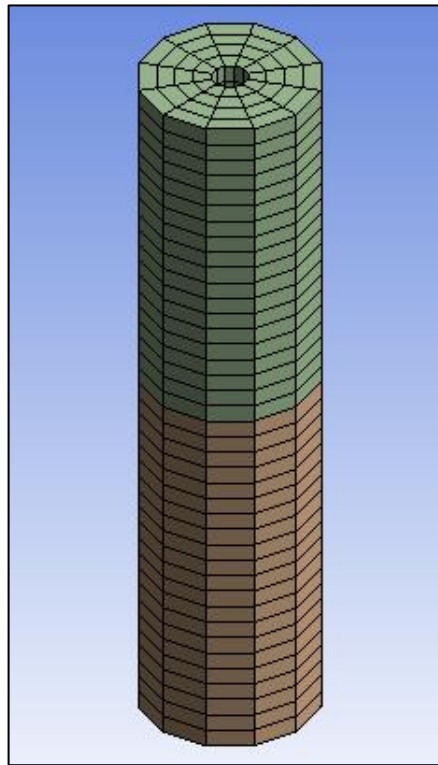
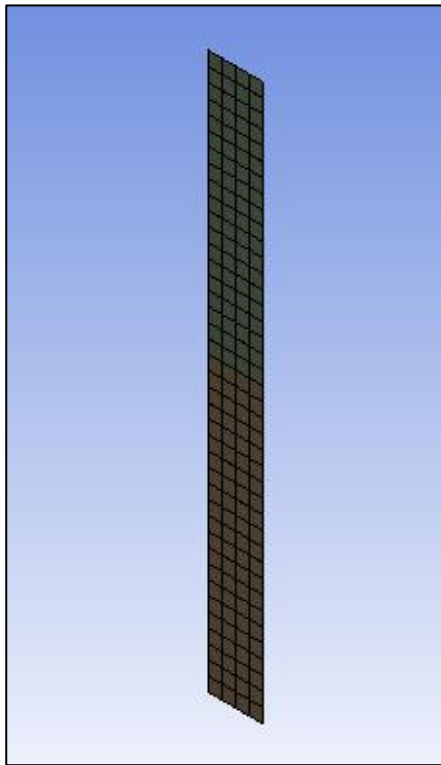
- Upon generating Pull, a line body is added in the tree
- Properties for this body are like a 3D surface body
- Model Type is set to Axisymmetric Shell (Read-Only)
- Send Shell 208/209 definition for these line coatings
- **Thickness** property definition is written to the input file when Stiffness Option is set to "Membrane and Bending" or "Membrane Only"
- The **Offset Type** property input is written to input file only when Stiffness Option is set to "Membrane and Bending"
- When Stress and Strain results scoped to Pull generated line coating bodies and the Position property is set to **Top/Bottom**, the contours displayed in Mechanical only correspond to the bottom layer.



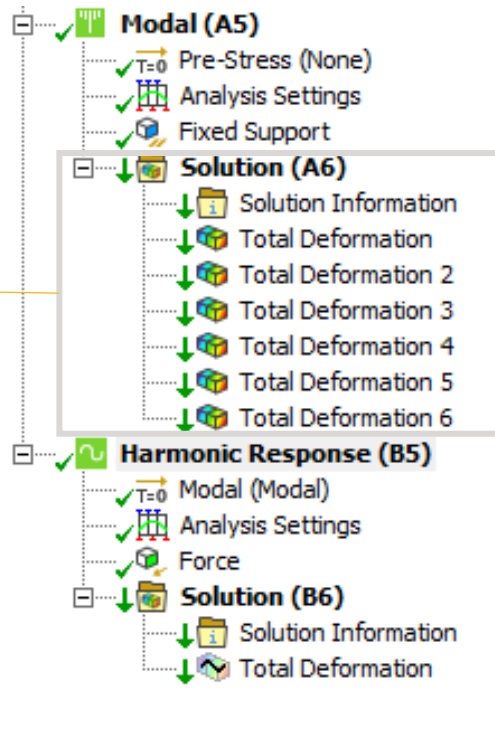


# / General Axisymmetric Enhancements

- General Axisymmetric mesh generation on an imported base mesh
- Force load scoping on edges and vertices of general Axisymmetric bodies. This force load behaves as a Nodal Force
- Bolt Pretension load allowed on General Axisymmetric bodies under Beta



# Workflows



If performed on the same server, the upstream (Modal) results don't have to be downloaded to submit the downstream (Harmonic) analysis

## 1. Both analyses are solved on DCS:

- Upstream system result files, required for the downstream analysis, are internally referred in the cloud storage for supported analyses, thus saving time in downloading/uploading files.
- The linked systems can be submitted individually or together by pressing solve on the most downstream system.

## 2. Only the downstream analysis is solved on DCS:

- Files needed for the downstream analysis from the upstream system are automatically uploaded to the DCS server.

# Analyses Supported

Without needing to download results of the upstream system, DCS currently supports:

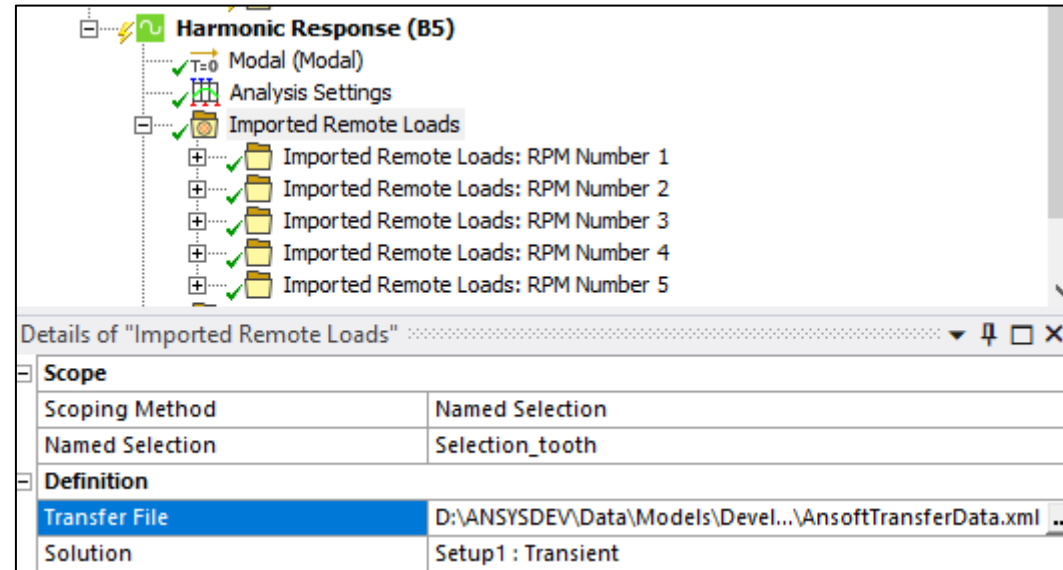
- Mode Superposition Harmonic Response, Transient, Random Vibration, Response Spectrum (including Prestress)
- Prestress Modal, Full Harmonic Response, Eigenvalue Buckling
- Prestress Coupled Field Modal, Coupled Field Harmonic
- Static Acoustics and Modal Acoustics, Harmonic Acoustics

Analyses in which upstream results are needed to be automatically downloaded:

- Downstream system having imported loads from upstream system such as Steady-State Thermal and Static Structural, and Electric Conduction and Steady-State Thermal systems
- Steady-State Thermal and Transient Thermal systems

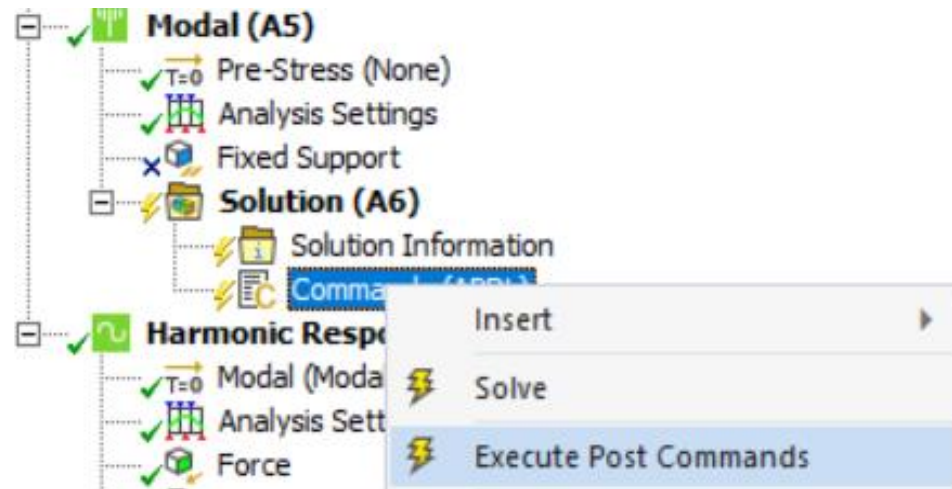
# / Maxwell: Mechanical Coupling

- Remote Forces / Moments can be generated from Maxwell exported file which eliminate the need of having both Maxwell and Mechanical models in the same Workbench project



# / Reinforcements and Execute Post Commands

- Reinforcements is now supported for Response Spectrum analysis
- Execute Post commands is supported to execute commands and extract information when the solution is partially solved or in a restart state



# NVH Toolkit



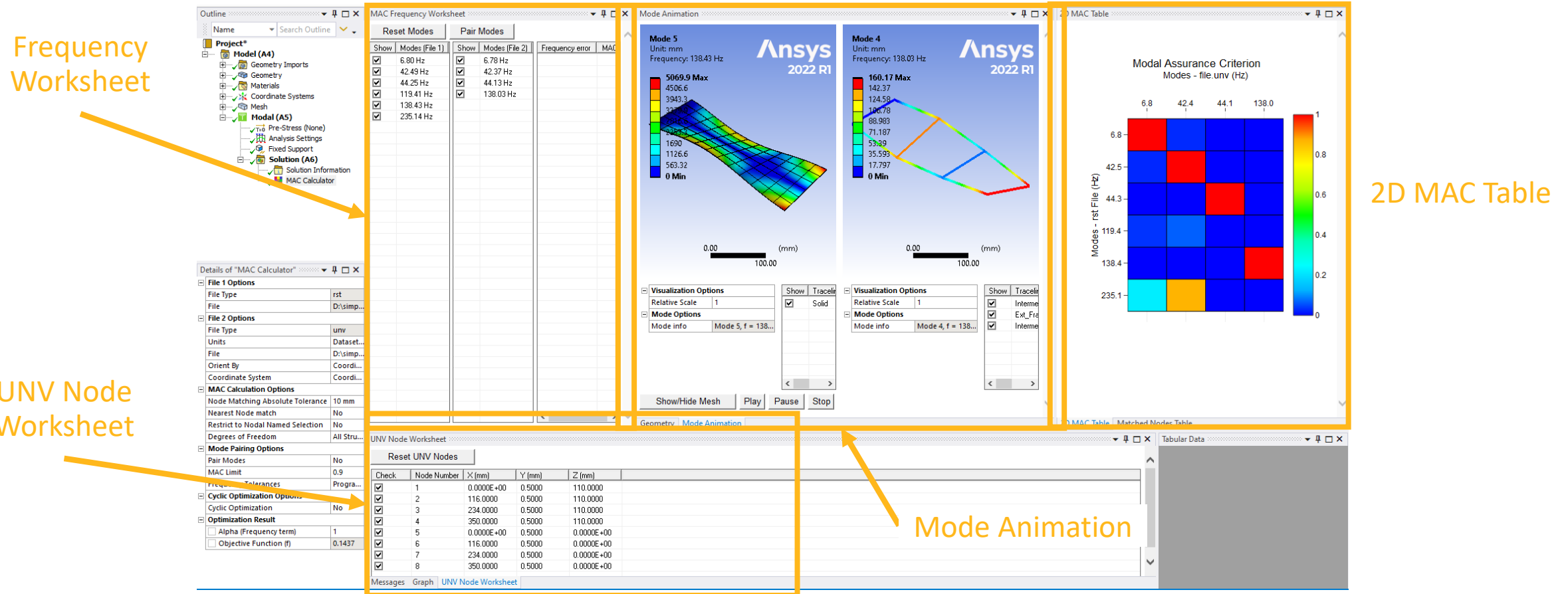


# NVH Toolkit: MAC Calculator

- The MAC Calculator result object computes the Modal Assurance Criterion (MAC) between a Modal Analysis (rst file) and a test (unv file). It allows you to:
  - Mode selection and flip to order them
  - Mode pairing to identify matches
  - Node selection and position tuning of unv nodes
  - UNV model orientation (Coord. System, Rigid Body Transformation or 3 Node Alignment)
  - Cyclic Optimization specific workflow
  - Interactive MAC Table
  - Interactive side-by-side Mode Animation

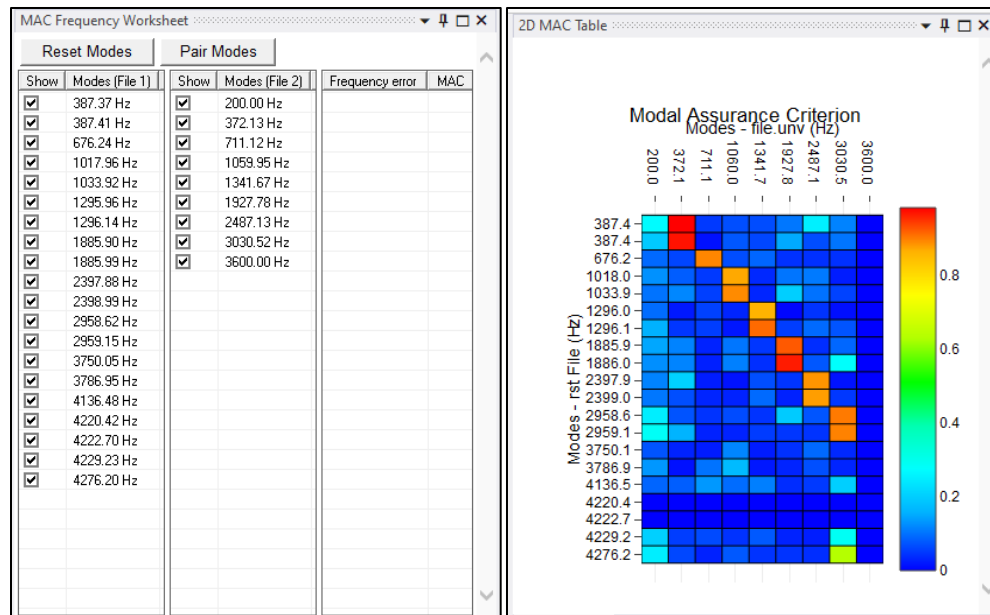
# NVH Toolkit: MAC Calculator – Comprehensive UI

- The UI has been tailored to provide enhance all the preprocessing and postprocessing capabilities:

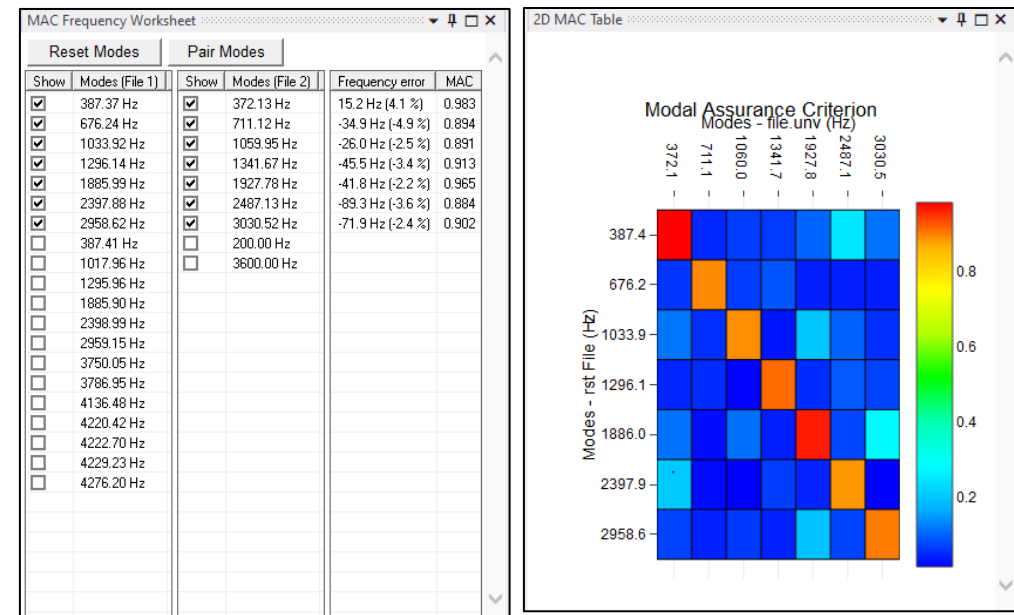


# NVH Toolkit: MAC Calculator – Mode pairing

- Automatic mode pairing identifies rst-unv mode matches based on MAC and Frequency values:



Before pairing



After pairing

# NVH Toolkit: MAC Calculator – UNV Node tuning

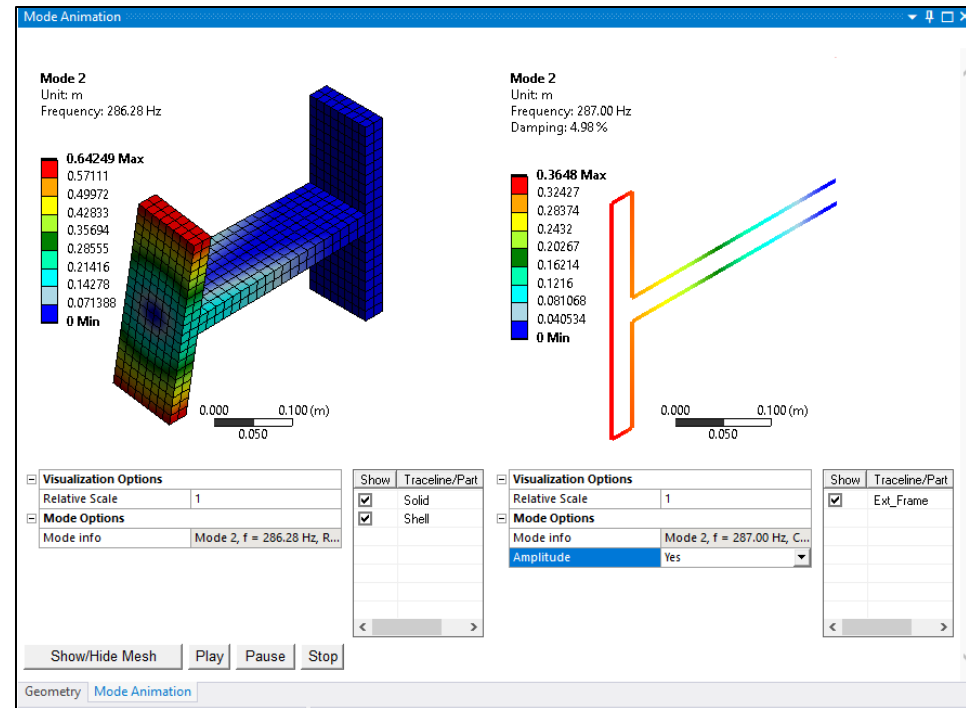
- The selected nodes are highlighted in green in the Geometry pane
- Coordinates are editable in the UNV Node Worksheet
- The UNV Model can be oriented in three ways:
  - By use of a Coordinate System already defined in the model
  - By entering the 3 displacements and 3 rotations that fully define a Rigid Body Transformation
  - By matching 3 rst and unv nodes

UNV Node Worksheet					
Reset UNV Nodes					
Check	Node Number	X (mm)	Y (mm)	Z (mm)	
<input checked="" type="checkbox"/>	1	0.0000E+00	0.5000	110.0000	
<input checked="" type="checkbox"/>	2	116.0000	0.5000	110.0000	
<input checked="" type="checkbox"/>	3	234.0000	0.5000	110.0000	
<input checked="" type="checkbox"/>	4	350.0000	0.5	110.0000	
<input checked="" type="checkbox"/>	5	0.0000E+00	0.5000	0.0000E+00	
<input checked="" type="checkbox"/>	6	116.0000	0.5000	0.0000E+00	
<input checked="" type="checkbox"/>	7	234.0000	0.5000	0.0000E+00	
<input checked="" type="checkbox"/>	8	350.0000	0.5000	0.0000E+00	

File 2 Options	
File Type	unv
Units	Dataset 164 (UNV File)
File	D:\h
Orient By	Coordinate System
Coordinate System	Coordinate System
MAC Calculation Options	
Rigid Body Transformation	
3 Node Alignment	

# NVH Toolkit: MAC Calculator – Mode Animation

- If a square in the 2D MAC Table is clicked, both Modes are Animated in a side-by-side window
- The user has the ability to control the graphical windows (rotate, pane) and to interact with the animation (Play/Pause/Stop)



# NVH Toolkit: Stress/Strain Recovery

- The Stress/Strain recovery results computes elastic Stress/Strain from the Modal Superposition of the results of a Modal Analysis according to a Modal Coordinates File (MCF)
- The user inputs the MCF and selects its units and the normalization method (Mass or Unity) employed to derived the Modal Coordinates
- The MCF can be indexed by either Time or Frequency

[-] Geometry	
Scoping Method	Geometry Selection
Geometry	1 Face
[-] rst Options	
rst File	D:\[redacted]...
[-] MCF Options	
MCF File	D:\[redacted]...
Skip Rows	10
Normalization	Mass Normalization
Units	Metric (mm-ton)
[-] Extract Options	
By	Frequency
Display Frequency	400 Hz
Amplitude	No
Sweeping Phase	0 °
[-] Stress Properties	
Type	Equivalent (von-Mises) Stress
Shell Layer	Top



# MAPDL Contacts



# / Dual Shape Function Based Contact Formulation

- The dual shape function-based algorithm is state of the art for projection-based contact
  - It reduces solution time and memory usage (performance improvement)
- Reduces dependent terms for each contact constraint:
  - Gap function/slip from dual mortar contact are closer to local geometric gap/slip at contact nodes
  - It is more efficient and remedy potential over-constraints.

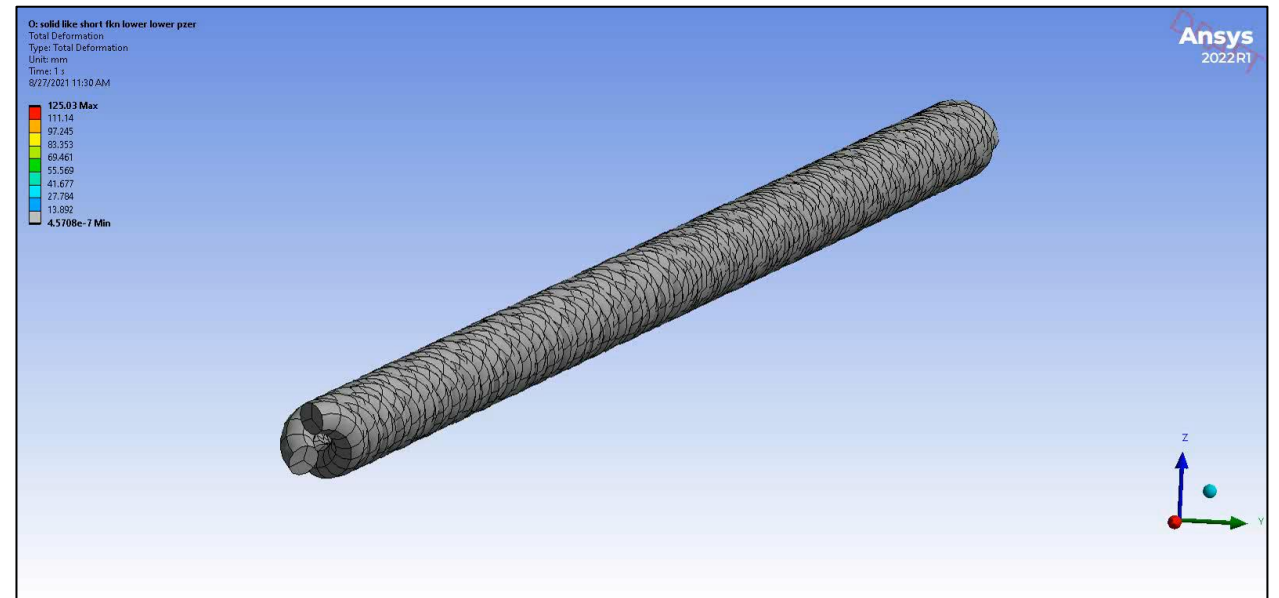
	Total Iteration	Max. Memory used	Max. Memory allocated	Run Time
Standard projection	69	89.1 GB	121.1 GB	12637 s
Dual shape function	68	56.7 GB	84.8 GB	5904 s (2.1x)

Three contact pairs Total Dofs 1,792,867, Augmented Lagrange method  
Running with DMP 32 CPUs

# New Adaptive Small Sliding Option

- Small sliding is assumed during substep. The nodal connectivity of the contact element reformed in the beginning of each substep and remains unchanged during iterations
- Bisection is performed if a large sliding distance is detected within a substep
- It can handle situations when contact pairs are initially in far-field and then come into contact
- The adaptive small-sliding logic often **improves robustness/performance** compared to the finite sliding logic and can also **improve solution accuracy** compared to the small sliding logic

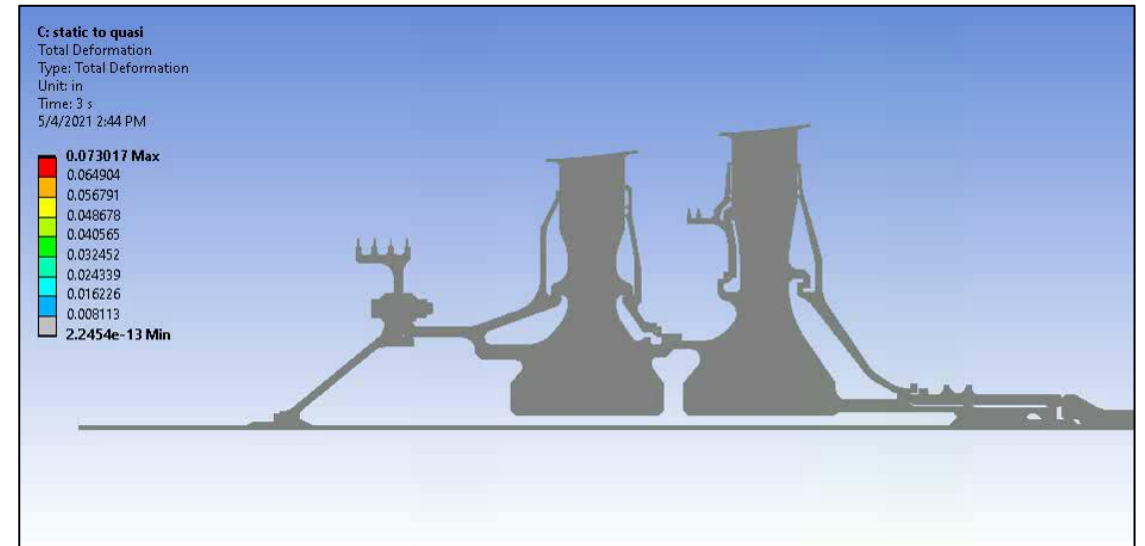
	Total Iterations	Elapsed Time
Pure small sliding	Fail at 4.5% loading	
Finite sliding	450	3 h 13 m
Adaptive small sliding	383	1 h 10 m (3x)



# Contact Robustness Improvements

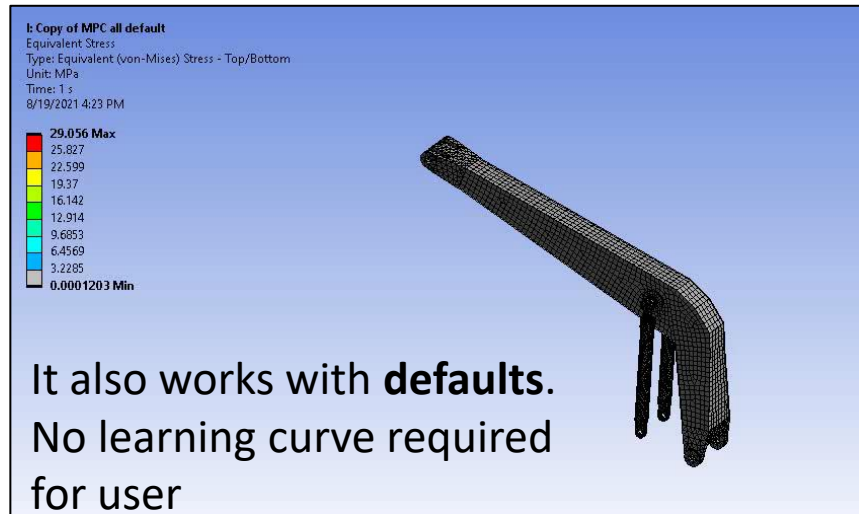
- Revise pinball radius for consistency between small and large deflection
  - Initial contact properties/results in Mechanical contact tools will be the same during solution
- Improve local contact searching to prevent potential spurious contact
  - Go over local target segments inside pinball to find the closest segment
- Improve solution for interference fits
  - Automatically turning on/off based on initial contact status/penetration
- Introduce cut-off contact stiffness for exponential relationship which no longer depends on the initial pressure

	Total Iterations	Elapsed Time
2021 R2	606	1621 s
2022 R1	514	1543 s



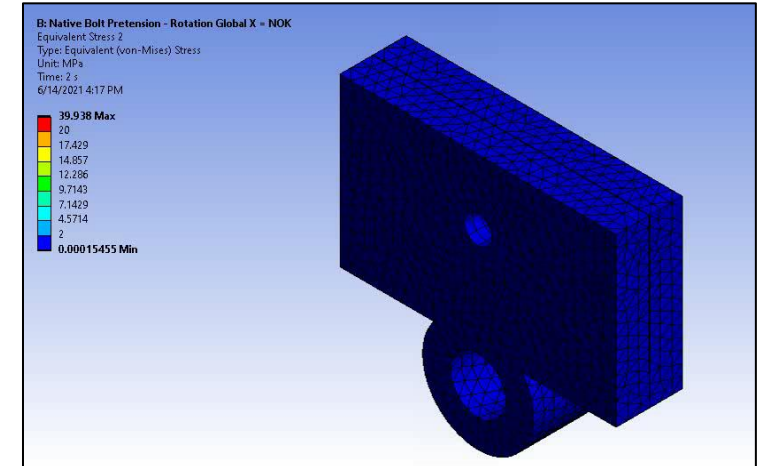
# Robustness and Accuracy Improvements for Shell Assemblies

- Motivation: to address an issue of stress-free rotation for a shell assembly model
- Enhance SHSD command:
  - Supports Line contact element (177) defined as shell edge
  - Supports gap/penetration close
- MPC is built based on large rotation based RBE3 constraints
  - Shell to shell, Shell to solid-surface, shell-edge to shell-edge



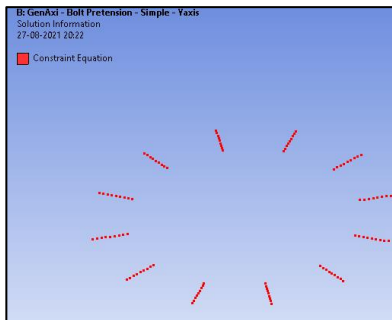
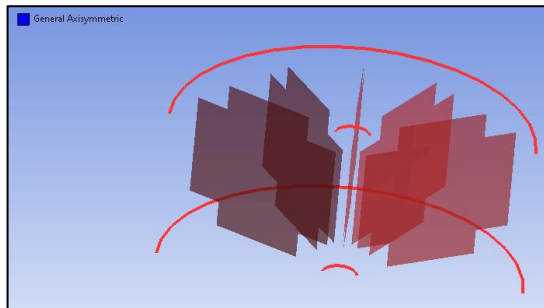
Stressman model

Method	Max Stress (MPa)
Bonded default formula (AL)	8064
Shared Topology	26
MPC Bonded (21R2)	Not solved
Bonded using beams	30
MPC bonded (22R2), all default	29

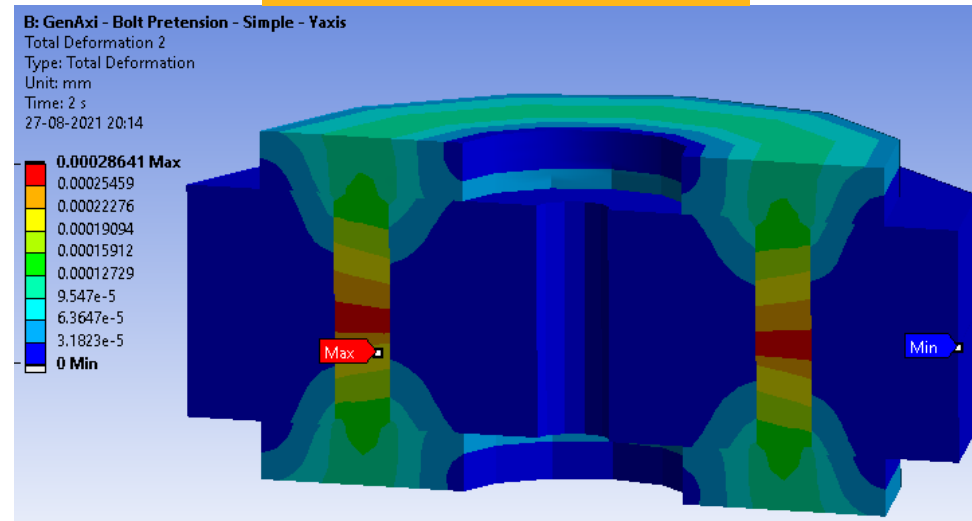


# Bolt pretension is Now Supported with General Axisymmetric Elements

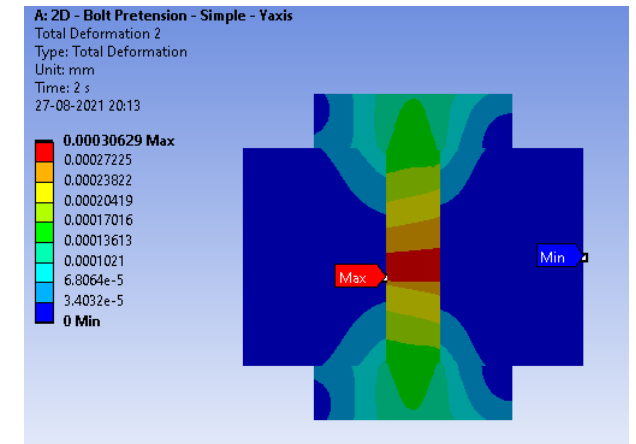
- PSMESH command is supported for general axisymmetric elements. It cuts the bolt and generate the pretension elements between all Fourier nodes.
- SLOAD command is also supported for general axisymmetric elements.
- Support adjustment and incremental load as well.
- Use Case: Apply preloads for part assemblies of gas turbines, automotive applications



General Axisymmetric



2D Axisymmetric



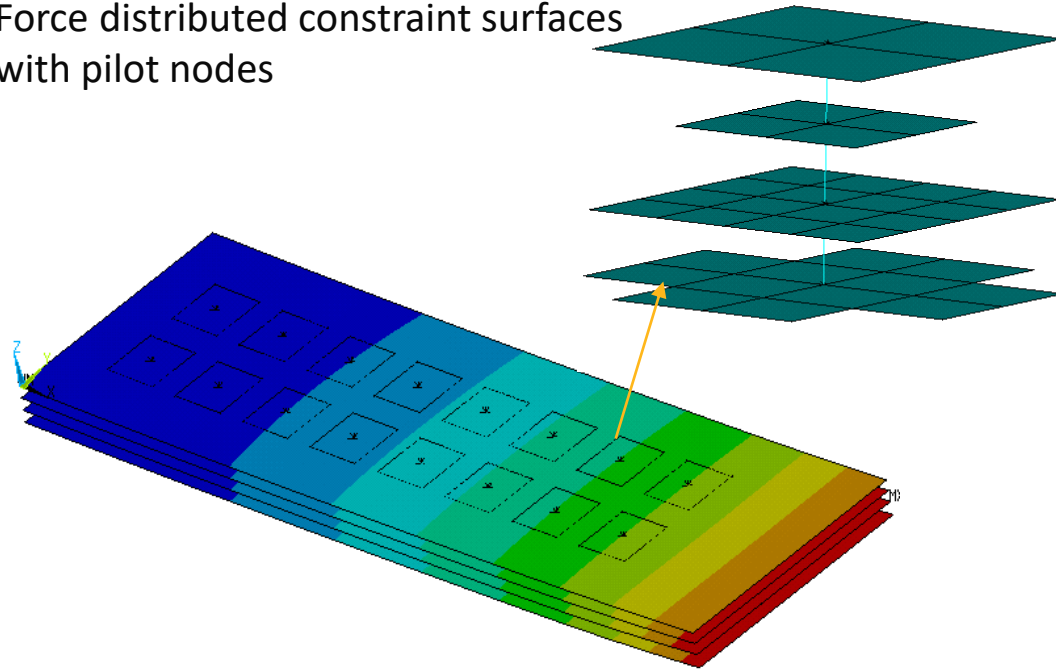
General axisymmetric results are similar as 2D axisymmetric or 3D solid



# / Mesh Independent Spot Weld Enhancements

- SWGEN/SWADD command:
  - Node-to-surface (175-170) MPC contact pairs are replaced by force distributed constraints with pilot nodes (174-170) to support large rotation framework and to make it more robust.
  - Spot weld generation is **2 times faster**

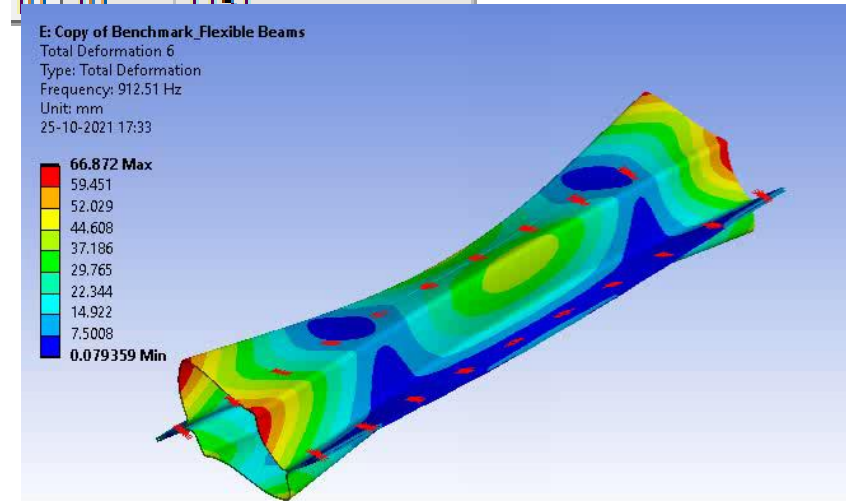
Force distributed constraint surfaces  
with pilot nodes



Mode shapes and frequencies are  
very close to reference results  
with new spot weld configuration

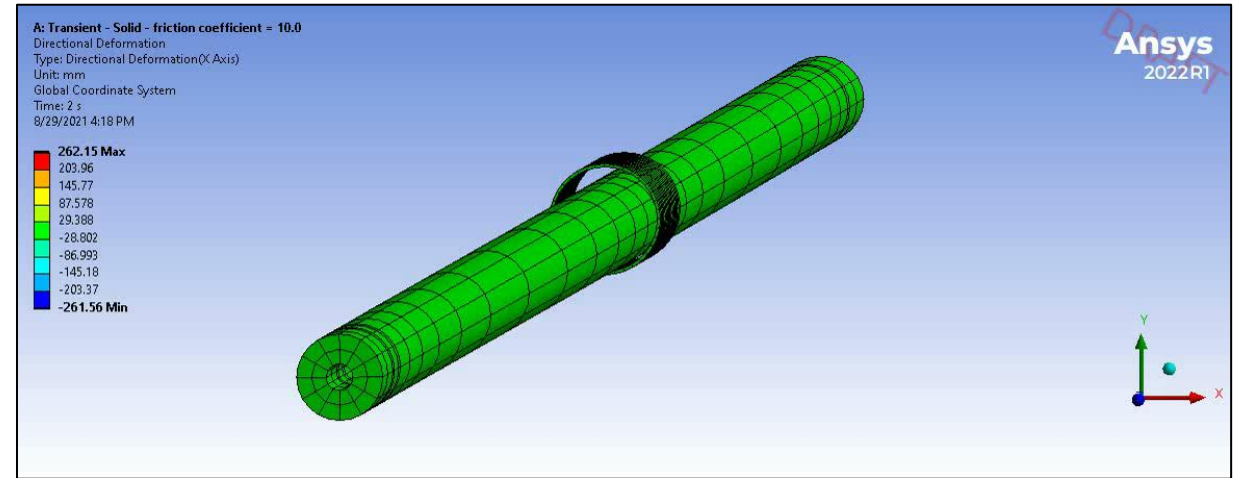
	Mode	<input checked="" type="checkbox"/> Frequency [Hz]
1	1.	474.41
2	2.	482.49
3	3.	537.25
4	4.	538.37
5	5.	771.37
6	6.	912.51
7	7.	917.97
8	8.	1077.
9	9.	1106.2
10	10.	1151

DH1 updating: natural frequency error		
Mode	Description	Experimental (Hz)
1	A1	474.6
2	A2	497.3
3	B2	525.4
4	B1	536.3
5	BOTTOM	768.4
6	TOP	899.1
7	B3	913.3
8	FLANGES	1060.0
9	C3	1110.0
10	C1	1170.0
11	2ND BEND.	1190.0

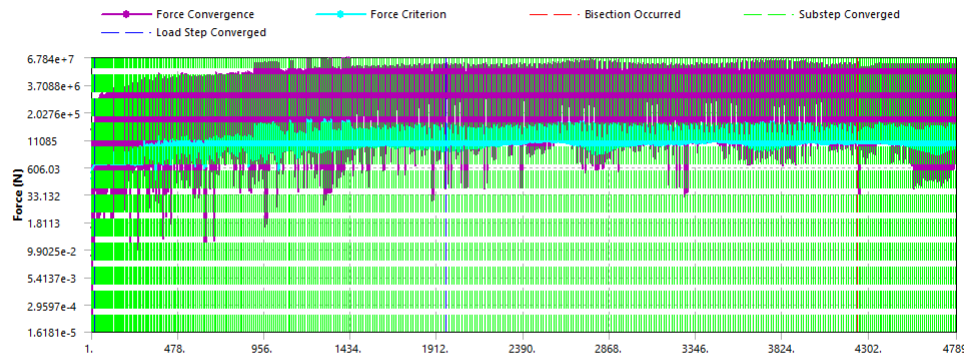


# Transient Dynamic Accuracy Improvements: HHT Algorithm

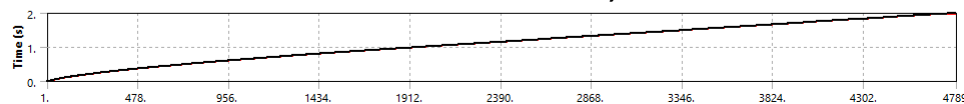
- HHT algorithm heuristics are improved. Stiffness matrix more consistent
- Start of new load step continues in HHT (used to go back to Newmark for 1<sup>st</sup> substep)
- Restart continues with HHT (used to go back to Newmark for 1<sup>st</sup> substep)



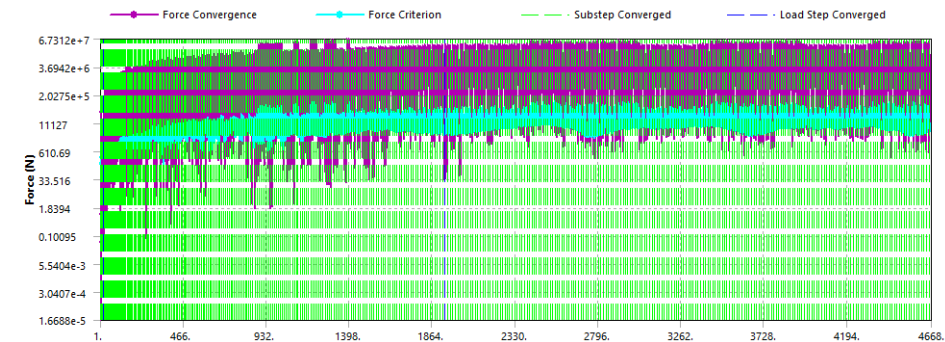
Force Convergence



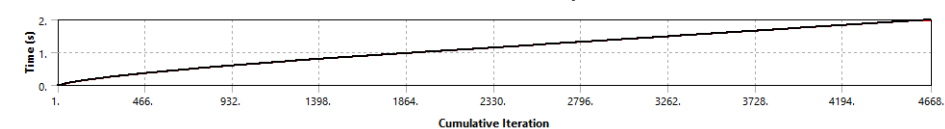
2021 R1 - More iterations, bisections



Force Convergence

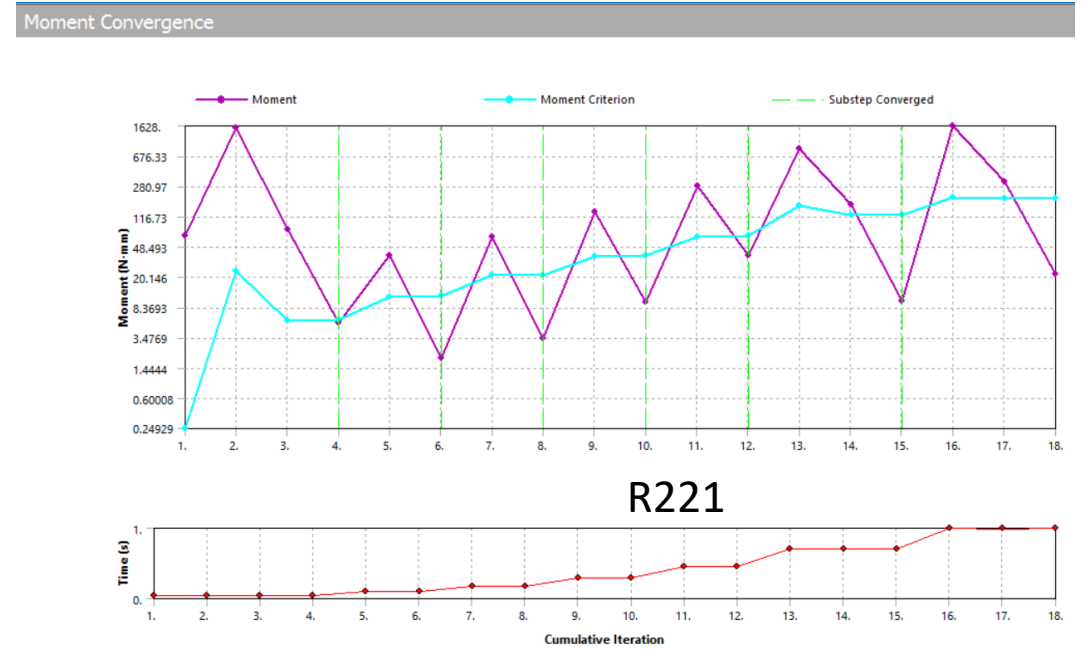
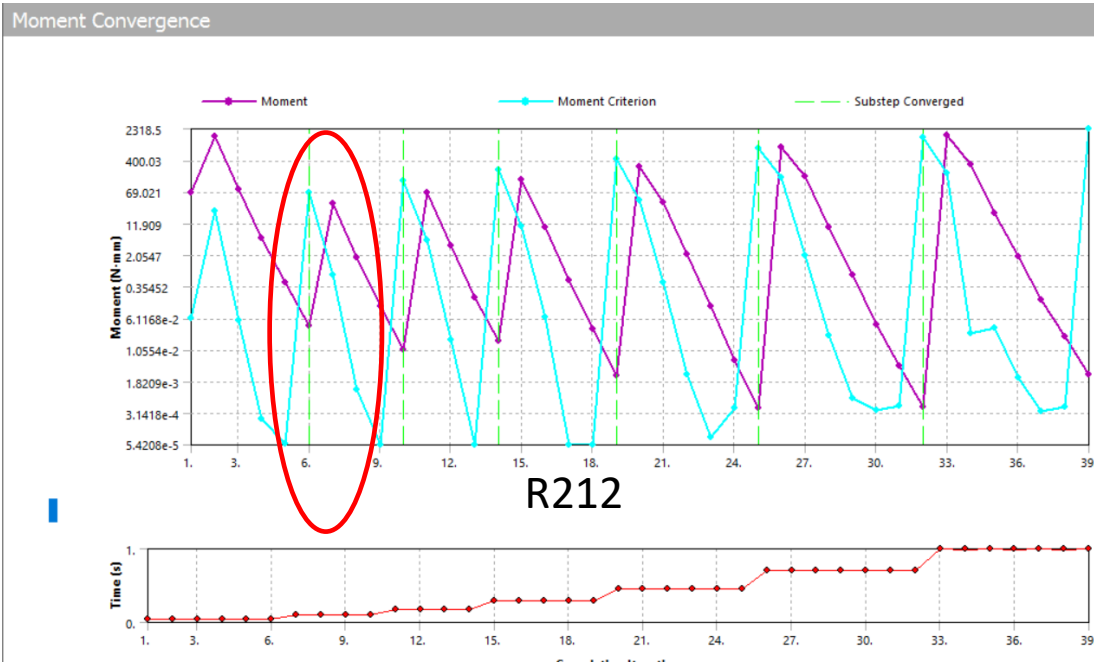


2021 R2- No bisections, ~3% less iterations



# Moment Convergence Reference Calculation (Robustness Improvement)

- Improved minimum reference calculation when Moment convergence reference is small
- Minimum reference based on mesh size and bounding box and force reference value results in smooth and accurate convergence



The sudden increase in moment convergence reference  
in 2021 R2 due to heuristics for “small values”

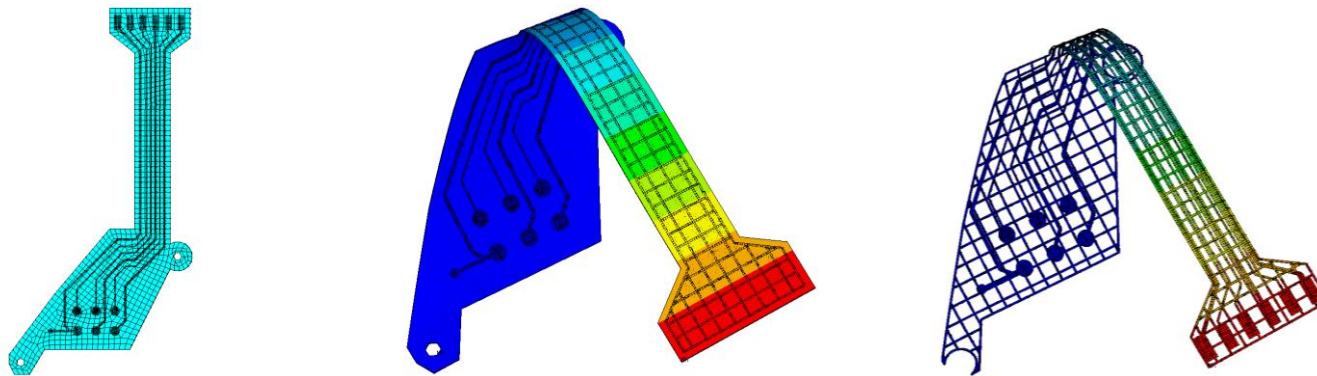
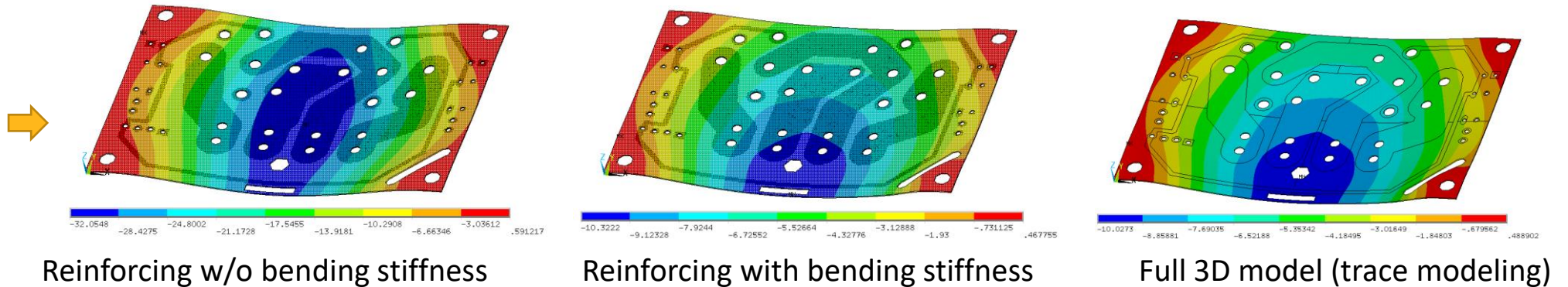
# MAPDL Elements



# Bending Stiffness for Smeared Reinforcement

- Greatly improves the solution accuracy with 3D smeared reinforcing (REINF) models
- Eliminates the need to use multiple REINF layers to capture the bending stiffness
- Enhances the REINF modeling usability in the new PCP/Chip simulation workflow

Accurate simulation results: REINF with bending vs. full 3D model

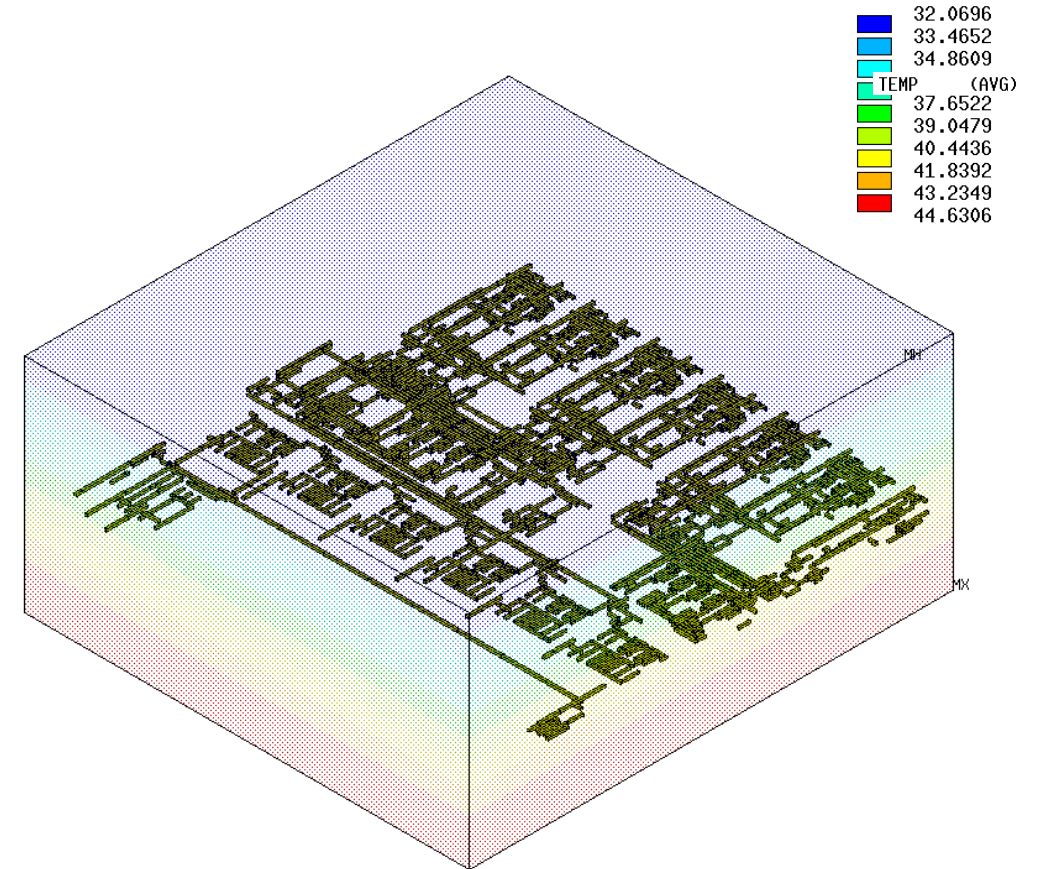


Robust and accurate simulation of flexible PCBs under large deformation



# / Reinforcing Performance Enhancements

- Motivated by the requirements to account for large models (full PCB and chip models)
- Improved performance in pre-processing
  - Allows large number of reinforcing members in one base element
  - Reduces time needed for load mapping
- Improved solution efficiency
- Improved performance in post-processing
  - Significantly reduced time for querying min/max member results
  - Improved inter-member result smoothing

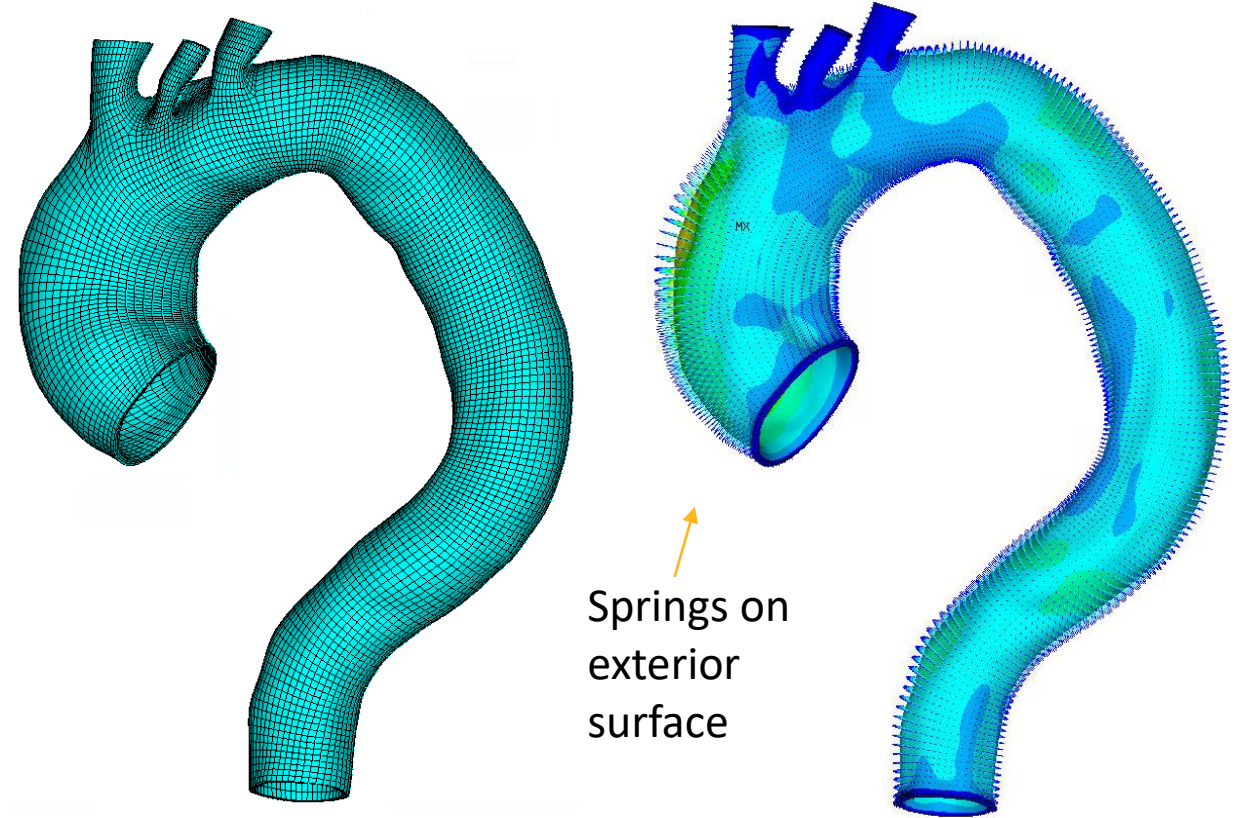




# / Inverse Solving with Uni-axial Stiffness Elements

- Motivated by the requirements from biomedical simulation to derive stress free configuration.
- LINK180 and COMBIN39 are now supported in inverse analysis
- Support user-defined nonlinear springs
- Capable of forward simulation with new loading after inverse solving

Inverse analysis of a thoracic aorta. Boundary conditions modeled with spring elements



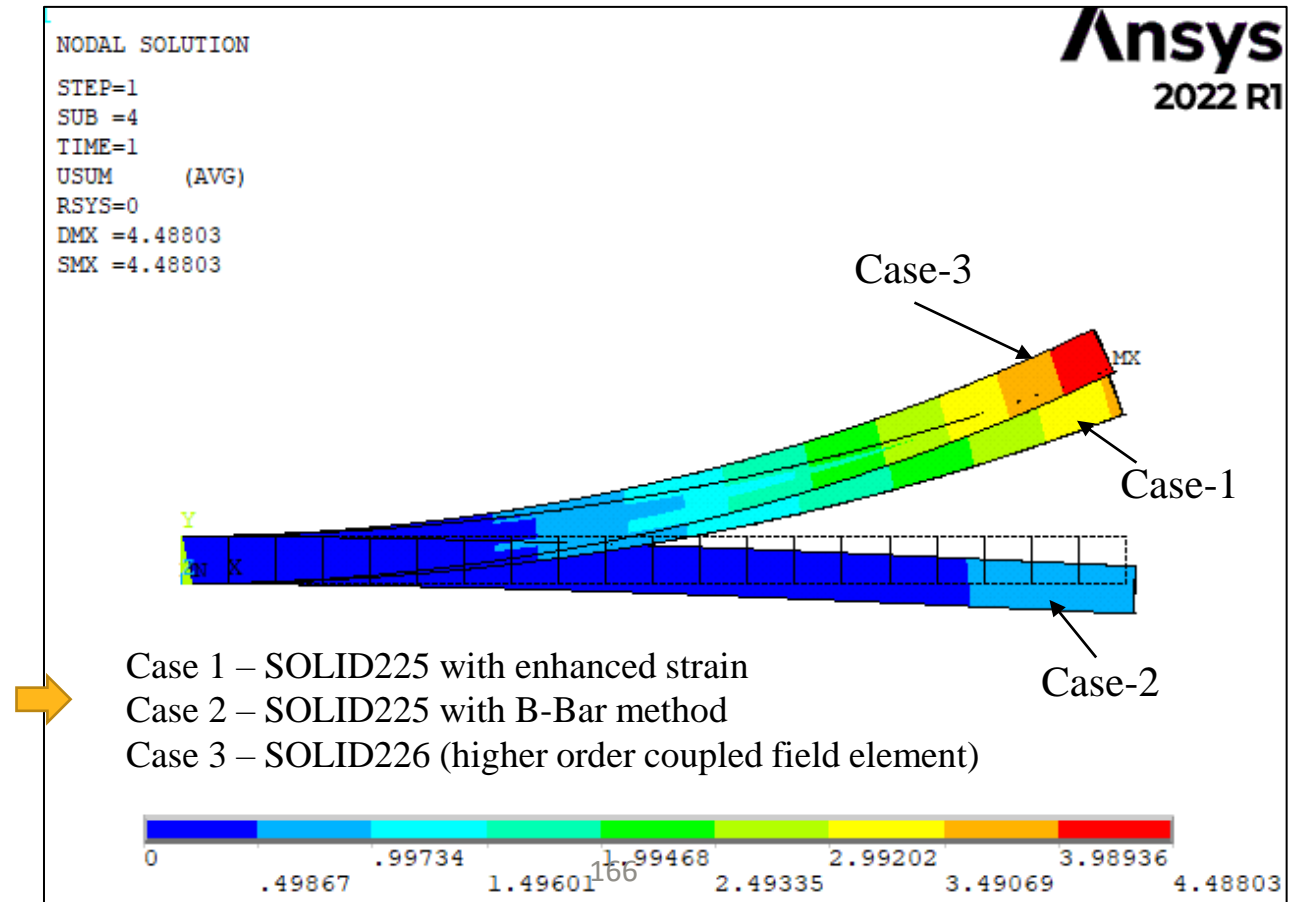
Input geometry under load

Recovered stress free state

# Enhanced Strain Formulation for PLANE222 and SOLID225 in Structural-Thermal Analysis

- New enhanced strain formulation improves solution accuracy of lower order coupled-field elements (PLANE222 and SOLID225) for structural-thermal analysis
- Requires less elements to capture the accurate response of bending-dominated problems
- Alleviates volumetric locking issues associated with incompressible materials

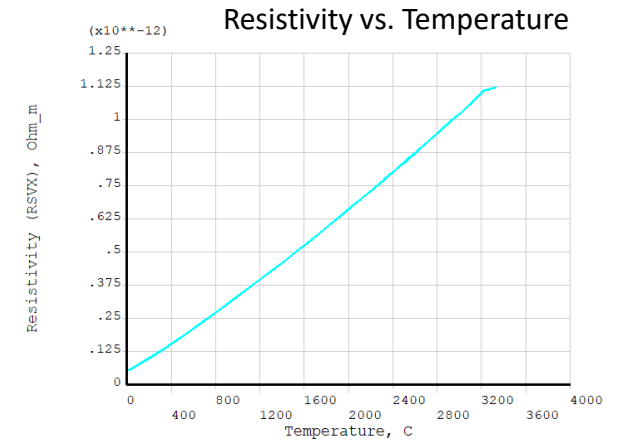
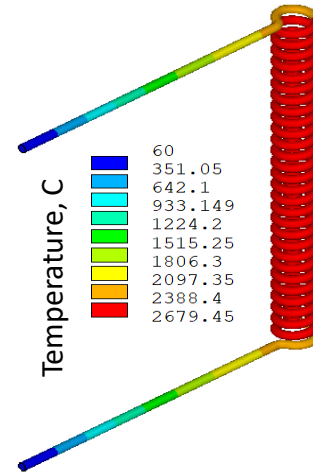
A cantilever beam under varying thermal load. Significant accuracy improvement with enhanced strain SOLID225



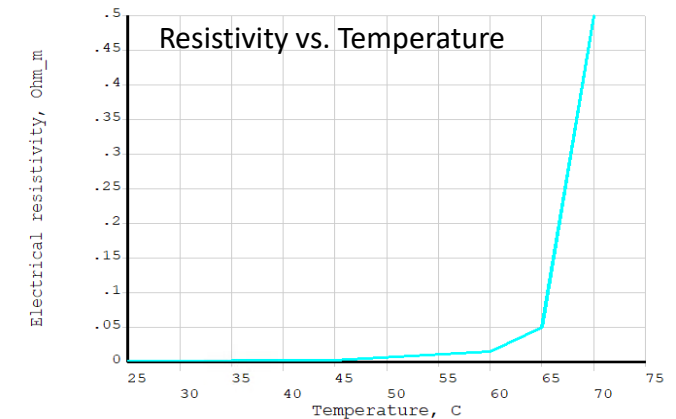
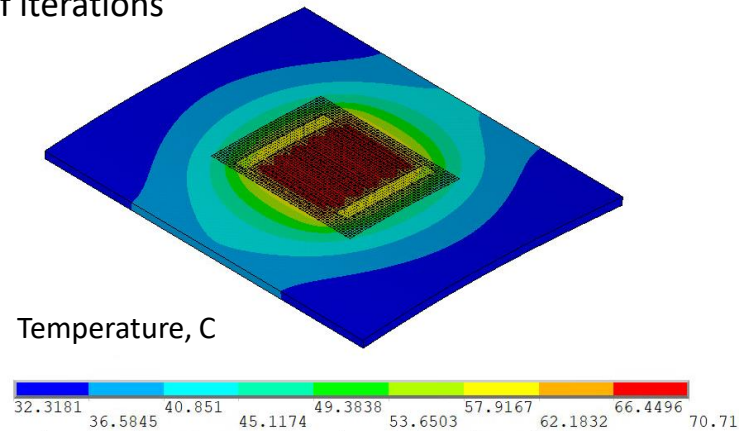
# Enhanced Robustness of Coupled-Field Analyses

- Consistent element stiffness matrices to account for nonlinear heat generation rates and electrical resistivity
- Greatly reduces number of nonlinear iterations. Overcomes previously non-converging problems
- Modified elements:
  - PLANE222, PLANE223, SOLID225, SOLID226, and SOLID227

Steady-State Heating of a Tungsten Coil. Solution failed to converge previously. Now solved successfully.



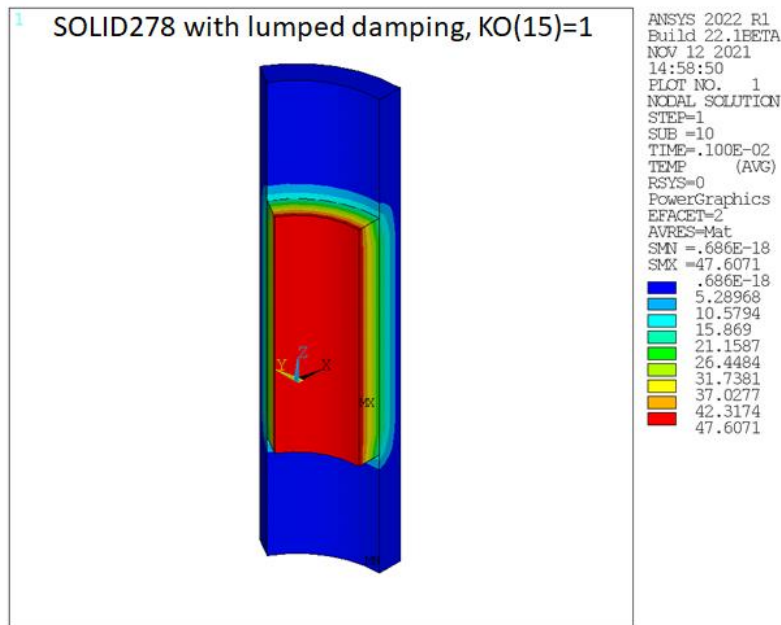
Transient analysis with car seat heating elements, converged with much fewer number of iterations



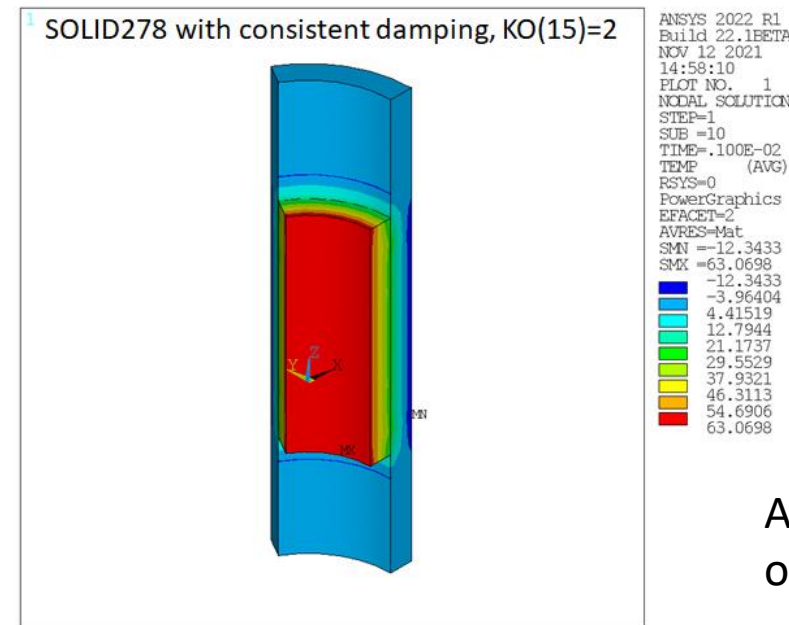
# / Lumped / Consistent Matrix Options for Thermal Elements

- Oscillations in thermal solution can be controlled with lumped/consistent form of damping (specific heat) matrix and convection matrix.
- A new option is now available to choose these options in current technology thermal elements
- With these settings user has better control on which regions should have lumped or consistent matrices.

Correct TEMP  
distribution



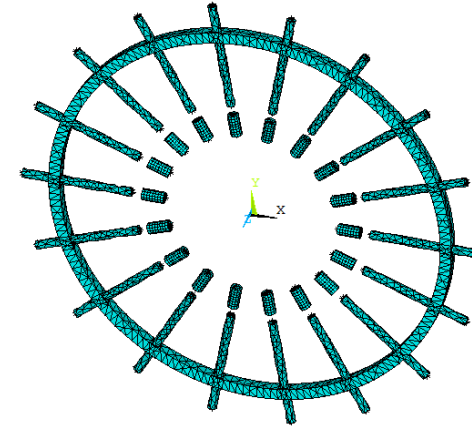
Artificial TEMP  
oscillation



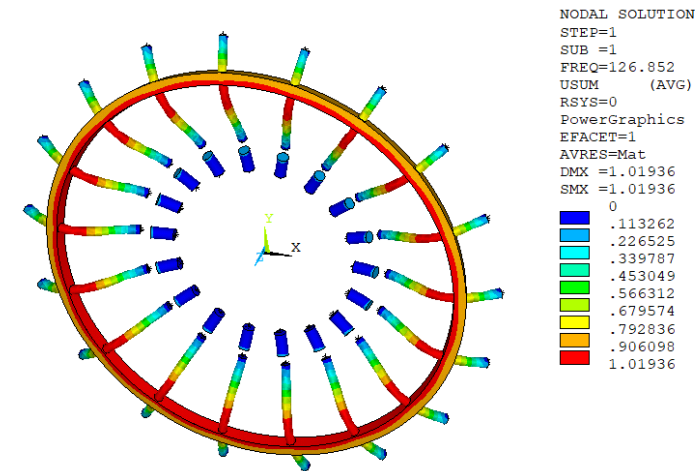
Transient thermal analysis of two concentric cylinders. The non-physical oscillation in temperature result is suppressed with a proper damping matrix option

# Joint Element Enhancements

- Motivated by the issue of overconstraint that affects the Lagrange Multiplier method approach, a penalty-based approach to joint elements was implemented
- This method also avoids the introduction of any new solution variables unlike the Lagrange Multiplier method
- The penalty-based joint elements are now supported in modal-based and linear perturbation procedures
- Additionally, the penalty-based spherical joint element now supports locks, stops, and friction



Cyclic Symmetric Boat Wheel Model



First Mode Shape



# Demonstration Example: Linear Perturbation Analysis of a Cyclic Symmetric Boat Wheel

**Model:** Each spoke has two cylindrical parts connected by a MPC184 revolute joint with linear stiffness specified.

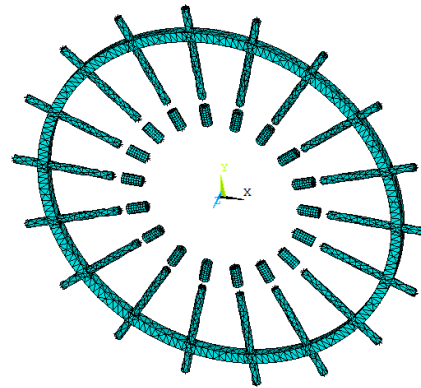
Loading:

**Static load step:** Radial displacements, axial displacements and surface pressure on outer cylinder

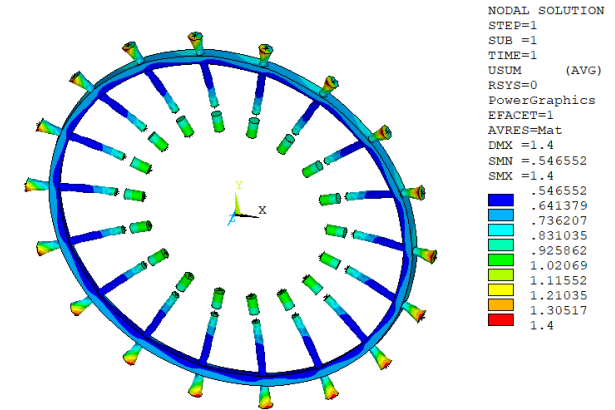
**Linear Perturbation step:** Modal analysis

LM vs Penalty Method

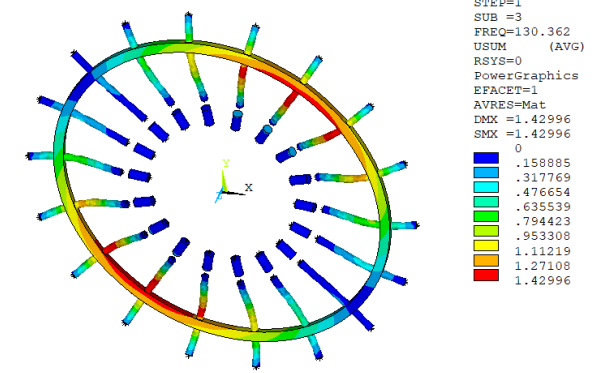
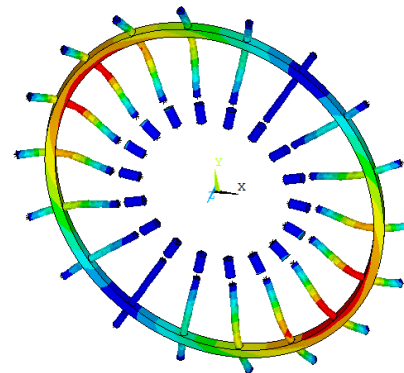
SET	TIME/FREQ	
	KO(2)1: Penalty	KO(2)0: LM
1	126.85	126.89
2	130.36	130.4
3	130.36	130.4
4	140.96	140.99
5	140.96	140.99



Geometry



Deformation at the end of static load step



2<sup>nd</sup> and 3<sup>rd</sup> eigen-mode shapes

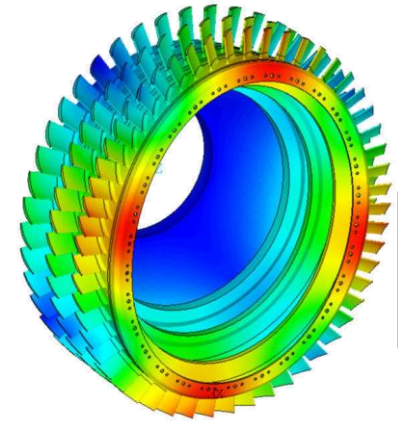
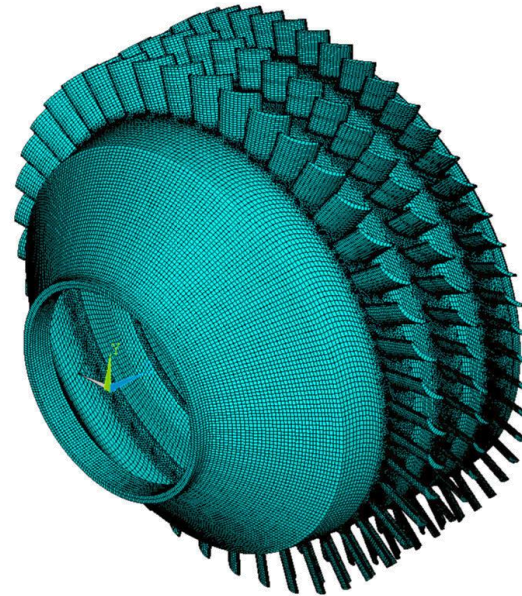
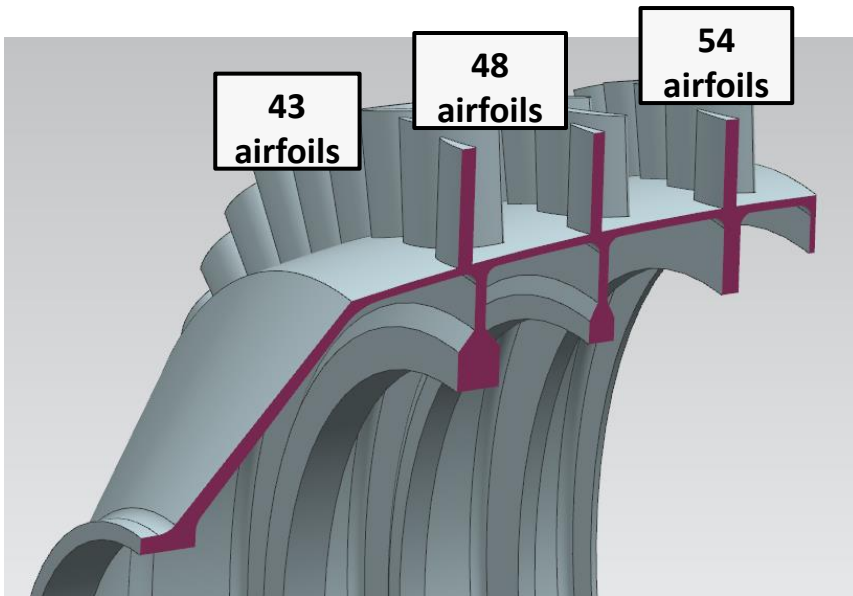


# MAPDL Linear Dynamics

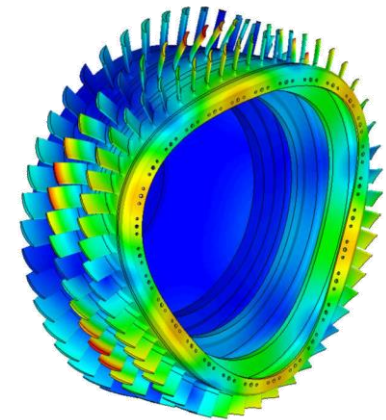


# Multistage Cyclic Symmetry

- Multistage cyclic symmetry involves combining distinct cyclic structures (stages) that do not have the same sector count
- This reduced model uses only cyclic sectors and special constraints at the stage interfaces
- This results in a dramatic reductions in model size, simulation time, and file storage



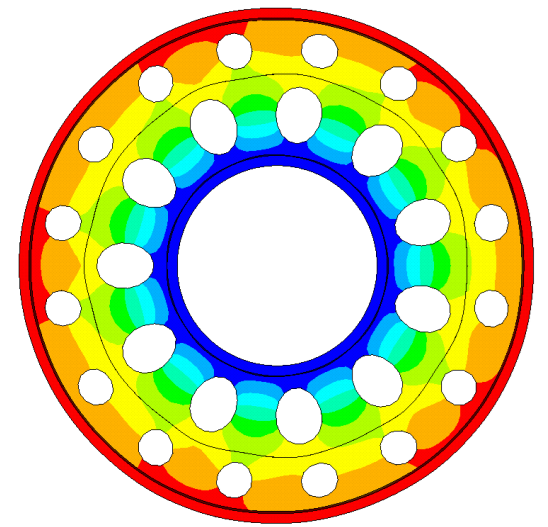
2 nodal  
diameter  
mode



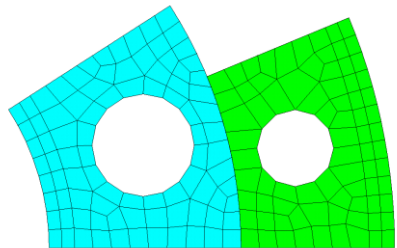
3 nodal  
diameter  
mode

# / Multiharmonic Multistage Cyclic Symmetry

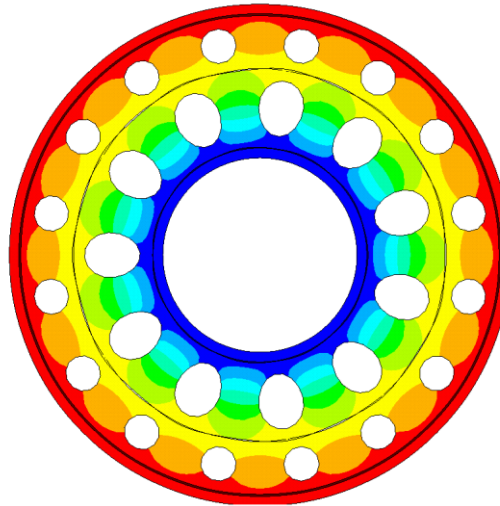
- Because there are different sector counts, multiple cyclic harmonics may contribute for each stage
- Multiple harmonics can be added to improve solution accuracy
- Results for each harmonic can be viewed individually to gain valuable engineering insight into contributions of the overall solution and how loading may affect the structure
- Harmonics can be automatically combined to achieve the expected physical solution



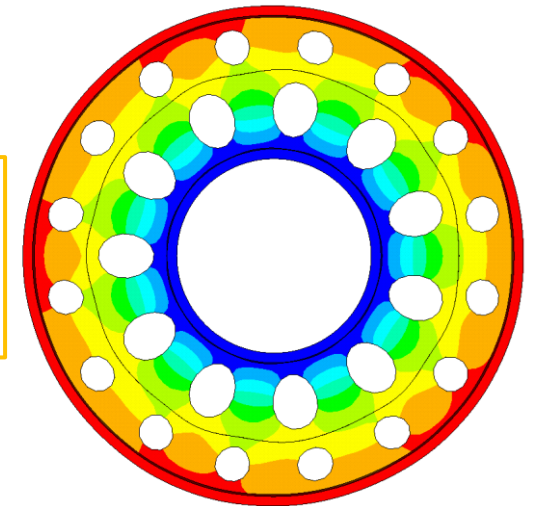
Reference Static Analysis:  
Static analysis under  
rotational load



Multistage:  
Static Analysis  
HI = 0  
Reduced Accuracy



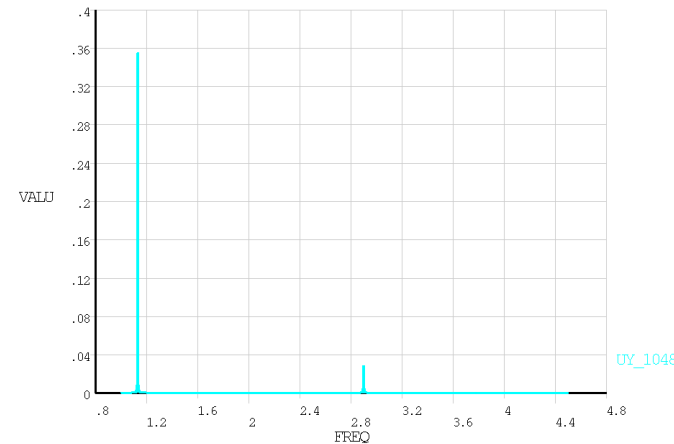
Multistage:  
Static Analysis  
HI = 0 + HI = 5  
High Accuracy



# Solution Comparison Using the Frequency Response Function (FRF) Correlation Method: RSTMAC

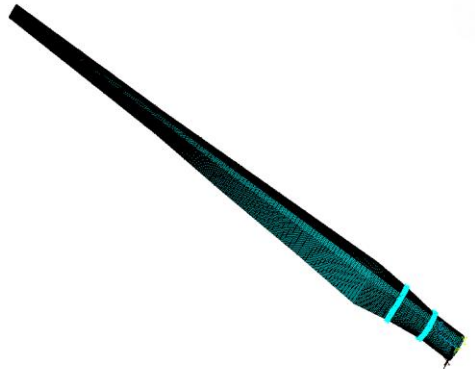
- FRF correlation analysis between experimental (.unv file) and FE generated FRF at each frequency point is supported.
- Two criteria:
  - Cross-signature assurance criterion (CSAC),
  - Cross-signature scale factor (CSF).
- CSAC is a measure of the shape correlation, an adaptation of MAC for the frequency domain.
- CSF is a measure of the difference in amplitude.

FRF graph at one of the measurement points



ANSYS 2021 R2

ELEMENTS

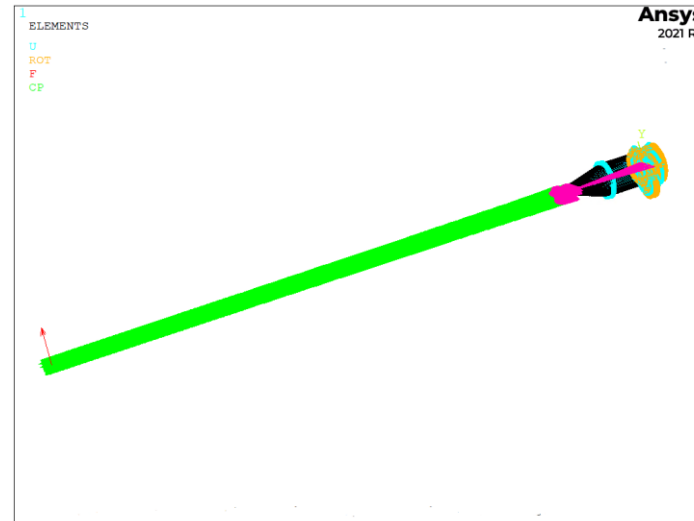


\*\*\*\* INDEX OF DATA SETS ON RESULTS FILE \*\*\*\*

SET	TIME/FREQ	LOAD STEP	SUBSTEP	CUMULATIVE
1	0.25969	1	1	1
2	0.95582	1	2	2
3	1.1308	1	3	3
4	2.8981	1	4	4
5	3.7871	1	5	5
6	4.9242	1	6	6
7	5.9346	1	7	7
8	6.4791	1	8	8
9	7.0171	1	9	9
10	7.4022	1	10	10

# Solution Comparison Using the Frequency Response Function (FRF) Correlation Method: RSTMAC

- This feature is enhancing the RSTMAC procedure and can be applied using FRF option on MACOPT command:  
**MACOPT,FRF**
- Results are presented for a wind turbine blade excited at its end and measurements are stored for different nodes in unv file format.



\*\*\*\*\* Frequency Response Function correlation \*\*\*\*\*  
Solutions are complex  
Number of substeps: 37

Frequency:	CSAC	CSF
1.000	1.000	1.000
1.088	1.000	1.000
1.175	1.000	1.000
1.263	1.000	1.000
1.350	1.000	1.000
1.438	1.000	1.000
1.525	1.000	1.000
1.613	1.000	1.000
1.700	1.000	1.000
1.788	1.000	1.000
1.875	1.000	1.000
1.963	1.000	1.000
2.050	1.000	1.000
2.138	1.000	1.000
2.225	1.000	1.000
2.313	1.000	1.000
2.400	1.000	1.000
2.488	1.000	1.000
2.575	1.000	1.000
2.663	1.000	1.000
2.750	1.000	1.000
2.838	1.000	1.000
2.925	1.000	1.000
3.013	1.000	1.000
3.100	1.000	1.000
3.188	1.000	1.000
3.275	1.000	1.000
3.363	1.000	1.000
3.450	1.000	1.000
3.538	1.000	1.000
3.625	1.000	1.000
3.713	1.000	1.000
3.800	1.000	1.000
3.888	1.000	1.000
3.975	1.000	1.000
4.063	1.000	1.000
4.150	1.000	1.000



# MAPDL Materials





# / Hyperelasticity with Embedded Fibers

- Isotropic hyperelastic materials can be augmented with embedded fiber terms

- For example:  $W = W_{iso} + \boxed{\frac{c_1}{2c_2} \left\{ \exp \left[ c_2 (\bar{I}_4 - 1)^2 \right] - 1 \right\}}$   $\longrightarrow$  Embedded fiber strain energy

- Up to 5 fibers can be added to the isotropic model
- The fibers can have different tension and compression properties

Tension  $\longrightarrow$   $W_{fib} = \frac{c_1^t}{2c_2^t} \left\{ \exp \left[ c_2^t (\bar{I}_4 - 1)^2 \right] - 1 \right\}$

Compression  $\longrightarrow$   $W_{fib} = \frac{c_1^c}{2c_2^c} \left\{ \exp \left[ c_2^c (\bar{I}_4 - 1)^2 \right] - 1 \right\}$

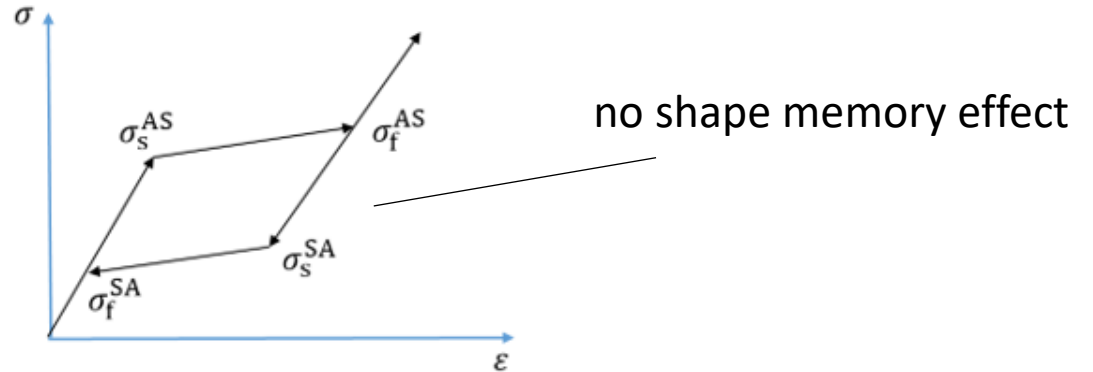
For typical fiber behavior, define the properties so the fibers only support tension:

$$c_1^t \neq 0, \quad c_2^t \neq 0, \quad c_1^c = 0, \quad c_2^c = 0$$

# Current Ansys SMA Material Models: TB,SMA

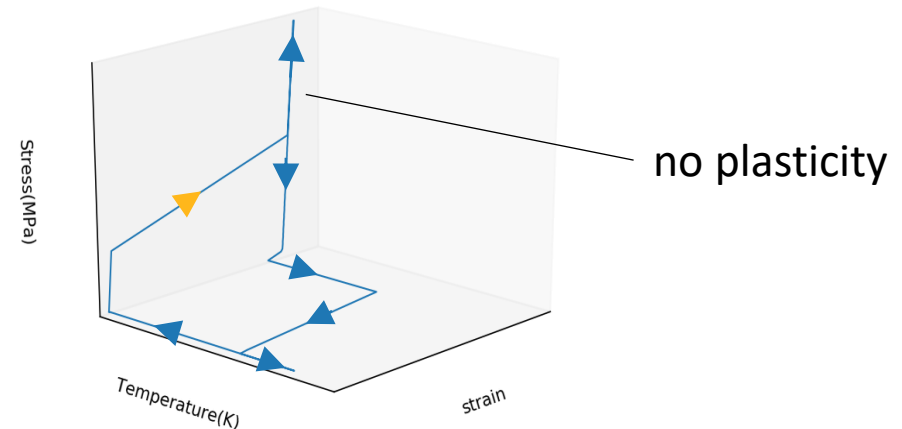
SMA model for superelasticity: **TB,SMA,,,,SUPE**

```
MP,EX,1,60000.0
MP,NUXY,1,0.36
Define SMA material properties
TB,SMA,1,,,SUPE
TBDATA,1, 520, 600, 300, 200, 0.07, 0.0
```



SMA model with shape memory Effect: **TB,SMA,,,,MEFF**

```
MP,EX,1,60000.0
MP,NUXY,1,0.36
Define SMA material properties
TB,SMA,1,,,MEFF
TBDATA,1,1000, 223, 50, 2.1, 0.04, 45000
TBDATA,7,0.05
```

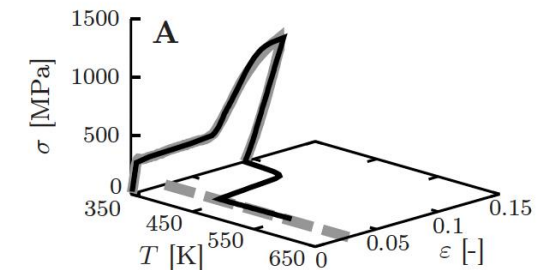
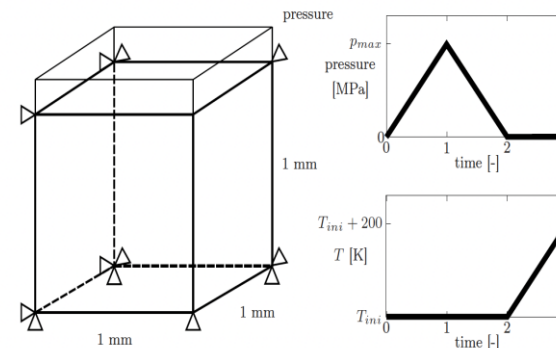
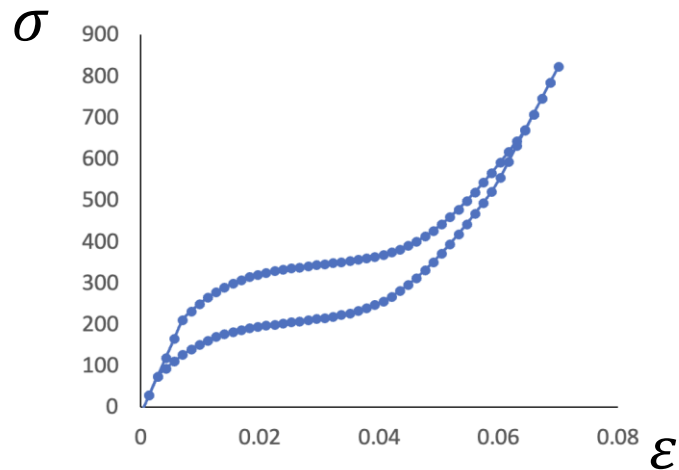
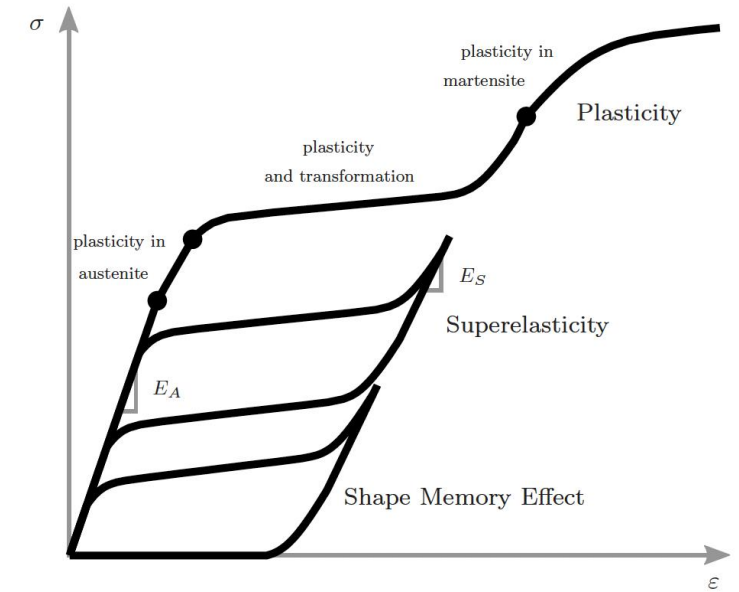


# New SMA Model

Table 1: Comparison of SMA models based on [main](#) features. Note that  $E$  is Young's modulus and TC is tension-compression.

Feature	Present Work	Ansys SUPE	Ansys MEFF	Abaqus
Superelasticity	✓	✓	✓	✓
SME	✓	X	✓	X
Plasticity	✓	X	X	✓
Reorientation	✓	✓	✓	✓
Phase dependent $E$	✓	✓	✓	✓
TC asymmetry	X	✓	✓	✓
Plasticity in austenite	✓	X	X	X
Simultaneous plasticity and transformation	✓	X	X	X
Smooth transformation	✓	X	X	X

main features of the present SMA model



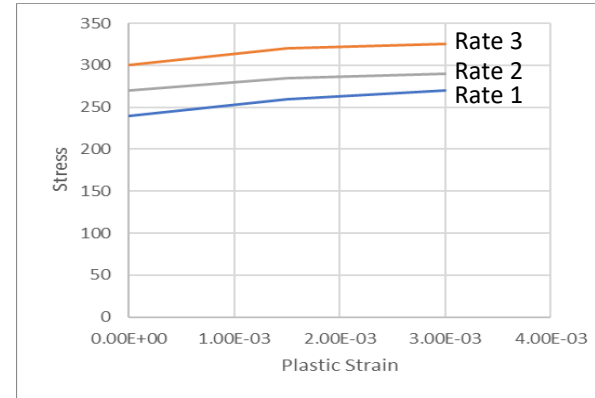
# Plastic-Strain-Rate-dependent Isotropic Hardening

- Direct entry of isotropic hardening data as a function of equivalent plastic strain rate
  - Define tabular data at various plastic strain rates using TBFIELD,PLSR
- Applicable to all isotropic hardening models
  - Bilinear isotropic hardening (TB,PLAS,,,,BISO)
  - Multilinear isotropic hardening (TB,PLAS,,,,MISO)
  - Nonlinear isotropic hardening (TB,NLISO,,,,VOCE/POWER)

- Consistent tangent considers plastic strain rate term:

$$\frac{\partial \sigma}{\partial \varepsilon^{pl}} = \frac{\partial \sigma}{\partial \varepsilon^{pl}} + \frac{\partial \sigma}{\partial \dot{\varepsilon}^{pl}} \frac{\partial \dot{\varepsilon}^{pl}}{\partial \varepsilon^{pl}} = \frac{\partial \sigma}{\partial \varepsilon^{pl}} + \boxed{\frac{1}{\partial t} \frac{\partial \sigma}{\partial \dot{\varepsilon}^{pl}}}$$

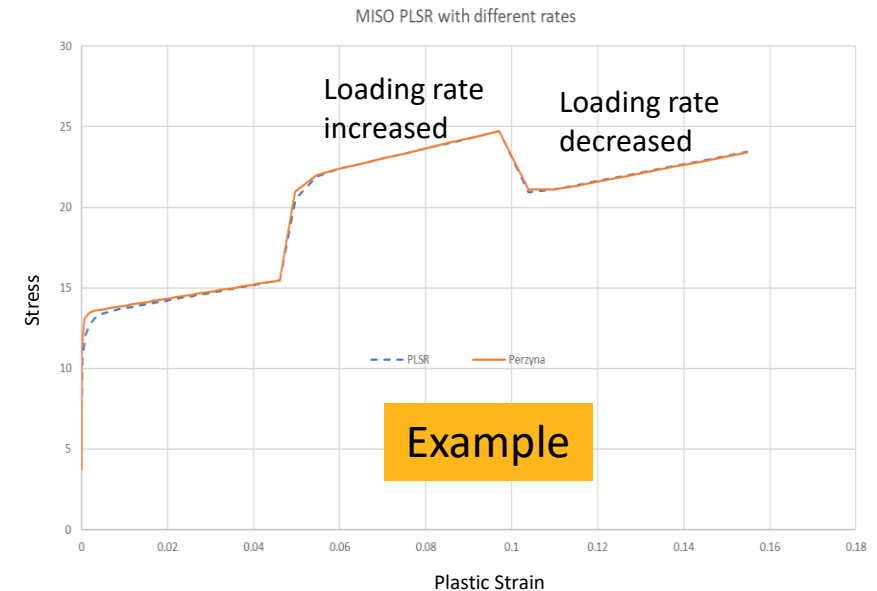
- The direct entry method can be used as an alternative to traditional rate-dependent models
  - Allows users to directly use experimental data at different rates
  - Avoids fitting of rate-dependent material model parameters



```
TB,PLAS,1,,,MISO
TBFIELD,PLSR,0.
TBPT,DEFI,0.0,15.0
TBPT...

TBFIELD,PLSR,rate2
TBPT,DEFI,0.0,18.624
TBPT...

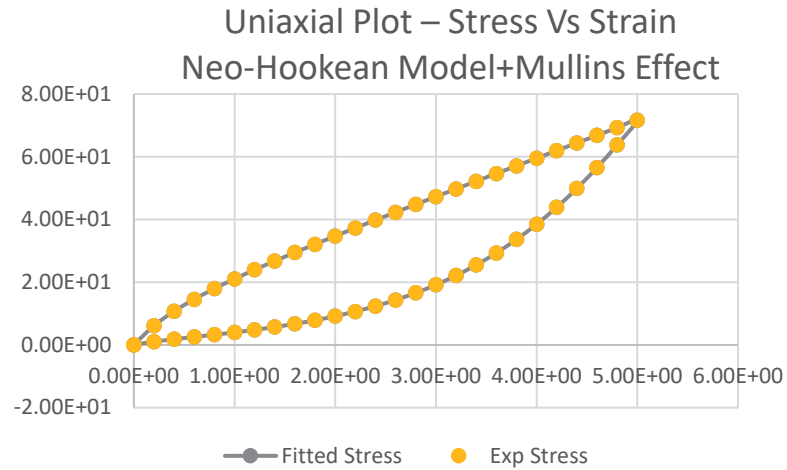
TBFIELD,PLSR,rate3
TBPT,DEFI,0.0,20.483
TBPT...
```



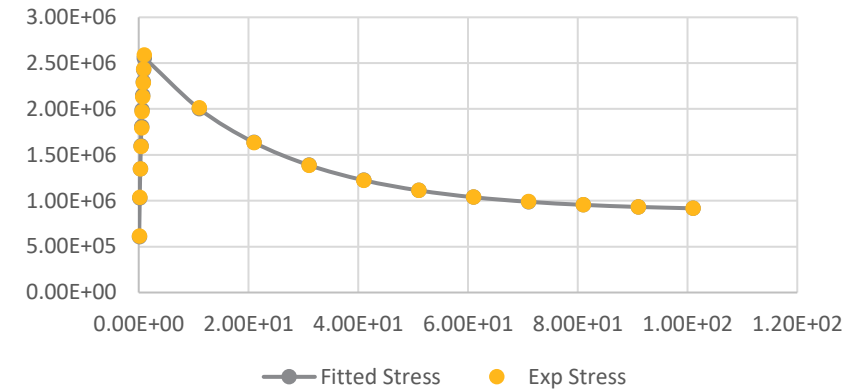
# / HyperElastic Parameter Fitting with AML

- Parameter Fitting has been expanded to support hyperelasticity and the following listed combinations
  - Hyperelasticity
  - Hyperelasticity with Mullins effect
  - Hyperelasticity with Prony Series
- Experimental Types
  - Uniaxial ; Biaxial; Pure Shear, Simple Shear and Volumetric
- Individual material models supported
  - Mooney Rivlin, Polynomial, Yeoh, Arruda Boyce, Gent, Ogden Foam, Blatz-Ko, NeoHookean, TNM Model(support extended to all exp types), Extended Tube, Bergstrom-Boyce
  - Mullins Effect
  - Prony Series ( Shear and Bulk Options )
- MAPDL GUI support via AML for previously supported hyperelastic models in the older GUI.
- Plotting available via TBFLOT macro for all models

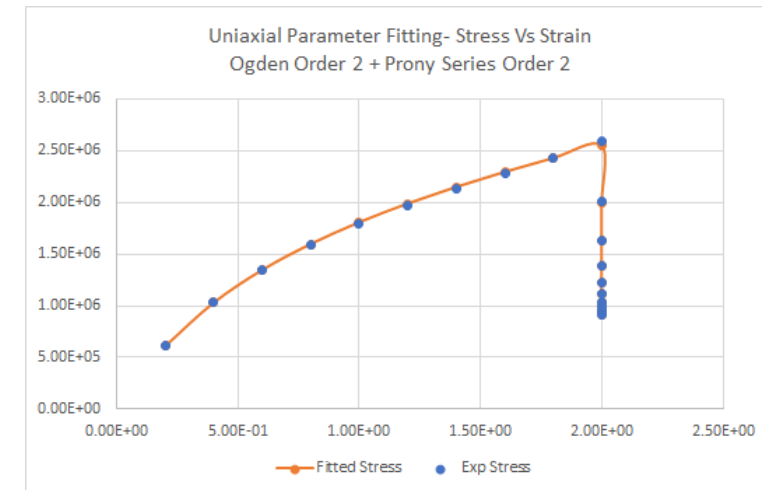
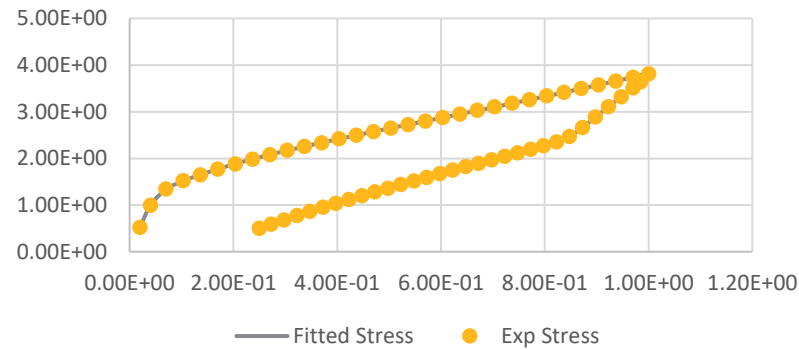
# HyperElastic Parameter Fitting with AML



Uniaxial Parameter Fitting- Stress Vs Time  
Ogden Order 2 + Prony Series Order 2



Biaxial Parameter Fitting - Stress Vs Strain  
TNM Model





# MAPDL Fracture



# Stress-Ratio Based Fatigue Crack Closure Models

## SMART – fatigue crack growth

Crack growth rate model  
(Paris/Tabular fatigue law)

+

**Crack-closure model**  
[optional]

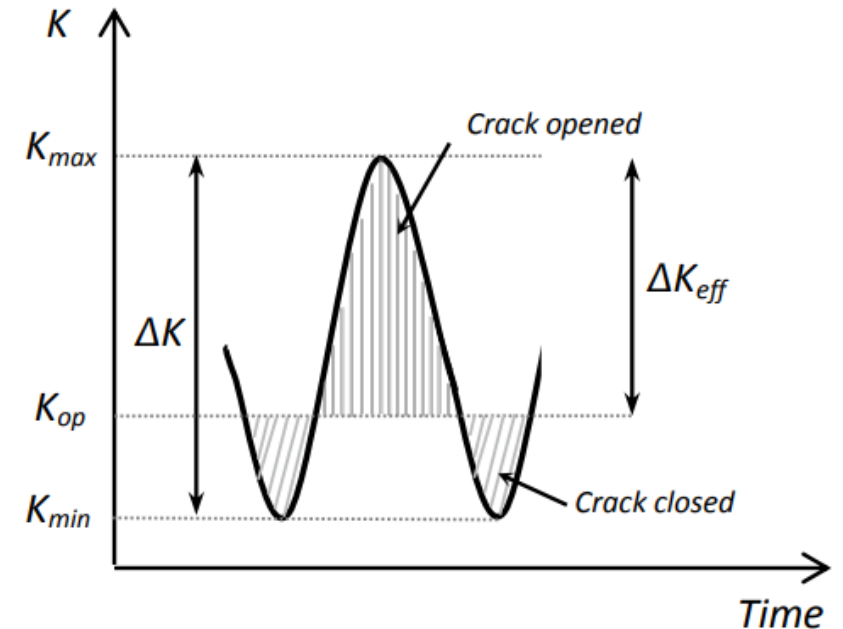
Measure of crack-closure

$$U = \frac{\Delta K_{eff}}{\Delta K}$$

$$\frac{da}{dN} = g(\Delta K_{eff}) = g(U\Delta K)$$

## Fatigue crack closure models

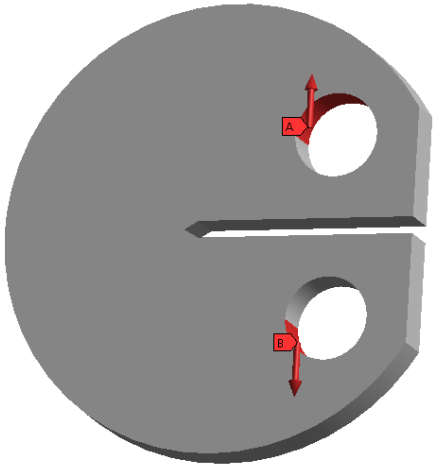
Elber function	$U = 0.5 + 0.4R$
Schijve function	$U = 0.55 + 0.33R + 0.12R^2$
Newman function	$U = \frac{1 - f(R)}{1 - R}$ <small><math>f(R)</math> is Newman crack opening function</small>
Polynomial function	$U = A_0 + A_1R + \dots + A_nR^n$



$$R = \frac{K_{min}}{K_{max}} \quad \text{stress ratio}$$

# Example: Paris Law Combined with Crack Closure Effect

Disk-shaped CT specimen



Paris law

( $C=1.3E-10$ ,  $m=2.08$ )

+

Polynomial crack-closure function

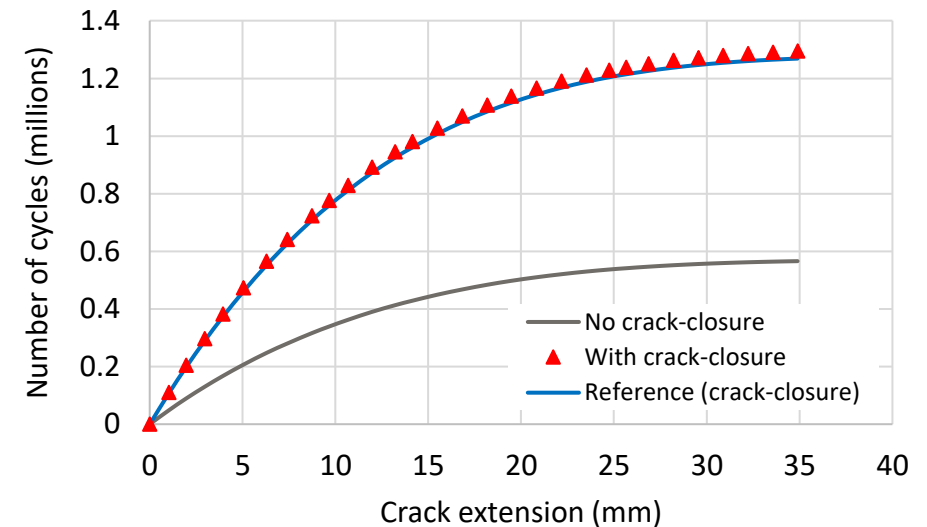
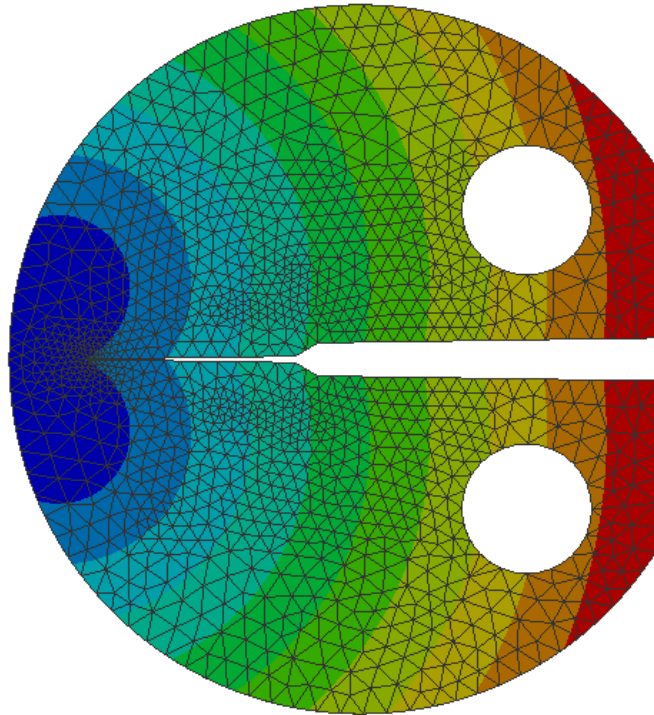
( $A_0=0.6$ ,  $A_1=0.35$ ,  $A_2=0.05$ )

$$\frac{da}{dN} = C(U\Delta K)^m$$

$$U = 0.6 + 0.35R + 0.05R^2$$

$P = 5\text{kN}$

Stress ratio  $R = 0.2$



# Stress-ratio dependent fatigue crack growth law

## SMART – fatigue crack growth

### New field variable - **SRAT** (stress ratio)

- useful to specify stress ratio  $R$  dependent parameters in a crack-growth rate model
- supported with Paris' law (PARIS) and tabular fatigue law (TFDK) options

TB,CGCR,matID,,,PARIS

TBFIELD,SRAT, $R_1$

TBDATA,1, $C_1$ , $m_1$

..

..

TBFIELD,SRAT, $R_n$

TBDATA,1, $C_n$ , $m_n$

TB,CGCR,matID,,,TFDK

TBFIELD,SRAT, $R_1$

TBPT,, $DK_{11}$ , $(da/dN)_{11}$

TBPT,, $DK_{12}$ , $(da/dN)_{12}$

..

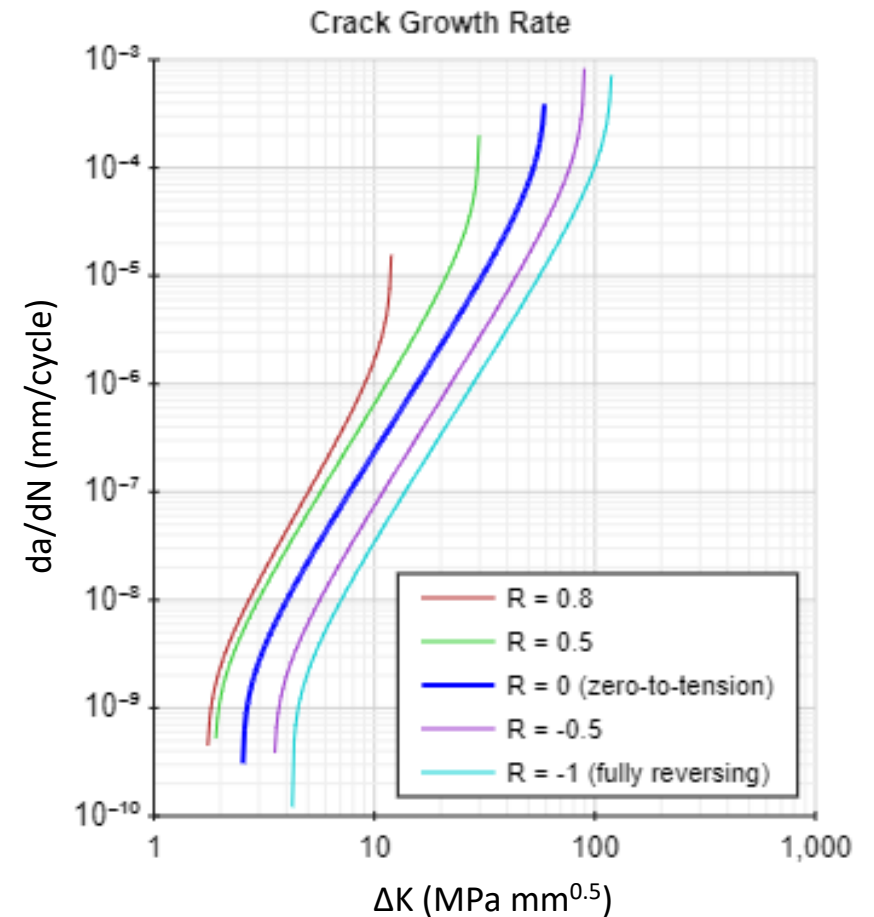
..

TBFIELD,SRAT, $R_n$

TBPT,, $DK_{n1}$ , $(da/dN)_{n1}$

TBPT,, $DK_{n2}$ , $(da/dN)_{n2}$

..

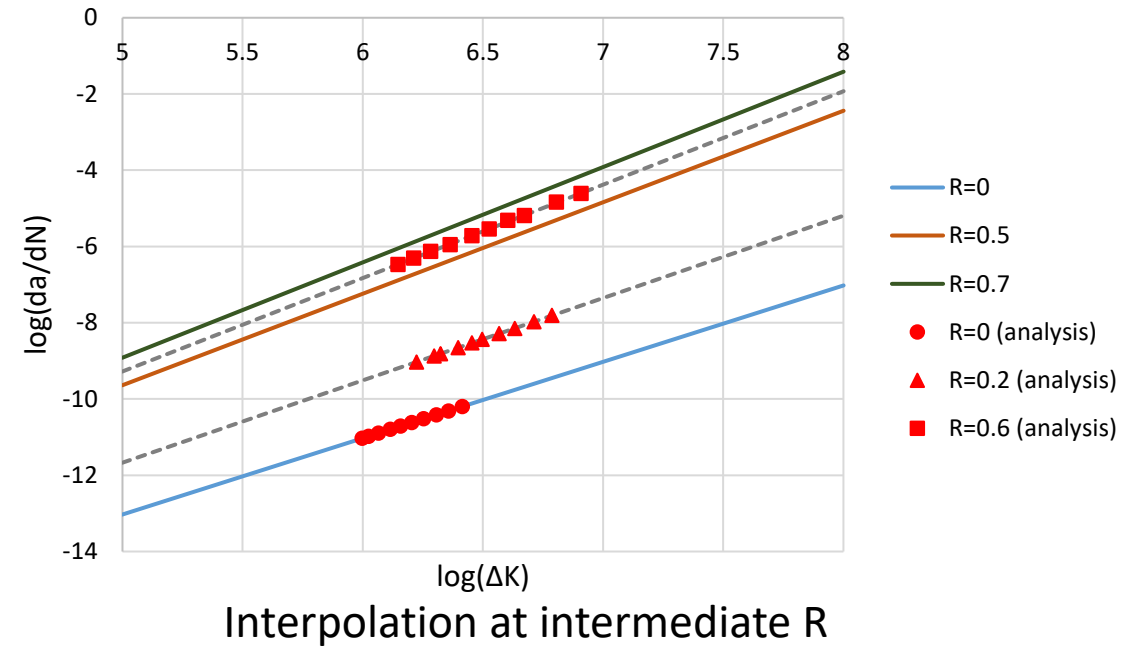
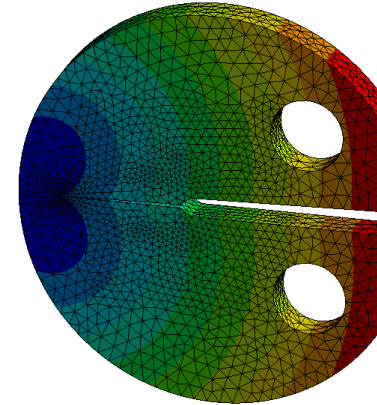
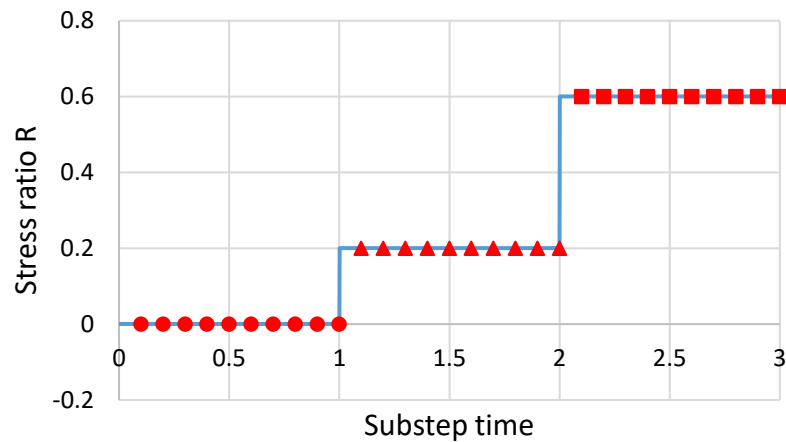


Fatigue crack-growth curves for a typical alloy at different stress ratios ( $R$ )

# Example: Stress-Ratio Dependent Fatigue Crack Growth Law

R dependent Paris' law

R	C	m
0	1E-10	2
0.5	4E-10	2.4
0.7	5E-10	2.5



# Temperature/Time Dependent Fracture Criterion for Static Crack Growth

**KIC** (temp, time)

**JIC** (temp, time)

## SMART – static crack growth

- Fracture criterion
  - KIC – critical stress-intensity-factor
  - JIC – critical J-integral
- 2022 R1 allows users to specify **KIC or JIC as function of temperature and/or time**
- Useful for crack-growth analysis, such as
  - non-uniform temperature field,  $T(x,y,z)$
  - fracture toughness varying with time (e.g., due to corrosion)
  - temperature varying with time,  $T(t)$

```
TB,CGCR,matID,,,KIC  
TBFIELD,TEMP (or TIME),val1  
TBDATA,1,KIC1
```

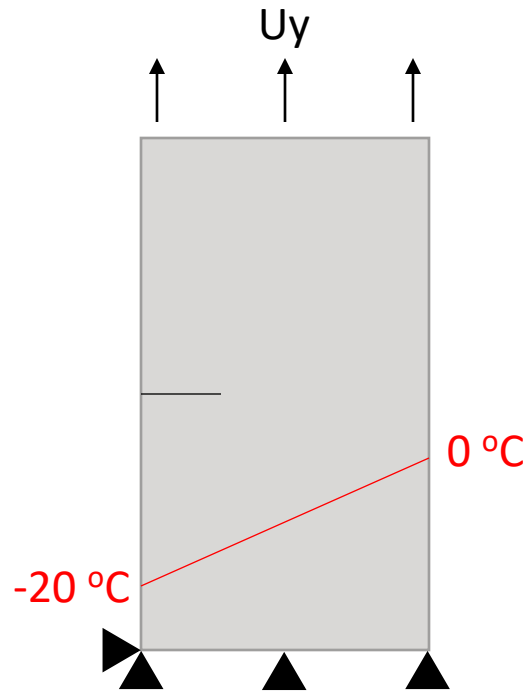
```
..  
TBFIELD,TEMP (or TIME),valn  
TBDATA,1,KICn
```

```
TB,CGCR,matID,,,JIC  
TBFIELD,TEMP (or TIME),val1  
TBDATA,1,JIC1
```

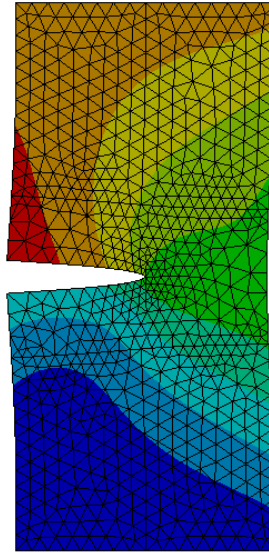
```
..  
TBFIELD,TEMP (or TIME),valn  
TBDATA,1,JICn
```



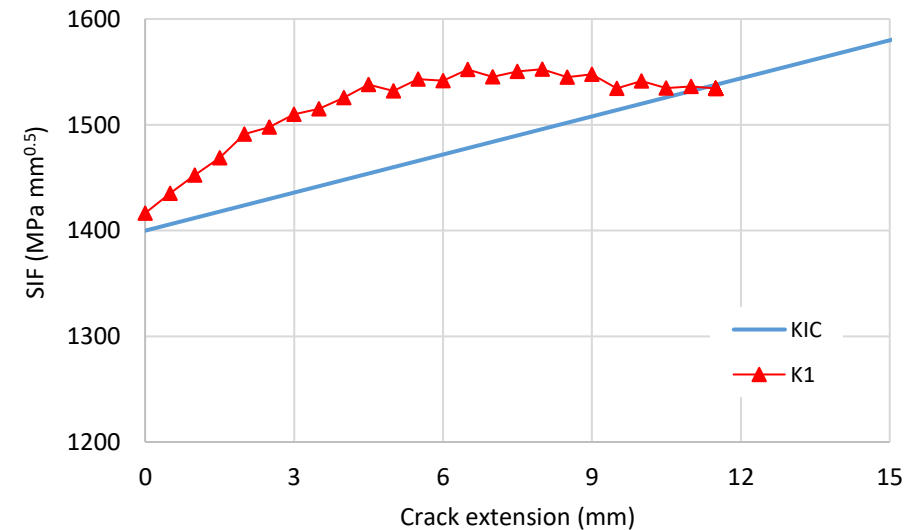
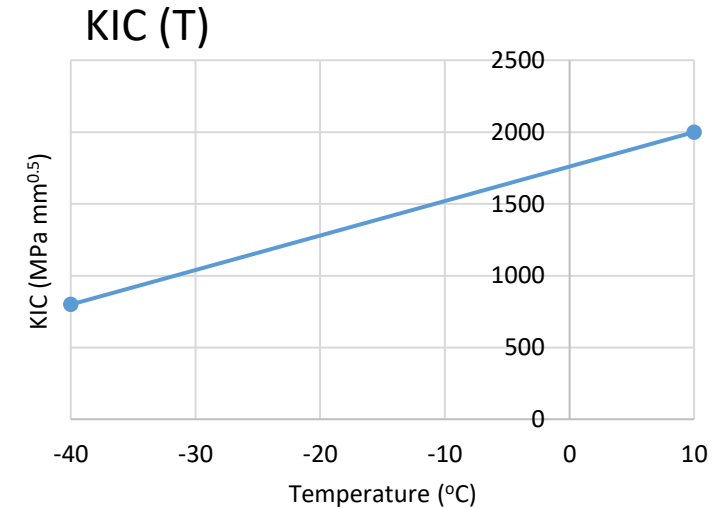
# Example: Temperature-Dependent Fracture Criterion



$a_0 = 10\text{ mm}$   
 $W = 40\text{ mm}$   
 $U_y = 0.075\text{ mm}$



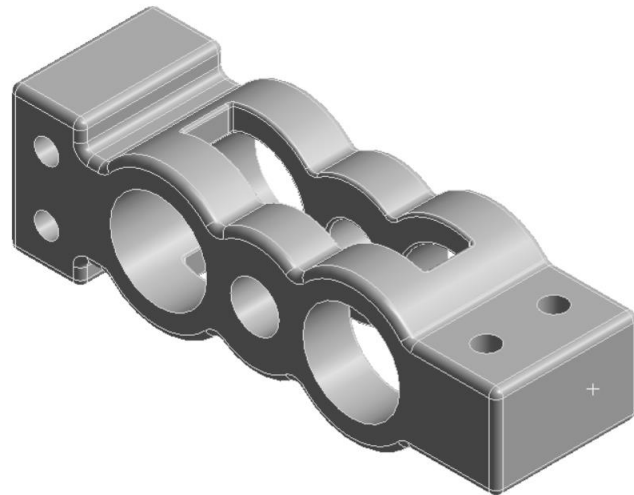
- Spatially-varying temperature field
- Temperature-dependent KIC
- Static crack growth stops when KIC exceeds K1



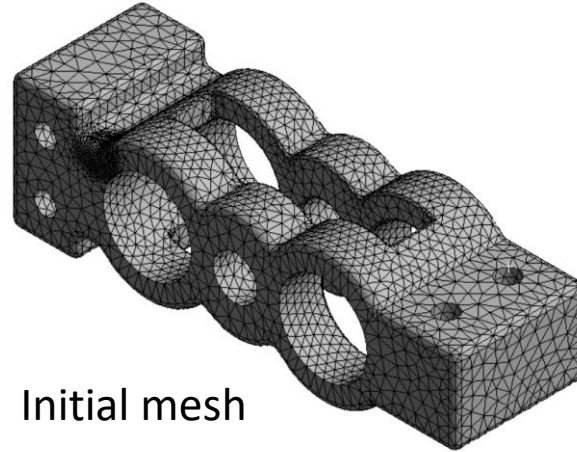
# Adaptive Crack Initiation/Insertion: Bracket Model

Automatic crack initiation based on principal stress criterion

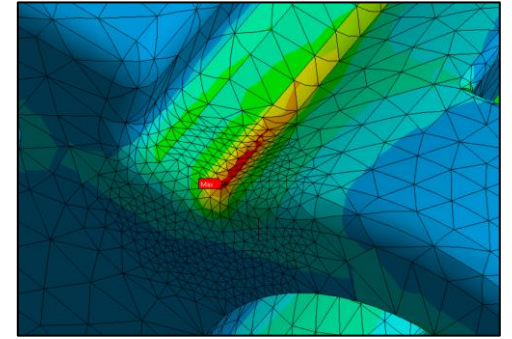
- Predefined elliptical crack will be inserted when the criterion is reached
- Subsequent crack growth follow SMART procedure



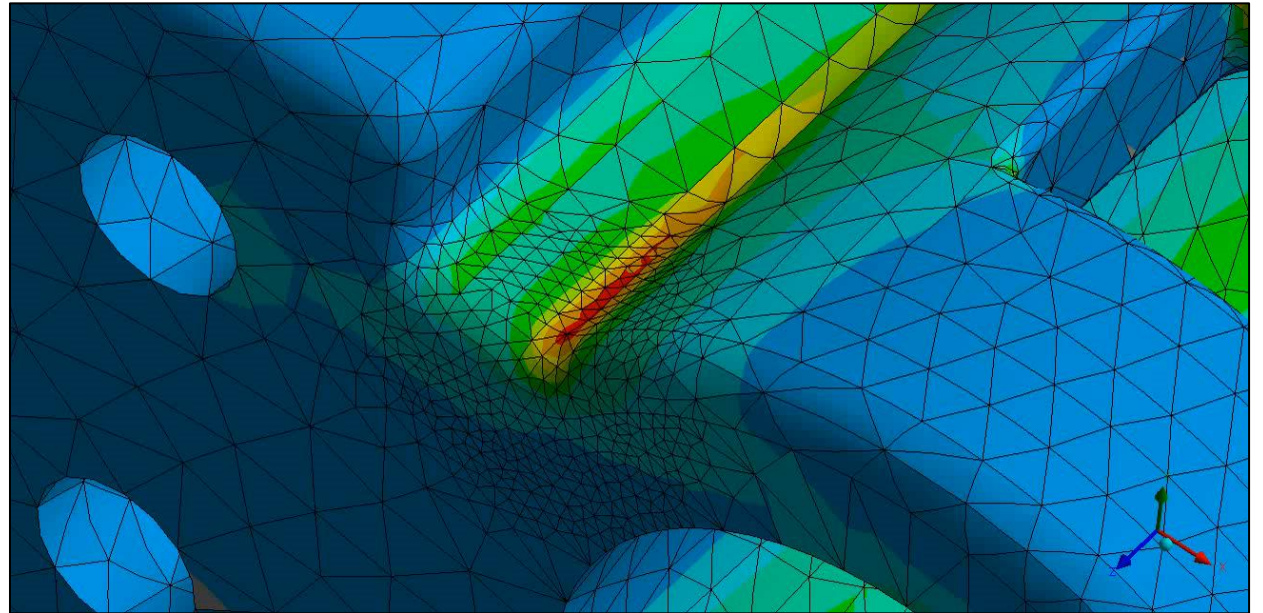
CAD model



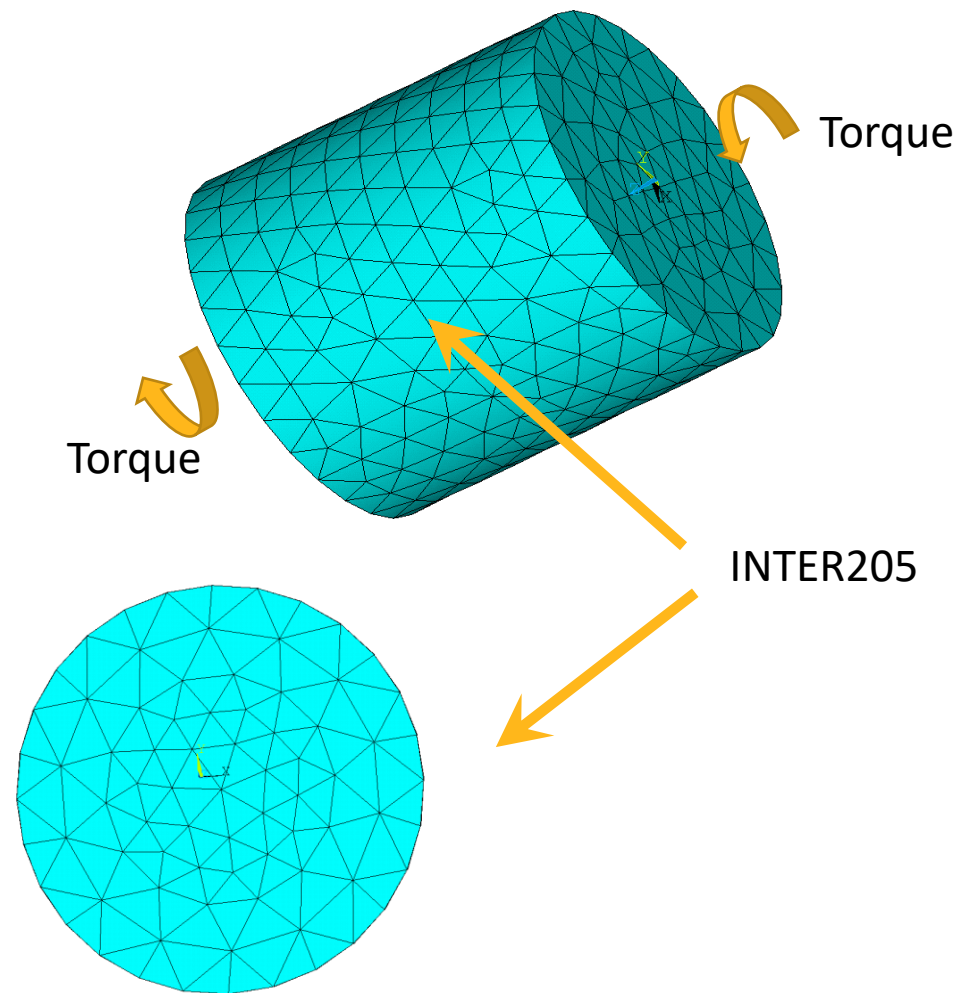
Initial mesh



The high-risk zone

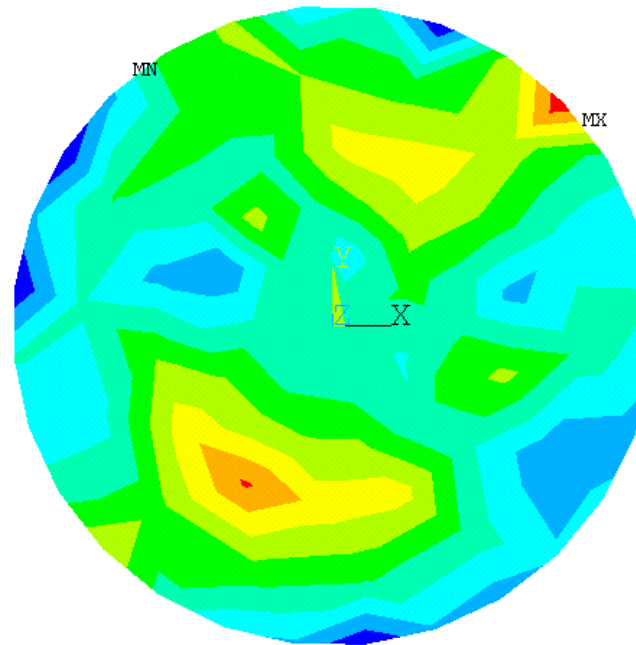


# ESYS for Interface Elements: INTER204 and INTER205

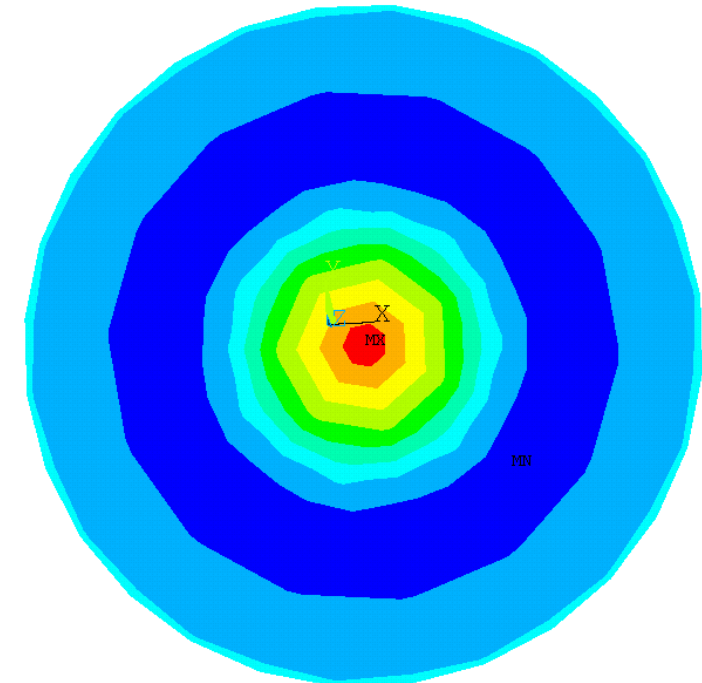


Two cylinders connected by INTER205 elements

Top view of INTER205 elements



(a) without ESYS



(b) with ESYS

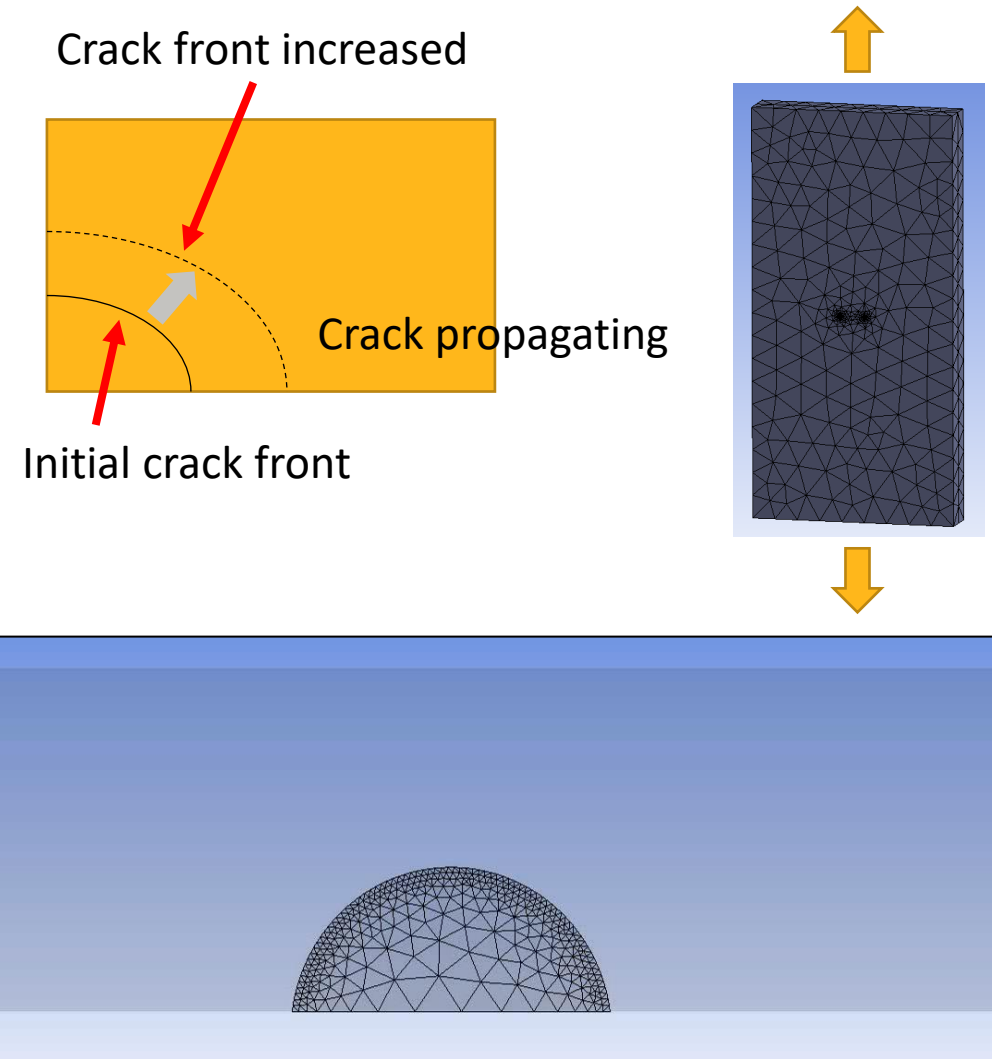
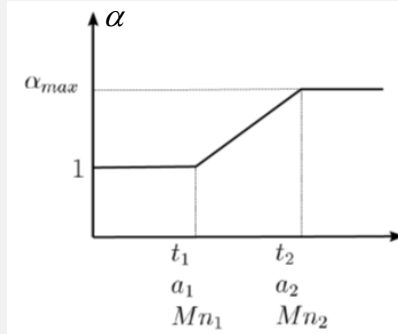
Display of shear traction of INTER205 calculated with and w/o ESYS

# Dynamic Crack Extension Size Control

## SMART crack growth

- Allow to change crack front element size and crack extension size along with crack propagating
- Useful for crack-growth analysis, such as
  - reducing the model size by increasing crack front element size with increased crack front as a result of crack growth
  - increase crack extension increment with crack growing deep into structure components – reduce the solution time to reach the desired crack extension

$$\Delta a = \alpha \cdot Esize_{ref}$$



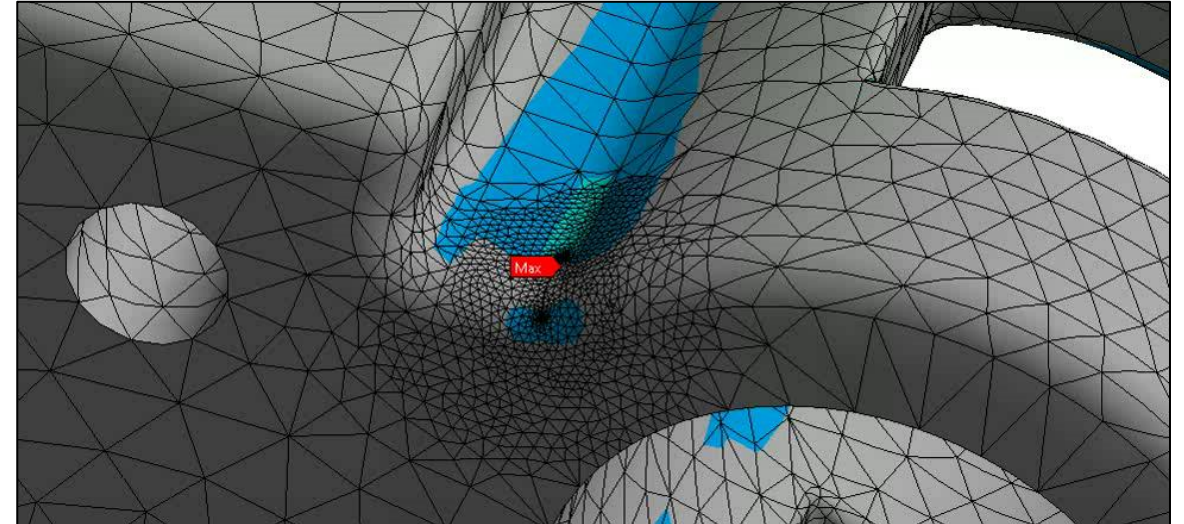
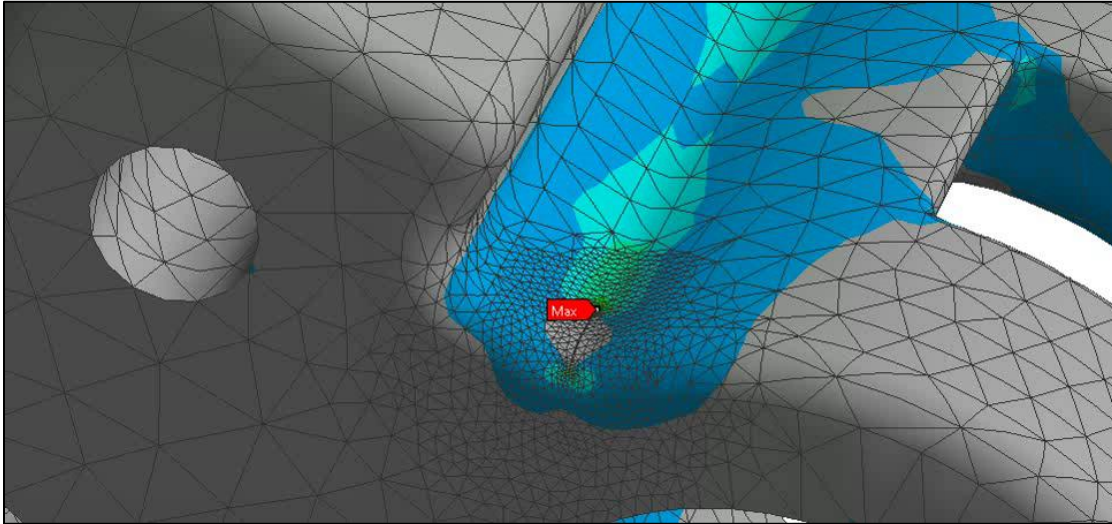


# Dynamic Crack Extension Size Control (Solution Comparison)

40 sub-steps solution  
Total crack increment = 12.87 mm  
Duration of the solution = 1922 sec

VS

40 sub-steps solution  
Total crack increment = 1.52 mm  
Duration of the solution = 4150 sec



With a multiplier factor of 20 after 40 remeshing

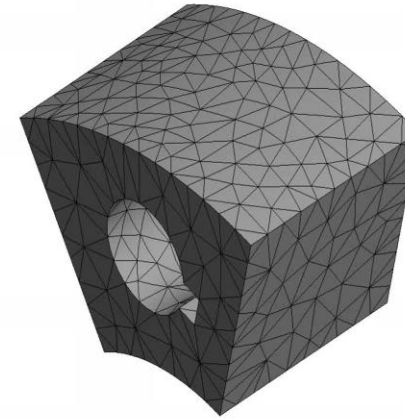
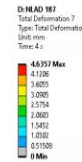
# **MAPDL Nonlinear Adaptivity (NLAD)**



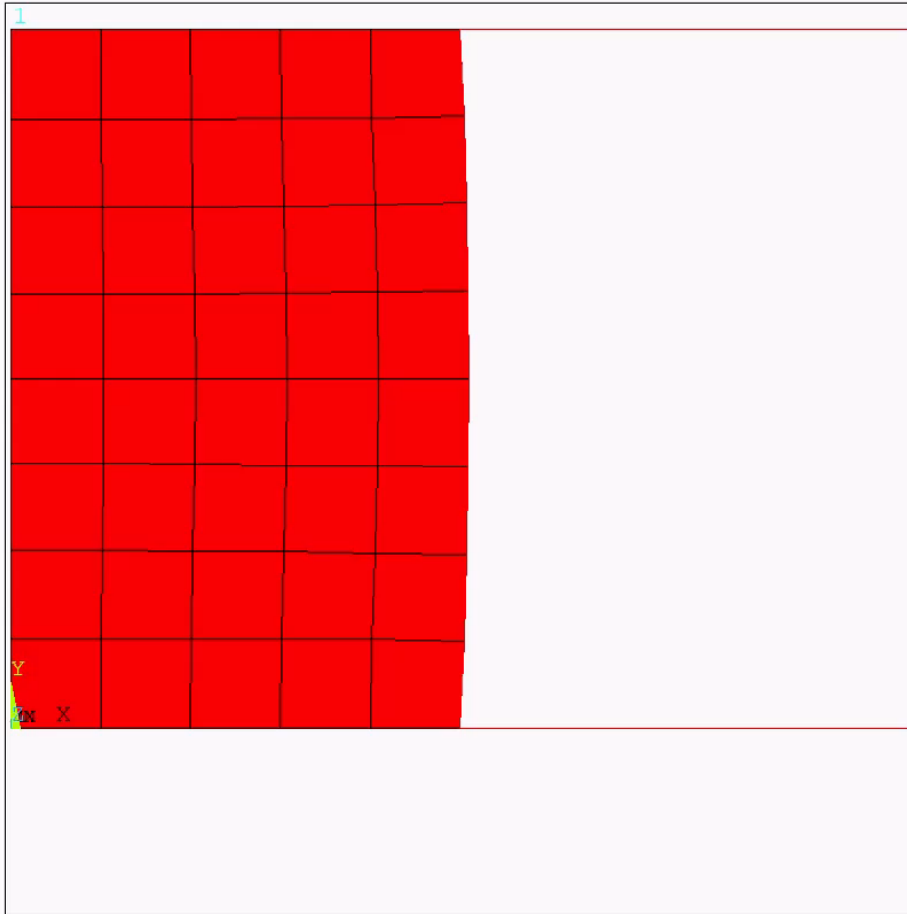


# Loads and Constraints on Initial Mesh with Multiple Load Steps for Nonlinear Adaptivity Analysis

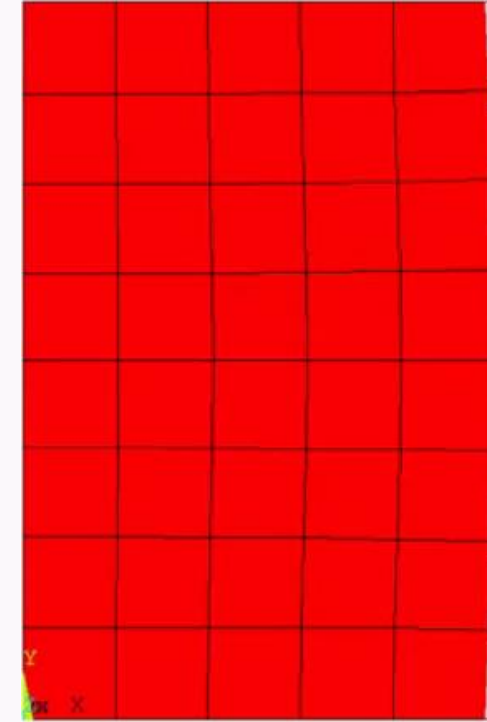
- Motivated by the challenges to apply proper load and boundary conditions on successive mesh configurations in a nonlinear adaptivity framework
- Extended to support pressure load
- Provide a feasible and user-friendly workflow
- Enhanced the capability to handle large deformation nonlinear analysis
  - All supported load and boundary conditions
- Robust and accurate load transfer from initial mesh to current mesh even for large deformations



# Demonstration Example: Initial Mesh-Based Body Temperature with Large Deformation



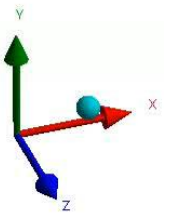
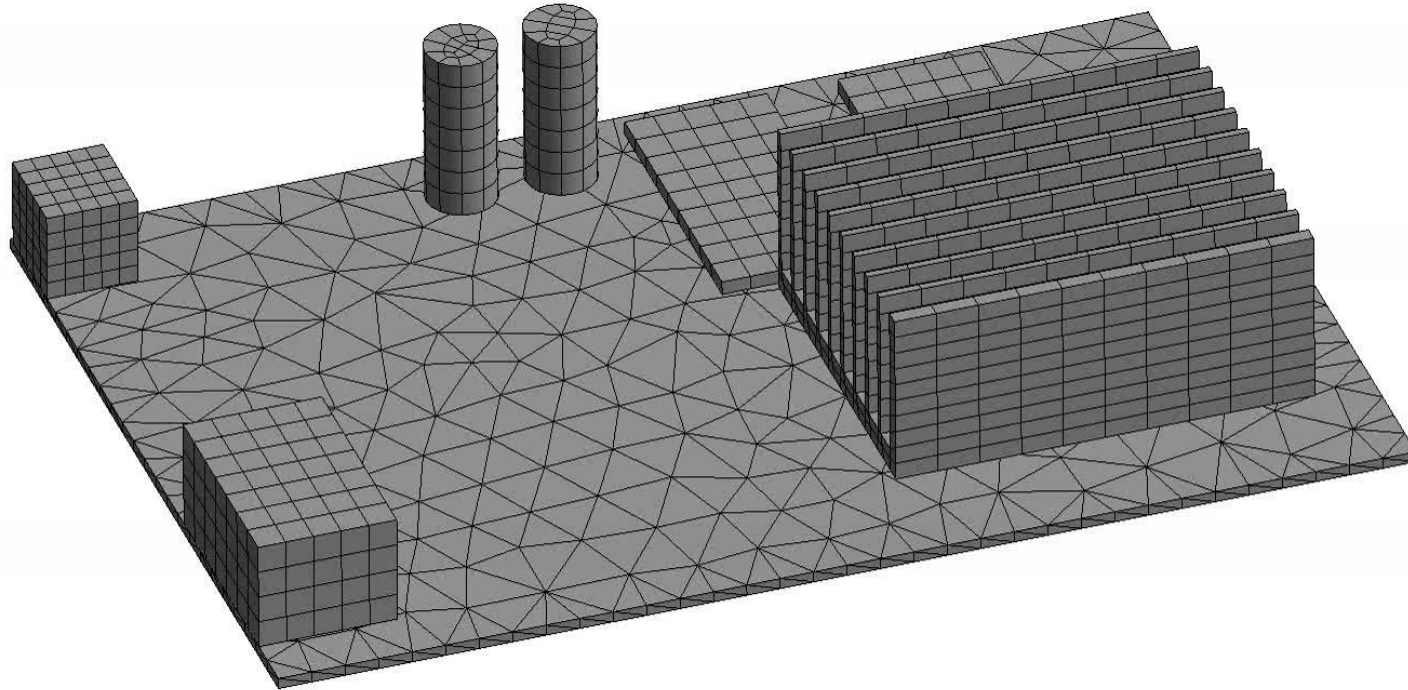
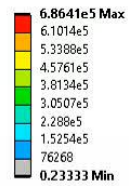
```
ANSYS 2022 R1  
Build 22.1BETA  
ELEMENT SOLUTION  
STEP=1  
SUB =1  
TIME=.041667  
BFETEMP (NOAVG)  
RSYS=0  
PowerGraphics  
EFACET=1  
DMX =.213333  
SMN =301.625  
SMX =301.625
```



Body temperatures are still applied on initial mesh after large deformation and remeshing

# Demonstration Example: Initial Mesh-Based Pressure Loads

E: Direct NLAD  
Equivalent Stress  
Type: Equivalent (von-Mises) Stress  
Unit: Pa  
Time: 4 s

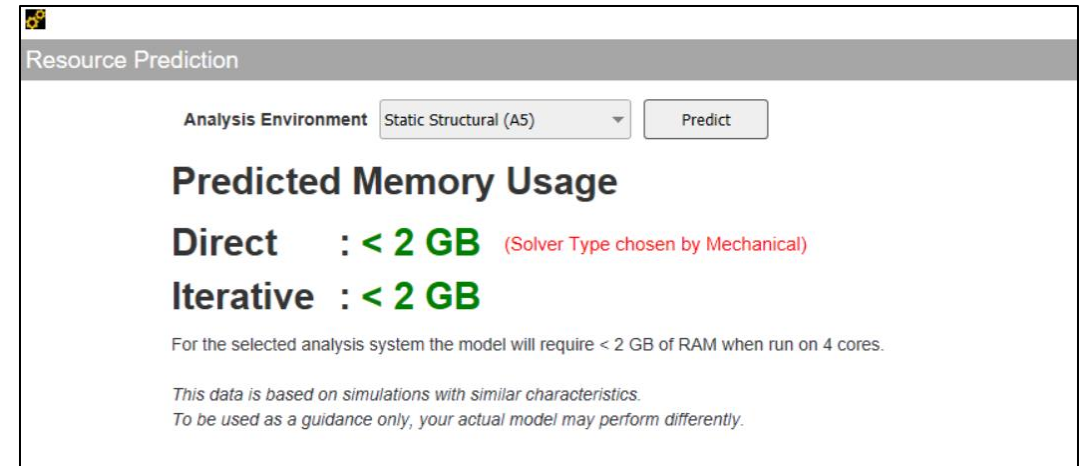
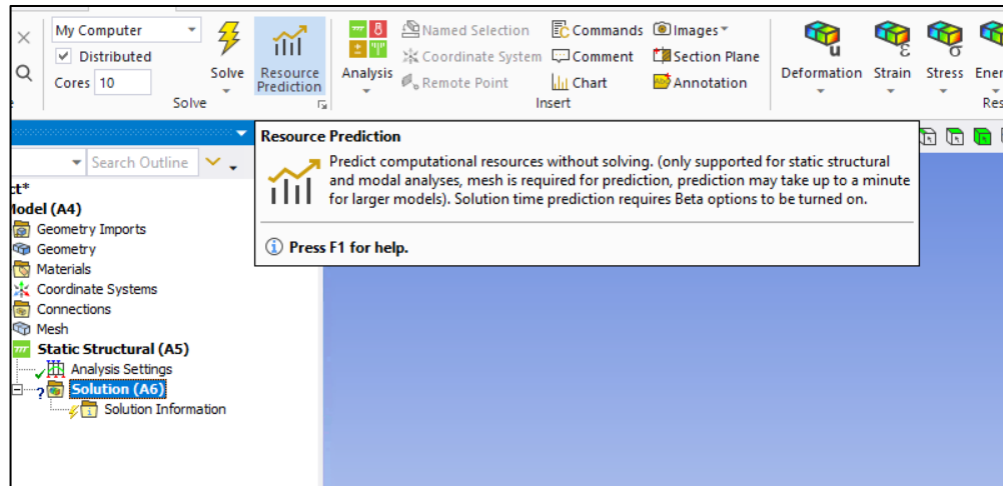


# MAPDL Solver



# Resource Prediction Enhancements

- Switched to new neural network algorithm by default
  - Exposure of resource prediction is only via Mechanical GUI
  - Improved accuracy for memory requirement predictions
  - Reduced installation size



# / Distributed Ansys Enhancements

- Introduction of new form of parallelism - Hybrid
  - Combines DMP and SMP (distributed + shared memory parallelism)
  - Activated via a new command line argument → `-nt <#>`
  - **SMP** → `-smp & -np N` to specify using “N” OpenMP threads
  - **DMP** → `-np N` to specify using “N” MPI processes
  - **Hybrid** → `-np N` to specify using “N” MPI processes  
`-nt M` to specify using “M” OpenMP threads per process during SOLVE command

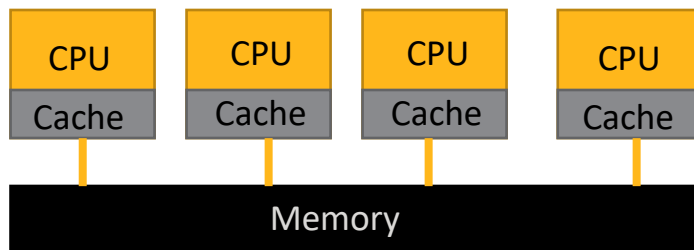
**Total core count** =  $N \times M = P$  cores

*ansys221 -b -np 8 -nt 2 <test.dat> out* will use 8 processes with 2 threads per process  
for a total of 16 CPU cores

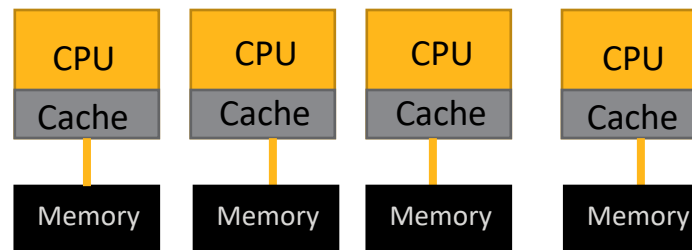


# / Distributed Ansys Enhancements

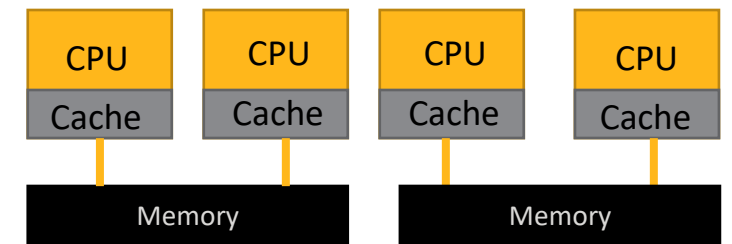
- Hybrid parallel
  - Wide applicability → Works for all features supported in DMP mode
  - Supported with all platforms and MPI libraries
  - Reduces memory requirements → Run larger jobs on your cluster
  - Improves scalability → Utilize more cores in your compute nodes



SMP



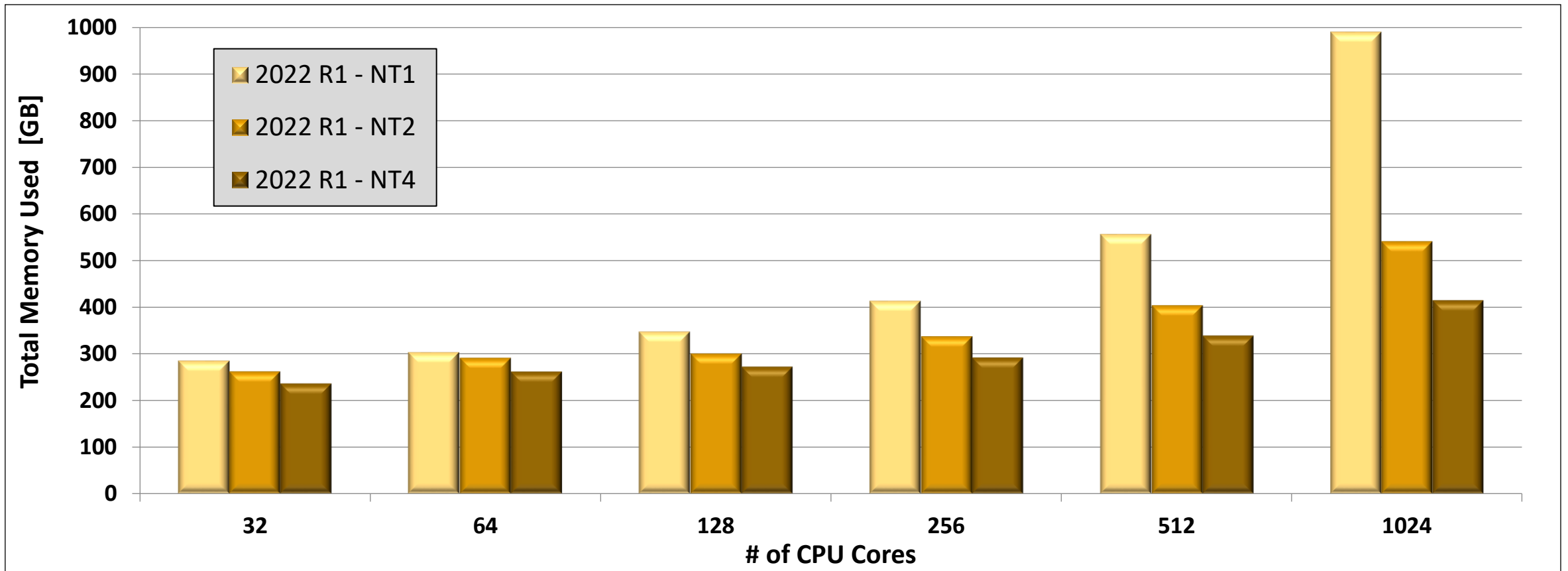
DMP



Hybrid

# / Distributed Ansys Enhancements

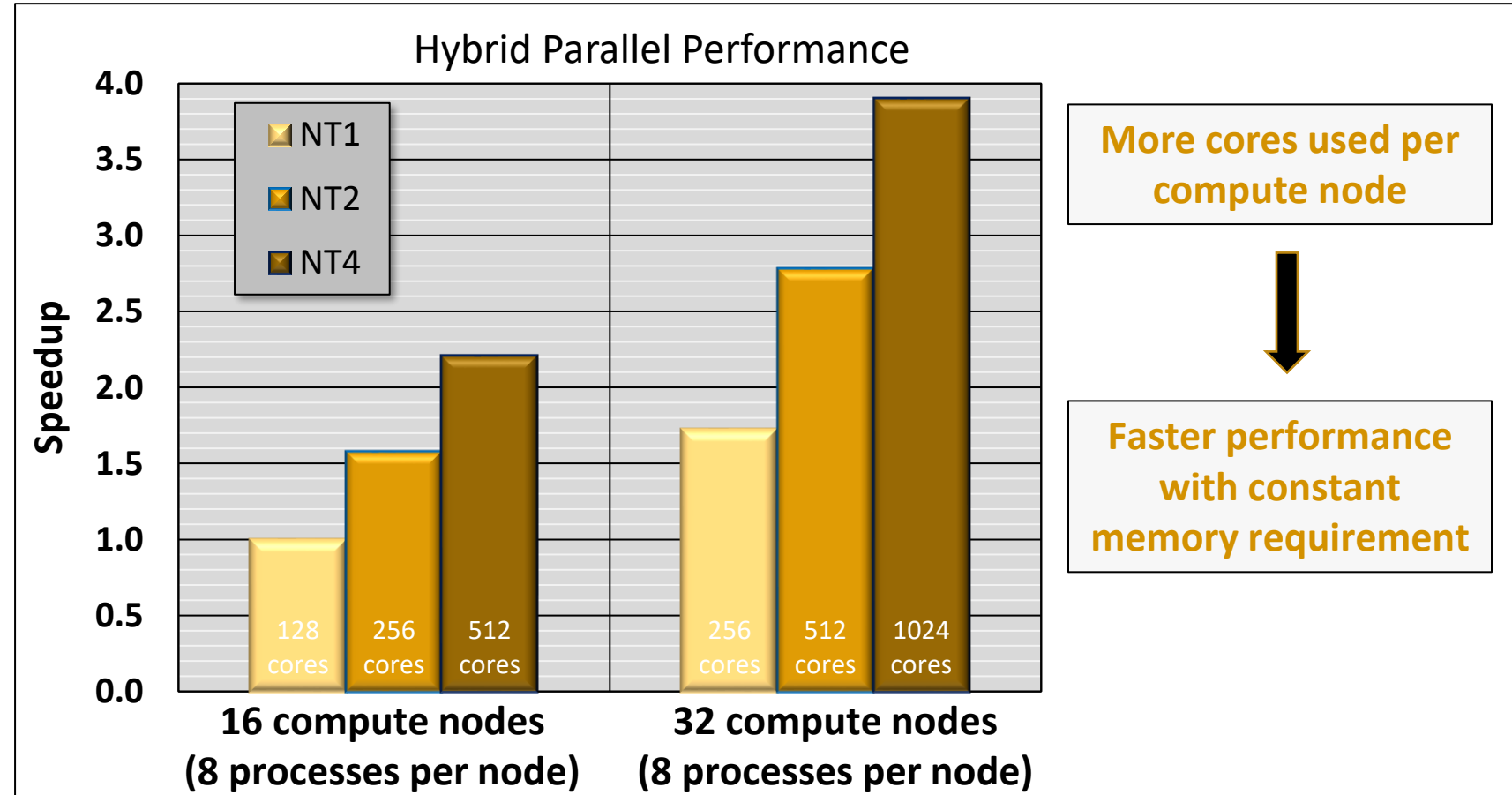
- Hybrid parallel reduces total memory requirements (Engine benchmark)



# / Distributed Ansys Enhancements

- Hybrid parallel → use more cores per compute node (equal RAM use)

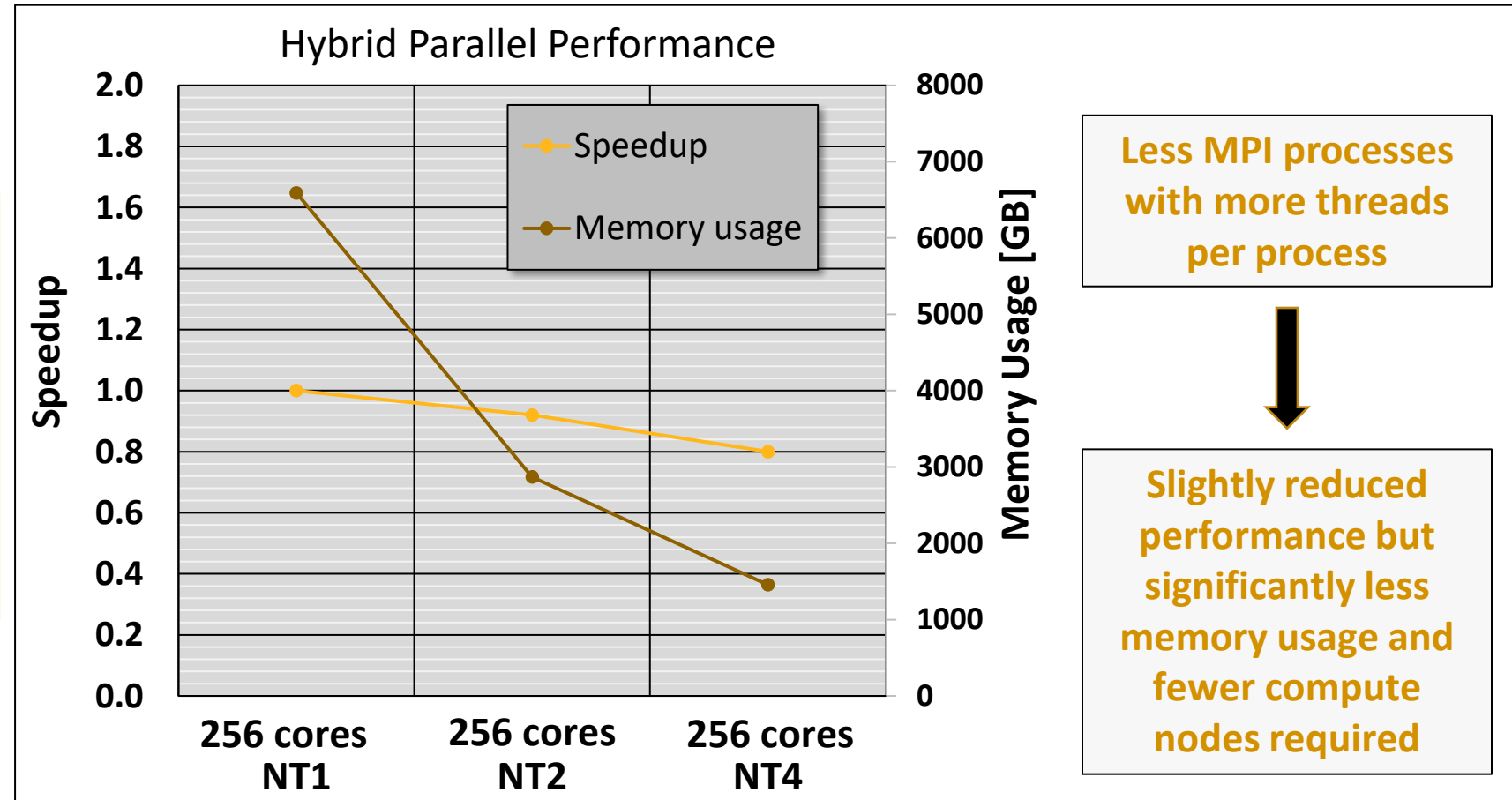
- 88 million DOF; sparse solver
- Nonlinear static analysis involving contact, large deflections, 126 Newton-Raphson iterations to converge
- Linux cluster; each compute node contains 2 Intel Xeon Platinum 8168 processors (44 cores), 346GB RAM, SSD, CentOS 7.9



# / Distributed Ansys Enhancements

- Hybrid parallel → reduce memory and hardware required (same core count)

- 88 million DOF; sparse solver
- Nonlinear static analysis involving contact, large deflections, 126 Newton-Raphson iterations to converge
- Linux cluster; each compute node contains 2 Intel Xeon Platinum 8168 processors (44 cores), 346GB RAM, SSD, CentOS 7.9



# / Distributed Ansys Enhancements

- **MPI library support**

- Upgraded to Intel MPI 2019 Update 10 on Windows
- Intel MPI 2018 Update 3 support is unchanged at this release on Linux (**default**)
- Support has been added for Intel MPI 2019 Update 12 on Linux (-mpi intelmpi2019)
- Open MPI v4.0.5 support on Linux is unchanged at this release
- Microsoft MPI v10.0 support on Windows is unchanged at this release

# Distributed Ansys Enhancements

- **Improved scaling**

- Significantly faster performance and reduced memory & I/O requirements when constraint and/or coupling equations are present in the model
- Assembled matrix (.full) file contains more efficient data storage
- Mostly affects sparse direct solver and related eigensolvers (Block Lanczos, Subspace, etc.)
- Symmetric matrices supported in 2021 R2
- **Unsymmetric matrices supported in 2022 R1**

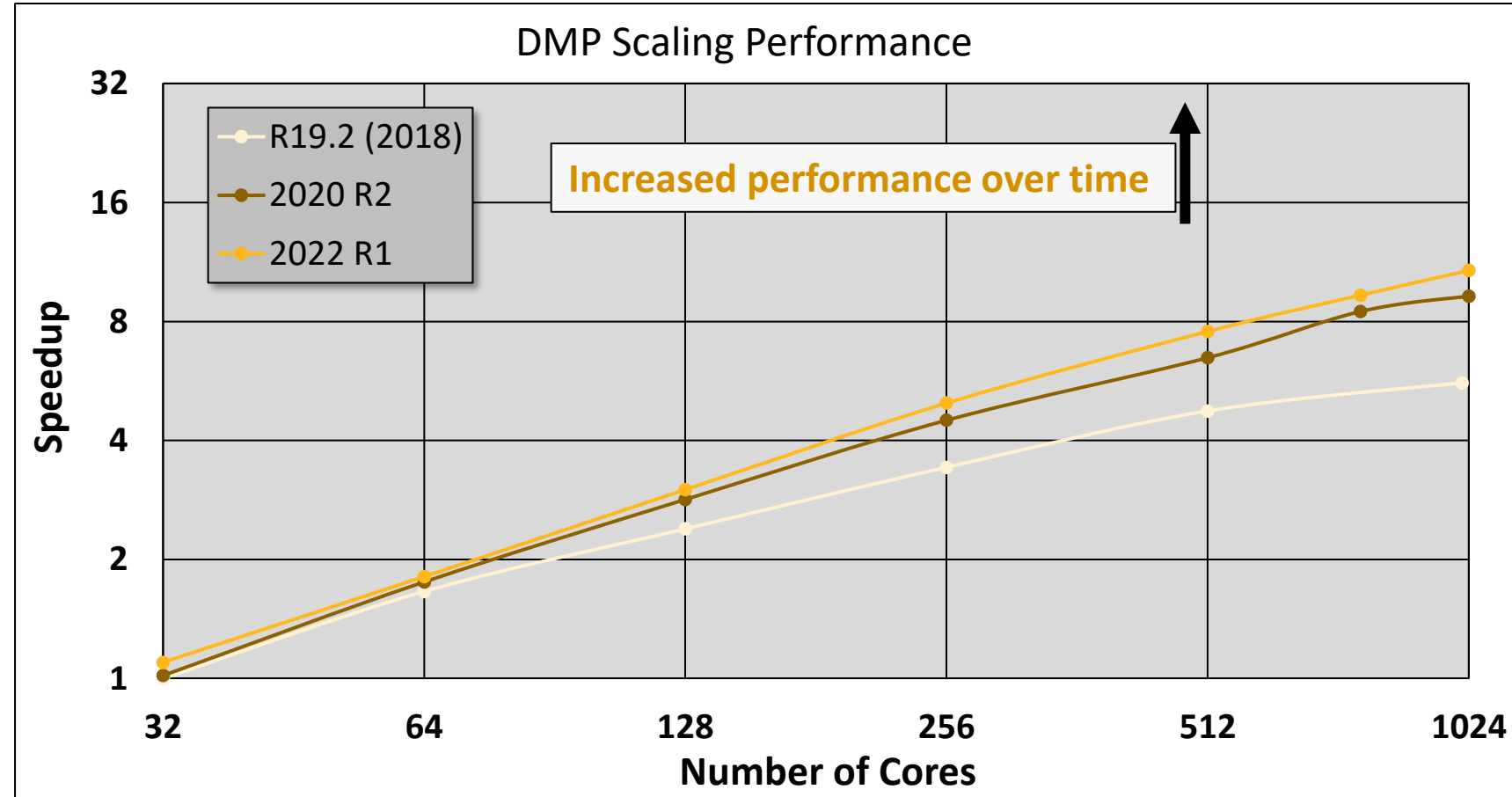


# Distributed Ansys Enhancements

- Improved scaling at higher core counts



- 5.6 million DOF; sparse solver
- Nonlinear static analysis involving contact, constraint equations, unsymmetric matrices
- Linux cluster; each compute node contains 2 Intel Xeon Gold 6148 processors (40 cores), 384GB RAM, SSD, Mellanox Infiniband, CentOS 7.6

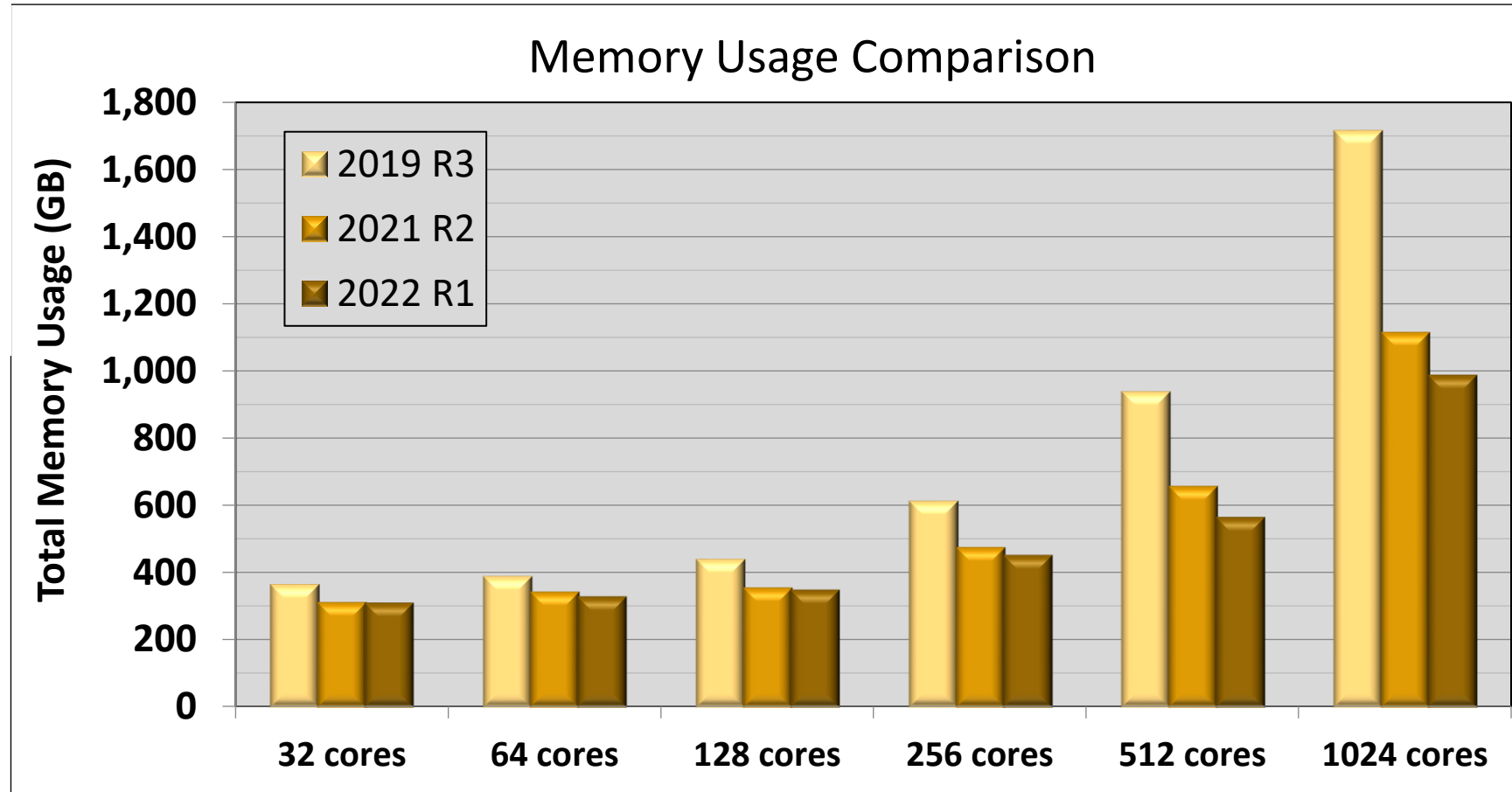


# / Distributed Ansys Enhancements

- Reduced memory usage at higher core counts



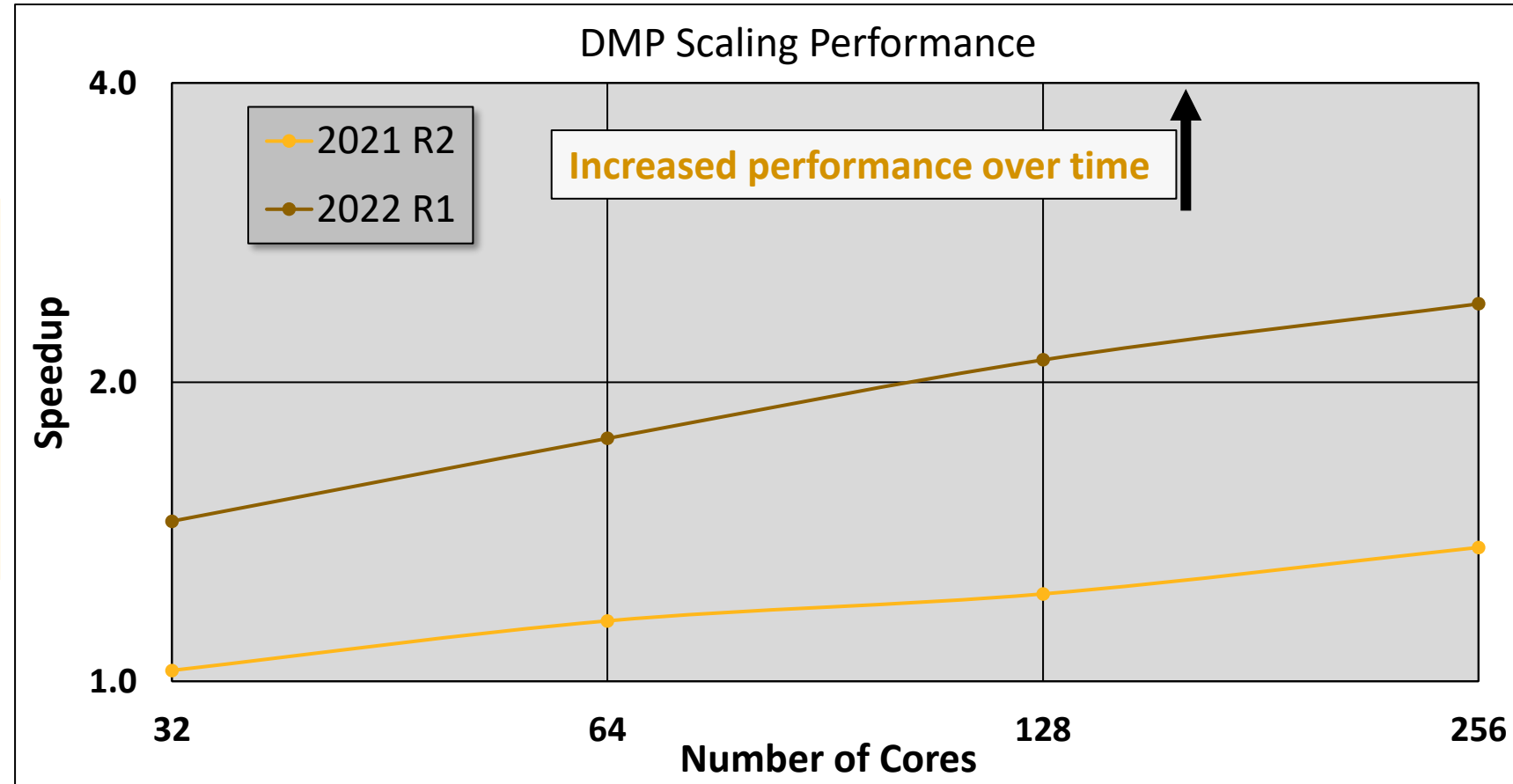
- 5.6 million DOF; sparse solver
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# Distributed Ansys Enhancements

- Improved scaling at higher core counts

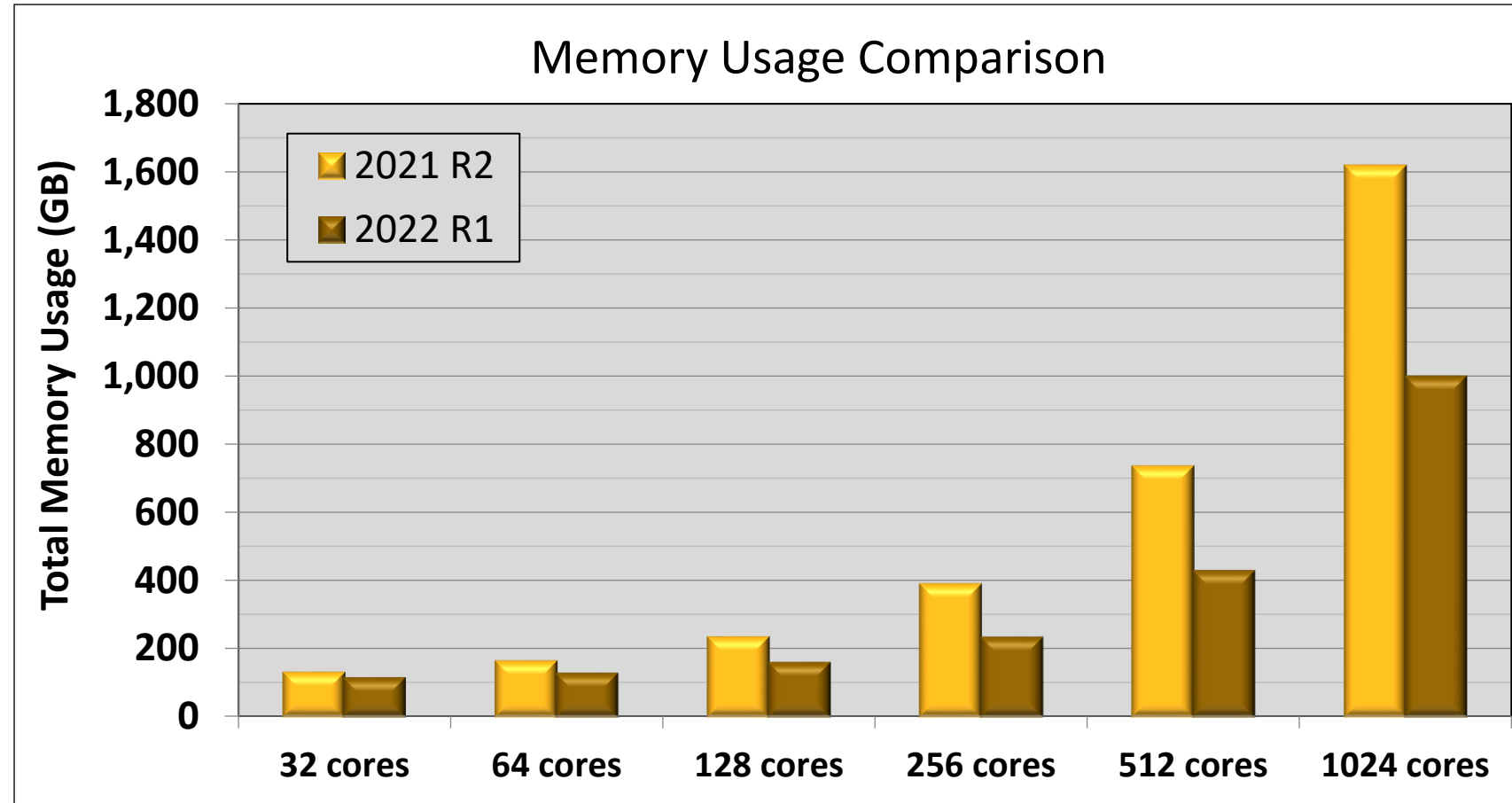
- 3.5 million DOF; UNSYM eigensolver
- Modal analysis requesting 200 mode shapes
- Linux cluster; each compute node contains 2 Intel Xeon Gold 6148 processors (40 cores), 384GB RAM, SSD, Mellanox Infiniband, CentOS 7.6



# / Distributed Ansys Enhancements

- Reduced memory usage at higher core counts

- 3.5 million DOF; UNSYM eigensolver
- Modal analysis requesting 200 mode shapes
- Linux cluster; each compute node contains 2 Intel Xeon Gold 6148 processors (40 cores), 384GB RAM, SSD, Mellanox Infiniband, CentOS 7.6



## Miscellaneous Enhancements

- PCG solver support added for MPC184 joint elements using penalty formulation
- Improved over-constraint detection for user-defined constraint equations

# **Ansys Aqwa**

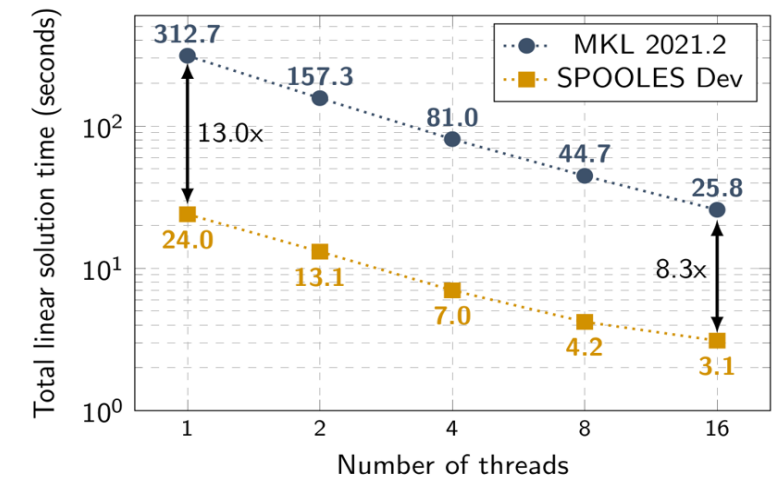
Integrated system for hydrodynamic, mooring  
and structural analysis of marine structures

**Ansys**



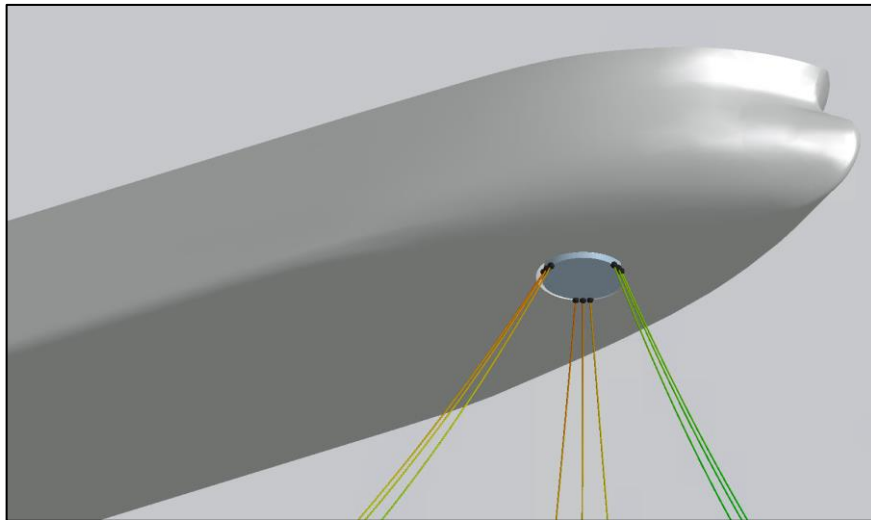
# Block Low-Rank Iterative Solver in Hydrodynamic Diffraction Analysis

- Integrate the Block Low-Rank Generalized Minimal Residual solver of SPOOLES library for solving the large-scale complex full matrix equation for the source strength distribution
- Replaces the Intel MKL solver
- OpenMP parallel calculation, including multithreading to the assembly process
- New solver is faster and requires less memory while keeping the same accuracy (6 digits)
- Integrated in both Windows and Linux Aqwa versions
- For a single ship model with 22000 diffracting elements and 41 wave directions:
  - 10x faster and 5.5x memory reduction for equation solving

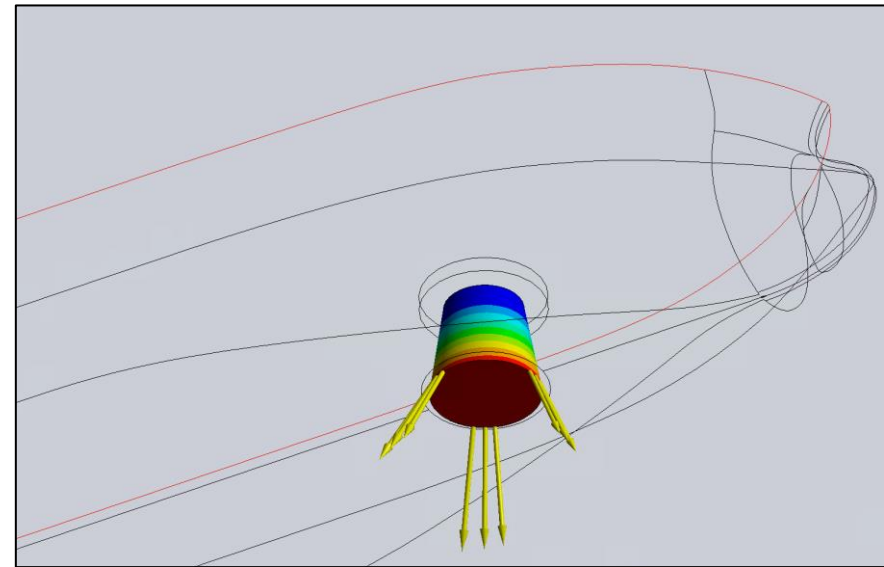
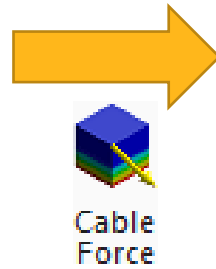


# Time Domain Hydrodynamic Load Mapping to Static Structural for Cables, Tethers and Joints

- Complementary to Hydrodynamic Pressure mapping in Mechanical
- Previously, pressure mapping was for freely-floating structures only
- In Release 2022 R1 we can now transfer instantaneous forces on cable attachments, and forces/moments on tether or joint attachments, as well as pressure on surfaces
- This provides a more complete description of the hydrodynamic loading for FE analysis



Hydrodynamic model



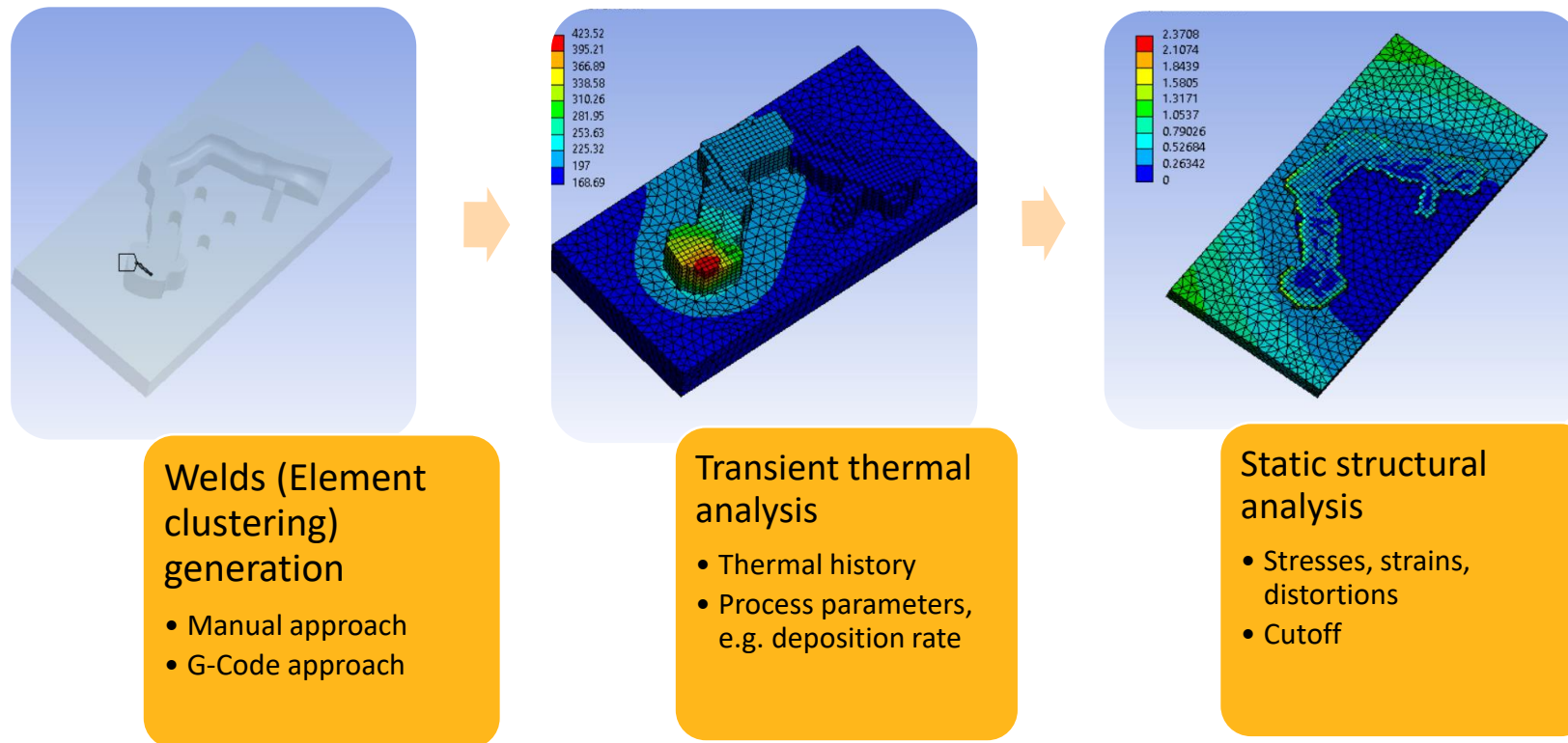
Structural model

# **Workbench Additive**



# / Directed Energy Deposition (DED) Simulation: Full Release

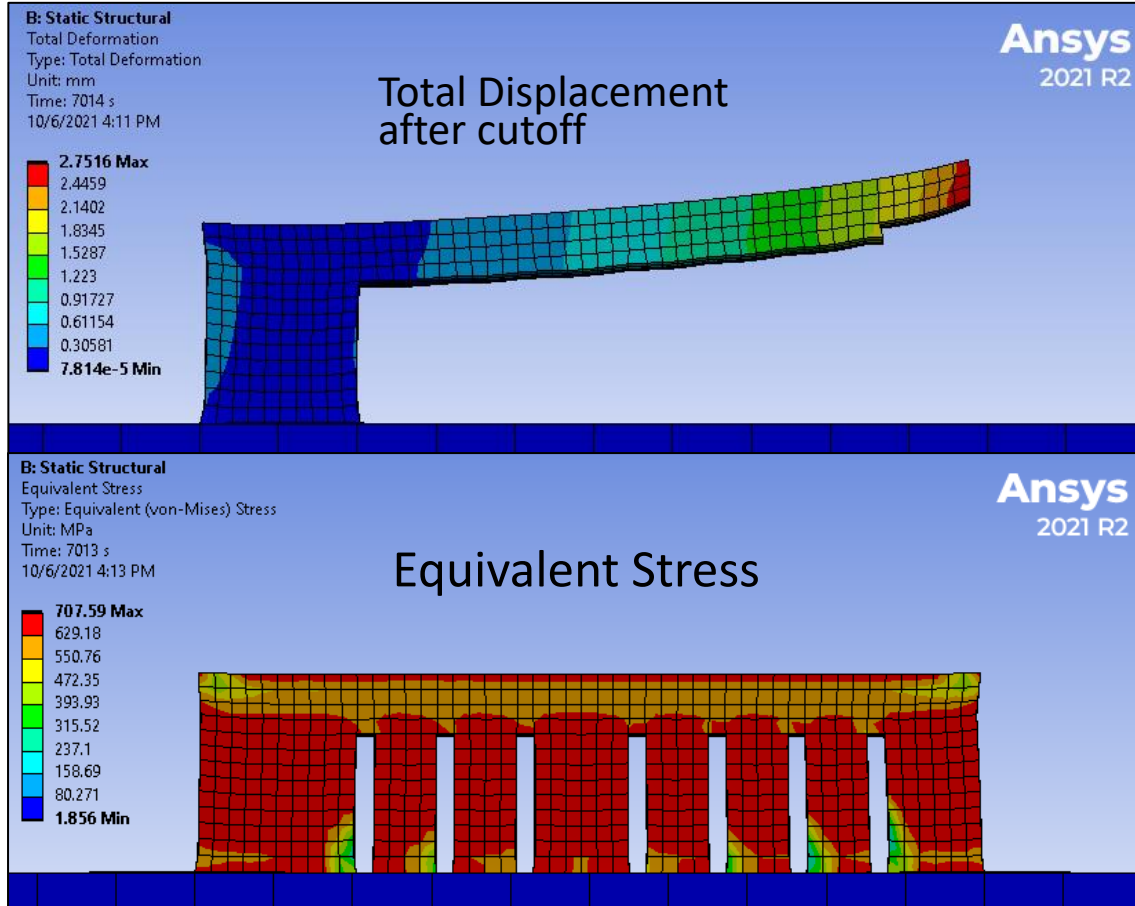
- Process simulation in Mechanical Workbench for Directed energy process (DED)
- Predict the macro-level temperature-level distortions and stresses in parts to prevent build failures and provide trend data for improving designs for additive manufacturing including part orientation and part build order.



# Improved Simplified Heat Treatment (Relaxation Temperature)

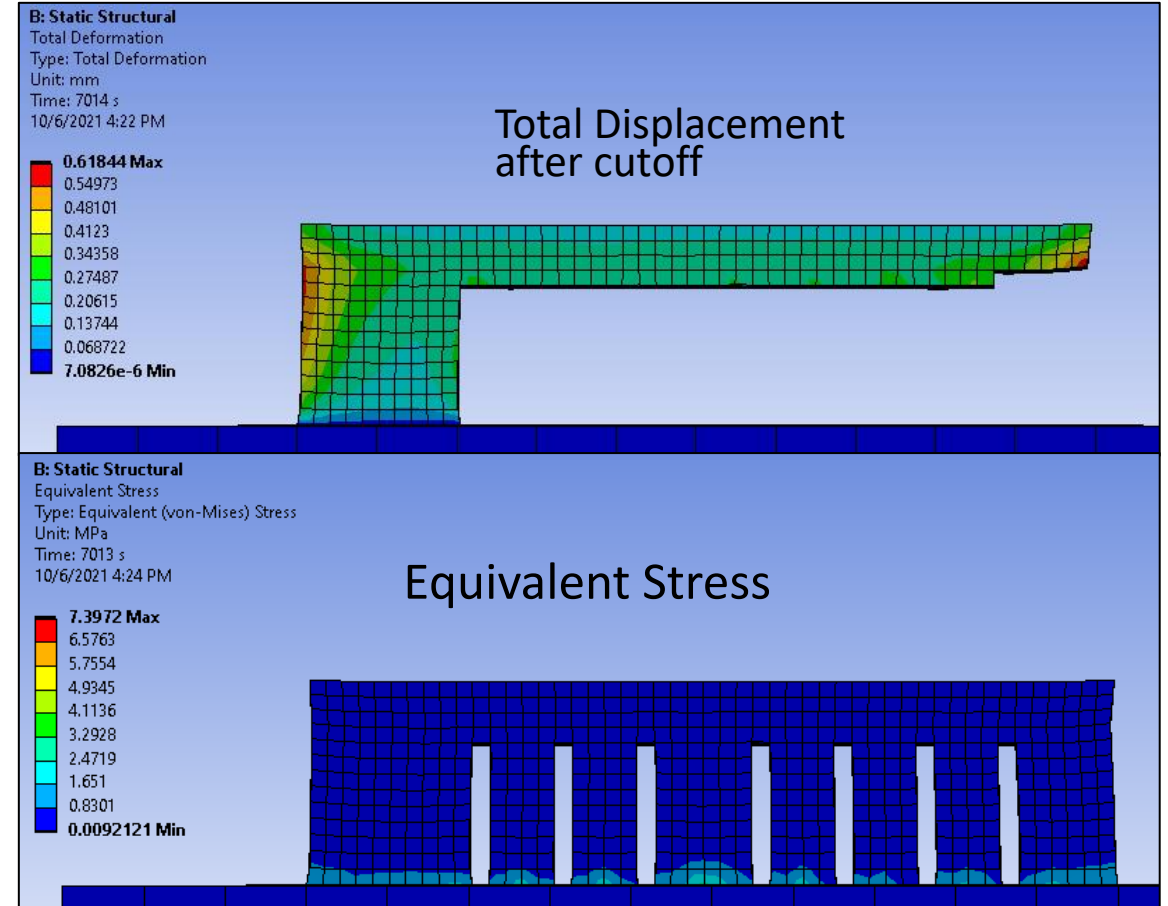
- Zero out only plastic strain
- No heat treatment for base plate

2021 R2



- Zero out elastic strain, plastic strain, stress
- Heat treatment for base plate

2022 R1



# / Import of Additive Supports from CAD

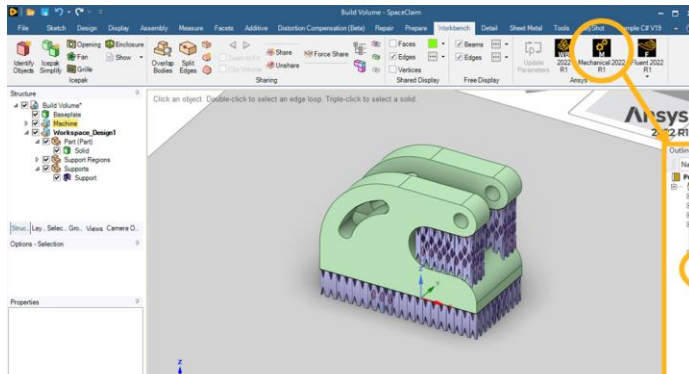
- Supports created using the Additive Prep SpaceClaim add-in are now automatically imported into Mechanical
- This removes the need to export supports and manually scope to the files in Mechanical
- This also simplifies the workflow when transitioning to Mechanical from Additive Prep to perform an AM simulation



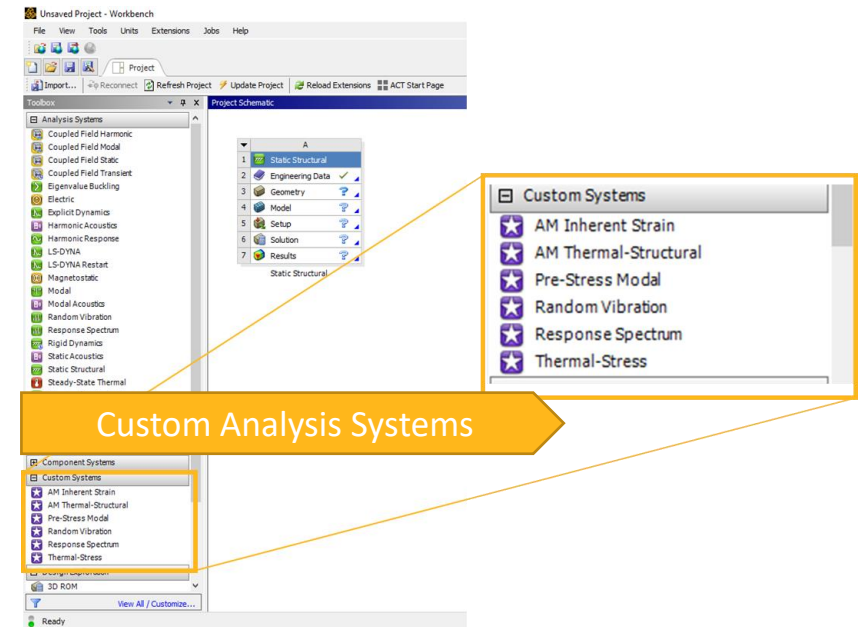
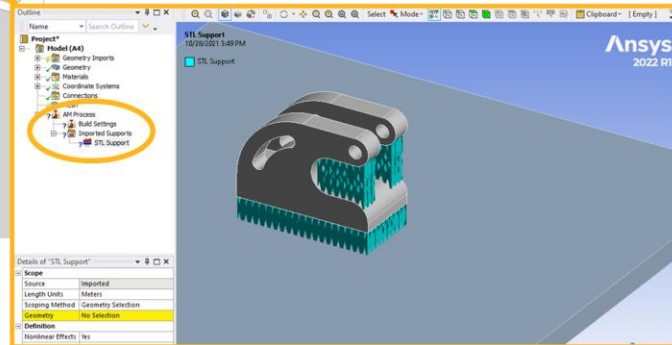


# 2022 R1 Additive Enhancements

- Smoother transfer of supports from Additive Prep to Workbench/Mechanical
- New Material!
  - 17-4PH in Additive Science (Stainless Steel)
- Custom Additive Manufacturing (AM) Analysis Systems
  - Custom systems automatically load AM sample materials and adds AM Process object in Mechanical
- Parallelization: 2D Microstructure solver in Additive Science
  - Fully threaded all the core component of the solver
- Improvements to MAPDL



Smooth Transfer to Mechanical

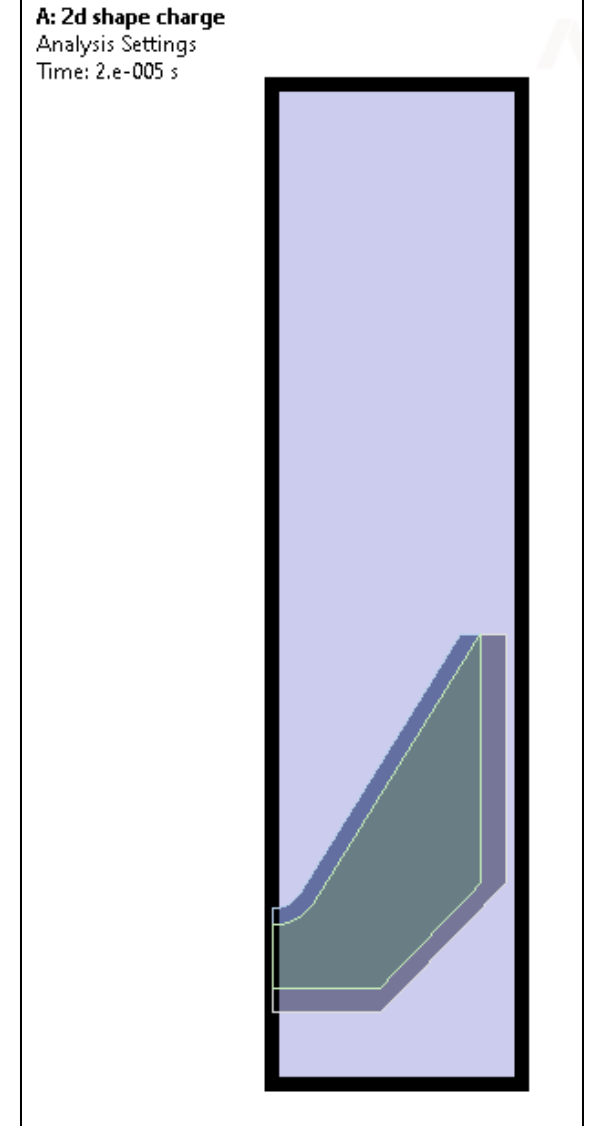
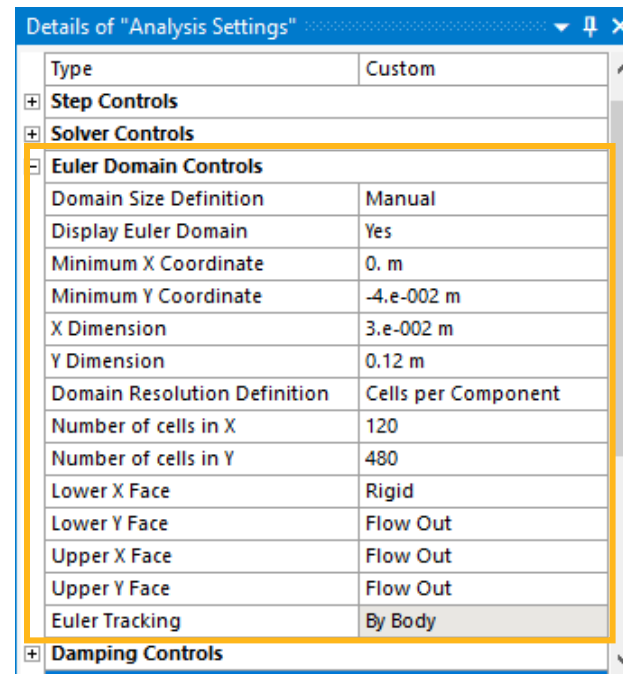
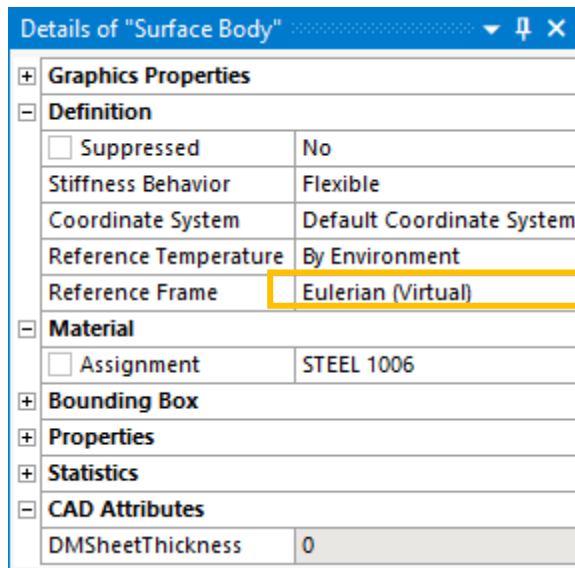


# Explicit Dynamics



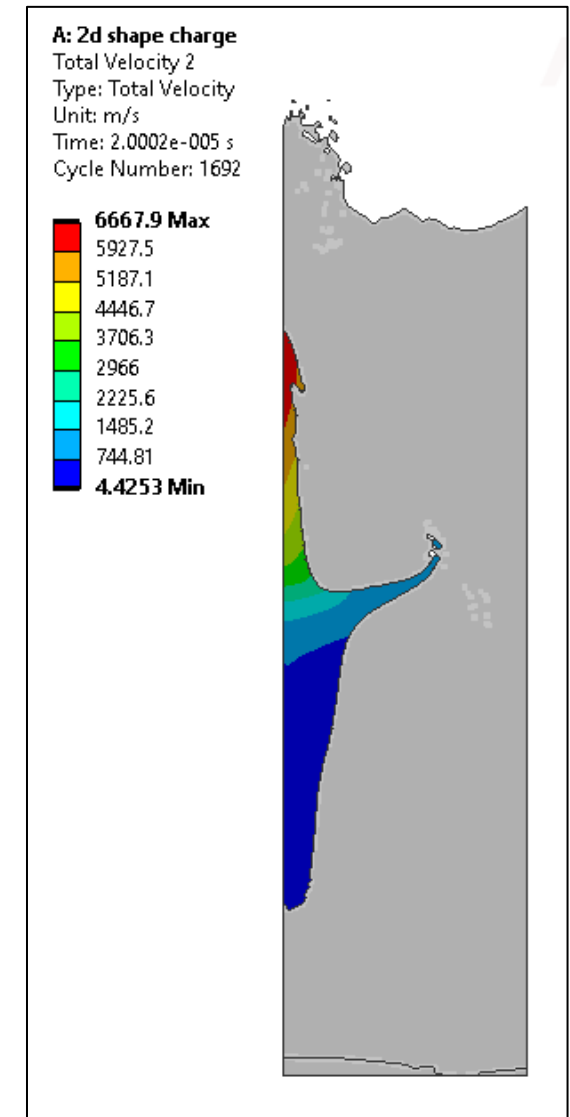
# Multi-Material Euler Exposed in Mechanical for Explicit Dynamics

- A virtual Multi-Material Eulerian (MME) domain is created in any 2d analysis if any of the bodies have the reference frame set to Eulerian (Virtual)
- Euler-Lagrange coupling is enabled automatically



# / Multi-Material Euler Exposed in Mechanical for Explicit Dynamics

- Enables the simulation of extreme deformation events such as fluid or gas flow without suffering from high mesh distortion or tangling which may occur when using a Lagrangian mesh
- Allows for significant speed-up of solution times for analyses that can be considered as 2d axisymmetric or 2d plane strain compared to an equivalent 3d MME analysis



# **Workbench LS-DYNA**

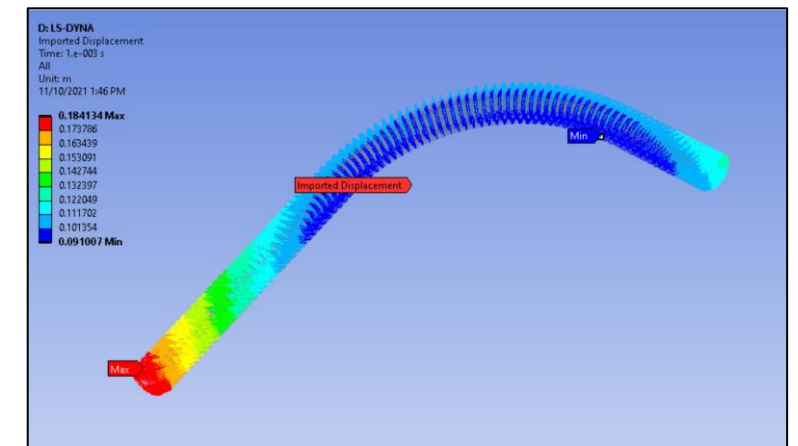
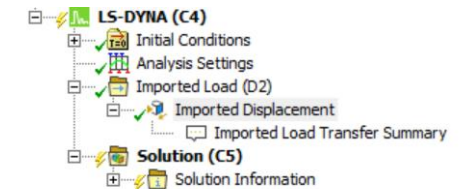
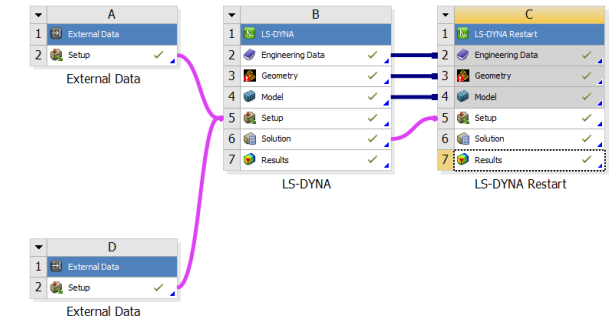


# Imported Displacement

- LS-DYNA now supports Imported Displacement, allowing you to setup the model in non-trivial ways
- The mesh can either be initialized towards a final mesh, allowing the solver to prestress the model, or it can be used to specify the reference geometry in foams applications, if the user has the deformed mesh instead
- The solver will automatically calculate the stresses at the beginning of the calculation
- Create an external data system and import a file containing displacements defined by coordinates or mesh reference
- Link to an LS-DYNA system in the project schematic
- The Imported Load folder is automatically added and Imported Displacement is available for selection in the context menu along with the previously supported Imported Pressure
- Includes the standard options for mapping and display

Graphics Controls	
By	Active Row
Active Row	1
Component	All
Display Source Points	Off
Display Source Point Ids	Off
Settings	
Mapping Control	Manual
Mapping	Profile Preserving
Weighting	Direct Assignment
Legend Controls	
Legend Range	Program Controlled
Named Selection Creation	
Unmapped Nodes	Off
Mapped Nodes	Off

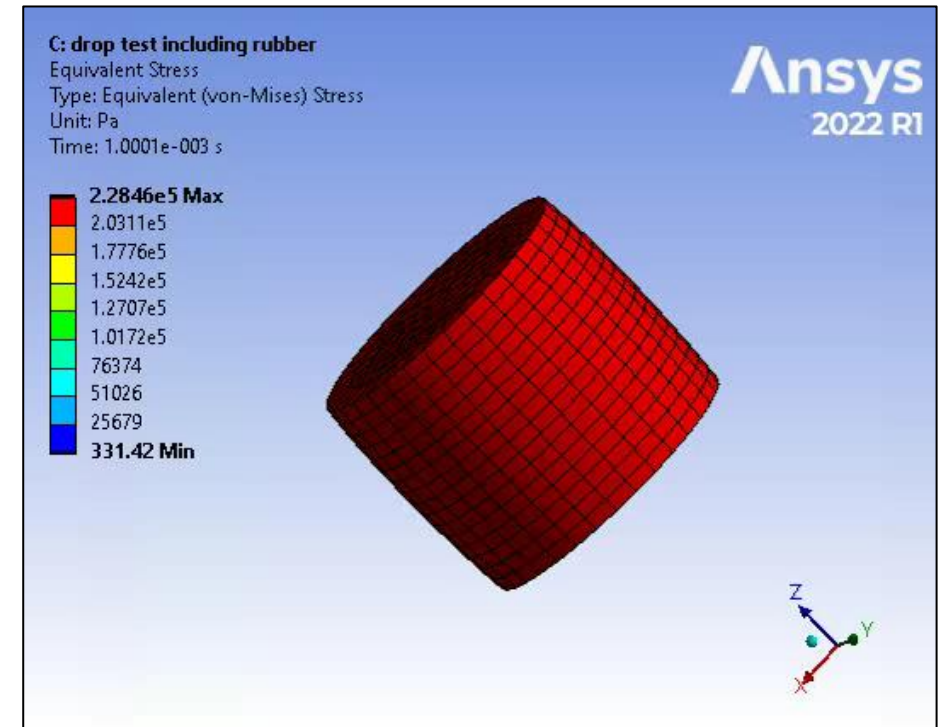
	A	B	C	D	E
1	Column	Data Type	Data Unit	Data Identifier	Combined Identifier
2	A	X Coordinate	m		File1
3	B	Y Coordinate	m		File1
4	C	Z Coordinate	m		File1
5	D	Displacement	mm	Displacement1	File1:Displacement1





# Imported Displacement

- Two options are available:
  - Boundary Prescribed Final Geometry** allows to move the scoped nodes from their initial positions to the final geometry along a straight-line trajectory. The geometry can be prestressed in a nontrivial way.
  - Loading can be applied in a series of Stepped or Ramped sections
  - A single displacement file is allowed for each component in the load table
  - Initial Foam Reference Geometry**, which is only valid when used with foams materials. The mesh in Mechanical is the deformed geometry. The external file contains the coordinates of the reference (undeformed) geometry. The stresses are initialized by the solver
  - A single row is allowed in the load table
  - Tabular Loading is not applicable in this case. The solver determines the rate at which the displacement is applied
  - Reference field in material is automatically selected when the Imported Displacement is scoped to the relevant body. When using a material defined using a command snippet the reference field is not modified

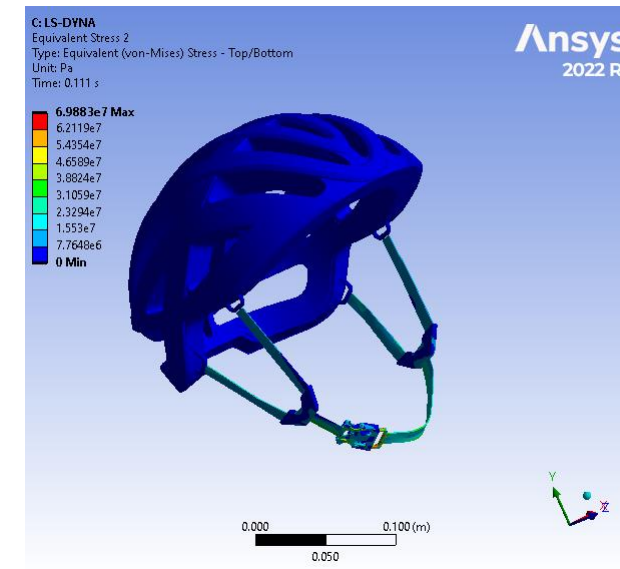
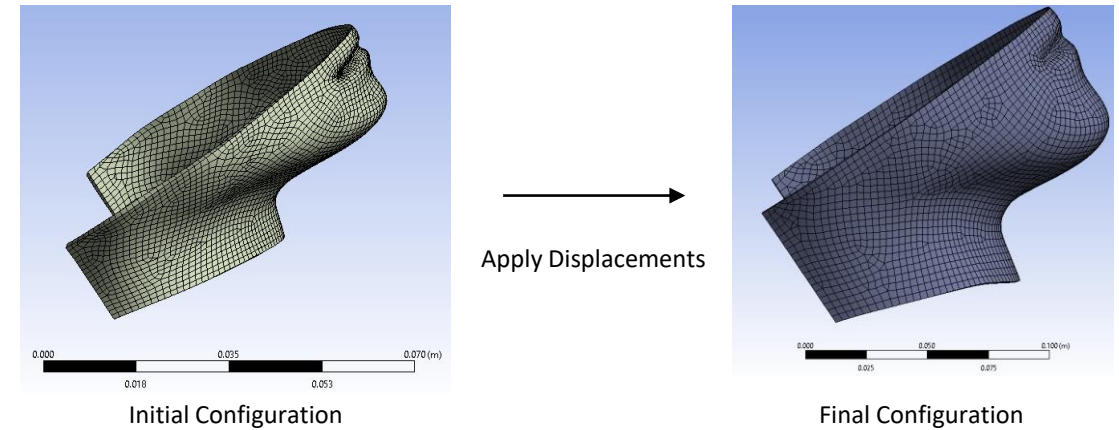
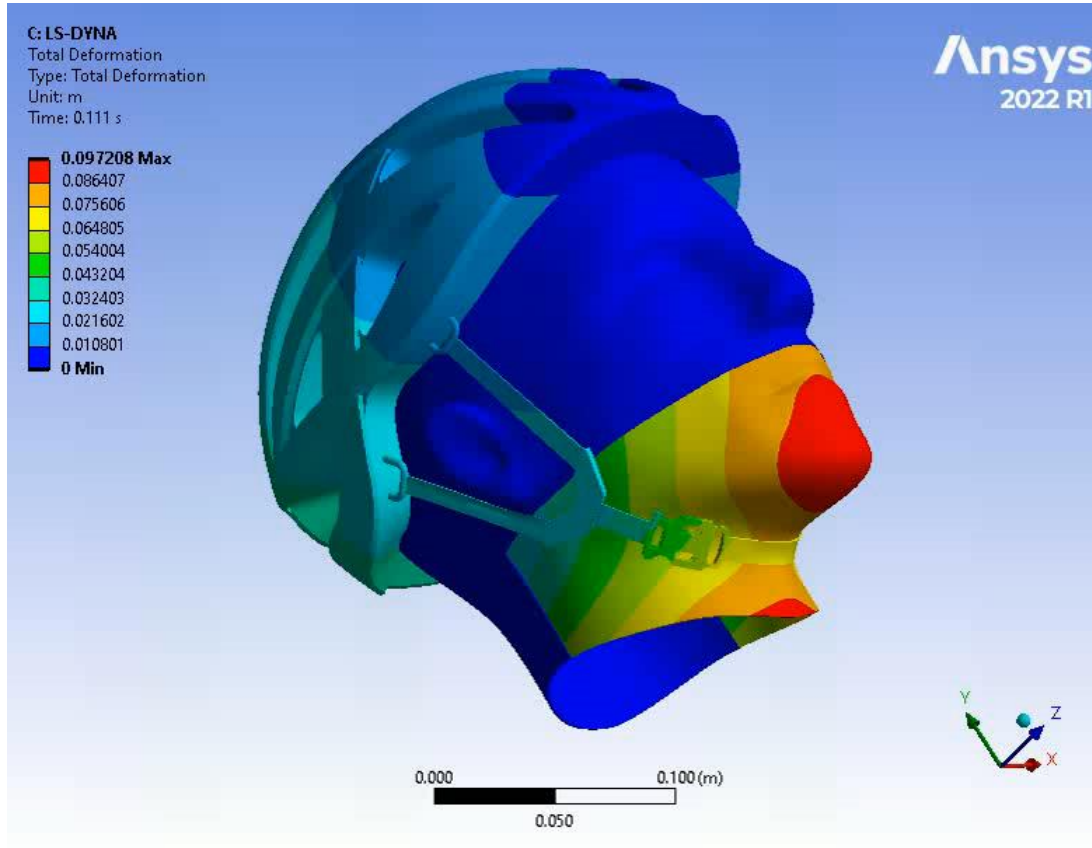


Details of "Imported Displacement"	
Scope	
Scoping Method	Geometry Selection
Geometry	1 Body
Definition	
Type	Imported Displacement
Displacement Type	Boundary Prescribed Final Geometry
Tabular Loading	Ramped
Suppressed	No
Override Constraints	No
Coordinate System	Source Coordinate System

	X Component (m)	Y Component (m)	Z Component (m)	Analysis Time (s)	Scale
1	File1:Displacement1	File1:Displacement2	File1:Displacement3	0.005	0.25
2	File1:Displacement1	File1:Displacement2	File1:Displacement3	0.001	1
*					

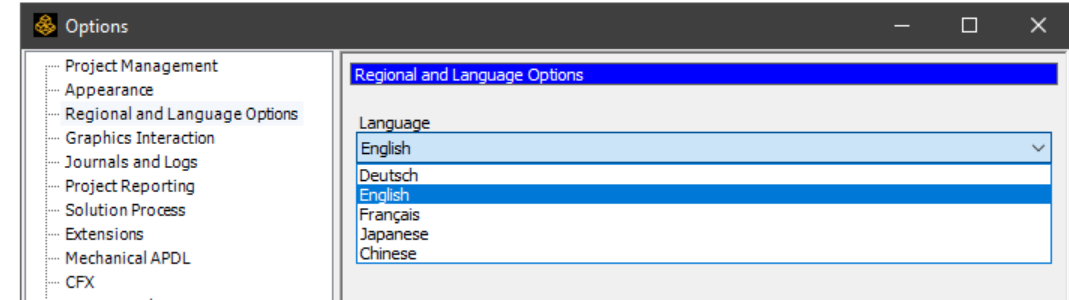
# Imported Displacement – Boundary Prescribed Final Geometry

- The displacements between a scaled version of the chin and the full size geometry are calculated and imported via an External Data system
- An imported displacement load is used to apply the displacement from the initial to the final configuration pulling the helmet into place and pre-stressing the chin straps



# Localization

- The LS-DYNA Worbench system now supports additional languages
- Enhancement covers items specific to the LS-DYNA analysis system; Analysis Settings, Load Objects, Toolbars



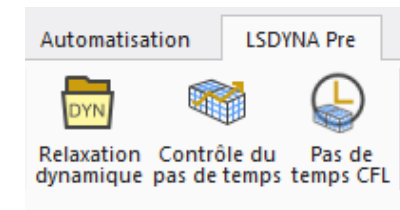
Language Selection

Détails de "Réglages de l'analyse"	
[-] Contrôles de pas	
Heure de fin	0.001 s
Coefficient de sécurité de l'incrément de temps	0.9
Nombre de cycles maximal	10000000
Mise à l'échelle automatique de la masse	Non
[-] Gestion du temps CPU et de la mémoire	
Allocation de Mémoire	Contrôlé par le programme
Nombre de CPUs	1
Type de traitement	Contrôlé par le programme
[-] Contrôles de Solveur	
Type de solveur	Contrôlé par le programme
Précision du Solveur	Contrôlé par le programme
Système d'unités	nmm
Explicite Solution Seulement	Oui
Numérotation de Nœud Invariable	Désactivé
Mise à jour de la contrainte de second degré	Non
Version du solveur	Contrôlé par le programme

Analysis Settings

Détails de "Contrôle Hourglass"	
[-] Geometry	
Méthode de champ d'application	Sélection de géométrie
Géométrie	1 Corps
[-] Définition	
Type Hourglass	Contrôlé par le programme
LS-DYNA ID	0
[-] Coefficients	
<input type="checkbox"/> Hourglass	0
<input type="checkbox"/> Compression quadratique	0
<input type="checkbox"/> Allongement Linéaire	0

Analysis Specific Loads



Analysis Specific Toolbars

# Localization

Includes all 4 additional languages that currently available in Workbench, French, German, Chinese, and Japanese

Détails de "Réglages de l'analyse"	
Contrôles de pas	
Heure de fin	0.001 s
Coefficient de sécurité de l'incrément de temps	0.9
Nombre de cycles maximal	10000000
Mise à l'échelle automatique de la masse	Non
Gestion du temps CPU et de la mémoire	
Allocation de Mémoire	Contrôlé par le programme
Nombre de CPUs	1
Type de traitement	Contrôlé par le programme
Contrôles de Solveur	
Type de solveur	Contrôlé par le programme
Précision du Solveur	Contrôlé par le programme
Système d'unités	mm
Explicite Solution Seulement	Oui
Numérotation de Nœud Invariable	Désactivé
Mise à jour de la contrainte de second degré	Non
Version du solveur	Contrôlé par le programme
Vitesses Initiales	
Contrôles de l'amortissement	
Amortissement Global	Non
Contrôles Hourglass	
Type Hourglass	Contrôlé par le programme
LS-DYNA ID	0
Coefficient de Hourglass défaut	0.1
Contrôles ALE	
Contrôles de Liaison	
Formulation	Contrôlé par le programme
Contrôles des composites	
Contrôles de sortie	
Format de sortie	Contrôlé par le programme
Facteur d'échelle de taille du fichier binaire	70
Contraintes	Oui
Déformations	Non
Déformation plastique	Oui
Historique des Variables	Non
Calculer les résultats à	Contrôlé par le programme
Fichier de contrainte pour pièces flexibles	Non
Contrôles de Sortie de l'Historique des Temps	
Calculer les résultats à	Non
Gestion de données d'analyse	

French

Details von "Analyseeinstellungen"	
Schrittsteuerungen	
Endzeit	0.001 s
Zeitschritt-Sicherheitsfaktor	0.9
Maximale Zyklenanzahl	10000000
Automatische Massenskalierung	Nein
CPU und Speicherverwaltung	
Speicherallokierung	Programmgesteuert
Anzahl CPUs	1
Verarbeitungstyp	Programmgesteuert
Solver-Steuerungen	
Solvertyp	Programmgesteuert
Solver-Genauigkeit	Programmgesteuert
Einheitensystem	mm
Nur explizite Lösung	Ja
Invariante Knotennummerierung	Aus
Spannungsupdate zweiter Ordnung	Nein
Solver Version	Programmgesteuert
Anfangsgeschwindigkeiten	
Dämpfungssteuerungen	
Globale Dämpfung	Nein
Hourglass-Steuerungen	
Hourglass-Typ	Programmgesteuert
LS-DYNA-ID	0
Standard-Hourglass-Koeffizient	0.1
ALE Steuerung	
Gelenkkontrollen	
Formulierung	Programmgesteuert
Composite-Steuerungen	
Ausgabesteuerungen	
Ausgabeformat	Programmgesteuert
Skalierungsfaktor für Binärdateigröße	70
Spannung	Ja
Dehnung	Nein
Plastische Dehnung	Ja
Zeitverlauf-Variablen	Nein
Ergebnisse berechnen für	Programmgesteuert
Spannungsdatei für flexible Teile	Nein
Ausgabesteuerungen für den Zeitlichen Verlauf	
Ergebnisse berechnen für	Nein
Analysedatenverwaltung	

German

"分析设置"的详细信息	
步骤控制	
结束时间	0.001 s
时步安全系数	0.9
最大周期数	10000000
自动质量缩放	没有
CPU和内存管理	
内存分配	程序控制的
CPU数	1
处理类型	程序控制的
求解器控制	
求解器类型	程序控制的
求解器精度	程序控制的
单位系统	mm
仅显式解	是
不变节点编号	关闭
二阶应力更新	没有
求解器版本	程序控制的
初始速度	
阻尼控制	
全局阻尼	没有
沙漏控制	
沙漏类型	程序控制的
LS-DYNA ID	0
默认沙漏系数	0.1
ALE控制	
连接控制	
公式	程序控制的
复合控制	
输出控制	
输出格式	程序控制的
二进制文件大小比例因子	70
应力	是
应变	没有
塑性应变	是
历史变量	没有
计算结果	程序控制的
柔性部件的应力文件	没有
时间历史输出控制	
计算结果	没有
分析数据管理	

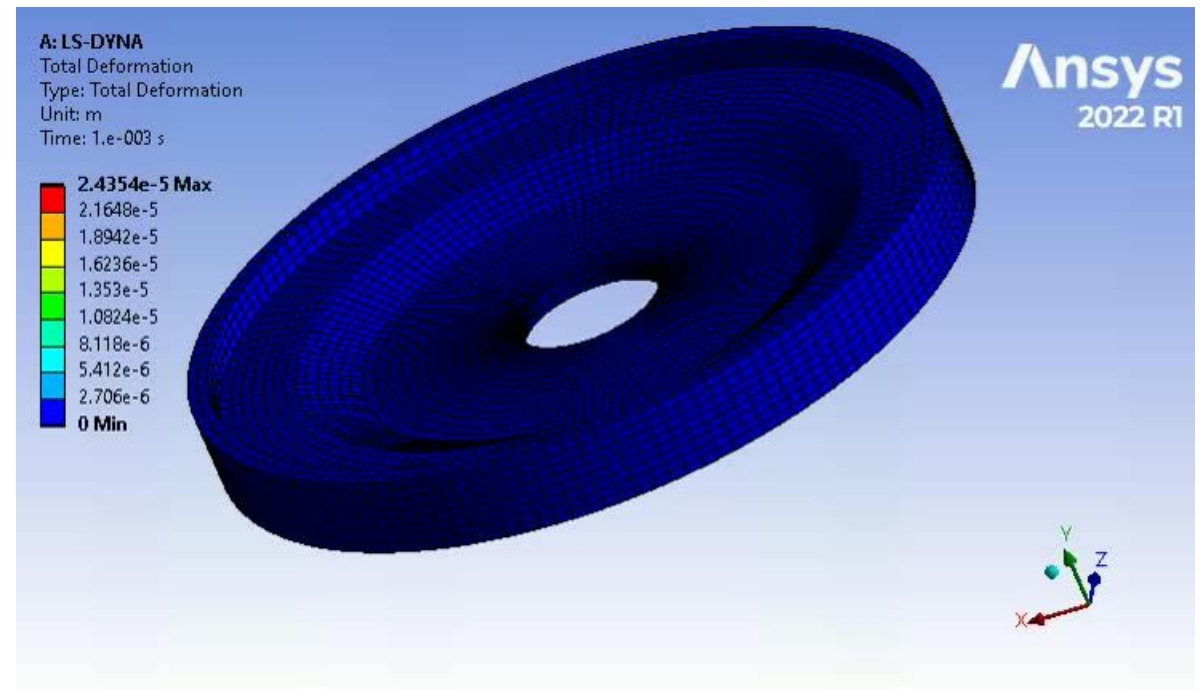
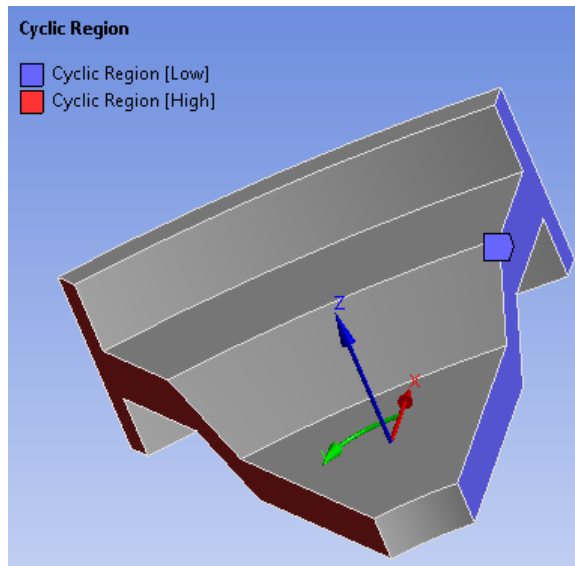
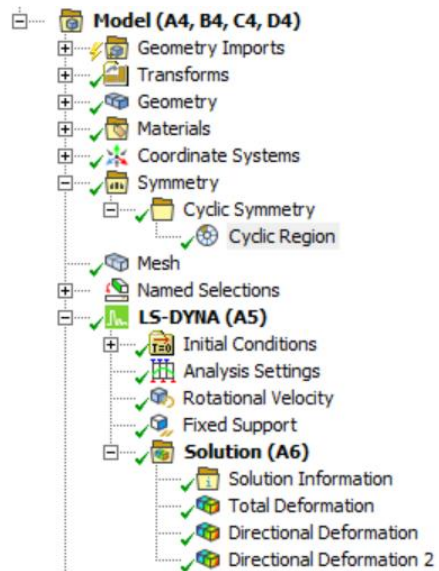
Chinese

"解析設定"の詳細	
ステップコントロール	
終了時間	0.001 s
時間ステップ安全率	0.9
最大サイクル数	10000000
自動質量スケール	No
CPUおよびメモリー管理	
メモリー割り当て	プログラムによるコントロール
CPU数	1
処理タイプ	プログラムによるコントロール
ソルバーコントロール	
ソルバータイプ	プログラムによるコントロール
ソルバー精度	プログラムによるコントロール
単位系	mm
陽解法解析のみ	Yes
不変節点番号	オフ
2次応力の更新	No
ソルバーバージョン	プログラムによるコントロール
初期速度	
減衰コントロール	
グローバル減衰	No
アワーグラス制御	
アワーグラスタイプ	プログラムによるコントロール
LS-DYNA ID	0
デフォルトのアワーグラス係数	0.1
ALEコントロール	
ジョイントコントロール	
定式化	プログラムによるコントロール
複合材コントロール	
出力コントロール	
出力形式	プログラムによるコントロール
バイナリファイルサイズのスケールファクター	70
応力	Yes
ひずみ	No
塑性ひずみ	Yes
履歴変数	No
結果の計算点	プログラムによるコントロール
弾性体の応力ファイル	No
時刻歴出力コントロール	
結果の計算点	No
解析データ管理	

Japanese

# Cyclic Symmetry

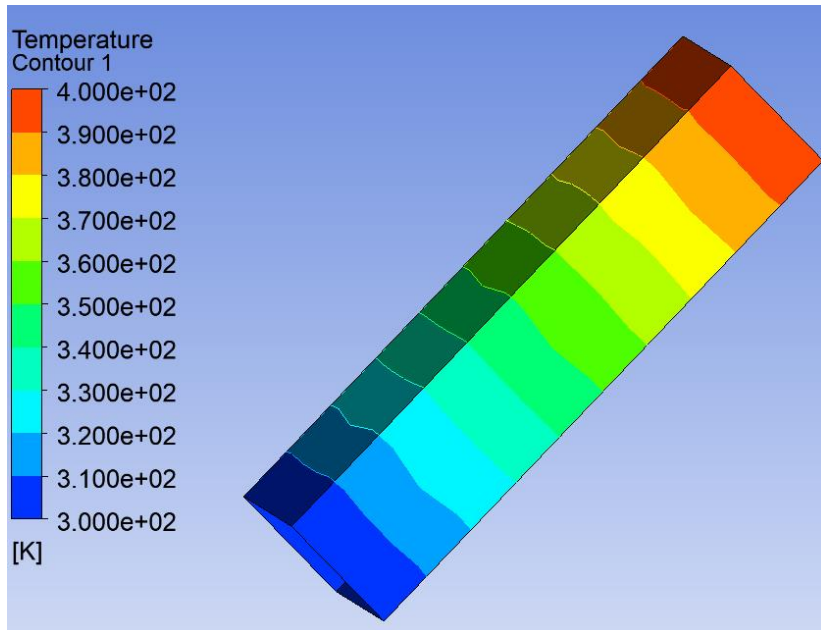
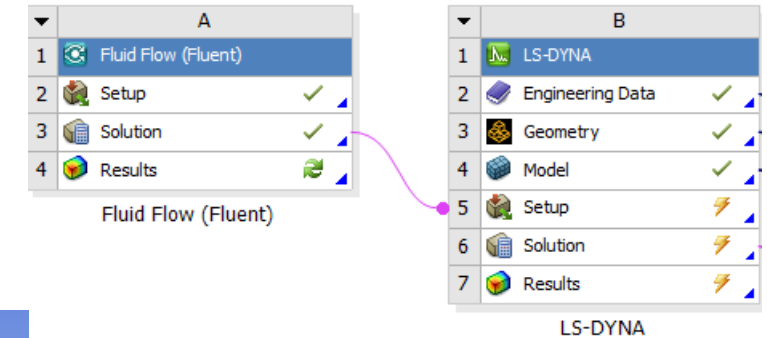
- Cyclic Symmetry is now supported in the LS-DYNA Workbench system and can be used in turbomachinery applications for faster running times
- Option to display the mesh and results with full symmetry (Beta)



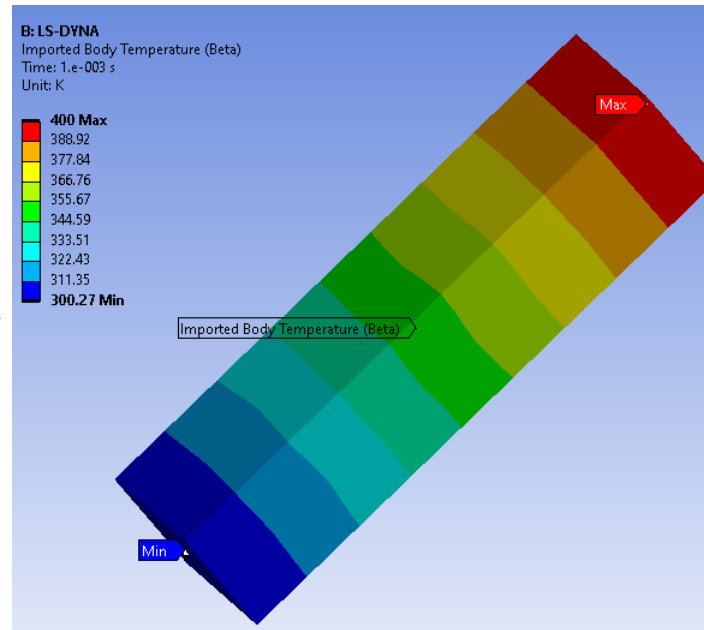


# Fluent to LS-DYNA 1 Way Thermal Transfer

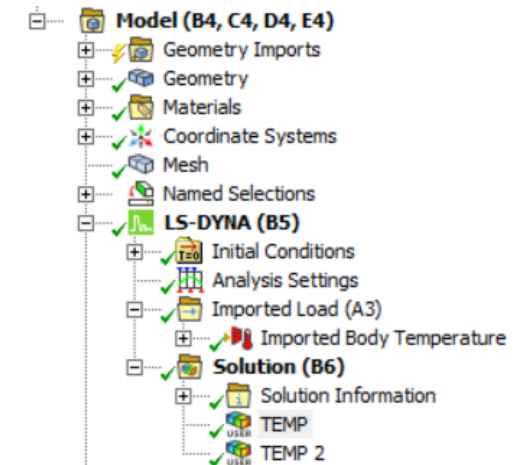
- Body Temperatures can be now imported from CFD calculations allowing to take in accurately temperatures effects in LS-DYNA simulations
- Link systems in the project schematic, imported load folder is added automatically, body temperature load can be inserted from the context menu
- Standard imported load features are available; Stepped\Ramped loading with the option to apply scale factor and offset



Fluent



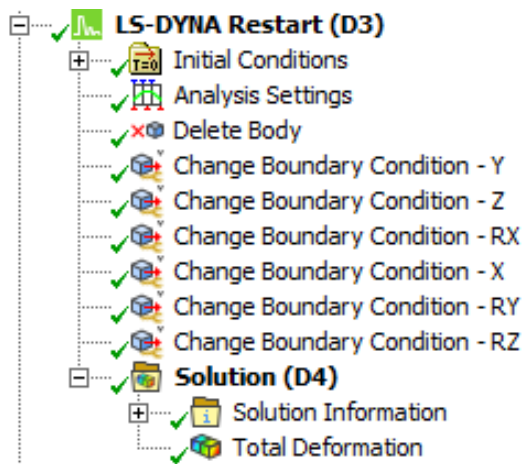
LS-Dyna





# / Restarts Improvements

- Displacements and Remote Displacements can be modified in restarts allowing to simulate complex movements
- Location method allows the selection of a boundary condition or a curve (added using the keyword manager)
- Each component of a boundary condition can be independently redefined by a curve

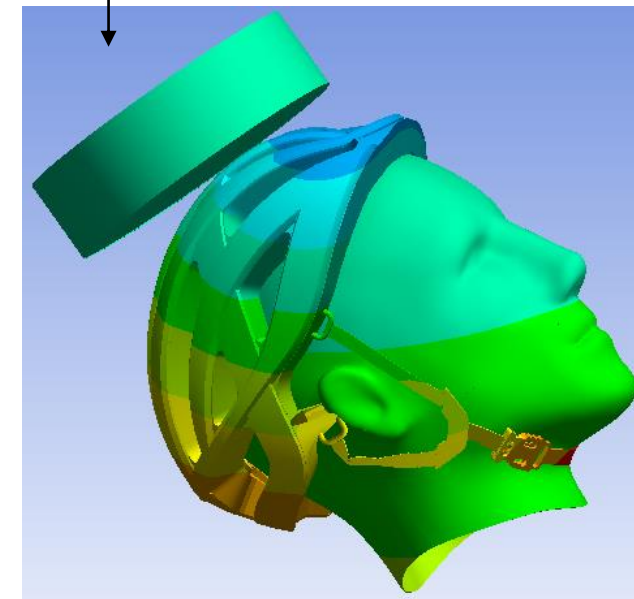
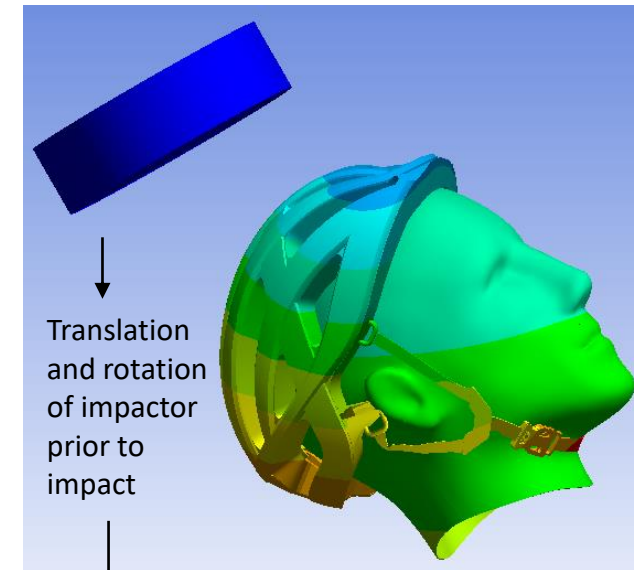


Details of "Change Boundary Condition 2"

Definition	
Location Method	Boundary Condition
Boundary Condition	Remote Displacement_Round_Impactor
Component	Y Component

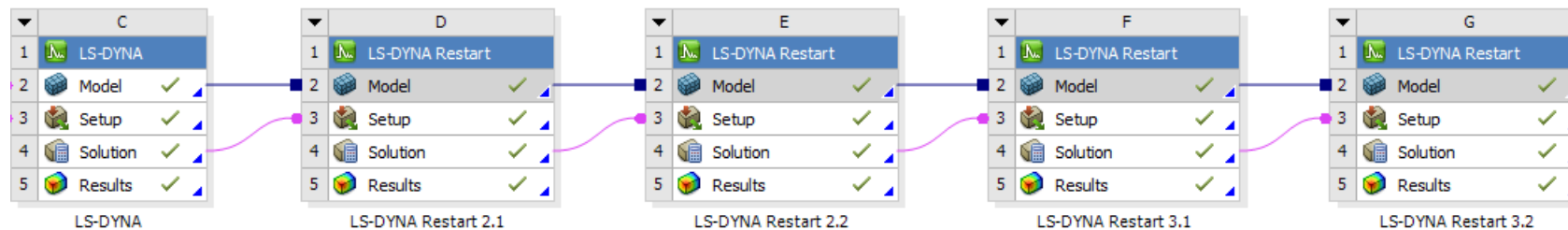
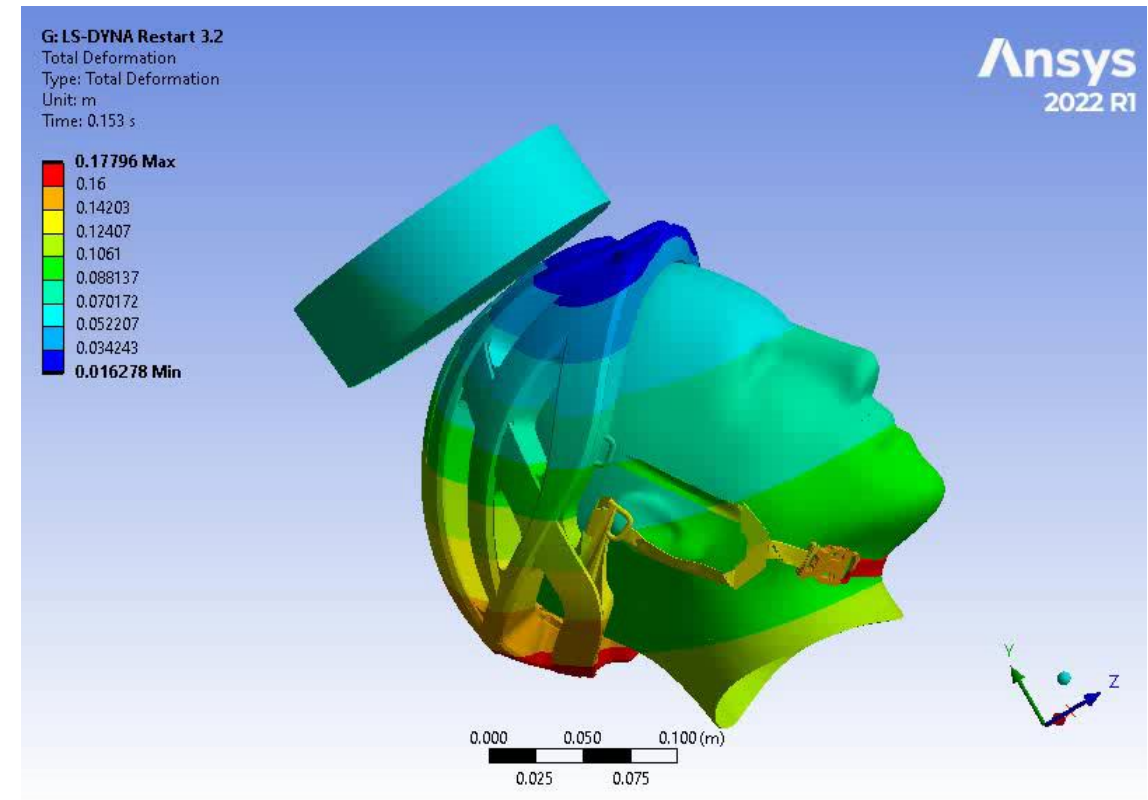
Tabular Data

		Time [s]	<input checked="" type="checkbox"/> Displacement [m]
1	1	0.	= 0.
2	1	0.132	0.
3	1	0.133	3.5e-002
4	N/A	3.	3.5e-002
5	N/A	4.	3.5e-002
*			



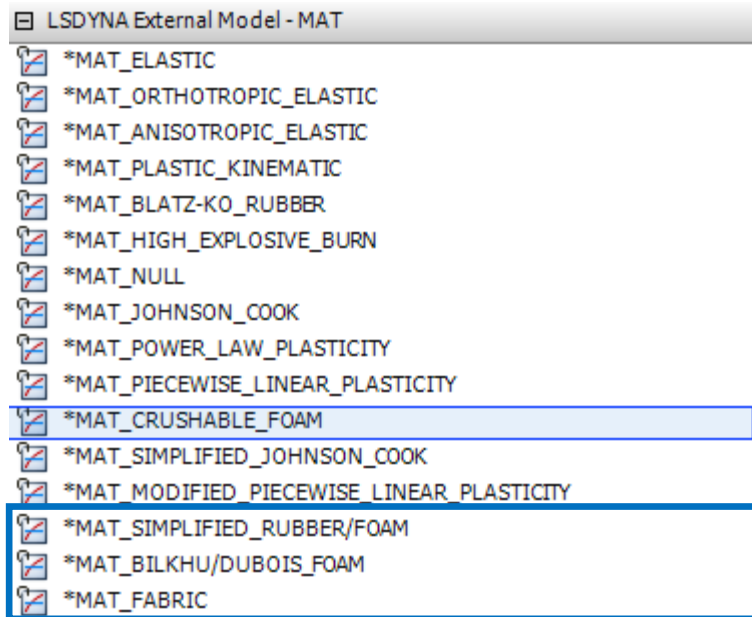
# Restarts Improvements

- Multi-step impact analysis using a series of small restarts
  - Pre-Stress
  - Impactor 1 Positioning
  - Impact 1
  - Impactor 2 Positioning
  - Impact 2
- Uses the new Change Boundary Condition object along with the Delete Body and Change Velocity objects



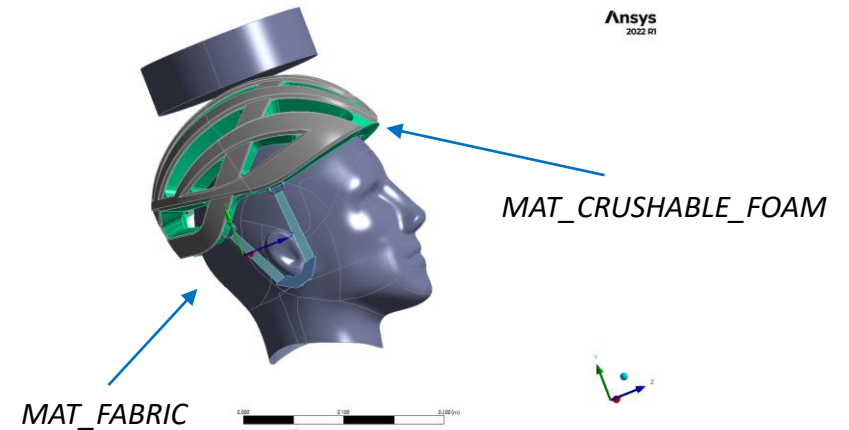
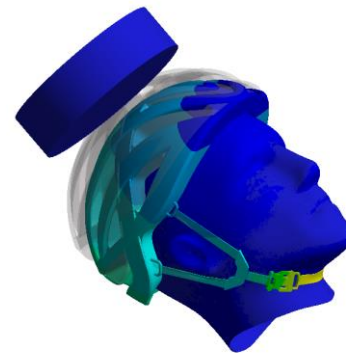
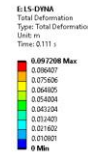
# Enhanced Material Support

- 4 additional material models have been introduced for applications using Fabric\Foam
- Can be imported to Engineering Data from .k file using External Model
- These definitions generally follow the input card with the variable names added as a suffix



Properties of Outline Row 8: MAT\_CRUSHABLE\_FOAM

	A	B	C	D	E
1	Property	Value	Unit		
2	Material Field Variables	Table			
3	Density	70	kg m <sup>-3</sup>		
4	Isotropic Elasticity				
5	Derive from	Young's Mod...			
6	Young's Modulus	2.5E+07	Pa		
7	Poisson's Ratio	0.2			
8	Bulk Modulus	1.3889E+07	Pa		
9	Shear Modulus	1.0417E+07	Pa		
10	*MAT_CRUSHABLE_FOAM				
11	Definition				
12	Tensile Stress cutoff, tsc	3.9E+05	Pa		
13	Rate Sensitivity via damping coefficient, damp	0			
14	Yield Stress versus Volumetric Strain, Icid	Tabular			
15	Scale	1			
16	Offset	0	Pa		



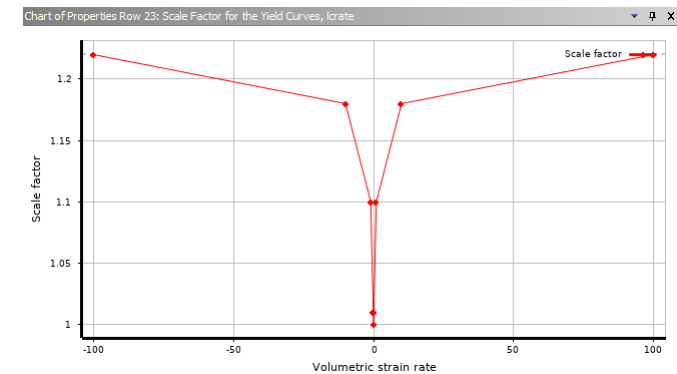
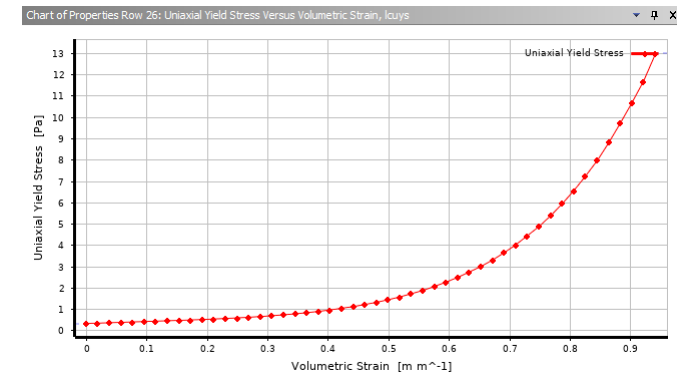
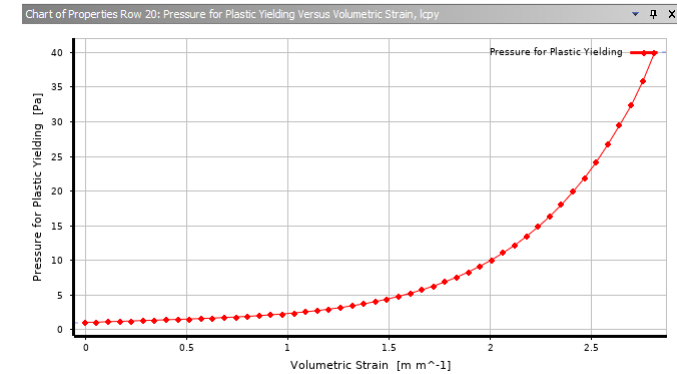
# Enhanced Material Support

- Associated load curves are available
- Added by default or available under the additional tabular data when selecting the material
- Not all options are supported when there are optional cards

MAT_BILKHU/DUBOIS_FOAM		
Definition		
Tensile Response	Volumetric Strain	
Viscous Damping Coefficient, vc	0	
Pressure Cutoff, pc	0	Pa
Variable Pressure Cutoff as a Fraction of Pressure Yield Value, vpc	0	Pa
Tension Cutoff for Uniaxial Tensile Stress, tsc	0	Pa
Variable Tension Cutoff as a Fraction of the Uniaxial Compressive Yield Strength, vtsc	0	Pa
Stiffness Coefficient for Contact Interface Stiffness, kcon	0	Pa
Number of Cycles to Determine the Average Volumetric Strain Rate, ncycle	0	
Pressure for Plastic Yielding Versus Volumetric Strain, lcpy	Tabular	
Uniaxial Yield Stress Versus Volumetric Strain, lcuys	Tabular	

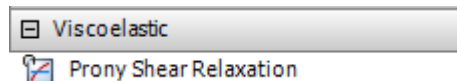
Additional Tabular Data

Scale Factor for the Yield Curves



# Enhanced Material Support: Prony Series

- Prony Series (Shear Relaxation) are supported for the following material properties, allowing to simulate time-dependent stress relaxation effects:
  - Mooney-Rivlin
  - Ogden
  - Yeoh
  - Polynomial
  - Arruda-Boyce
  - MAT\_SIMPLIFIED\_RUBBER/FOAM (new addition at 2022 R1)



Properties of Outline Row 3: Admat\_OD1

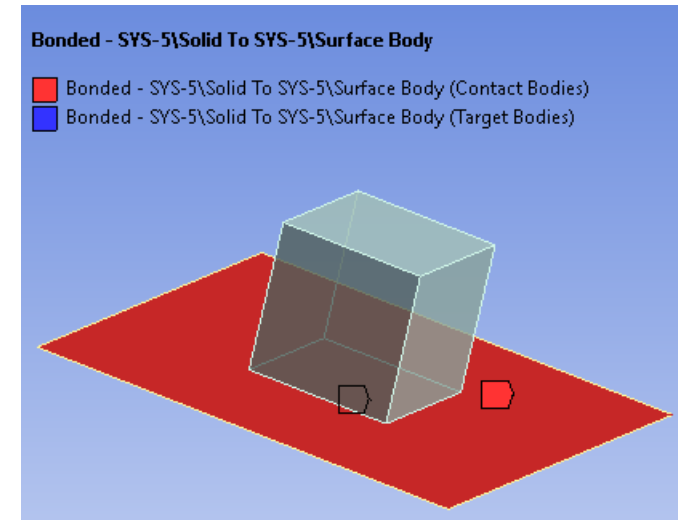
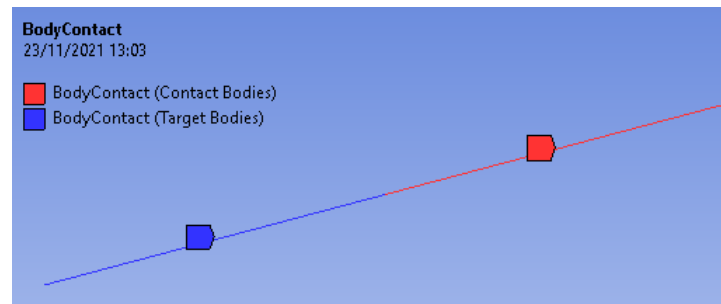
	A	B	C	D	E
1	Property	Value	Unit		
2	Material Field Variables	Table			
3	Density	1000	kg m^-3		
4	Ogden 1st Order				
5	Material Constant MU1	23845	Pa		
6	Material Constant A1	2.1151			
7	Incompressibility Parameter D1	1.4429E-07	Pa^-1		
8	Prony Shear Relaxation	Tabular			
9	Number of Terms	12			
10	Relative Moduli(): Scale	1			
11	Relative Moduli(): Offset	0			
12	Relaxation Time(): Scale	1			
13	Relaxation Time(): Offset	0	s		

	B	C	D
1	Index i	Relative Moduli()	Relaxation Time(i) (s)
2	1	1000	0.0001
3	2	10000	0.00011
4	3	9000	0.002
5	4	8000	0.003
6	5	7000	0.004
7	6	6000	0.005
8	7	5000	0.006
9	8	4000	0.007
10	9	3000	0.008
11	10	2000	0.009
12	11	1000	0.00111
13	12	2050	0.0123

# Contact Scoping

- Additional contact scoping options are available for LS-DYNA
- Contacts can now be scoped to Shell and Beam bodies in a 3D Analysis

Details of "BodyContact" ▾ ▴ □ ×	
Scope	
Scoping Method	Geometry Selection
Contact	1 Edge
Target	1 Body
Contact Bodies	Beam2
Target Bodies	Beam1
Protected	No
Definition	
Type	Bonded
Scope Mode	Manual
Trim Contact	Program Controlled
Maximum Offset	1.e-002 m
Breakable	No
Suppressed	No
Advanced	
Formulation	Program Controlled



Details of "Bonded - SYS-5\Solid To SYS-5\Surface Body" ▾ ▴ □ ×	
Scope	
Scoping Method	Geometry Selection
Contact	1 Body
Target	1 Face
Contact Bodies	SYS-5\Surface Body
Target Bodies	SYS-5\Solid
Contact Shell Face	Program Controlled
Protected	No
Definition	
Type	Bonded
Scope Mode	Manual
Behavior	Program Controlled
Trim Contact	Program Controlled
Maximum Offset	1.e-007 m
Breakable	No
Suppressed	No
Advanced	
Formulation	Program Controlled

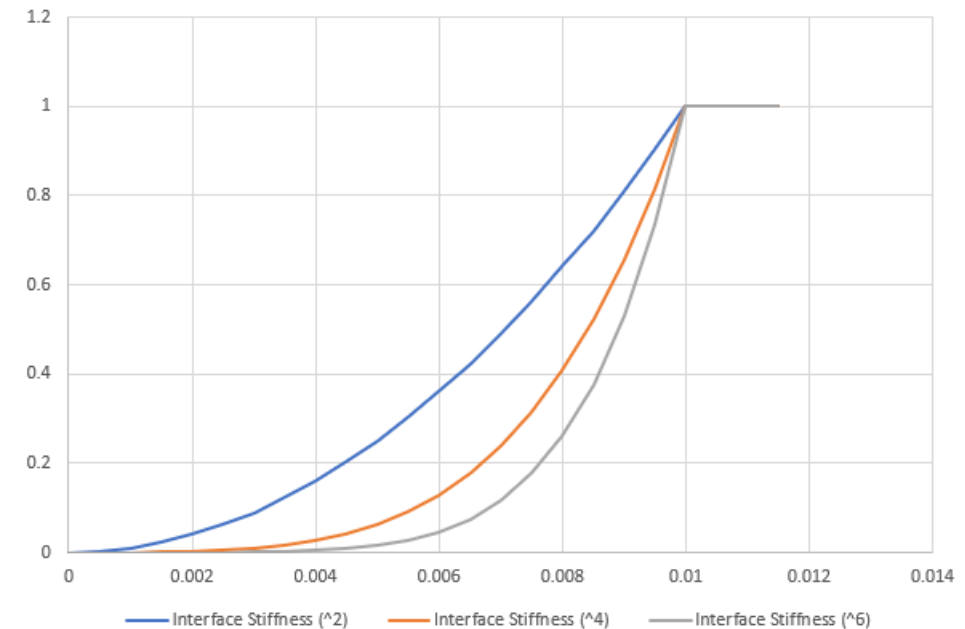


# Interference Contact Properties

- New options added to give additional flexibility in contact stiffness curve definition
- End time for contact. Previously this just used analysis time
- Stiffness curve function exponent to vary the transition to peak stiffness

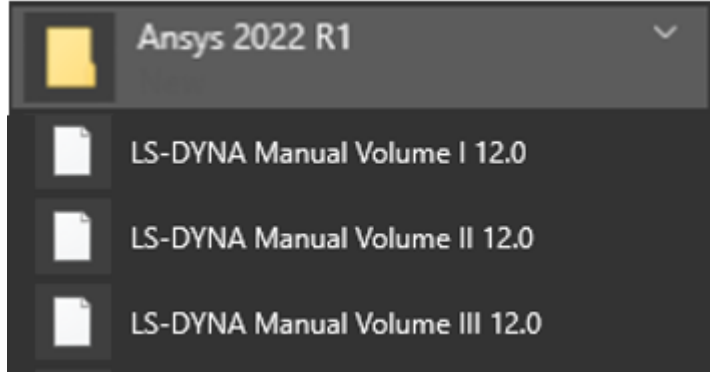
```
*DEFINE_CURVE_FUNCTION
$      ID      sidr
      3        0
min(1, (time / 0.01)**4)
```

Details of "Contact Properties 2"	
<b>Definition</b>	
Contact	Frictional - Multiple To Multiple (Interference)
Type	Interference
Formulation	SURFACE_TO_SURFACE_INTERFERENCE
<b>Common Controls</b>	
<input type="checkbox"/> Birth Time	0 s
<input type="checkbox"/> Death Time	0 s
<input type="checkbox"/> Viscous Damping Coefficient	10
<input type="checkbox"/> Contact Penalty Scale Factor	4
<input type="checkbox"/> Target Penalty Scale Factor	4
<b>Advanced Controls</b>	
<input type="checkbox"/> Optional Thickness for Contact Surface	0 m
<input type="checkbox"/> Optional Thickness for Target Surface	0 m
<input type="checkbox"/> Optional Solid Element Thickness	0 m
Soft Constraint Formulation	Program Controlled
<input type="checkbox"/> Soft Constraint Scale Factor	0.1
Depth	5
<b>Interference Controls</b>	
<input type="checkbox"/> Stiffness Scale factor At End of Dynamic Relaxation	1
<input type="checkbox"/> Interference End Time	0.01 s
<input type="checkbox"/> Stiffness Curve Exponent	4



# New Solver Version

- Default solver version for 2022 R1 is 12.1
- Refer to the documentation for a summary of new developments \ improvements to stability and release notes for further information



Details of "Analysis Settings"	
[-] Step Controls	
End Time	0.01 s
Time Step Safety Factor	0.9
Maximum Number Of Cycles	10000000
Automatic Mass Scaling	No
[+] CPU and Memory Management	
[-] Solver Controls	
Solver Type	Program Controlled
Solver Precision	Program Controlled
Unit System	nmm
Explicit Solution Only	Yes
Invariant Node Numbering	Off
Second Order Stress Update	No
Solver Version	Program Controlled
[-] Initial Velocities	Program Controlled
Initial Velocities are applied immediately	12.1
[+] Damping Controls	
[+] Hourglass Controls	
[+] ALE Controls	
[+] Joint Controls	
[+] Composite Controls	
[+] Output Controls	
[+] Time History Output Controls	
[+] Analysis Data Management	

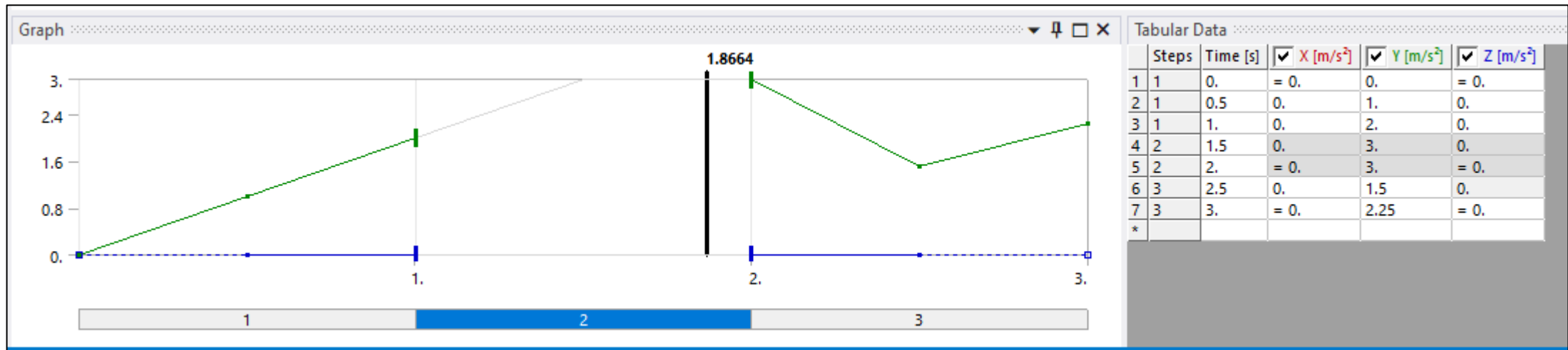
# **Rigid Body Dynamics**

**Ansys**

# Time Varying Acceleration Load

Remove the RBD solver specific limitations on the Acceleration and Standard Earth Gravity loads. Now the definition of Acceleration or Standard Earth Gravity is fully consistent with the other solvers, particularly the MAPDL solver, allowing smooth transition from one solver to another.

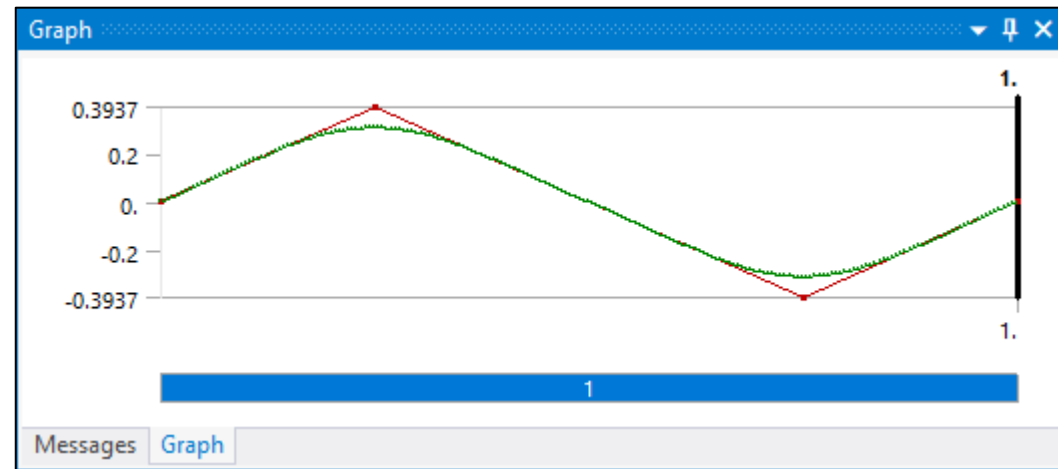
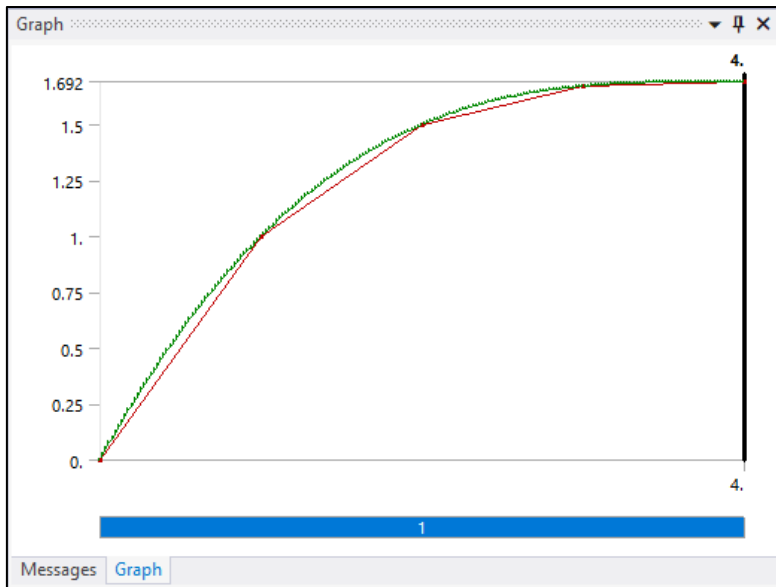
- Allows time varying accelerations specified using functions or tabular data
- Allows activating/deactivating acceleration or Standard Earth Gravity per load steps



# / Display of Joint Condition Fitting

- In the RBD solver, imposed joint conditions (displacement, velocity, rotations, rotational velocities) given by tables are internally fitted
- ➔ consequently, the results may differ from the piecewise curve display on the joint condition.

The results of the fitting is now overlaid on the joint condition curve.



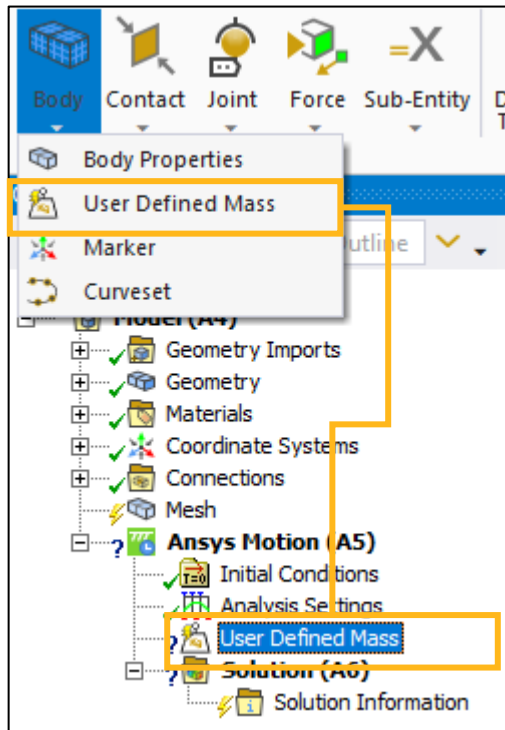
# **Motion Integration with Mechanical**

**Ansys**



# User Defined Mass

- Change Mass/Inertia property
  - A common practice in rigid dynamics analysis is to replace a complex geometry by a simpler one, but it is important to use the right geometry properties, thus User Defined Mass object allows to modify the Mass, Moments of Inertia and centroid of selected rigid bodies

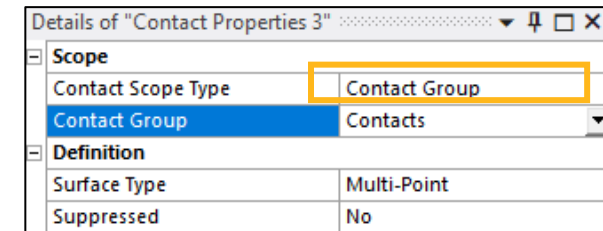
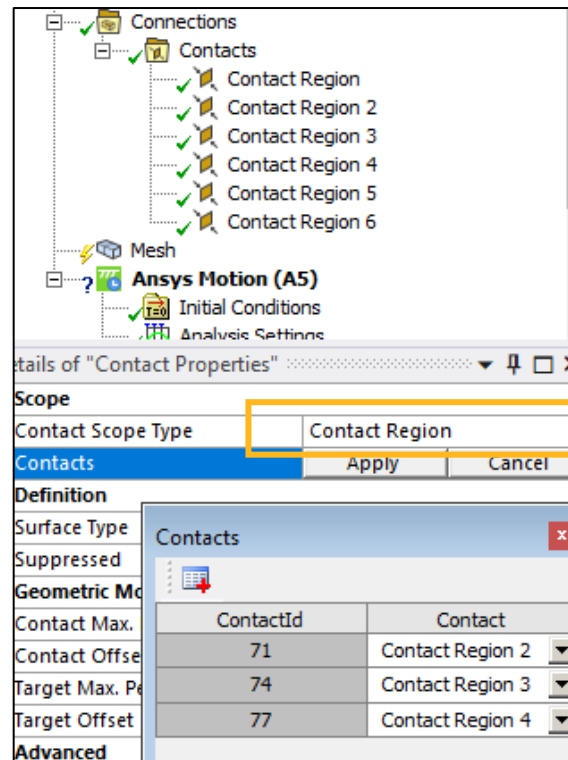


Details of "User Defined Mass"	
<b>Scope</b>	
Scoping Method	Geometry Selection
Geometry	1 Body
<b>Definition</b>	
<input type="checkbox"/> Mass	9.8125 kg
<input type="checkbox"/> Moment of Inertia Ip1	0.408854117927411 kg·m <sup>2</sup>
<input type="checkbox"/> Moment of Inertia Ip2	0.408854069188203 kg·m <sup>2</sup>
<input type="checkbox"/> Moment of Inertia Ip3	0.408854069188204 kg·m <sup>2</sup>
Centroid	Unchanged

# Contact Group Scoping

- Improvement in selection Method

- Contact Properties, Contact Friction Properties and Tie Properties objects are used to modify contact settings. For important number of contact regions, it was cumbersome to select all desired contacts to modify, thus now it is possible to modify all contacts of a given Connection Group improve a lot user friendliness

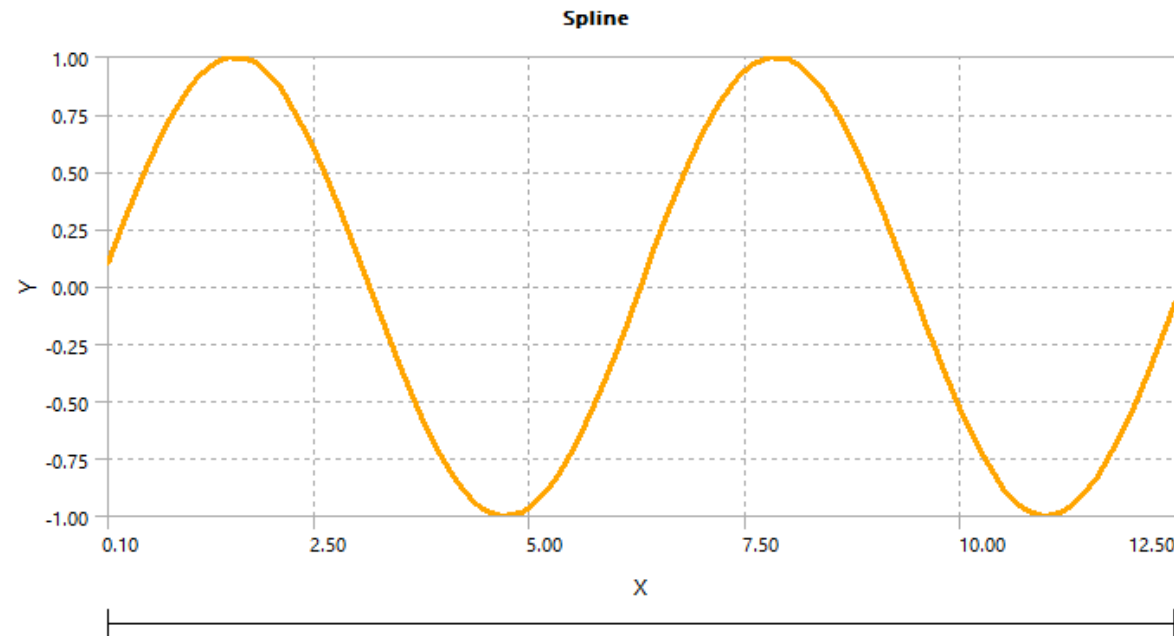


New – Contact group

Previous – Contact Region with tabular data

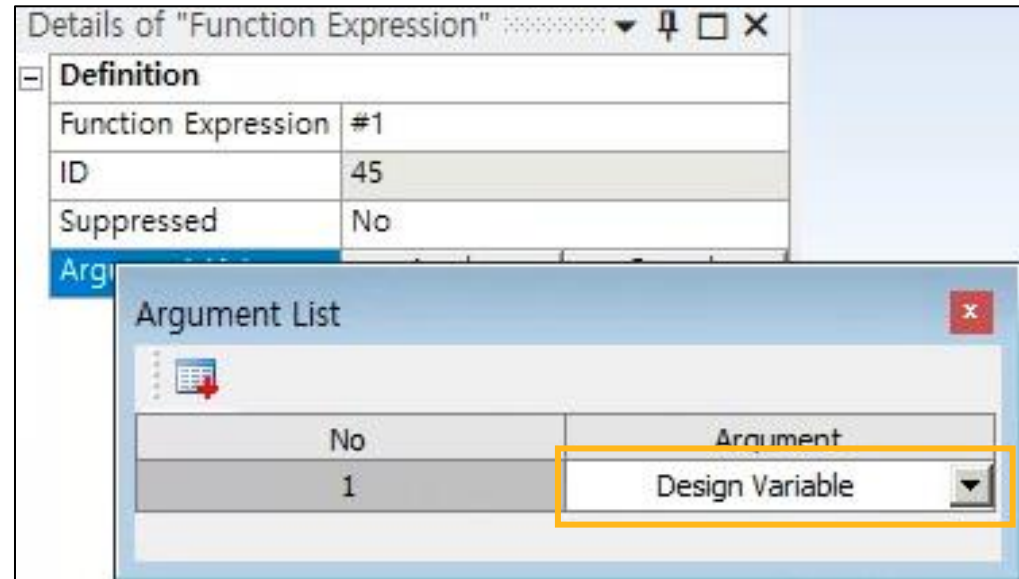
# / Function Previewer

- Preview function expression
  - Evaluation of function expression expressed by Mathematical(sin, cos, tan and so on), Logical (If) and several constant value are available
  - Integration variables are not available and will be extended for STEP, LININT in next version



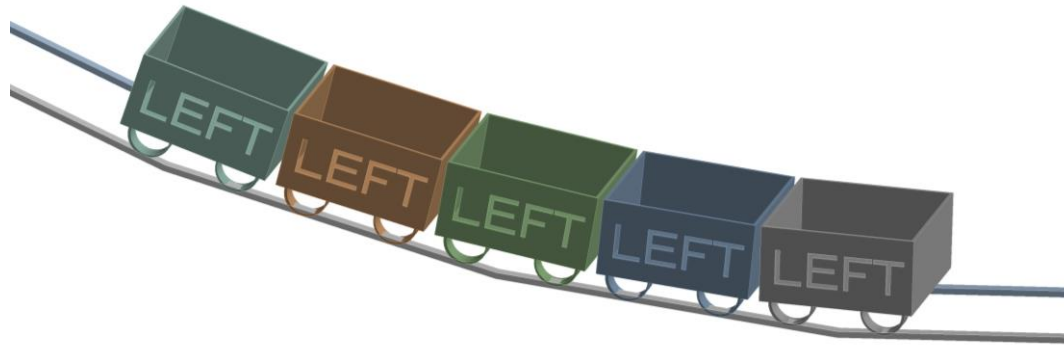
# / Design Variable

- Parametric study with design variable
  - A Design Variable(DV) object can be assigned to “Parameter Set”.
  - The DV can be used within Function Expression, and it enables to perform design experiments with complex loading conditions



# / Links: Path Follower

- A new way to assemble segment
  - Links toolkit enables you to assemble segment along with imaginary line passing “Path”. On the other hands, A Path Follower function enables you to place segments on the predefined Curvesets
  - It can be useful to build a roller coaster or train applications for placing cabin on the rail



# Additional Improvements

- Extended Joint Friction
  - Joint Friction Properties has been extended to Translational and Cylindrical joints
  - Transition Velocity coefficient is now available

Details of "Joint Friction Properties 2"	
[-] Scope	
Joint	Joint 2 [Translational]
[-] Definition	
<input type="checkbox"/> Static Friction Coefficient	0.5
<input type="checkbox"/> Dynamic Friction Coefficient	0.3
<input type="checkbox"/> Stiction Transition Velocity	0.0001 m/s
<input type="checkbox"/> Transition Velocity Coefficient	1.5
<input type="checkbox"/> Max Stiction Deformation	1E-05 m
Reaction Force	Use
Bending Moment	Use
Torsional Moment	Use
Friction Effect	Sliding And Stiction
<input type="checkbox"/> Reaction Arm	0.001 m
Overlap Option	Constant
<input type="checkbox"/> Initial Overlap	0.001 m
<input type="checkbox"/> Pre Force	0 N

- FMI 2.0
  - FMI 2.0 is available. Interface Time Step and Model description are available

Details of "Co-Simulator"	
[-] Definition	
Interface Type	FMI
FMI Version	2.0
<input type="checkbox"/> Interface Time Step	0.01 s
Model Description	pendulum
Signal Input	Tabular Data
Signal Output	Tabular Data
Suppressed	No



# Postprocessing Improvements

- Custom Result

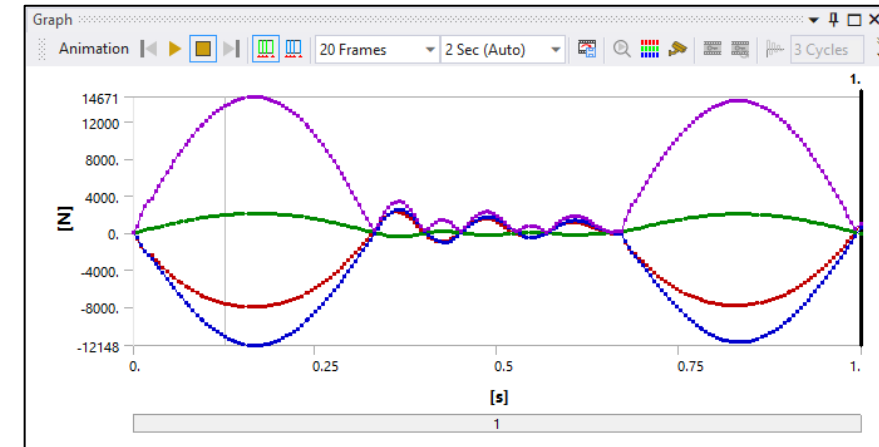
- Custom Result object allows you to evaluate output of Function expression and User Subroutine
- Result values can also be used as output parameter

Details of "Custom Result"	
Definition	
Type	Function Expression
Function Expression	output_Knuckle_right_dz
Unit Name	Length
Suppressed	No
Results	
<input type="checkbox"/> Result	0.150153694 m
Maximum Value Over Time	
<input type="checkbox"/> Result	0.242967413 m
Minimum Value Over Time	
<input type="checkbox"/> Result	0.142779276 m

- Joint Probes

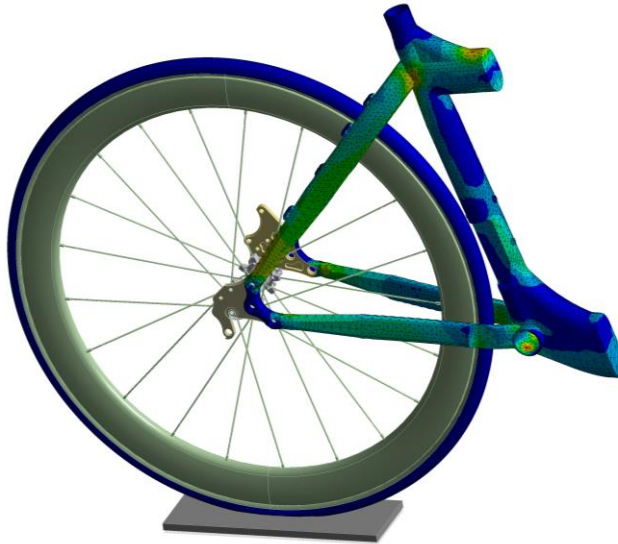
- Joint Probe results can now be used to analyze joint results

Details of "Joint Probe"	
Definition	
Type	Joint Probe
Boundary Condition	BUSHING_05
Orientation Method	Joint Reference System
Suppressed	No
Options	
Result Type	Total Force
Result Selection	All
<input type="checkbox"/> Display Time	End Time
Results	
Maximum Value Over Time	
<input type="checkbox"/> X Axis	2295.6 N
<input type="checkbox"/> Y Axis	2091.3 N
<input type="checkbox"/> Z Axis	2485.4 N
<input type="checkbox"/> Total	14671 N
Minimum Value Over Time	
<input type="checkbox"/> X Axis	-7961.4 N
<input type="checkbox"/> Y Axis	-393.5 N
<input type="checkbox"/> Z Axis	-12148 N
<input type="checkbox"/> Total	0.11785 N



# / Postprocessing Improvements

- Nodal Averaged Stress & Strain
  - Stresses and strains are now evaluated either at the nodes or at the element
  - Thus, now Mechanical postprocessing is consistent with standalone Postprocessor



**nCode DesignLife**

**Ansys**



# Vibration Fatigue, Direct Harmonic Case with PSD Loading

DesignLife Mechanical UI allows Direct Harmonic calculation with PSD Loading:

- 4 Analysis Types supported - Stress, Strain, Solid and Shell Seam Weld
- Single Loading Event and Multiple Loading Events
- Harmonic Environment selection
- PSD Cycle Counting Method: Lalanne, Dirlik, Narrow Band or Strainberg
- Table Definition: Input Frequency vs Values

Details of "Analysis Settings" ▾ 🔍 □ ×

Definition	
Analysis Domain	Time based ▾
Analysis Type	Time based ▾
<input type="checkbox"/> Scale Factor	Frequency based
Calculate Safety Factor	No
Number of Threads	2
Solver Directory	0
+ Analysis Data Management	

Details of "Loading Event" ▾ 🔍 □ ×

Definition	
Domain	Frequency
Event	Loading Event
Type	PSD ▾
<input type="checkbox"/> Repeat Count	1
+ Add Load	
Add Load	Add

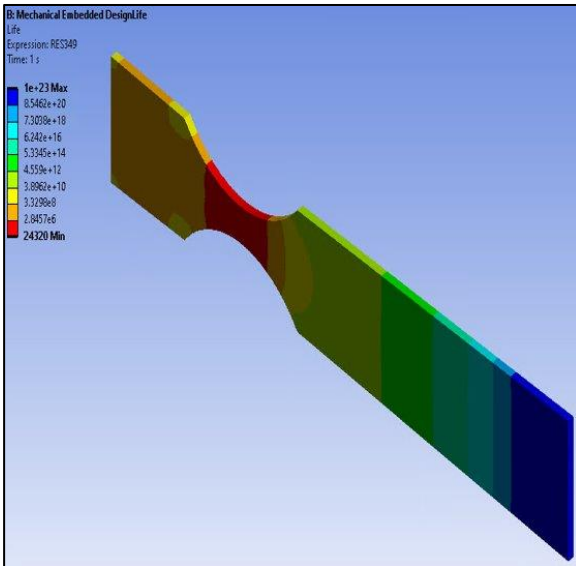
Details of "PSD Load" ▾ 🔍 □ ×

Definition	
Environment	Harmonic Response
PSD Cycle Counting Method	Lalanne
Table Definition	Tabular Data
Use Static Load Case	Yes
Static Load Case Environment	Static Structural
Static Load Case Step	2

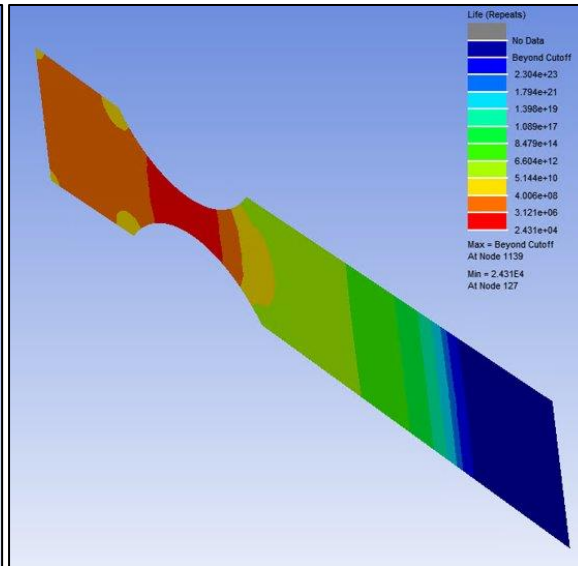
# Vibration Fatigue, Direct Harmonic Case with PSD Loading for Stress Analysis Type

## Life Result

Mechanical UI



Standalone



## Overall – Result comparison

### Mechanical UI

Node	Damage	RMS stress [MPa]	Irregularity factor	Largest Stress Cycle Amplitude [MPa]	Life [Repeats]
127	4.11E-05	60.09	0.741	173.2	2.4320E+04
192	3.94E-05	59.84	0.7415	172.5	2.5370E+04
126	3.53E-05	59.23	0.9286	165.8	2.8340E+04
193	3.45E-05	59.1	0.9284	165.4	2.8970E+04
128	3.42E-05	58.63	0.425	180.6	2.9240E+04
191	3.30E-05	58.48	0.4505	179	3.0310E+04
295	2.35E-05	56.77	0.7387	163.7	4.2590E+04
552	2.23E-05	56.5	0.8782	159.4	4.4890E+04

### Standalone

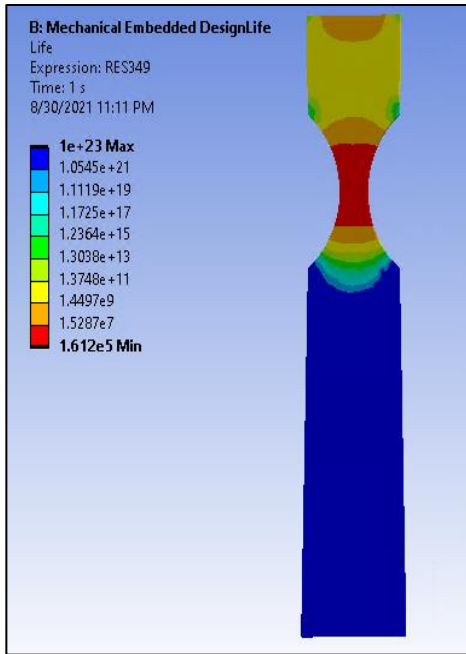
Node	Damage	RMS stres	Irregularity factor	Largest Stress Cycle Amplitude	Life
127	4.11E-05	60.09	0.7237	173.7	2.4310E+04
192	3.94E-05	59.84	0.7289	172.8	2.5360E+04
126	3.52E-05	59.22	0.9372	165.6	2.8410E+04
193	3.45E-05	59.09	0.9349	165.3	2.9020E+04
128	3.42E-05	58.62	0.4273	180.5	2.9290E+04
191	3.32E-05	58.48	0.4392	179.5	3.0170E+04
295	2.35E-05	56.77	0.711	164.5	4.2530E+04
552	2.22E-05	56.5	0.8993	158.9	4.4970E+04

# Vibration Fatigue, Direct Harmonic Case with PSD Loading for Strain Analysis Type

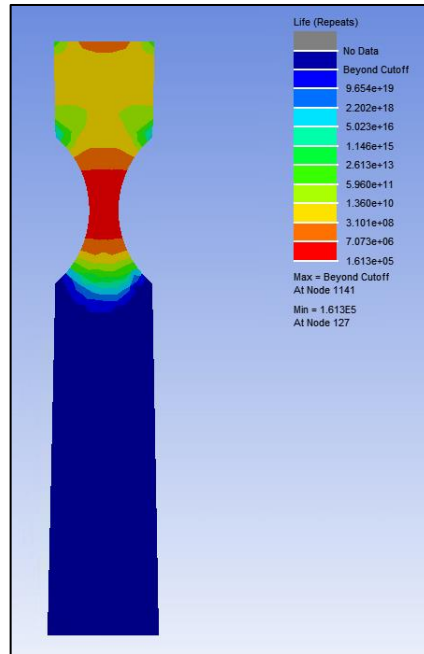
## Overall – Result comparison

### Life Result

#### Mechanical UI



#### Standalone



### Mechanical UI

Node	Damage	RMS stress [Pa]	Mean stress [Pa]	Return Period to Exceed UTS [s]	Irrregularity factor	Largest Stress Cycle	Life [Repeats]
127	6.2040E-06	5.86E+07	0	1.71E+05	0.7382	1.70E+08	1.6120E+05
192	5.9750E-06	5.83E+07	0	1.98E+05	0.739	1.69E+08	1.6740E+05
126	5.4000E-06	5.77E+07	0	3.56E+05	0.927	1.63E+08	1.8520E+05
128	5.3010E-06	5.72E+07	0	2.23E+05	0.4225	1.77E+08	1.8860E+05
193	5.2870E-06	5.76E+07	0	3.87E+05	0.9268	1.62E+08	1.8920E+05
191	5.1860E-06	5.70E+07	0	2.44E+05	0.4249	1.77E+08	1.9280E+05
295	3.8180E-06	5.55E+07	0	1.19E+06	0.7362	1.61E+08	2.6190E+05
552	3.6480E-06	5.52E+07	0	1.69E+06	0.8762	1.57E+08	2.7410E+05
397	3.6280E-06	5.52E+07	0	1.65E+06	0.8287	1.58E+08	2.7570E+05
296	3.4610E-06	5.49E+07	0	2.21E+06	0.9267	1.55E+08	2.8890E+05

### Standalone

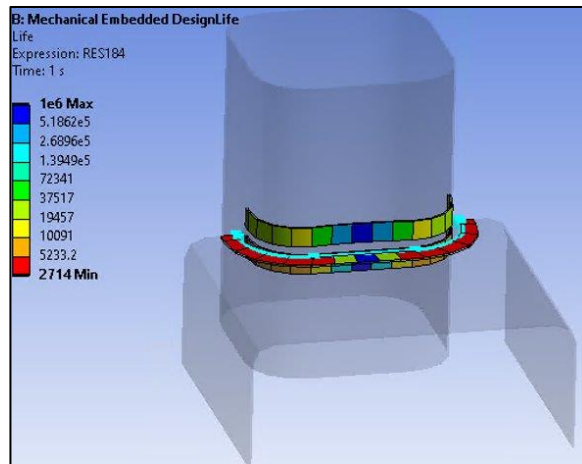
NodeId	Damage	RMSStress	MeanStress	ReturnPeriodToExceedUTS	IrrFactor	LargestStressCycle	Life
127	6.198E-06	5.86E+07	0	1.74E+05	0.7499	1.70E+08	1.613E+05
192	5.967E-06	5.83E+07	0	2.04E+05	0.7597	1.69E+08	1.676E+05
126	5.386E-06	5.77E+07	0	3.62E+05	0.9369	1.62E+08	1.857E+05
193	5.275E-06	5.76E+07	0	3.93E+05	0.9352	1.62E+08	1.896E+05
128	5.264E-06	5.71E+07	0	2.31E+05	0.4351	1.77E+08	1.900E+05
191	5.150E-06	5.70E+07	0	2.52E+05	0.4375	1.76E+08	1.942E+05
295	3.817E-06	5.55E+07	0	1.20E+06	0.7378	1.61E+08	2.620E+05
552	3.639E-06	5.52E+07	0	1.75E+06	0.9056	1.56E+08	2.748E+05
397	3.626E-06	5.52E+07	0	1.65E+06	0.8311	1.58E+08	2.758E+05
296	3.454E-06	5.49E+07	0	2.24E+06	0.9347	1.55E+08	2.895E+05



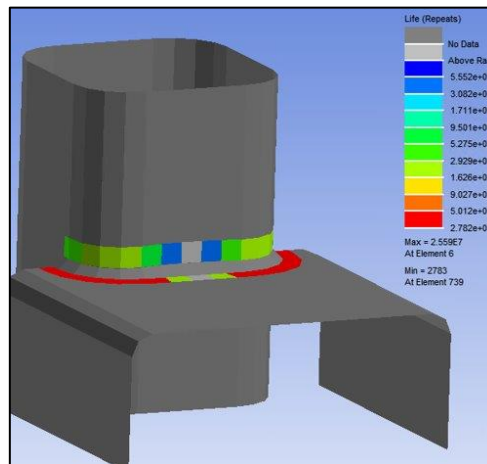
# Vibration Fatigue, Direct Harmonic Case with PSD Loading for Shell Seam Weld Analysis Type

## Life Result

Mechanical UI



Standalone



## Overall – Result comparison

### Mechanical UI

Element	Damage	RMS bending ratio	RMS stress [Pa]	Mean stress [Pa]	Irregularity factor	Largest Stress Cycle Amplitude [Pa]	Life [Repeats]
739	3.684E-04	0.02089	5.08E+07	0	0.8541	1.83E+08	2714
745	3.641E-04	0.02083	5.06E+07	0	0.854	1.82E+08	2747
728	3.629E-04	0.04665	4.80E+07	0	0.9623	1.73E+08	2756
729	3.588E-04	0.04701	4.79E+07	0	0.9624	1.73E+08	2787
748	3.208E-04	0.003562	4.66E+07	0	0.9604	1.68E+08	3117
736	3.182E-04	0.003561	4.65E+07	0	0.9604	1.67E+08	3143
390	2.751E-04	0.2171	4.61E+07	0	0.8128	1.68E+08	3634
737	2.665E-04	0.0477	4.48E+07	0	0.9444	1.61E+08	3752
747	2.663E-04	0.04748	4.48E+07	0	0.9446	1.61E+08	3755
300	2.606E-04	0.1987	4.52E+07	0	0.8145	1.64E+08	3838

### Standalone

ElementId	Damage	RMSBendingRatio	RMSStress	MeanStress	IrrFactor	LargestStressCycle	Life
739	3.59E-04	1.26E-04	5.04E+07	0	0.8542	1.82E+08	2783
728	3.58E-04	1.16E-04	4.78E+07	0	0.9643	1.73E+08	2794
745	3.55E-04	1.25E-04	5.02E+07	0	0.8541	1.81E+08	2816
729	3.54E-04	1.17E-04	4.76E+07	0	0.9644	1.72E+08	2827
748	3.20E-04	2.44E-05	4.65E+07	0	0.9612	1.68E+08	3129
736	3.17E-04	2.43E-05	4.64E+07	0	0.9612	1.67E+08	3155
390	2.72E-04	5.96E-04	4.59E+07	0	0.8132	1.67E+08	3682
737	2.67E-04	6.95E-05	4.49E+07	0	0.9449	1.61E+08	3745
747	2.67E-04	6.84E-05	4.49E+07	0	0.9451	1.61E+08	3748
300	2.57E-04	5.48E-04	4.49E+07	0	0.8149	1.63E+08	3889

# Vibration Fatigue, Direct Harmonic Case with PSD Loading for Solid Seam Weld Analysis Type

## Overall – Result comparison

### Mechanical UI

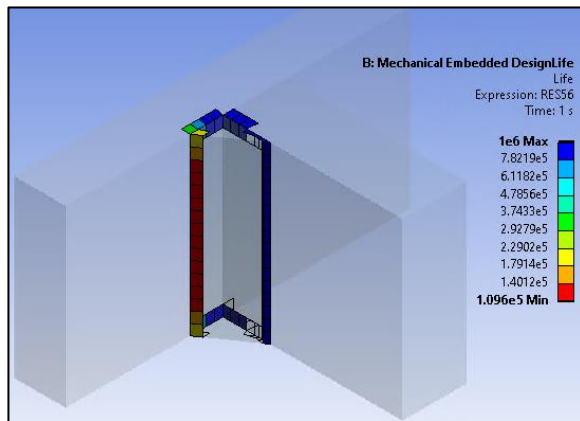
Element	Damage	RMS stress [Pa]	Irregularity factor	Largest Stress Cycle Amplitude [Pa]	Life [Repeats]
5081	9.1270E-06	1.37E+07	0.8841	5.03E+07	1.096E+05
5080	9.1270E-06	1.37E+07	0.8841	5.03E+07	1.096E+05
5082	9.0410E-06	1.36E+07	0.8841	5.01E+07	1.106E+05
5079	9.0410E-06	1.36E+07	0.8841	5.01E+07	1.106E+05
5083	8.8620E-06	1.35E+07	0.8841	4.98E+07	1.128E+05
5078	8.8620E-06	1.35E+07	0.8841	4.98E+07	1.128E+05
5084	8.5830E-06	1.34E+07	0.8841	4.93E+07	1.165E+05
5077	8.5830E-06	1.34E+07	0.8841	4.93E+07	1.165E+05
5085	8.1920E-06	1.32E+07	0.8841	4.85E+07	1.221E+05
5076	8.1920E-06	1.32E+07	0.8841	4.85E+07	1.221E+05

### Standalone

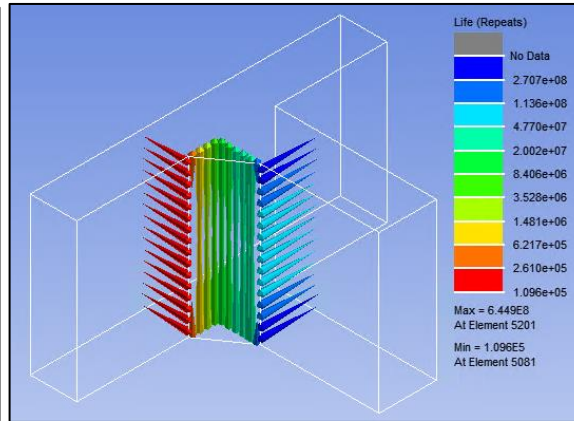
ElementId	Damage	RMSStress	IrrFactor	LargestStressCycle	Life
5081	9.127E-06	1.37E+07	0.8841	5.03E+07	1.096E+05
5080	9.127E-06	1.37E+07	0.8841	5.03E+07	1.096E+05
5082	9.041E-06	1.36E+07	0.8841	5.01E+07	1.106E+05
5079	9.041E-06	1.36E+07	0.8841	5.01E+07	1.106E+05
5083	8.862E-06	1.35E+07	0.8841	4.98E+07	1.128E+05
5078	8.862E-06	1.35E+07	0.8841	4.98E+07	1.128E+05
5084	8.583E-06	1.34E+07	0.8841	4.93E+07	1.165E+05
5077	8.583E-06	1.34E+07	0.8841	4.93E+07	1.165E+05
5085	8.192E-06	1.32E+07	0.8841	4.85E+07	1.221E+05

## Life Result

### Mechanical UI



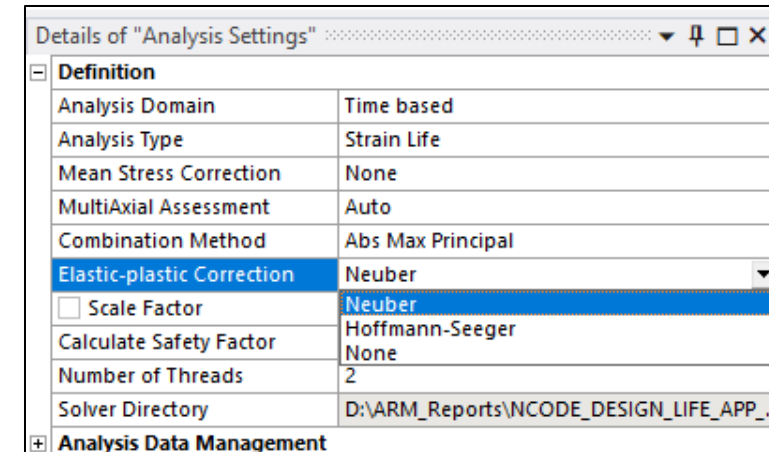
### Standalone



# / Ansys nCode DesignLife Mechanical UI: EPC Options

## Elastic-Plastic Correction:

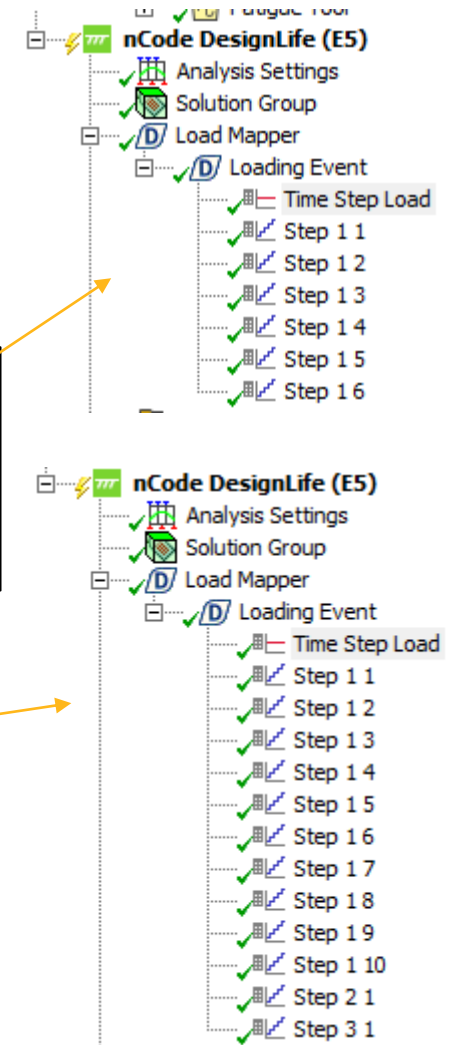
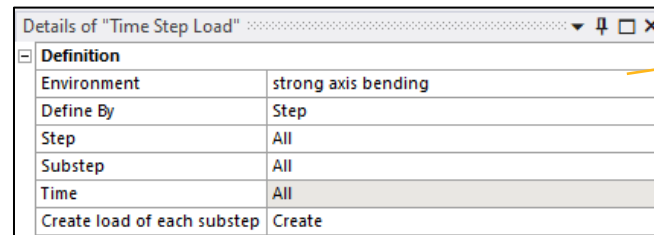
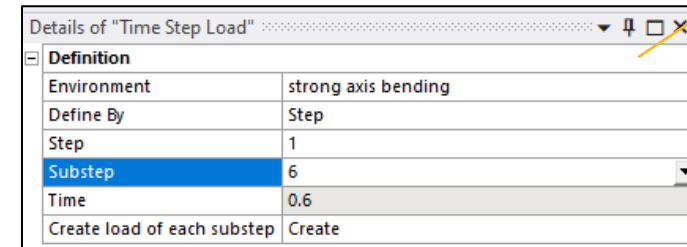
- For Strain Life analysis, the Elastic-plastic Correction option is exposed
  - Choose between Neuber, Hoffmann-Seeger or None option
  - The None option should be used when plastic stress or strain are present
  - None option is only available for Mechanical Premium or Mechanical Enterprise license
- Analysis of rst files containing plastic results:
  - For Stress Life analysis, if the rst files used for loading contain plastic results, the user should check the results carefully. The extension notifies the user by issuing the warning message:
    - “FE contain plastic stress/strain results, which are invalid for SN fatigue. Please check results carefully.”
  - For Strain Life analysis, if the rst files used for loading contain plastic results, the user should use None option or check results carefully. The extension notifies the user by issuing the warning message:
    - “FE contain plastic stress/strain results. Set the Elastic-Plastic correction to None for accurate results. This option is not available with PRO license. Please check results carefully.”



# Ansys nCode DesignLife Mechanical UI: Load for all sub step

**Time Step Load:** Define the loading cases by Time or Step

1. If the load is defined by “Time”, select the desired time as loading conditions or select “All” to use full results
  2. If the load is defined by “Step” the user will have the options:
    - ‘Step’, ‘Substep’ and ‘Create load for each substep’.
    - Set the Step and Substep desired for each loading condition
- To setup multiple loading conditions, use the “Create load of each substep” button. It creates a loading case per step defined and per substep, until the set substep
  - If you want to create a loading case for every step and its substep, then select “All”, “All” and hit the button
  - Once you have all the loading per step and substep you can unsuppress or delete the undesired ones



# Ansys nCode DesignLife Mechanical UI: Effect of Stress Averaging

Results between DesignLife UI and Mechanical UI can be different due to stress averaging

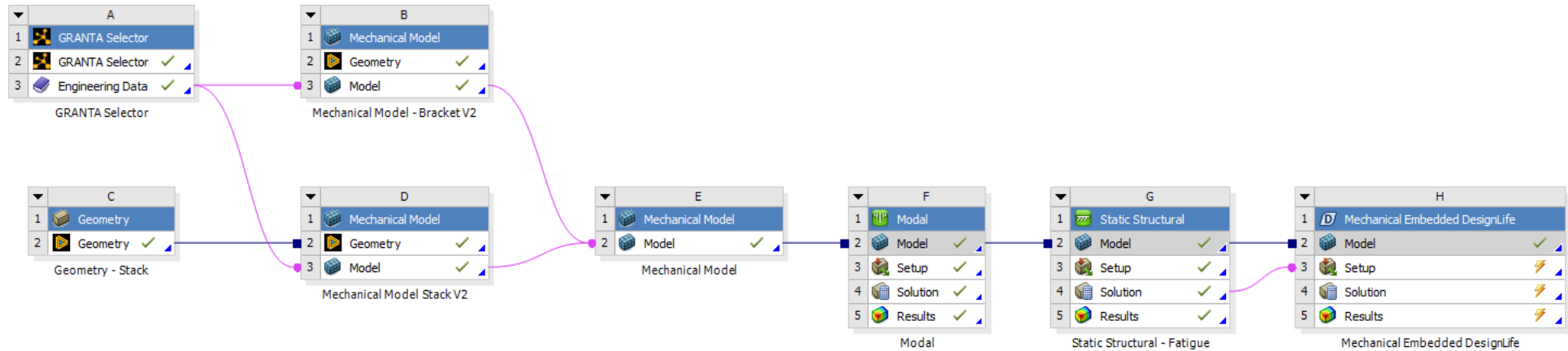
- DesignLife UI calculates the surface nodal stresses by averaging the stresses from all elements that share that node
- Mechanical UI only includes the stresses from elements that have a face on the surface

## NOTES:

- For refined, well-shaped meshes, the effect of the nodal stress averaging method should be minimal
- For coarse tet meshes, the averaged nodal stresses can be affected by the averaging scheme. Thus, the calculated fatigue results may be different

# Ansys nCode DesignLife Mechanical UI: Usability

- Non-compatible systems can be linked with nCode extension if they are not selected as loading events
- Improvements focusing on Performance, Robustness, Accuracy & Documentation





 **Ansys**

